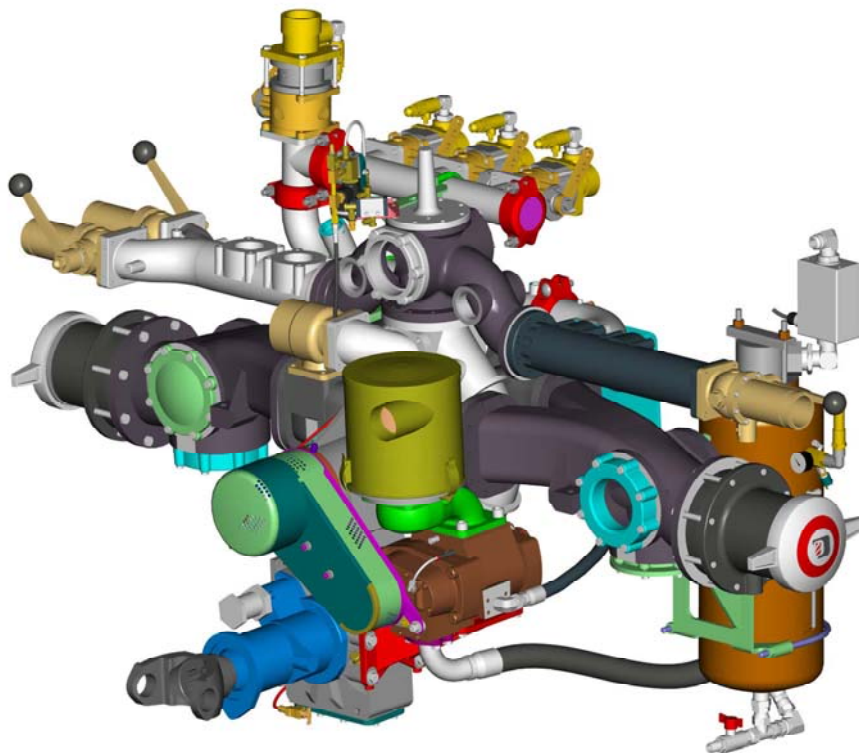


DARLEY
AutoCAFS II
Compressed Air Foam System
OPERATION INSTRUCTIONS
EMBC
AUTOCAFS COMMANDER CONTROL SYSTEM



WWW.DARLEY.COM

Corporate Office:

325 Spring Lake Drive
Itasca, Illinois 60143-2072
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CAFS Applications:

920 Kurth Rd.
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Pump Manufacturing:

1051 Palmer St.
Chippewa Falls, WI. 54729
800-634-7812, Fax (715) 726-2656

This manual is for DARLEY FIRE PUMP:
Model: EMBC Pump Serial Number: _____

Prepared by: EAS
Approved by:
Revised by:

Introduction

This manual provides information for the correct operation, use, and maintenance, of the Darley EMBC AutoCAFS II compressed air foam system including the new AutoCAFS Commander Control. Please read and understand these instructions thoroughly before putting the system in service. Doing so will ensure optimal performance and long life of your CAFS equipped apparatus.

The manual is divided into four sections plus an appendix. Each section details the operation, use, and maintenance of the individual CAFS components that comprise the EMBC compressed air foam system. The appendix includes supplementary information.

Section 1 EM Fire Pump

Operation

Maintenance

Components

Section 2 Air Compressor System

Components

Operation

Maintenance

Section 3 AutoCAFS Commander Control Module

Operation

Installation

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Section 5 Operation of Apparatus Compressed Air Foam System

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Electric Clutch Maintenance and Repair Guide

AutoCAFS II Test Reference Guide

Detailed Specifications

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SECTION 2 – Air Compressor System

Description, Operation & Maintenance ----- 1200638

Related Drawings-----DLD0503 Sht 1 & 2
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DCM0501
DCM0700
DCM0803
DCM0901
DCM1002
5204803
DCM1300
DCM1400
DCS0502
DCS0503

SECTION 3 – AutoCAFS Commander Control Module

Description, Operation & Installation reference -----

SECTION 4 – Foam Proportioner

Description, Operation & Maintenance -----(Apparatus Manufacturer Supplied Proportioner Manuals)

SECTION 5 – Operation of Apparatus Compressed Air Foam System

Description and Operation ----- 1200638

Related Drawings-----DCS0200
DCS0201
1205510
DCS0502
DCM0300
DCM0301
DCM0302

Trouble-shooting ----- 1200638

SAFETY

Always read safety instructions indicated by  symbol.



WARNING: When using *Compressed Air Foam*, the initial reaction force of opening the hose nozzle, is much greater than the normal operating force. The hose nozzle operators should brace themselves as if opening a nozzle on a high-pressure water line. The force on the operator will drop off quickly, becoming much easier to handle than a typical water line.



WARNING:

- 1) Open and close valves slowly
- 2) Do not run with just air/water
- 3) Shut off air when foam tank is empty
- 4) Be prepared for high nozzle reactions – open nozzle slowly



WARNING:

- 1) Do not exceed system rated pressure, capacity or speed.
- 2) Observe local regulations on the use of hearing protection.
- 3) Use only hoses with pressure rating higher than their intended use.
- 4) Remove all pressure from hoses before disconnecting.
- 5) Do not blow pressurized air against the skin.
- 6) Shutdown and depressurize completely before attempting maintenance.
- 7) Compressor oil and components are very hot during operation. Do not touch during or immediately after use.



CAUTION: Avoid immediate restart of Compressor after shutdown.

Allow a 1-minute minimum time period between compressor shutdown and restart for system blow-down.

If maximum compressor speed is exceeded, compressor is automatically disengaged. The compressor will automatically re-engage if engine speed is reduced to 900 rpm or lower and system blow-down is completed.



CAUTION:

- Do not over speed compressor - Input RPM should not exceed that required to produce rated air flow of 220 cfm at 150 psi maximum pressure.
- Disengage air compressor when service testing or performing UL test on CAFS equipped vehicle.



Darley
Pump Division
1051 Palmer St.
Chippewa Falls, WI 54729
715-726-2650

Technical Bulletin on Midship Mounted Fire Pump Drivelines

1202519

FEB, 25 2016

The driveline torque rating is 19,230 lb-ft (26,072 Nm) – exceeding this torque rating can result in a driveline failure.

Great care must be taken in the layout of pump drivelines. Interference and driveline vibration must be considered. An experienced installer with knowledge of driveline considerations, proper layout and recommended guidelines should be utilized as well as proper CAD systems for driveline layouts. Installation of the driveline should not occur until a proper analysis is performed by either a qualified driveline specialist or W.S. Darley. W.S. Darley utilizes, can distribute and can train qualified individuals to use the Allison Multiple Joint Driveline Analysis program.

W.S. Darley requires that midship driven pumps have at most 500 radians per second² torsional vibration, at most 1000 radians per second² inertial drive torsional vibration and at most 1000 radians per second² inertial coast torsional vibration, as calculated by the Allison Multiple Joint Driveline Analysis program, for a completed driveline installation. A completed driveline installation includes the entire multi-driveshaft assembly from the power source on apparatus transmission output flange to the input flange of the rear axle.

Failure to design and analyze a proper driveline layout could result in severe injury and damage to equipment, including but not limited to: the water pump, the water pump transmission, drive tubes, hanger bearings, u-joint crosses, gears, the rear differential, and the main truck transmission.

Questions can also be directed to our Customer Service Department at
800-634-7812 or 715-726-2650.

SECTION 1

Pump Assembly

Description of Pump Type

The Type EMBC pump is a high speed, single stage, UL rated, centrifugal Fire Fighting Pump with an integral belt driven rotary screw air compressor for compressed air foam generation.

Inherent characteristics of the EMBC are compactness, lightweight, high efficiency, and a wide range of pumping capabilities.

The EMBC pump is midship mounted and powered via the chassis engine/transmission.

OPERATION AND MAINTENANCE OF TYPE EMBC FIRE PUMP

Operation of Pump

The pump gearshift consists of a sliding clutch gear splined to the transmission shaft. The sliding clutch gear can be moved forward to engage the pump clutch gear (PUMP position) or to the rear engaging the rear drive shaft (ROAD position). A neutral position is half way between.

The sliding clutch gear is moved either by direct mechanical linkage from a shift lever, or an air powered cylinder controlled by a selector switch. The shift lever must be moved all the way and locked into either ROAD position to drive the truck, or PUMP position to power the pump.

The truck clutch must always be disengaged to stop the rotation of the truck transmission output shaft before shifting into either ROAD or PUMP gear to prevent clashing and damaging the gear teeth. With the manual shift lever, a butt tooth position of gears may be encountered occasionally preventing engagement. If this occurs, move pump shift lever to neutral (half way) position, engage truck clutch momentarily, then disengage the truck clutch and try to shift the pump again.

The pump is always operated with the truck transmission in direct (high) gear, such as 4th on a 4-speed, or 5th on a 5-speed manual transmission, and D or 2.5 on an automatic transmission.

Review the following instruction sheet "*PUMP SHIFTING PROCEDURE*" for step-by-step shifting instructions.



CAUTION: Never run the pump dry except momentarily and at low speeds.

Do not use this pump for hose testing.



CAUTION: Do not over speed compressor - Input RPM should not exceed that required to produce rated air flow of 220 cfm at 150 psi maximum pressure.

Disengage air compressor when service testing or performing UL test on CAFS equipped vehicle

Pump Gear Case Lubrication

Maintain gear case oil level to a point between the two grooves on the oil level dipstick. When checking oil level, dipstick must be screwed all the way in for accurate readings.

Check the oil level every 25 hours or every three months. Change the oil every 50 hours or 6 months.

Service the pump transmission with SAE 80W/90, GL4/GL5 gear lubricant. Do not use grease.



CAUTION: Do not overfill. Overfilling may cause excessive gear case operating temperatures.

Inject grease in zerk fittings on the driveline universal joints once a year.

Mechanical Shaft Seal

This pump assembly incorporates high quality mechanical shaft seal(s) separating the pump housing components from atmosphere. Depending on the pump design, there may be one or two seals on each impeller shaft.

The seal size, design type, component materials, and housing configuration have been specifically designed for this pump application and rated operating parameters.

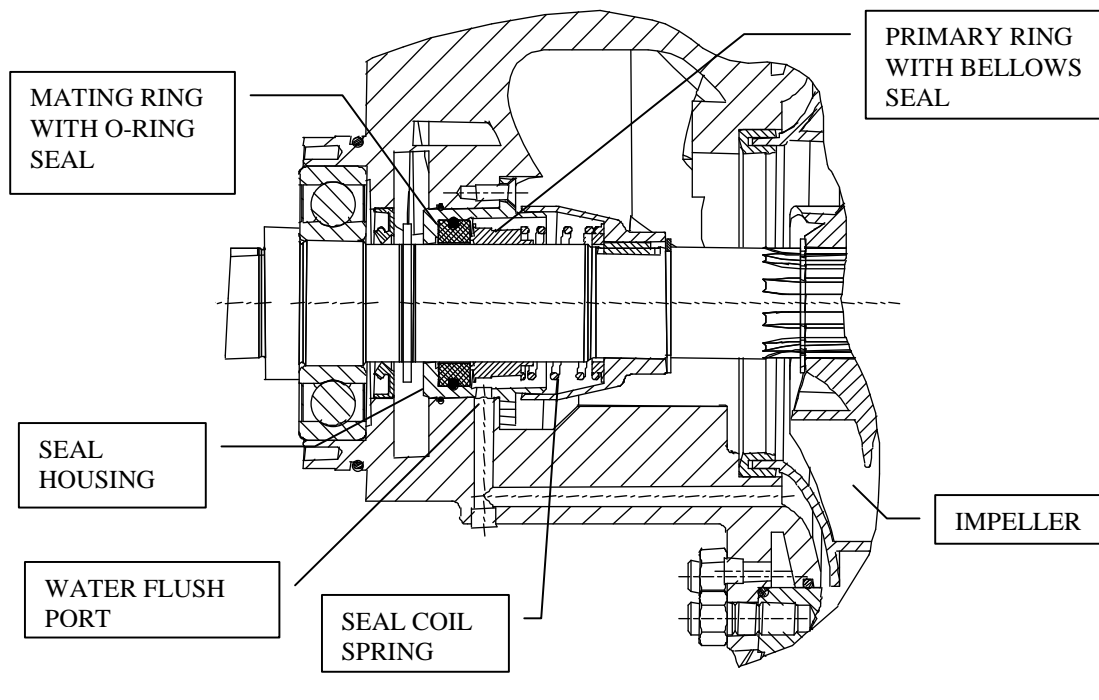
Mechanical Seal Basics

A mechanical seal is a device that houses two highly polished components (known as faces). One face rotates, the other is stationary. A secondary elastomer bellows seals the primary ring to the shaft. An o-ring or cup seal seals the mating ring in the housing. The polished seal faces of the primary and mating rings are pressed together by a spring mechanism to provide adequate force to affect a seal. The force acting between the seal faces increases in direct proportion to product pressure.

The elastomer bellows seal utilized in this pump has the following design features:

- Mechanical drive of the primary seal ring. The drive band's notch design eliminates overstressing the elastomer sealing bellows.
- Bellows design provides automatic compensation for shaft endplay, run out, and primary ring wear.
- Seal face contact pressure is controlled by a single, non-clogging coil spring. This coil spring has been custom welded per Darley specifications to eliminate high-speed spring distortion.

The seal housing is designed and ported to provide optimal water flow and pressure assuring proper cooling and flushing of the seal components.



Operation and Maintenance

When operated within rated operating conditions of this pump, these seals will provide trouble free service for extended periods.

Properly selected and applied mechanical shaft seals are leak free and require no adjustment. Should the seal area develop a leak, investigate the cause as soon as possible. Seal failure, leakage, may be the result of; worn seal faces, leaking bellows, or damaged o-rings. These failures may be attributed to bearing failure, impeller blockage, impeller imbalance, seal housing contamination, operating beyond pump design rating, or dry running,

Mechanical shaft seal design relies on the sealed media, in this case, water, to cool and lubricate the sealing surfaces. Therefore, extended dry operation may cause overheating and scoring or damage to the sealing surfaces, resulting in excessive leakage or a much shortened seal life.

To maximize seal life, minimize operation at pump pressures higher than pump rating. While operating at pressures beyond rating will not immediately damage the seal, it will increase sealing surface wear rate.



CAUTION: DO NOT RUN THE PUMP DRY EXCEPT MOMENTARILY AND AT LOW SPEEDS



CAUTION: DO NOT USE THIS PUMP FOR HOSE TESTING



CAUTION: THE MECHANICAL SEAL SHOULD NOT BE RUN DRY, WHILE THE PUMP IS NOT ENTRAINED WITH WATER, FOR A PERIOD LONGER THAN 2 MINUTES. FAILURE TO FOLLOW THIS RECOMMENDATION WILL LEAD TO PREMATURE WEAR AND FAILURE OF YOUR MECHANICAL SHAFT SEAL.



DARLEY

INSTALLATION OF MECHANICAL FACE SEAL WITH O'RING

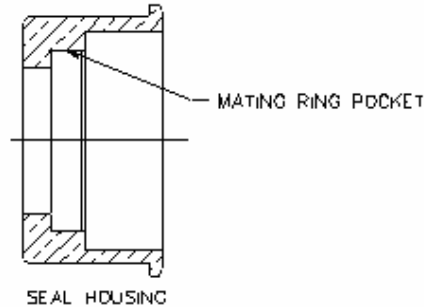
SPECIAL HANDLING

Study the engineering layout before installing the seal. This shaft seal is a precision product and should be handled and treated with care. Take special care to prevent scratches on the lapped faces of the primary and mating ring. Provide a very clean work area where the assembly will take place. Clean hands prior to assembly.

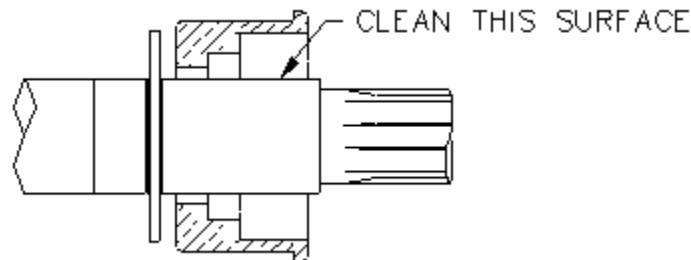
INSTRUCTION STEPS:

Instructions for Installing a Mechanical Shaft Seal

1. Inspect mating ring pocket in seal housing ensuring it is clean, free of chips, and nick free, to provide a proper sealing surface. Isopropyl alcohol may be used to clean the surfaces if required.



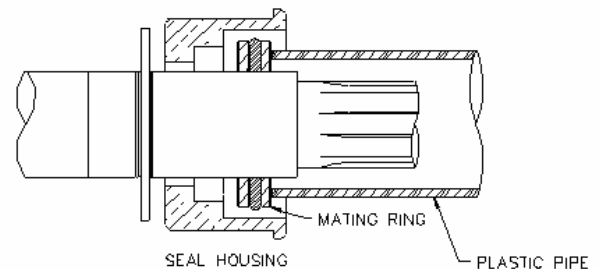
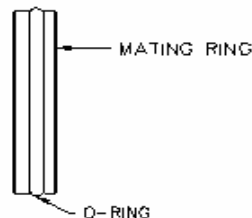
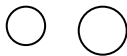
2. Inspect the pump shaft surface under the bellows, ensuring it is clean and nick free to provide a proper sealing surface. Isopropyl alcohol may be used to clean surface if required.



3. Lightly lubricate the o-ring on the mating ring with a single drop of P-80 water soluble rubber lubricant (do not over lubricate) and push it into the cavity using the recommended installation tool or other suitable plastic tube free of contaminants, firmly seating the mating ring square.

Note: The polished face of the mating ring must face out – away from the pump's gear case. Try to not touch the polished sealing face with your fingers; the oils from your fingerprint can cause the seal to leak. Remove any P-80 from the sealing face after installation.

The approximate size of a drop should be between the sizes of these two circles.

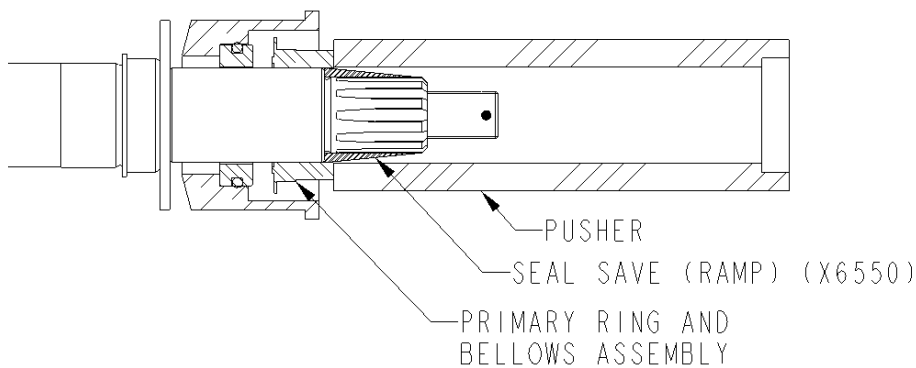
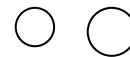


4. Clean the mating ring surface with isopropyl alcohol to remove any fingerprints and any other contaminants left on mating ring.

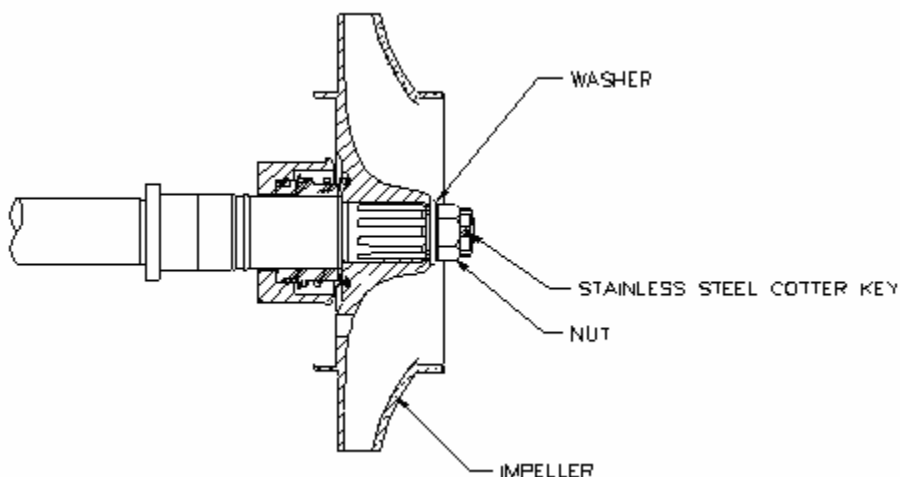
Note: Steps 5 – 9 need to all be completed with in 15 minutes or less.

5. Apply a small drop of P-80 rubber lubricant or water-soluble lubricant (not soapy water) to the inside diameter of the bellows assembly allowing it to be pushed easily into position.
6. Clean the polished sealing face of the primary ring with a clean lint free rag with isopropyl alcohol to remove all fingerprints and other contaminants.
7. Slide a seal save, similar to X6550, over the shaft splines to ensure that the seal is not damaged during installation. Place the primary ring and lubricated bellows assembly (without the spring) on the shaft, using a proper pusher - push the assembly into position so that the seal surfaces are in contact. Remove the seal save from the shaft.

The approximate size of a drop should be between the sizes of these two circles.



8. Put the spring in place, seated tight against the spring retainer on the primary ring.
Note: Some springs may be slightly tapered, so one end fits the seal better than the other. The end of the spring that best fits the seal should go towards the seal to ensure even spring pressure all the way around.
9. Slide impeller onto impeller shaft, engage the spring into the groove of the impeller hub and install impeller washer, impeller nut, and stainless steel cotter key.



*** Reference pump configuration for individual mechanical seal instructions.*

*** Reference pump assembly drawings and pump assembly tips for further assembly.*

Note: If the seal leaks slightly after assembly, it may be necessary to run the pump for approximately 30 minutes at 50-60 psi to rinse out excess lubricant and other contaminants.

Once a mechanical seal has been installed, it is recommended that it not be reused.

If further information is needed, call **DARLEY** in Chippewa Falls, WI. at 800-634-7812 or 715-726-2650



W. S. DARLEY & CO.

DARLEY INJECTION TYPE STUFFING BOX ADJUSTMENT

⚠ Prop 65 Warning: This product contains lead, a chemical known to the State of California to cause cancer, birth defects, and other reproductive harm. Wash hands after handling.

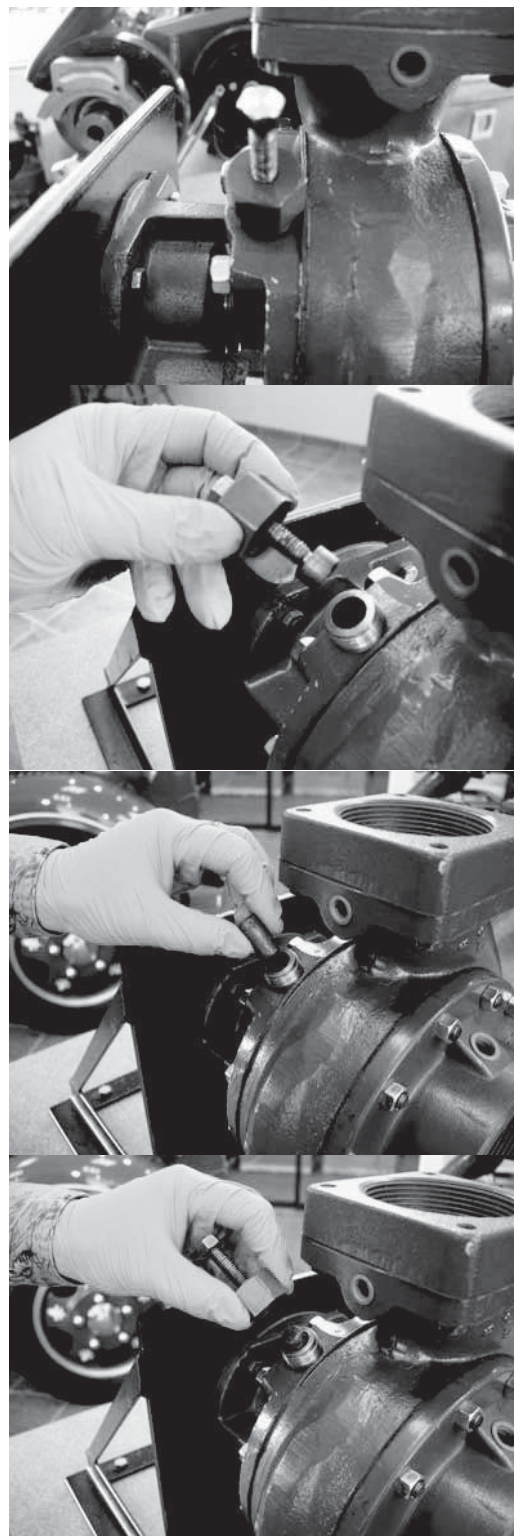
⚠ Caution: Do not attempt to use anything but Darley injection packing. Using the wrong packing material in your pump may cause catastrophic failure of the pump shaft sealing components.

Only use W.S. Darley & Co.'s plastallic injection packing material. It is made of a special composition of shredded fibers, and a special bonding and lubricating compound.

It is important that the stuffing box is completely filled solid with packing and compressed firm during adjustment to prevent formation of voids and excessive leakage.

To pack the stuffing box when empty and assembled in the pump, remove the packing screw and nut assembly, and insert pellet form packing into the packing plunger guide. Replace the packing screw assembly and use a hand speed wrench to force the pellets into the gland. DO NOT USE A POWER TOOL! Repeat pellet additions while turning the impeller shaft by hand until resistance to turning is felt when the stuffing box is almost full. Continue turning packing screw by hand using a standard 6" long 9/16" end wrench until 4 lb. of force is felt at the end of the wrench. This is equivalent to 2 ft-lb or 24 in-lb torque. Continue turning until a few flakes of packing are extruded out the opening between the impeller shaft and the stuffing box hole. The gland is now ready for pressure testing or pumping.

After priming the pump with water, start the pump and raise the discharge pressure to 50 psi. Tighten the packing screw using a 6" long 9/16" end wrench until 4 lb. force is felt at the end of the wrench (24 in-lb torque). Continue operating the pump at 50 psi for 5 minutes to dissipate packing pressure against the shaft and permit cooling water to flow between the shaft and stuffing box hole. Make sure that water actually does come through before operating pump at any higher pressure. The normal drip rate may vary between 5 and 60 drops per minute.



Operate the pump for 10 minutes at the highest normal operating pressure flowing sufficient water to prevent overheating. Do not run the pump blocked tight. Lower discharge pressure to 50 psi and repeat the packing screw tightening procedure outlined above.

The pump may now be operated for any time period required within its rated capacity. However, the drip rate should be monitored more frequently during the first few hours, and adjusted if necessary to achieve a stable flow rate. Several more adjustments may be required.



For a list of approximate quantity of packing pellets required by model (completely repacked), see below:

Model	Approximate # Packing Pellets
A	6
2BE	6
EM	15
H	8
JM	8
KD	10
KS	8
LD	15
LS	9
P	10
U2	5
U4	10

If further information is needed, call **W.S. DARLEY & CO.**
at Chippewa Falls, WI. at 800-634-7812 or 715-726-2650

PUMP SHIFTING PROCEDURE

For trucks equipped with manual transmissions, the following shifting procedure should be followed for pump operation:

1. Set parking brake.
2. Disengage truck clutch to stop shaft rotation.
3. Move pump shift lever to PUMP position.
4. Move truck transmission shift lever to neutral position.
5. Engage truck clutch.
6. Prime the pump (see priming instructions).
7. Disengage the truck clutch.
8. Move truck transmission shift lever to direct drive position and lock in place with safety latch.
9. Engage truck clutch to begin pumping.

If the power pump shift is provided, the procedure is identical except green indicator light (if provided) will come on at step #3 to show pump gear has been engaged.

To return to road operation:

1. Disengage truck clutch to stop shaft rotation.
2. Move truck transmission shift lever to neutral position.
3. Move pump shift lever to ROAD position.

When the truck is equipped with an automatic transmission, a danger exists that if the operator forgets to move the pump shift lever to PUMP position, and at the same time place transmission selector lever in high gear before leaving cab, the engine will continue to run due to converter slip. Upon advancing the vernier throttle at the pump operators panel, the engine could overcome the parking brake and accidentally move the truck. To prevent this possibility, the following shifting procedure should be followed for pump operation:

1. Set parking brake.
2. Place automatic transmission shift selector in neutral.
3. Move pump shift lever to PUMP position. "Pump Engaged" light in cab should now come on.
4. Prime the pump (see Priming Instructions).
5. Move automatic transmission shift selector to direct drive position (See Automatic Transmission Instructions).
6. Lock automatic transmission shift selector in direct drive position with safety latch provided.
7. Check that the parking brake is fully engaged.
8. Depress foot accelerator and observe that speedometer registers MPH. If pump is not engaged, speedometer will not indicate MPH.
9. Listen for pump shift and sound of pump gears turning.
10. At pump operators position, observe that the green indicator light above vernier throttle control is on. **Do not** operate throttle unless light is on.
11. Observe discharge pressure gage on panel while advancing vernier throttle, to ensure that it is indicating pressure. If Pump is not engaged, no pressure will show.
12. Remember, the vernier throttle has a quick release emergency center button. Push it all the way in immediately, should the truck move.

To return to ROAD OPERATION:

1. Place the truck transmission selector lever in reverse position to stop forward rotation of transmission shaft.
2. Move transmission selector to neutral, and at the same time, move the pump shift lever from PUMP to the ROAD position.

OPERATION OF PUMP SHIFT WITH AUTOMATIC TRANSMISSION

The pump gear shift consists of a sliding clutch gear splined to the transmission shaft which can be moved forward to engage the pump clutch gear, or to the rear to engage the rear drive shaft connected to the truck drive axle.

The sliding clutch gear is moved either by direct mechanical linkage from a notched quadrant shift lever, or by an air power cylinder controlled by a selector valve. The shift lever or selector valve must be moved all the way and locked for either ROAD position to drive truck or PUMP position to pump.

MANUAL PUMP GEAR SHIFT PROCEDURE

With trucking parking brake set, the truck transmission shift selector must be in neutral position to stop rotation of truck transmission output shaft before shifting into either ROAD or PUMP gear to prevent clashing and damage to gear teeth. With a manual pump gear shift control, a butt tooth position of gears may be encountered preventing engagement and "Pump Engaged" light from coming on. If this occurs, move transmission shift selector momentarily into any forward gear position with engine idling, then return to neutral. Wait approximately 5 seconds until shaft stops turning. Moving the pump shift lever to PUMP position again should complete the pump shift and turn on the "Pump Engaged" light. Repeat this procedure if a butt tooth condition is again encountered. Pump priming should be completed before shift to PUMP position.

AIR POWER PUMP GEAR SHIFT PROCEDURE

With the air power pump gear shift control a butt tooth condition may also occur preventing engagement and "Pump Engaged" light from coming on. This can be easily overcome by momentarily placing truck transmission in any forward gear position with engine idling after the pump shift valve is placed in the PUMP position. Shaft rotation will complete the shift and turn on green "Pump Engaged" light. Transmission shift should be returned to neutral position after "Pump Engaged" light comes on for the pump priming period.

The above procedure insures that the pump shift is completed and the "Pump Engaged" light comes on. An alternate procedure in case of a butt tooth condition is simply to wait until the transmission is placed into pump drive gear position, when shaft rotation will immediately permit pump gears to mesh into full engagement.

The pump is usually operated with truck transmission in direct (high) gear such as D or 2-5. Overdrive may be required with very low speed engines.

When the truck is equipped with an automatic transmission, a danger exists that should the operator forget to move the pump shift valve to PUMP position, and at the same time place transmission selector in high gear before leaving cab, the engine will continue to run due to converter slip. If the operator advances the vernier throttle at the pump operators panel, the engine could overcome the parking brake and cause the truck to move. To prevent this possibility, the following shifting procedure should be followed for PUMP position:

1. Read and fully understand pump Operators Manual before proceeding.
2. Set parking brake and idle engine.
3. Place automatic transmission shift selector in neutral.
4. Move pump shift lever or valve to PUMP position. "Pump Engaged" green light in cab should now come on. If not, momentarily place truck transmission shift selector in a forward gear to complete pump shift, then return to neutral position. Green light will now be on.

5. Prime pump.
6. Move automatic transmission shift selector to direct drive position. "OK to Pump" green light in cab should now be on.
7. Lock automatic transmission shift selector in direct drive.
8. Listen for sound of pump gears turning. Speedometer will show MPH unless connected to the wheel.
9. At pump operators position, observe the green indicator warning light near vernier throttle control.
WARNING: DO NOT OPEN THROTTLE UNLESS LIGHT IS ON.
10. Advance throttle to provide a minimum of 900 rpm idle speed. Observe discharge pressure gage on panel while advancing vernier throttle to make sure it is indicating pressure. If pump is not engaged or the pump is not primed, no pressure will show.

The vernier throttle has a quick release emergency red center button. Push it all the way in to return the engine to idle if necessary in an emergency situation

To return to ROAD operation:

1. Throttle engine back to idle.
2. Place the truck transmission selector lever in neutral position. Wait approximately 5 seconds until drive shaft stops rotating.
3. Move pump shift lever or valve from PUMP to ROAD position. "Pump Engaged" green indicator light should be off. A butt tooth condition may require momentary engagement of transmission to complete the shift.

OPERATION OF PUMP SHIFT WITH MANUAL TRANSMISSION

The pump gear shift consists of a sliding clutch gear, splined to the transmission shaft which can be moved forward to engage the pump clutch gear, or to the rear to engage the rear drive shaft connected to the truck drive axle. A neutral position is half way between.

The sliding clutch gear is moved either by direct mechanical linkage from a notched quadrant shift lever, or by a vacuum or air power cylinder controlled by a selector switch. The shift lever must be moved all the way and locked for either ROAD position to drive truck or PUMP position to power pump.

The truck clutch must always be disengaged to stop rotation of truck transmission output shaft before shifting into either ROAD or PUMP gear to prevent clashing and damage to gear teeth. With the manual shift lever, a butt tooth position of gears may be encountered occasionally preventing engagement. If this occurs, move pump shift lever to neutral (half way) position, engage truck clutch momentarily, then disengage truck clutch and try pump shift again.

The pump is always operated with truck transmission in direct (high) gear, such as 4th on a 4 speed or 5th on 5 speed transmission.

The following shifting procedure should be followed for PUMP operation:

1. Set parking brake.
2. Disengage truck clutch.
3. Move pump shift lever (valve) to PUMP position. "PUMP ENGAGED" green light in cab should now come on.
4. Move truck transmission shift lever to neutral position.
5. Engage truck clutch.
6. Prime pump (See Priming Instructions).
7. Disengage truck clutch.
8. Move truck transmission shift lever to direct drive position and lock in place with safety latch provided.
9. Slowly engage truck clutch to begin pumping.

TO RETURN TO ROAD OPERATION

1. Disengage truck clutch to stop shaft rotation. Allow 10 seconds for shaft to stop rotating.
2. Move truck transmission shift lever to neutral position.
3. Move pump shift lever (valve) to ROAD position.

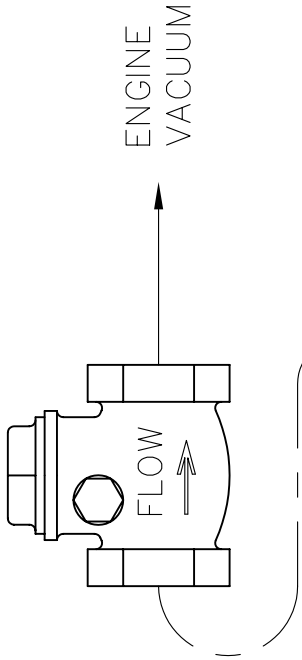
CAUTION

Follow the procedures step by step as indicated.

**IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT
CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650**

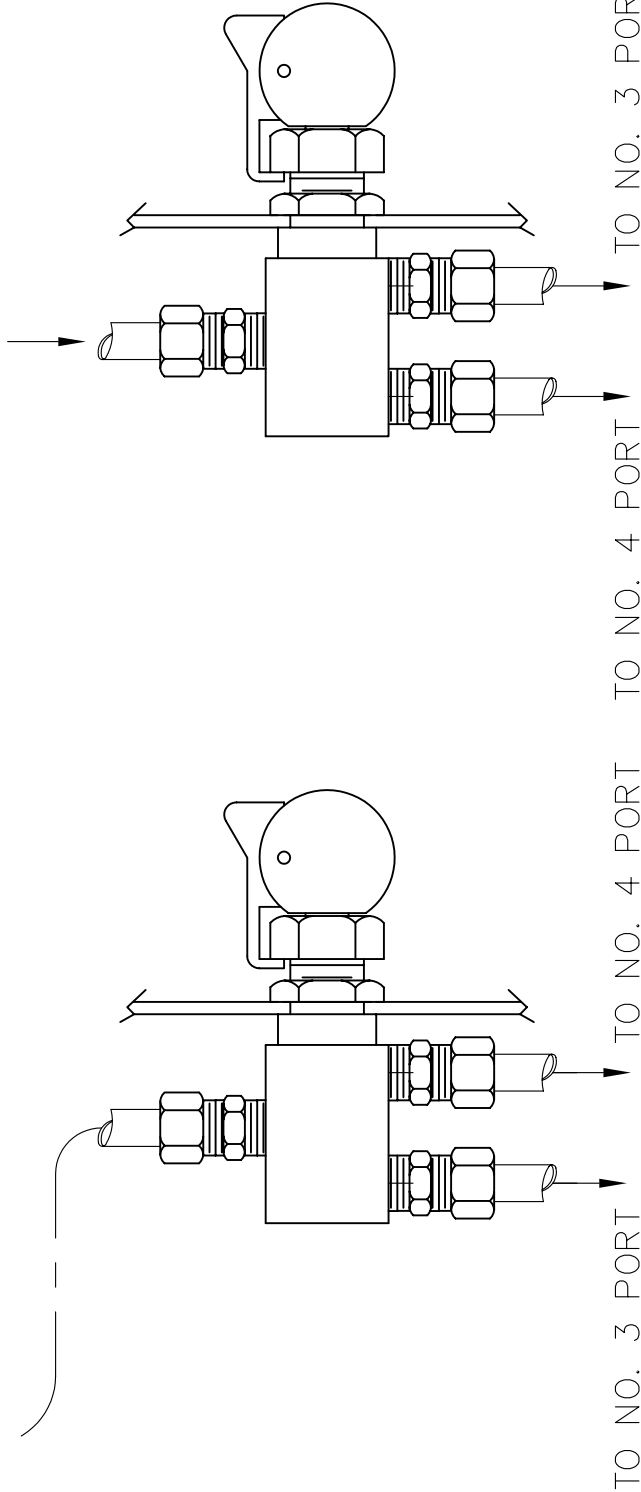
REVISIONS			
LTR	DESCRIPTION	DATE	CHG NO.
			APPR'D

CHECK VALVE



NOTE: CONNECT TO AIR SOURCE THAT HAS AN 80 PSI PRESSURE PROTECTION VALVE

75 TO 120 PSI AIR SUPPLY

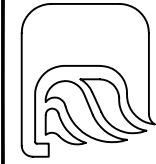


VACUUM SHIFT
PIPING DIAGRAM
3" DIA. CYLINDER

AIR PRESSURE SHIFT
PIPING DIAGRAM
1-1/8 DIA. CYLINDER

INCH
[MILLIMETER]

THIRD ANGLE PROJECTION



W.S. DARLEY & CO.
MELROSE PARK, IL - CHIPPEWA FALLS, WI

DWG - POWER SHIFT VALVE

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	DR'N	TED	DATE	MAR10,97
	CHKD		SCALE	NTS
				DGS0500

PARTS LIST			REVISIONS				
REP. NO.	DESCRIPTION	QTY.	LTR	DESCRIPTION	DATE	CHG NO.	APPR'D
110	PISTON ROD	1					
111	ORING - ROD	1					
112	OILITE SLEEVE BEARING	1					
113	O'RING	2					
114	CYLINDER	1					
115	QUAD RING SEAL	1					
116	CYLINDER END	1					
118	O'RING	1					
119	WIPER RING	1					
120	CYLINDER HEAD	1					
121	PISTON	1					
201	CYLINDER SCREWS	3					
202	ADAPTER	1					
203	ADAPTER SCREWS	4					

PORT NO. 4

PORT NO. 3

INCH [MILLIMETER]

THIRD ANGLE PROJECTION

	DR'N	DWS	DATE	JUN02,93
	CHKD		SCALE	1/1
			TRCD	

OLD PART NO. G2514

DO NOT SCALE PRINT

ALL DIMENSIONS IN INCHES UNLESS NOTED

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REFERENCE SPEC AY00201 (G427A1) & AY00202 (G427A2)

W.S. DARLEY & Co.
MELROSE PARK, IL - CHIPPEWA FALLS, WI
DWG - 1.75" AIR CYLINDER

DGC0701

WARNING: DO NOT USE THIS PUMP FOR HOSE TESTING

OPERATING THE ENGINE

After the pump has been primed, the engine speed should be increased gradually -- never jerk throttle wide open. Likewise, the engine speed should be decreased gradually when shutting down.

Watch the pump pressure gage and open throttle only enough to give the desired pressure. The pressure may rise high enough to burst the discharge hose, when using small nozzles, if the engine is given full throttle (except pumps equipped with pressure regulators set for desired pressure).

Never run engine at high speeds except when pump is primed and ready to discharge water.

COOLING THE ENGINE

NFPA 1901 requires that a supplementary heat exchanger cooling system be provided. On most models, this heat exchanger is an integral part of the pump, and the installation of two hoses from the engine cooling system to the pump is all that is required.

On some models an external heat exchanger must be used. In that case two hoses from the engine cooling system and two lines from the pump will run to the heat exchanger.

The cooling line should not be opened until pressure develops in the pump, and pump should never be operated under heavy loads prolonged without an adequate supply of cooling water flowing.

Coolant temperatures should never be allowed to exceed 200° F while pumping and 180° F is usually taken as a safe operating temperature.

Always shut off cooling line when through pumping.

SUCTION STRAINERS

A large suction strainer, which will prevent the passage of a body larger than the pump impeller ports, must always be used on the free end of the suction line when pumping from draft.

The small hydrant strainer must always be inserted in the suction manifold of pump, when pumping from hydrants and at all other times except when maximum capacity is required from draft.

Failure to use a strainer at all times when pumping will cause serious trouble by clogging the pump because, even in water mains, foreign matter is invariably present, and will be drawn into pump by the high velocity of the water entering.

SUCTION LINE

The suction line of a fire pump can be the source of more operating difficulties than all the rest of the pump when working with a suction lift. Faults in the suction line which cause trouble in operation are as follows:

AIR LEAKS:

A small amount of air, expanding in the vacuum of the suction line, displaces a considerable volume of water which subtracts from the capacity that the pump is able to deliver, making the priming difficult or causing the pump to lose its prime. Therefore, it is absolutely essential to keep the suction line and the suction side of pump casing air tight at all time when drafting water.

Air leakage into pump while operating is usually indicated by a rattling sound in pump casing, miniature explosions in stream issuing from the nozzle, or by losing of prime when operating at very low capacities.

The usual cause of leaky suction lines is carelessness in handling of suction hose. Bruising of hose threads by bumping against hard surfaces or sand in the coupling often prevents tightening of the joints up against the gaskets. The hose gaskets are often defective and are sometimes lost without being noticed by the operator.

INSUFFICIENT SUBMERGENCE:

The free end of suction hose must be submerged to a sufficient depth to prevent the entrance of air that may be sucked down from the surface of the water to a considerable depth when operating at large capacities.

Entrance of air into suction lines in this manner is indicated by a small whirlpool, or vortex, on the surface of the water over the end of the hose.

A minimum submergence of 4 times the hose diameter to the upper holes in suction strainer is recommended where full capacity of pump is required. Where sufficient submergence is not possible, a board or sheet of metal laid over end of suction line will keep air from entering.

SUCTION LINE ENTRANCE TOO CLOSE TO BOTTOM:

If the end of suction line is laid on the bottom of the source of supply, a part of the suction opening will be shut off; and if the bottom is soft, the hose will suck itself down into the earth closing more of the opening and loosening sand and mud to be carried into the pump.

The suction entrance should be suspended a foot or more above the bottom, or if this is not possible, it should be laid on a board or piece of sheet metal. A rope tied to the suction strainer is a convenient means of holding it off the bottom.

OBSTRUCTION OF SUCTION STRAINER BY FOREIGN MATTER:

The high velocity of water entering the suction line will carry loose foreign bodies in against the strainer from a considerable distance. Therefore, all weeds and refuse should be removed from close proximity of the suction entrance.

SUCTION LINE TOO SMALL OR TOO LONG:

The flow of water into the pump is opposed by the frictional resistance in the suction line. This friction loss must be added to the height of the pump above the water (static lift) to determine the "total lift" of the pump. When all of the vacuum in the pump (atmospheric pressure) is consumed in raising water through this total lift, then the limit of capacity has been reached. This capacity can be increased only by decreasing total lift. If the static lift cannot be reduced, then the friction loss must be reduced by using a shorter or larger suction hose.

The rated capacity of the pump is guaranteed for a static lift of 10 feet for ratings up to 1500 gpm, with 20 feet of recommended suction hose at 2000 feet. To increase the capacity without reducing the static lift, or to increase lift without sacrificing capacity, requires larger suction hose.

An excessively long suction line is a handicap to any pump, for besides reducing capacity through the added friction loss, it retards priming and it produces a detrimental effect known as "cavitation". This means a separation of the water column in the pump suction, or void spaces, produced by the inertia of the heavy mass of water in the line resisting sudden change in the velocity when the pump starts to deliver or when discharge valves are opened or closed. This phenomenon reduces

capacity further, and usually sets up a vibratory motion and “water hammer” as the water surges in and out of the void spaces.

When operating with a long suction line, the driving engine should be accelerated gradually, the discharge gates opened gradually, and the capacities of the pump should be held down to within the range of smooth performance.

AIR TRAP IN SUCTION LINE:

If the suction line is laid so that part of it is higher than any other part that is nearer to the pump, as when hose is laid over a high bridge rail, an air trap is formed at the highest part of the hose from which the air cannot be sucked out by the primer. This trapped air is expanded and carried into the pump with the first rush of water causing the pump to immediately lose its prime.

If suction line cannot be laid so that it slopes all the way from pump to water, it can still be primed easily by simply allowing the primer to continue to function until all the trapped air in the hose has been carried into the pump and picked up by the primer.

TESTING FOR AIR LEAKS

Tests for leakage should be made with the suction hose attached and capped, discharge gate open, and all other openings closed tightly.

Run electric priming pump with primer shut-off valve open, until 22" of Hg is shown on the gage. The vacuum should hold for no more than 10" of drop in 5 minutes before satisfactory performance of pump can be expected.

If excessive leakage of air occurs, the source of leaks can be located by shutting off primer motor, with vacuum at its highest point, and listening for the hiss of air.

In the absence of a vacuum gage, the vacuum in pump may be judged by closing suction opening with the flat of hand or a rubber pad.

Water or air pressure may be applied to pump casing to test for air leakage if more convenient. DO NOT pressurize with air beyond 10 PSI

SOURCE OF WATER SUPPLY

Water may be drafted from a pond, lake, stream, cistern, stock tank, or well; but whatever the source, the static lift must not exceed 20 feet from the center of the pump to the surface of the water and a lift not exceeding 10 feet is recommended. The source of supply should be reasonably clear and free from foreign matter. It is recommended that all water holes, which may be needed for fire protection, be deepened if necessary and kept free from weeds and refuse. In many fire protection areas, cisterns or reservoirs are built and allowed to fill up with rain water to be used in emergencies.

PUMPING IN COLD WEATHER

The first insurance against cold weather trouble is to keep fire apparatus stored in heated quarters. All water must be eliminated from pump casing and primer line between periods of operations.

When setting up for pumping, unnecessary delays should be avoided by having thoroughly trained pump operators. Be sure that primer and booster lines are kept closed until ready for use. Having discharge lines ready so that pump may be started as soon as it have become primed. Do not stop flow of water through the pump until ready to drain and return to the station.

Engine Coolant from the engine circulated through the heater jacket in pump casing prevents all ordinary freezing troubles.

WHEN FINISHED PUMPING

Drain water out of pump casing immediately. (Drain valve is located at lowest point in pump casing, and usually accessible from underneath operators panel.)

Don't forget to close all drain cocks after all water has been drained out. Trouble in priming will follow on the next run if this is forgotten.

Shut off cooling line to make pump ready for priming again.

If pump transmission is equipped with a transmission cooler it must be drained also. If the master drain is located below the cooler outlets it can be connected to the master drain, if not, two separate drains must be connected to the transmission cooler. **Failure to drain transmission cooler may result in water in the gearcase if water in the cooling coil freezes.**

If pump is equipped with a heat exchanger, drain heat exchanger using gravity and vacuum drain on all trucks as follows: Close all open lines and drain cocks. Open cooler valve and open air vent at top or drain cock at bottom of heat exchanger depending on model. With the pump air-tight, open primer with engine running for about a minute and then close primer. Drain pump of water which was deposited when heat exchanger and lines were being drained.

Pump not often used for fire service should be inspected and run periodically to ensure that they will be in readiness for an emergency.

PUMPING SALT WATER

The pump should be flushed out with fresh water immediately after pumping salt water to prevent excessive rusting. (Except pumps which are built of special materials, such as bronze, to resist the corrosive action of the brine.)

When measuring sea water with a Pitot Gage, capacities shown in Table No. 2 should be discounted approximately 1 1/2% to determine the correct capacity.

A centrifugal pump will show 3% higher pressure and require 3% more power when handling sea water than when handling fresh water if operated at the same speed and capacity.

TESTING OF EQUIPMENT FOR PRACTICE

It frequently happens that operators of fire apparatus, who are not thoroughly familiar with its operations, become confused under the stress of emergency and neglect some little detail that may cause trouble or delay in getting the equipment into operation. Therefore, we urge that practice tests be conducted repeatedly until operators are thoroughly trained. More than one person in the department should be a competent operator.

Practice should include pumping from low lifts, high lifts with short and long suction lines, with suction line elevated to form an air trap, and from hydrants, at large and small capacities.

It is important to note the effects of air leaks in hose, insufficient submergence and restriction of suction line. (Suction line can be restricted by placing a can or other strong closure around the suction strainer).

NEVER BREAK OR RESTRICT SUCTION OR ALLOW AIR TO ENTER SUCTION LINE WHILE ENGINE IS OPERATING WITH THROTTLE OPEN. This will release the load and allow engine to run away.

Do not allow personnel to hold a large nozzle while working at high pressures for serious accidents may result if hose breaks loose.

MEASURING PUMP PERFORMANCE

Pump performance is measured by the quantity of water it can deliver per minute against a certain pressure called "Total Head" or "Net Pump Pressure", as it is usually termed in fire pump testing.

The net pump pressure is the sum of the pump discharge pressure, as shown on the pressure gage with which the pump is regularly equipped, and the total suction lift converted to equivalent pounds per square inch. If pump is operating from a hydrant, the net pump pressure is the discharge pressure less the incoming pressure from hydrant measured at the suction entrance of pump.

Capacity of fire pump is measured in gallons per minute. The usual method of measurement is to determine the pressure of the jet of water leaving a given size of nozzle by means of a "Pitot Gage" from which the capacity is computed mathematically.

A Pitot Gage consists of a small tube adapted to a point directly into the hose nozzle from the center of the issuing stream, the other end of the tube being connected to an accurate pressure gage.

The nozzle jet drives straight into the Pitot tube and converts the velocity of the jet to pressure which is an accurate measure of velocity of the water as it leaves the nozzle. The tip of the Pitot tube should be one-half the diameter of the nozzle away from nozzle tip while taking reading. Table No. 2 gives nozzle capacities for various Pitot Gage readings.

If a Pilot gage is not available approximate pump capacities can be determined by reference to Table No.3

ACCEPTANCE TESTS

Acceptance tests require continuous tests of three hours duration: 2 hours at 100% rated capacity and 150 PSI net pump pressure; one-half hour at 70% capacity and 200 PSI; one-half hour at 50% capacity and 250 PSI; and a spurt test at 100% capacity and 165 PSI.

Table No. 1 shows recommended set-ups and gage readings for rating tests.

To adjust nozzle pressure for the correct capacity, while maintaining the correct pump pressure, it is necessary to make simultaneous adjustments of engine throttle and the discharge gate valve, partially closing the latter until just the right discharge resistance is built up.

ENGINES

A fire pump imposes heavy loads on the engine that drives it, sometimes absorbing all of the power the engine is capable of delivering at full throttle. Continuous pumping gives the engine no time to rest. Therefore, a new engine and pump unit must be thoroughly broken-in before it is required to deliver prolonged maximum pump performance.

We recommend a minimum break in period of 20 hours at light pumping loads, with occasional spurt tests and interruptions. Temperature and lubrication should be checked during this period.

Engine manufacturers' power ratings usually show maximum performance of a selected, factory adjusted engine, operating without fan, generator, muffler or other accessories, and corrected for "ideal" conditions, i.e. sea level barometer (29.92" of mercury) 60°F and high humidity. Therefore, the actual power delivered by an average truck mounted engine is considerably lower than the manufacturers' rating, and allowances must be made in predicting pump performance.

EFFECTS OF ATMOSPHERIC CONDITIONS ON ENGINE AND PUMP PERFORMANCE

Each one inch of drop in Barometric pressure or each 1000 feet of elevation of the pumping site reduces engine power approximately 3 1/2% for engines not equipped with a turbo charger.

Each 12° rise in temperature above 60° F of carburetor intake air reduces engine power approximately 1%.

Lowering of humidity reduces power slightly.

Each one inch drop in Barometric pressure or each 1000 feet of elevation reduces the maximum possible static lift of a pump approximately one foot.

Temperature of the water supply affects the attainable suction lift of a pump. The effect is slight at low water temperatures but becomes increasingly detrimental as the temperature rises.

A 10° rise from 70°F will subtract about 1/2 foot from the maximum attainable suction lift, while an equal rise from 100°F will reduce the lift at least 1 1/2 feet.

Temperature is an important consideration when pumping from a test pit where the water is heated by recirculation.

**IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT
CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650**

DEFINITIONS

HEAD OF WATER -- vertical depth of water measured in feet or in pressure per unit or area. In hydraulics, head always represents pressure and it is expressed interchangeably in feet of water or pounds per square inch and sometimes in inches of depth of mercury.

STATIC HEAD -- the pressure that is exerted by a stationary column of water of a given height or depth.

TOTAL HEAD OR TOTAL DYNAMIC HEAD -- the maximum height above the source of supply to which the pump would elevate the water plus all the resistance to flow in the pipe or hose line.

DISCHARGE HEAD -- the pressure measured at the discharge outlet of a pump.

SUCTION HEAD -- the positive pressure measured at the suction entrance of a pump (when pumping from an elevated tank or hydrant).

VELOCITY HEAD -- the equivalent pressure represented by fluid in motion as measured by means of a Pitot Gage.

STATIC LIFT -- the vertical height of the center of the pump above the source of supply (when pump from draft).

TOTAL SUCTION LIFT -- the static lift plus the friction in suction line plus entrance losses.

NET PUMP PRESSURE -- the total dynamic head of the pump.

EFFECTIVE NOZZLE PRESSURE -- the pump discharge pressure minus hose friction plus or minus the difference in elevation above or below pump.

WATER HORSEPOWER - the theoretical power required to deliver a given quantity of water per minute against a given head.

BRAKE HORSEPOWER -- Actual power as delivered by a motor or engine to a driven machine.

PUMP EFFICIENCY -- The quotient of the water horsepower divided by brake horsepower required to produce it.

WATER HAMMER -- a series of shock waves produced in a pipeline or pump by a sudden change in water velocity. A sudden change in flow velocity can result from rapid closure of valves. A pressure wave is set up which travels back and forth in the water column at extremely high speed producing rapid vibrations that may be violent and destructive if the water column is long.

THE MAXIMUM THEORETICAL LIFT of a pump is 34 feet, which is the pressure of the atmosphere at sea level. The maximum practical total lift at sea level is 20 to 25 feet (depending on the type and condition of the pump) and this decreases with drops in barometric pressure.

OPERATING CHARACTERISTICS OF PUMPS

CENTRIFUGAL PUMPS: A centrifugal pump develops pressure by centrifugal force of the liquid rotating in the impeller wheel. The pressure developed depends upon the peripheral speed of the impeller (increasing as the square of the speed) and it remains fairly constant over a wide range of capacities up to the maximum output of the pump, if speed remains constant.

If the discharge outlet of a centrifugal pump is entirely shut off, with speed kept constant, there is a small rise in pressure, the water churns in the pump casing and the power drops to a low value. If the discharge is opened wide, with little resistance to flow the pressure drops while the capacity and power both increase to their maximum.

A centrifugal pump is an extremely simple mechanism mechanically, but rather complex hydraulically; in that many factors enter into the design of the impeller and water ways which will affect the pump's efficiency.

DISPLACEMENT PUMPS: Rotary and piston pumps are termed "Positive Displacement" pumps because each revolution displaces or discharge (theoretically) an exact amount of liquid, regardless of the resistance. The capacity is, therefore, proportional to the number of revolutions of the pump per minute and independent of the discharge pressure except as it is reduced by "slip" (leakage past the pistons or rotors). For a given speed the power is directly proportional to the head. If the discharge is completely shut off, the pressure, power, and torque climb indefinitely until the drive power is stalled or breakage occurs.

Slip is the greatest factor affecting efficiency of a displacement pump, and this factor is greatly influenced by the condition of and wears on the working parts.

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CONVERSION FACTORS

One pound per square inch	=	2.31 feet of water
	=	2.04 inches of mercury
	=	27.7 inches of water
One foot of water	=	0.43 pounds per square inch
One inch of mercury	=	1.13 feet of water
	=	0.49 pounds per square inch
One cubic foot of water	=	62.4 pounds
	=	7.5 gallons
One gallon of water	=	231 cubic inches
	=	0.13 cubic feet
	=	8.34 pounds
	=	3.8 liters
One Imperial Gallon	=	1.2 U.S. gallons
Atmospheric Pressure (Sea Level)	=	14.8 pounds per square inch
	=	29.9 inches of mercury
	=	34 feet of water

TABLE NO. 1
NFPA 1901 TEST

Class A							
TEST No.	GPM	Recom- mended Nozzles	Min. Nozzle Press. PSI	Min. Disch. Press. PSI	Min. Net Pump Press. PSI	Disch. Lines	Suction Hose
250 GPM Fire Pump							
1	250	(1), 1”	72	143	150	(1), 50'	20’ of 3”
2	175	(1), 7/8”	62	194	200		
3	125	(1), 3/4”	56	244	250		
4	250	(1), 1”	72	158	165		
350 GPM Fire Pump							
1	350	(1), 1-1/4”	58	144	150	(1), 50'	20’ of 4”
2	245	(1), 1”	69	195	200		
3	175	(1), 7/8”	62	245	250		
4	350	(1), 1-1/4”	58	159	165		
500 GPM Fire Pump							
1	500	(1), 1-1/2"	57	143	150	(1), 50'	20' of 4"
2	350	(1), 1-1/4"	58	194	200		
3	250	(1), 1"	72	245	250		
4	500	(1), 1-1/2"	57	158	165		
750 GPM Fire Pump							
1	750	(1), 1-3/4" or (2), 1-1/4"	68 66	142	150	(2), 50'	20' of 4-1/2"
2	525	(1), 1-1/2"	62	193	200	or	
3	375	(1), 1-1/4"	66	244	250	(2), 100'	
4	750	(1), 1-3/4" or (2), 1-1/4"	68 66	157	165	Siamesed	
1000 GPM Fire Pump							
1	1000	(1), 2" or (2), 1-1/2"	71 57	142	150	(2), 50'	20' of 5"
2	700	(1), 1-3/4" or (2), 1-1/4"	60 58	193	200	or	
3	500	(1), 1-1/2"	57	244	250	(3), 100'	
4	1000	(1), 2" or (2), 1-1/2"	71 57	157	165	Siamesed	

Min. discharge pressures listed above are for pumps operating with full 10' static suction lift. These pressures must be increased by 1 PSI for each 2.3 ft. less than 10' of lift.

TABLE NO. 1
NFPA 1901 TEST

Class A							
TEST No.	GPM	Recom- mended Nozzles	Min. Nozzle Press. PSI	Min. Disch. Press. PSI	Min. Net Pump Press. PSI	Disch. Lines	Suction Hose
1250 GPM Fire Pump							
1	1250	(1), 2-1/4" or (2), 1-1/2"	69 88	143	150	(3), 50'	20' of 6"
2	875	(1), 2" or (2), 1-3/8"	55 61	194	200	or	
3	625	(1), 1-1/2" 2-1/4"	88 69	245	250	(3), 100'	
4	1250	or (2), 1-1/2"	88	158	165	and (1), 50'	
						Siamesed	
1500 GPM Fire Pump							
1	1500	(2), 1-3/4" or (3), 1-1/2"	68 57	142	150	(3), 50'	20' of
2	1050	(1), 2" or (2), 1-1/2"	78 62	194	200	or	6" Min
3	750	(1), 1-3/4" or (2), 1-1/4"	68 66	245	250	(3), 100' and (1), 50'	or (2) 20' of
4	1500	(2), 1-3/4" or (3), 1-1/2"	68 57	157	165	Siamesed	6" Max

Min. discharge pressures listed above are for pumps operating with full 10' static suction lift. These pressures must be increased by 1 PSI for each 2.3 ft. less than 10' of lift.

TABLE NO. 1
NFPA 1901 TEST

Class A							
TEST No.	GPM	Recom- mended Nozzles	Min. Nozzle Press. PSI	Min. Disch. Press. PSI	Min. Net Pump Press. PSI	Disch. Lines	Suction Hose
1750 GPM Fire Pump							
1	1750	(2), 2" or (3), 1-1/2" (2), 1-5/8" or	55 76 61	143	150	(4), 50'	(2) 20' of 6"
2	1225	(2), 1-1/2" or (3), 1-1/4"	84 79	194	200	or	
3	875	(1), 2" or (2), 1-3/8"	55 61	245	250	(4), 100'	
4	1750	(2), 2" or (3), 1-1/2"	55 76	158	165		
2000 GPM Fire Pump							
1	2000	(2), 2" or (4), 1-1/2" (2), 1-3/4"	71 57 60	147	150	(4), 50'	(2) 20' of 6"
2	1400	or (3), 1-1/2" (1), 2"	49 71	199	200	or	
3	1000	or (2), 1-1/2" (2), 2"	57 71	249	250	(4), 100'	
4	2000	or (4), 1-1/2"	57	163	165		
2250 GPM Fire Pump							
1	2250	(2), 2-1/4"	56	144	150	(2 Groups) (3), 100'	20' of 8"
2	1575	(2), 1-3/4"	76	196	200	Siamesed	
3	1125	(2), 1-1/2"	72	246	250		
4	2250	(2), 2-1/4"	56	153	165		

Min. discharge pressures listed above are for pumps operating with full 10' static suction lift. These pressures must be increased by 1 PSI for each 2.3 ft. less than 10' of lift.

TABLE NO. 1
NFPA 1901 TEST

Class A							
TEST No.	GPM	Recom- mended Nozzles	Min. Nozzle Press. PSI	Min. Disch. Press. PSI	Min. Net Pump Press. PSI	Disch. Lines	Suction Hose
2500 GPM Fire Pump							
1	2500	(2), 2-1/4"	69	144	150	(2 Groups) (3), 100'	20' of 8"
2	1750	(2), 2"	55	195	200	Siamesed	
3	1250	(2), 1-1/2"	88	246	250		
4	2500	(2), 2-1/4"	69	159	165		
3000 GPM Fire Pump							
1	3000	(2), 2-1/2"	65	146	150	(2 Groups) (3), 100'	(2) 20' of 8"
2	2100	(2), 2"	78	196	200	Siamesed	
3	1500	(2), 1-3/4"	68	247	250		
4	3000	(2), 2-1/2"	65	161	165		
3000 GPM Industrial Fire Pump							
1	3000	(2), 2-1/2"	65	96	100	(2 Groups) (3), 100'	(2) 20' of 8"
2	2100	(2), 2"	78	146	150	Siamesed	
3	1500	(2), 1-3/4"	68	197	200		
3500 GPM Industrial Fire Pump							
1	3500	(2), 2-1/2" and (1), 2-1/4"	45 44	95	100	(2 Groups) (3), 100' Siamesed	(2) 20' of 8"
2	2450	(2), 2-1/4"	67	146	150	&	
3	1750	(2), 2"	55	197	200	(2)-50' Siamesed	

Min. discharge pressures listed above are for pumps operating with full 10' static suction lift. These pressures must be increased by 1 PSI for each 2.3 ft. less than 10' of lift.

**IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT
CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650**

TABLE NO. 2
DISCHARGE FROM SMOOTH BORE NOZZLE
Pressures measured by Pitot gage.

Nozzle Pressure PSI	1/4	3/8	1/2	5/8	3/4	7/8	1	1 1/8	1 1/4	1 3/8	1 1/2	1 5/8	1 3/4	2	2 1/4	2 1/2
GALLONS PER MINUTE DELIVERED																
5	4	9	16	26	37	50	66	84	103	125	149	175	203	266	337	415
6	4	10	18	28	41	55	72	92	113	137	163	192	223	292	369	455
7	4	11	19	30	44	59	78	99	122	148	176	207	241	315	399	491
8	5	11	21	32	47	64	84	106	131	158	188	222	257	336	427	525
9	5	12	22	34	50	67	89	112	139	168	200	235	273	357	452	557
10	6	13	23	36	53	71	93	118	146	177	211	248	288	376	477	587
12	6	15	25	40	58	78	102	130	160	194	231	271	315	412	522	643
14	7	15	27	43	63	84	110	140	173	210	249	293	340	445	564	695
16	7	16	29	46	67	90	118	150	185	224	267	313	364	475	603	743
18	7	17	31	49	71	95	125	159	196	237	283	332	386	504	640	788
20	8	18	33	51	75	101	132	167	206	250	298	350	407	532	674	830
22	8	19	34	54	79	105	139	175	216	263	313	367	427	557	707	871
24	8	20	36	56	82	110	145	183	226	275	327	384	446	582	739	909
26	9	21	37	59	85	115	151	191	235	286	340	400	464	606	769	947
28	9	21	39	61	89	119	157	198	244	297	353	415	481	629	799	982
30	10	22	40	63	92	123	162	205	253	307	365	429	498	651	826	1017
32	10	23	41	65	95	127	167	212	261	317	377	443	514	673	854	1050
34	11	23	43	67	98	131	172	218	269	327	389	457	530	693	880	1082
36	11	24	44	69	100	135	177	224	277	336	400	470	546	713	905	1114
38	11	25	45	71	103	138	182	231	285	345	411	483	561	733	930	1144
40	11	26	46	73	106	142	187	237	292	354	422	496	575	752	954	1174
42	11	26	47	74	109	146	192	243	299	363	432	508	589	770	978	1203
44	12	27	49	76	111	149	196	248	306	372	442	520	603	788	1000	1231
46	12	28	50	78	114	152	200	254	313	380	452	531	617	806	1021	1259
48	12	28	51	80	116	156	205	259	320	388	462	543	630	824	1043	1286
50	13	29	52	81	118	159	209	265	326	396	472	554	643	841	1065	1313
52	13	29	53	83	121	162	213	270	333	404	481	565	656	857	1087	1339
54	13	30	54	84	123	165	217	275	339	412	490	576	668	873	1108	1364
56	13	30	56	86	125	168	221	280	345	419	499	586	680	889	1129	1389
58	13	31	56	87	128	171	225	285	351	426	508	596	692	905	1149	1414
60	14	31	57	89	130	174	229	290	357	434	517	607	704	920	1168	1437
62	14	32	58	90	132	177	233	295	363	441	525	617	716	936	1187	1462
64	14	32	59	92	134	180	237	299	369	448	533	627	727	951	1206	1485
66	14	33	60	93	136	182	240	304	375	455	542	636	738	965	1224	1508
68	14	33	60	95	138	185	244	308	381	462	550	646	750	980	1242	1531
70	15	34	61	96	140	188	247	313	386	469	558	655	761	994	1260	1553
72	15	34	62	97	142	191	251	318	391	475	566	665	771	1008	1278	1575
74	15	35	63	99	144	193	254	322	397	482	574	674	782	1023	1296	1597
76	15	35	64	100	146	196	258	326	402	488	582	683	792	1036	1313	1618
78	15	36	65	101	148	198	261	330	407	494	589	692	803	1050	1330	1639

TABLE NO. 2
DISCHARGE FROM SMOOTH BORE NOZZLE
Pressures measured by Pitot gage.

Nozzle Pressure PSI	GALLONS PER MINUTE DELIVERED															
	1/4	3/8	1/2	5/8	3/4	7/8	1	1 1/8	1 1/4	1 3/8	1 1/2	1 5/8	1 3/4	2	2 1/4	2 1/2
80	16	36	66	103	150	201	264	335	413	500	596	700	813	1063	1347	1660
82	16	37	66	104	152	204	268	339	418	507	604	709	823	1076	1364	1681
84	16	37	67	105	154	206	271	343	423	513	611	718	833	1089	1380	1701
86	16	37	68	107	155	208	274	347	428	519	618	726	843	1102	1396	1721
88	16	38	69	108	157	211	277	351	433	525	626	735	853	1115	1412	1741
90	17	39	70	109	159	213	280	355	438	531	633	743	862	1128	1429	1761
92	17	39	70	110	161	215	283	359	443	537	640	751	872	1140	1445	1780
94	17	39	71	111	162	218	286	363	447	543	647	759	881	1152	1460	1800
96	17	40	72	113	164	220	289	367	452	549	654	767	890	1164	1476	1819
98	17	40	73	114	166	223	292	370	456	554	660	775	900	1176	1491	1838
100	18	41	73	115	168	225	295	374	461	560	667	783	909	1189	1506	1856
105	18	42	75	118	172	230	303	383	473	574	683	803	932	1218	1542	1902
110	19	43	77	121	176	236	310	392	484	588	699	822	954	1247	1579	1947
115	19	43	79	123	180	241	317	401	495	600	715	840	975	1275	1615	1991
120	19	44	80	126	183	246	324	410	505	613	730	858	996	1303	1649	2033
125	20	45	82	129	187	251	331	418	516	626	745	876	1016	1329	1683	2075
130	20	46	84	131	191	256	337	427	526	638	760	893	1036	1356	1717	2116
135	21	47	85	134	195	262	343	435	536	650	775	910	1056	1382	1750	2157
140	21	48	87	136	198	266	350	443	546	662	789	927	1076	1407	1780	2196
145	21	49	88	139	202	271	356	450	556	674	803	944	1095	1432	1812	2235
150	22	50	90	141	205	275	362	458	565	686	817	960	1114	1456	1843	2273

TABLE NO. 3
Approximate Discharge Flow From Different Nozzles
At the end of Fifty Feet of Average, 2 1/2"
Rubber Lined Fire Hose, for Various
Pump Pressures with Discharge
Valve Wide Open

PUMP PRESSURE LBS	SIZE OF		NOZZLE &		GALLONS PER		MINUTE
	3/4	7/8	1	1 1/8	1 1/4	1 3/8	1 1/2
30	90	119	153	187	217	250	282
40	103	137	177	216	253	290	327
50	115	153	198	242	284	325	367
60	126	168	216	265	311	357	402
70	136	182	234	287	337	385	435
80	145	194	250	308	361	414	465
90	154	206	265	325	383	437	492
100	162	217	280	343	405	462	520
110	171	228	295	360	425	485	549
120	179	239	307	377	444	510	572
130	186	249	318	392	462	530	596
140	193	258	330	407	480	549	618
150	200	267	341	421	497	567	
175	215	288	374	455	538		
200	230	309	395	486			
225	243	328	420				
250	257	345					

This table is offered as an aide in testing pump performance where facilities for accurate measurement of capacity are not available. The capacities given above are conservative, and will not vary more than 5% from actual capacities with any of the standard hose that might be used.

TABLE NO. 4

**Pump or Hydrant Pressure required to give
Effective Nozzle Pressure through various
Lengths of Rubber Lined Hose.**

Size of Hose		1	1 1/2			2		2 1/2					3	
Size of Nozzle		1/4	3/8	1/2	5/8	5/8	3/4	3/4	7/8	1	1 1/4	1 1/2	1 1/4	1 1/2
Nozzle Press PSI	Length of Hose Feet	PUMP OR HYDRANT PRESSURE - PSI												
40	100	45	43	48	60	42	50	44	46	51	64	88	51	62
	200	49	46	56	79	43	60	47	52	60	86	130	59	78
	400	58	51	73	118	46	79	53	62	79	129	212	75	110
	600	67	57	89	158	50	99	59	74	97	172		92	143
	800	76	62	106	196	53	119	65	85	116	215		108	176
	1000	85	68	122	235	56	138	72	96	134	258		124	208
	1500	108	72	142		64	187	87	118	181			165	
	2000	130	96	204		72	226	103	151	227			205	
60	100	67	64	72	89	63	73	65	69	75	95	132	76	92
	200	74	68	84	117	65	86	70	78	89	126	196	88	115
	400	87	76	107	173	69	112	79	94	116	188		111	161
	600	101	85	131	231	74	138	88	111	143	250		135	208
	800	114	93	153		79	164	98	127	170			158	
	1000	127	101	178		83	190	107	143	197			182	
	1500	161	122	237		95	155	130	184	264				
	2000	195	142			106		153	225					
80	100	88	85	96	117	83	99	87	92	99	126	175	101	103
	200	97	91	112	154	86	117	93	103	115	167		116	154
	400	115	102	143	228	92	154	105	125	148	249		147	
	600	132	112	174		98	191	117	147	181			178	
	800	150	123	206		104	228	129	167	214			209	
	1000	167	134	238		110		141	191	247				
	1500	211	161			125		171	245					
	2000	254	188			140		201						
100	100	111	107	120	146	104	123	108	115	125	157		126	152
	200	122	113	139	192	108	145	116	128	150	209		146	190
	400	143	127	177	284	115	190	130	154	200			184	
	600	165	140	217		123	235	145	180	250			223	
	800	186	154	256		131		159	206					
	1000	208	167			138		174	232					
	1500	262	200			157		211						
	2000		234			175		253						

TABLE NO. 5
REACH OF FIRE STREAMS

Size of Nozzle	1/4"	3/8"	1/2"	5/8"	3/4"	7/8"	1"	1-1/4"	1-1/2"
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NOZZLE

PRESSURE EFFECTIVE VERTICAL REACH - Feet

40	30	35	40	50	59	62	64	65	69
60	35	40	45	60	74	77	79	84	87
80	38	42	48	65	81	85	89	94	96
100	40	44	50	68	84	89	94	100	102

NOZZLE

PRESSURE MAXIMUM VERTICAL REACH - Feet

40	60	65	70	75	78	79	80	80	80
60	70	75	85	95	105	106	108	110	110
80	78	83	95	105	117	125	132	140	140
100	80	88	100	110	122	135	145	155	155

NOZZLE

PRESSURE EFFECTIVE HORIZONTAL REACH - Feet

40	20	25	30	40	44	50	55	62	66
60	25	32	37	50	54	61	67	75	80
80	28	35	40	57	62	70	76	84	88
100	30	37	42	60	66	76	84	93	95

NOZZLE

PRESSURE MAXIMUM HORIZONTAL REACH - Feet

40	65	80	90	100	108	120	125	138	140
60	80	95	95	120	127	142	156	176	183
80	90	105	105	135	143	160	175	201	210
100	95	110	110	140	153	180	205	215	223

TABLE NO. 6
Friction Loss in Fire Hose
 Loss in PSI per 100 Feet of Hose

SIZE HOSE	LINEN HOSE			BEST RUBER LINED HOSE							
G.P.M.	1 1/2	2	2 1/2	3/4	1	1 1/2	2	2 1/2	3	3 1/2	(2)-2 1/2
10	1.0			13.5	3.5	0.5	.1				
15	2.2			29.0	7.2	1.0	0.3				
20	3.6			50.0	12.3	1.7	0.4				
25	5.5			75.0	18.5	2.6	0.6				
30	8.0	1.9		105.0	26.0	3.6	0.9				
40	13.0	3.2		180.0	44.0	6.1	1.5				
50	20.0	4.9	1.6		67.0	9.3	2.3				
60	28.0	7.0	2.2		96.0	13.5	3.3				
70	37.0	9.0	3.1		131.0	17.0	4.3				
80	47.0	11.5	3.8		171.0	23.0	5.6				
90	59.0	14.5	5.0		217.0	29.0	7.0				
100	72.0	17.5	5.9		268.0	33.0	8.4				
120		25.0	8.3		386.0	47.0	11.7				
140		34.0	11.0			62.0	16.0	5.2	2.0	0.9	1.4
160		43.0	14.0			78.0	20.0	6.6	2.6	1.2	1.9
180		53.0	17.7			97.0	25.0	8.3	3.2	1.5	2.3
200		63.0	21.5			121.0	30.6	10.1	3.9	1.8	2.8
220						146.0		12.0	4.6	2.1	3.3
240						173.0		14.1	5.4	2.5	3.9
260						204.0		16.4	6.3	2.9	4.5
280						237.0		18.7	7.2	3.3	5.2
300						272.0		21.2	8.2	3.7	5.9
320								23.8	9.3	4.2	6.6
340								26.9	10.5	4.7	7.4
360								30.0	11.5	5.2	8.3
380								33.0	12.8	5.8	9.2
400								36.2	14.1	6.3	10.1
425								40.8	15.7	7.0	11.3
450								45.2	17.5	7.9	12.5
475								50.0	19.3	8.7	13.8
500								55.0	21.2	9.5	15.2
525									23.2	10.5	16.6
550									25.2	11.4	18.1
575									27.5	12.4	19.6
600									29.9	13.4	21.2
650									34.5	15.5	24.8
700									39.5	17.7	28.3
750									45.0	20.1	32.2
800									50.5	22.7	36.2
850									56.5	25.4	40.7
900									63.0	28.2	45.2
1000									76.5	34.3	55.0

Losses in rough walled, rubber hose may be 50% higher than values given above.

TABLE NO. 7
Friction Loss in 15-year-old Steel Pipe
 Loss in PSI per 100 Feet of Pipe

PIPE SIZE G.P.M.	1/8	1/4	3/8	1/2	3/4	1	1 1/4	1 1/2	2	2 1/2	3	4	6	8
1	52.0	12.0	2.8	0.9										
2		45.0	10.0	3.2	4.0									
5			55.0	18.0	4.5	1.4	0.4							
10				64.0	16.0	5.0	1.3	0.6						
15				135.0	34.0	11.0	2.7	1.3	0.5					
20					59.0	18.0	4.7	2.2	0.8					
25					89.0	27.0	7.1	3.4	1.2					
30					125.0	39.0	10.0	4.7	1.7	0.6				
35						51.0	13.0	6.3	2.2	0.7				
40						66.0	17.0	8.0	2.9	0.9				
45						82.0	21.0	10.0	3.6	1.2				
50						99.0	26.0	12.0	4.3	1.4	0.6			
60						140.0	38.0	17.0	6.1	2.0	0.8			
70							49.0	23.0	8.0	2.7	1.1			
80							63.0	29.0	10.0	3.4	1.5			
90							78.0	36.0	13.0	4.3	1.8			
100							96.0	44.0	15.0	5.1	2.2	0.5		
125							144.0	66.0	24.0	7.8	3.3	0.8		
150								93.0	33.0	11.0	4.6	1.1		
175								125.0	44.0	15.0	6.1	1.5		
200									56.0	19.0	7.8	1.9		
250									84.0	28.0	12.0	2.9		
300									114.0	40.0	16.0	4.0	0.6	
350										53.0	22.0	5.4	0.8	
400										68.0	28.0	6.9	1.0	
450										84.0	35.0	8.6	1.2	
500										102.0	42.0	10.0	1.4	0.4
600											60.0	15.0	2.1	0.6
800												25.0	3.5	1
1000												37.0	5.2	1.3
1500													11.0	2.7
2000													19.0	4.7
2500													29.0	7.1
3000														10

TABLE NO. 8
Resistance of Fittings
Equivalent Lengths of Straight Pipe - Feet

PIPE SIZE	1/2	3/4	1	1 1/4	1 1/2	2	2 1/2	3	4	5	6	8
Gate Valve	0.4	0.6	0.8	1.1	1.4	1.8	2.2	2.8	4.1	5.3	6.7	9.4
Global Valve	3.0	4.5	6.0	8.5	10.5	14.0	17.0	22.0	32.0	42.0	53.0	75.0
Angle Valve	1.4	2.0	2.7	3.8	4.8	6.3	7.9	10.5	14.5	18.5	23.0	33.0
Std. Elbow	1.1	1.5	2.0	2.8	3.5	4.7	5.8	7.5	11.0	14.0	18.0	24.0
45 Elbow	0.6	0.8	1.0	1.4	1.6	2.1	2.5	3.1	4.2	5.2	6.3	8.5
Long Sweep EI Str Run Tee	0.5	0.8	1.0	1.4	1.7	2.3	2.8	3.7	5.3	7.0	9.0	12.5
Std. Tee Thru Side Outlet	2.1	2.9	3.9	5.5	6.9	9.1	11.6	14.8	21.0	27.0	34.0	49.0
Sudden Enlargement or contraction	1.8	2.5	3.2	4.2	5.0	6.5	7.5	9.5	13.0	16.0	19.0	25.0
Entrance to Pipe	1.0	1.3	1.6	2.2	2.6	3.3	3.9	4.9	6.5	8.2	10.0	13.0

TABLE NO. 9
To Convert Pounds per Square Inch to
Feet Elevation of Water

2.308ft head = 1.0 psi
1ft head = .433psi

Feet	5	10	15	20	25	30	35	40	45	50	60	70	80	90
Pounds	2.2	4.3	6.5	8.7	11	13	15	17	20	22	26	30	35	39
Feet	100	120	130	140	150	160	170	180	190	200	220	240	260	280
Pounds	43	52	56	61	65	69	74	78	82	87	95	104	113	121
Feet	300	320	340	360	380	400	425	450	475	500	525	550	600	700
Pounds	130	139	147	156	165	173	184	195	206	217	227	238	260	303

Table NO. 10
American National Fire Hose Connection Screw Thread - NH

Size of Hose	4-Mar	1	1 1/2	2 1/2	3	3 1/2	4	4 1/2	5	6	8
Thr's per inch	8	8	9	7.5	6	6	4	4	4	4	4
Thread Designation	0.75-8 NH	1-8 NH	1.5-9 NH	2.5-7.5 NH	3-6 NH	3.5-6 NH	4-4 NH	4.5-4 NH	5-4 NH	6-4 NH	8-4 NH
Max. O.D. Male	1.375	1.375	1.99	3.0686	3.6239	4.2439	5.0109	5.7609	6.26	7.025	9.05

Ref. NFPA 1963

Underwriters Nozzle Tip Thread: 2.1875 O.D. - 12 threads per inch.

IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT
CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650

W.S. DARLEY & CO.

OPERATING INSTRUCTIONS - ELECTRIC PRIMING PUMP

The Darley electric primer will develop up to 25 in. Hg. in an air tight pumping system.

The Primer is activated by a combination spring return on-off valve and electric switch. Pulling the valve out opens the valve and closes the electrical circuit to start the motor.

Before the pump can be primed, booster line valves, drain valves, cooling line valve, and all other openings into the pump must be closed and absolutely air tight. The discharge side of the pump is sealed by a check valve, therefore the main discharge valves need not be closed.

When operating from draft, suction hose connections must be tight and free of air leaks.

Make certain the suction hose strainer is properly submerged and free of foreign material.

The main pump drive should remain disengaged until priming is complete to prevent possible damage to impeller seal rings by running "dry".

Pull the primer shutoff valve all the way out to start priming and hold open until water discharges from primer pump exhaust port. Push valve all the way in to shut off primer motor and seal tight.



CAUTION: FOR PRIMING UP TO 10' OF LIFT:

If water does not discharge from the primer exhaust within about 30 seconds (45 seconds with 2 – 20' lengths of hose) stop the primer pump, check for air leaks and make sure primer pump is receiving lubricant from its reservoir, if one is present. **MAX PRIMER OPERATION TIME = 90 seconds every 5 minutes. DO NOT EXCEED 90 SECONDS OF PRIMER OPERATION.**



CAUTION: FOR PRIMING 10' OF LIFT AND HIGHER:

If water does not discharge from the primer exhaust within 90 seconds stop the primer pump, check for air leaks and make sure primer pump is receiving lubricant from its reservoir, if one is present. **DO NOT EXCEED 90 SECONDS OF PRIMER OPERATION.**



CAUTION: The primer pump and motor will begin to generate heat as soon as operation begins. Extended run times (up to 90 seconds) and repeating priming cycles consecutively or within short time periods may lead to premature failure of the primer pump assembly: such failures include but are not limited to: overheating of the motor, seizure of the rotor, and cracking of primer vanes. To avoid this, after your first priming attempt, thoroughly inspect the pump system for air leaks, check that the primer is

**receiving lubricant from its reservoir if such is present,
and resolve the issue before attempting re-prime.**

Engage "Pump" shift to start pumping water.

When pumping from hydrants, the primer is not needed and must be kept closed.

It may be necessary to use the primer momentarily when pumping from a booster tank when the suction head is insufficient to force all the air out of the pump.

LUBRICATING SYSTEM - ELECTRIC PRIMING PUMPS WITH FLUID RESERVOIR

The electric motor rotary van primer pump creates a high vacuum by continuous lubrication of rotor and vanes. Therefore the primer lubricant supply tanks (4 quarts) should be kept full at all times. Recommended primer system lubricant is Darley PRIME GREEN. PRIME GREEN is an environmentally safe, non-toxic, biodegradable lubricant. Its use assures proper primer vane lubricant while minimizing environmental effects.

After the main pump is drained, run the primer motor to drain primer lines and re-lubricate the primer pump.

The vent hole on the lubricant tank cap should be kept open at all times to prevent siphoning lubricant from the tank after the pump is stopped. Do not increase the size of the hole.

Locate the lubricant tank where it may be conveniently inspected and filled.

Should water appear in the lubricant supply tank, the primer valve is leaking. Check and replace valve plug seal o-ring if necessary.

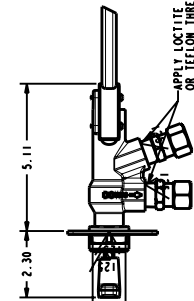
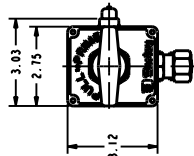
ELECTRIC PRIMING PUMPS WITHOUT FLUID RESERVOIR

The fluidless electric-motor rotary-vane primer pump creates a high vacuum by using a special material for the vanes and an initial factory applied lubricant film. This film must be present in order for the primer to operate properly and to provide maximum life for the primer components.

This film should not wash away completely if the pump is used to pump clean water. If the priming pump is disassembled for any reason, all internal surfaces of the housing and end caps must be cleaned and coated completely with Dow Corning #111 Silicone valve lubricant prior to operating the primer. If a degradation of performance is noticed, performance may be restored by re-applying the film in this manner. It is recommended to service the primer annually to clean and re-apply the silicone film to the inside of the primer housing and end caps. Do not apply grease to the rotor slots, or the sides of the vanes.

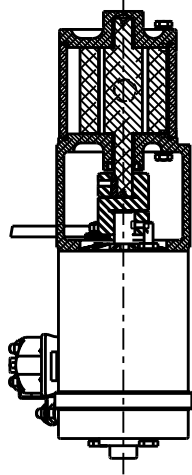
After the main pump is drained, run the primer motor to drain primer lines.

NO.	DESCRIPTION	PART NO.	QTY.
1	ASSEMBLY - PRIMER, MOTOR	425105	1
2	BEARING - OILITE, 0.625 ID	176023	2
3	COUPLING - 844093	240450	1
4	CYLINDER - VACUUM PUMP	102800	1
5	HEAD - CYLINDER, INBOARD	200904	1
6	HEAD - CYLINDER, OUTBOARD	200800	1
7	PISTON - 313-18 X 1.00, SST	540810	4
8	KEY - 50, 0.19 X 0.42 GR2	340225	1
9	KEY - WOODRUFF, 606	340220	1
10	MOTOR - PRIMER, 24V	422500	1
11	MUT - MET., 313-15, SST	540302	2
12	O-RING - 3.50 X 3.69 X 0.09	360101	2
13	OIL SEAL - 0.625 ID X 1.128 OD	360055	1
14	TUBE FITTING - 5/16, 25 X .19	350021	1
15	STRAP - GROUND, 12V PRIMER	260516	1
16	WASHER - FLAT, 5/16, STEEL	440160	4
17	WASHER - LOCK, 0.313 ID	340302	2
18	WASHER - LOCK, 0.313 ID	340302	2
19	WASHER - LOCK, 0.313 ID	340302	2



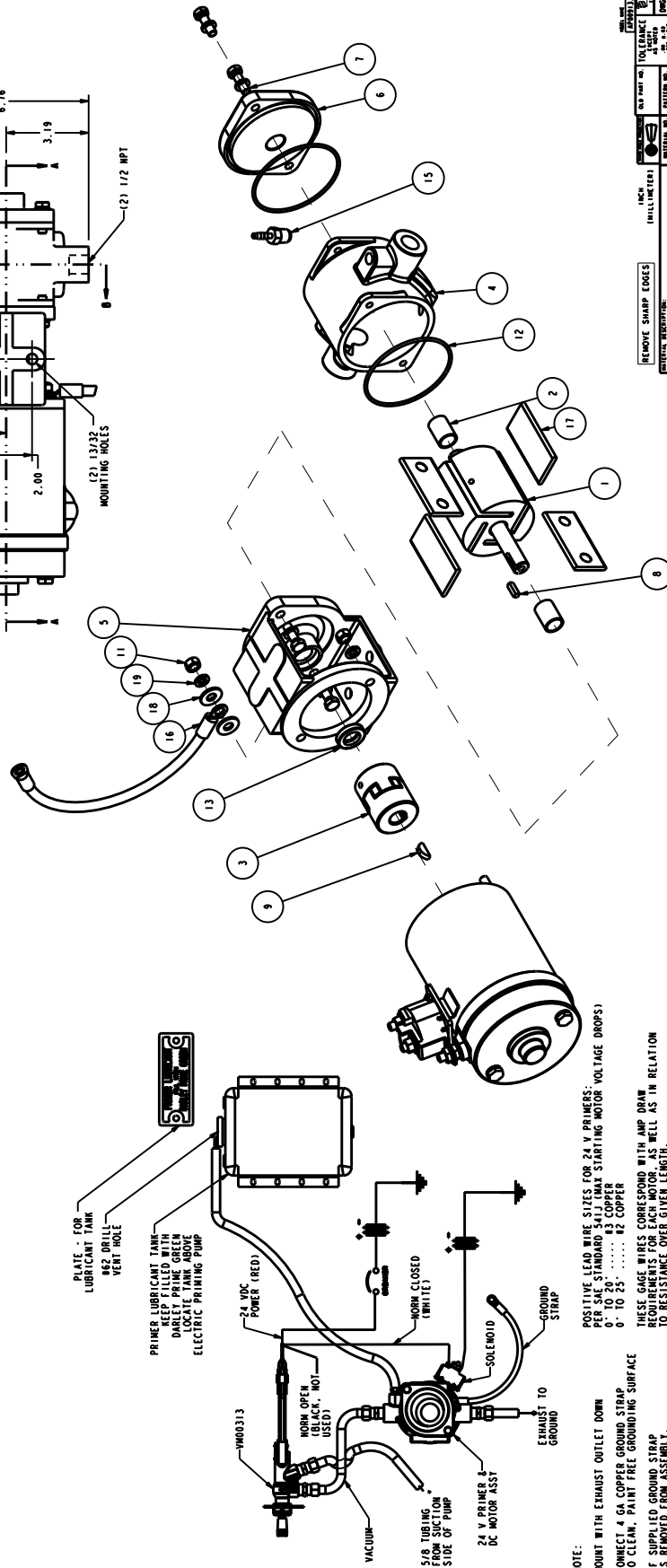
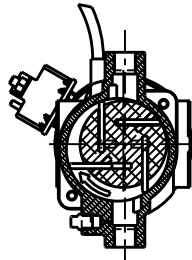
APPLY LOCTITE PST 565
OR TEFLON THREAD TAP
TO THE MOTOR AND 1/2 TURN
DO NOT OVER TIGHTEN

NOTE:
- FOR CROSS SECTION AND EXPLODED VIEW DRAWING
- SEE DRAWING DVC0203
- ALSO, WHEN REPLACING THE OLD VERSION OF THIS VALVE WITH
THE NEW VERSION, THE EXISTING 1762500 CAN BE USED TO RETROFIT
IT TO THE EXISTING PANEL.



SECTION A-A
SCALE 1/2

SECTION B-B



NOTE:
MOUNT WITH EXHAUST OUTLET DOWN
CONNECT 4 GA COPPER GROUND STRAP
TO CLEAN, PAINT FREE GROUNDING SURFACE
IF SUPPLIED GROUND STRAP
REQUIREMENTS FOR EACH MOTOR, AS WELL AS IN RELATION
TO 90 IN. LB.
THESE GAGE WIRES CORRESPOND WITH AMP DRAW
REQUIREMENTS FOR EACH MOTOR, AS WELL AS IN RELATION
TO 90 IN. LB.
24 V = 140 AMP

POSITIVE LEAD WIRE SIZES FOR 24 V PRIMERS:
PER SAE STANDARD 541J (MAX STARTING MOTOR VOLTAGE DROPS)
0 TO 20 #3 COPPER
0 TO 25 #2 COPPER

WEIGHT : 27 LB (12.1 KG)

REV.	DESCRIPTION	DATE	BY	CHKD.	APP'D.
1	INITIAL RELEASE	12 SEP 2008	111	111	111
2	REWORK FOR 1/2	12 SEP 2008	111	111	111
3	REWORK FOR 1/2	12 SEP 2008	111	111	111
4	REWORK FOR 1/2	12 SEP 2008	111	111	111
5	REWORK FOR 1/2	12 SEP 2008	111	111	111
6	REWORK FOR 1/2	12 SEP 2008	111	111	111
7	REWORK FOR 1/2	12 SEP 2008	111	111	111
8	REWORK FOR 1/2	12 SEP 2008	111	111	111
9	REWORK FOR 1/2	12 SEP 2008	111	111	111
10	REWORK FOR 1/2	12 SEP 2008	111	111	111
11	REWORK FOR 1/2	12 SEP 2008	111	111	111
12	REWORK FOR 1/2	12 SEP 2008	111	111	111
13	REWORK FOR 1/2	12 SEP 2008	111	111	111
14	REWORK FOR 1/2	12 SEP 2008	111	111	111
15	REWORK FOR 1/2	12 SEP 2008	111	111	111
16	REWORK FOR 1/2	12 SEP 2008	111	111	111
17	REWORK FOR 1/2	12 SEP 2008	111	111	111
18	REWORK FOR 1/2	12 SEP 2008	111	111	111
19	REWORK FOR 1/2	12 SEP 2008	111	111	111

REV.	DESCRIPTION	DATE	BY	CHKD.	APP'D.
1	INITIAL RELEASE	12 SEP 2008	111	111	111
2	REWORK FOR 1/2	12 SEP 2008	111	111	111
3	REWORK FOR 1/2	12 SEP 2008	111	111	111
4	REWORK FOR 1/2	12 SEP 2008	111	111	111
5	REWORK FOR 1/2	12 SEP 2008	111	111	111
6	REWORK FOR 1/2	12 SEP 2008	111	111	111
7	REWORK FOR 1/2	12 SEP 2008	111	111	111
8	REWORK FOR 1/2	12 SEP 2008	111	111	111
9	REWORK FOR 1/2	12 SEP 2008	111	111	111
10	REWORK FOR 1/2	12 SEP 2008	111	111	111
11	REWORK FOR 1/2	12 SEP 2008	111	111	111
12	REWORK FOR 1/2	12 SEP 2008	111	111	111
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14	REWORK FOR 1/2	12 SEP 2008	111	111	111
15	REWORK FOR 1/2	12 SEP 2008	111	111	111
16	REWORK FOR 1/2	12 SEP 2008	111	111	111
17	REWORK FOR 1/2	12 SEP 2008	111	111	111
18	REWORK FOR 1/2	12 SEP 2008	111	111	111
19	REWORK FOR 1/2	12 SEP 2008	111	111	111

W.S. DARLEY & CO.

OPERATING INSTRUCTIONS - ELECTRIC PRIMING PUMP

PUSH BUTTON ELECTRIC ACTUATED VALVE

The Darley electric primer will develop up to 25 in. Hg. in an air tight pumping system.

The Primer is activated by a push button, 0.8 second cycle time, electric-actuated valve. This valve has three wires: ground (black), +12 or +24 VDC constant power (red), and +12 or +24 VDC energizing power (white). The valve has two internal micro-switches that cut the power to the valve when it is either fully closed or fully open. The push button is a simple SPST switch, that bypasses the red (constant power) wire with the white wire and energizes the valve to the open position when it is pushed; and cuts power to the white wire and resumes power to the red wire, closing the valve when it is released. When the button is pushed, the circuit for the primer motor is also completed, priming will begin instantaneously, and likewise when the button is released, the primer motor will shut off instantaneously.

Before the pump can be primed, booster line valves, drain valves, cooling line valve, and all other openings into the pump must be closed and absolutely air tight. The discharge side of the pump is sealed by a check valve; therefore the main discharge valves need not be closed.

When operating from draft, suction hose connections must be tight and free of air leaks.

Make certain the suction hose strainer is properly submerged and free of foreign material.

The main pump drive should remain disengaged until priming is complete to prevent possible damage to impeller seal rings by running "dry".

Push the primer/valve activation button, located on your control panel, and hold until water discharges from primer pump exhaust port.

If water does not discharge from primer exhaust within about 30 seconds (45 seconds with 2-20' lengths) stop primer pump, check for air leaks and make sure primer pump is receiving lubricant from its reservoir.

NOTE: Do not run the primer for more than one minute; it will burn up the motor, if prime is not reached within one minute, repeat the steps above.

Engage "Pump" shift to start pumping water.

When pumping from hydrants, the primer is not needed and must be kept closed.

It may be necessary to use the primer momentarily when pumping from a booster tank when the suction head is insufficient to force all the air out of the pump.

LUBRICATING SYSTEM - ELECTRIC PRIMING PUMPS WITH FLUID RESERVOIR

The electric motor rotary van primer pump creates a high vacuum by continuous lubrication of rotor and vanes. Therefore the primer lubricant supply tanks (4 quarts) should be kept full at all times. Recommended primer system lubricant is Darley PRIME GREEN. PRIME GREEN is an environmentally safe, non-toxic, biodegradable lubricant. Its use assures proper primer vane lubricant while minimizing environmental effects.

After the main pump is drained, run the primer motor to drain primer lines and re-lubricate the primer pump.

The vent hole on the lubricant tank cap should be kept open at all times to prevent siphoning lubricant from the tank after the pump is stopped. Do not increase the size of the hole.

Locate the lubricant tank where it may be conveniently inspected and filled.

Should water appear in the lubricant supply tank, the primer valve is leaking. Check and replace valve plug seal o-ring if necessary.

ELECTRIC PRIMING PUMPS WITHOUT FLUID RESERVOIR

The fluidless electric-motor rotary-vane primer pump creates a high vacuum by using a special material for the vanes and an initial factory applied lubricant film. This film must be present in order for the primer to operate properly and to provide maximum life for the primer components.

This film will not wash away completely if the pump is used to pump water. If the priming pump is disassembled for any reason, all internal surfaces of the housing and end caps must be coated completely with Dow Corning #111 Silicone valve lubricant prior to operating the primer. If after several years, a degradation of performance is noticed, performance may be restored by re-applying the film in this manner.

After the main pump is drained, run the primer motor to drain primer lines.

REVISIONS			
LTR	DESCRIPTION	DATE	CHG NO. APPR'D

SUPPLIED BY KZCO

White: Switched 12 VDC (+) = Relay Energized "OPEN"
 0 VDC = Relay De-energized "CLOSE"

Black: Constant 12 VDC (-)

Red: Constant 12 VDC (+)

CUSTOMER SUPPLIED

2-POSITION SPST Switch

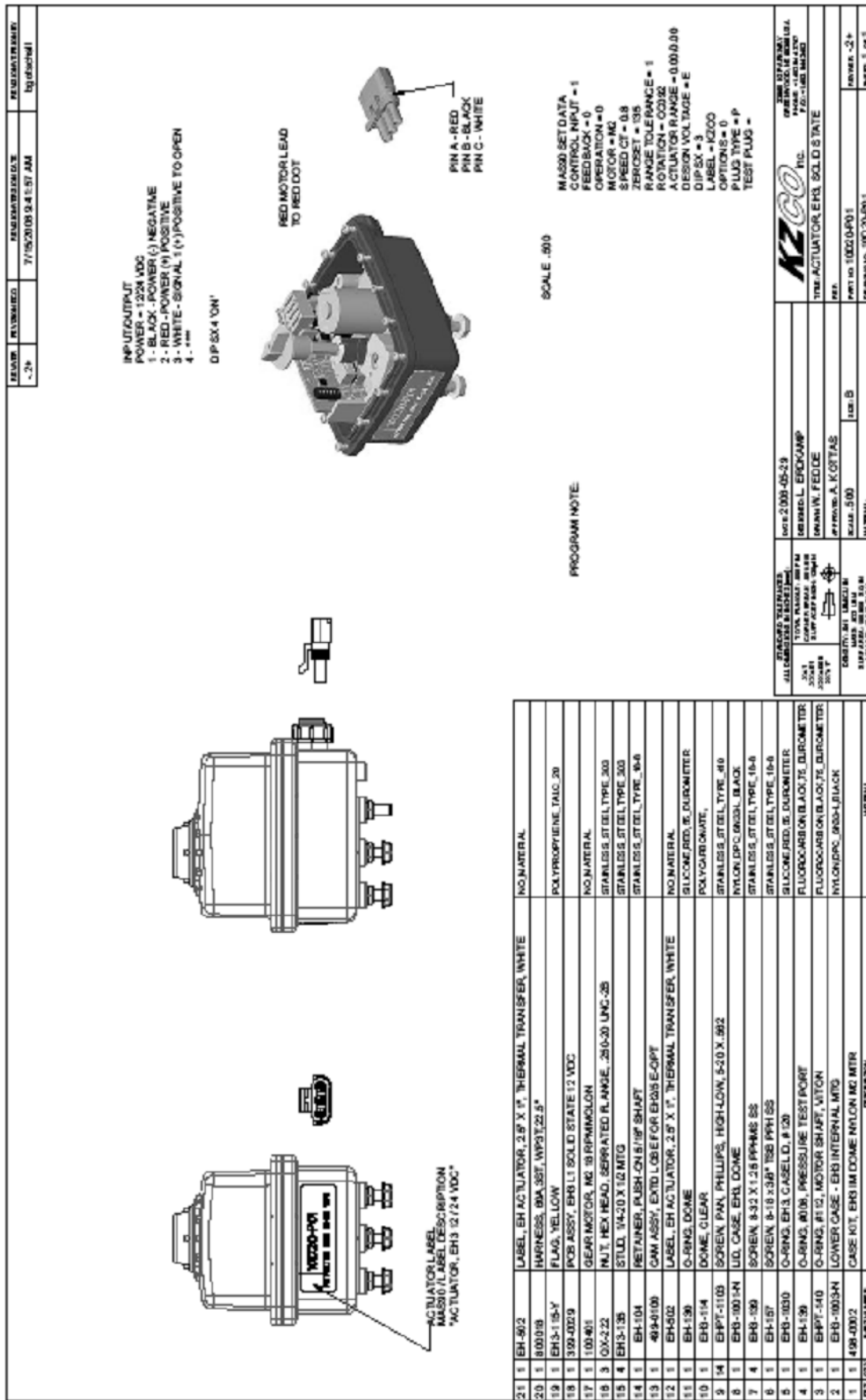
ON OFF

12 VOLTS DC

Top View Dimensions: (1.88) (6.43) (5.03) .50 NPT FE

Side View Dimensions: (7.04) (4.00) (2.05) (2.18)

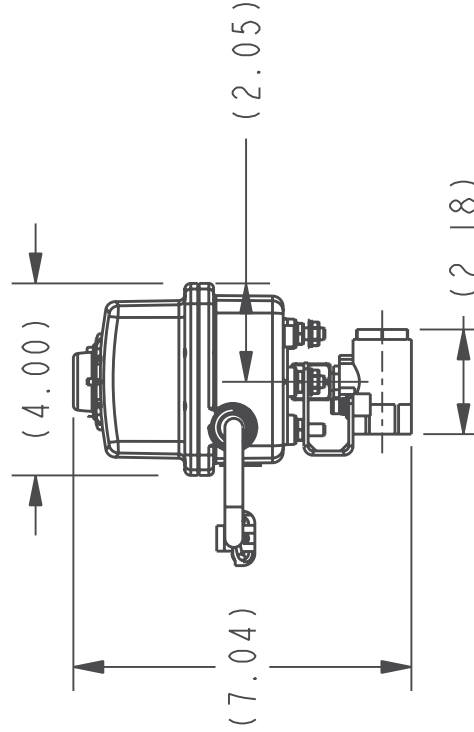
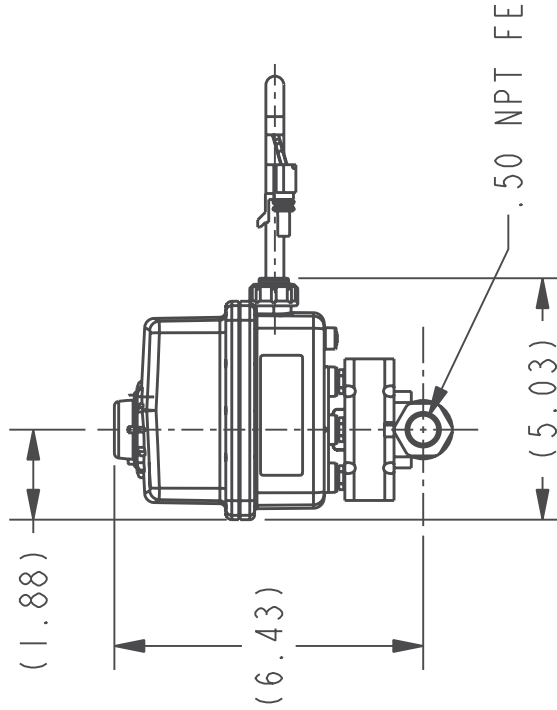
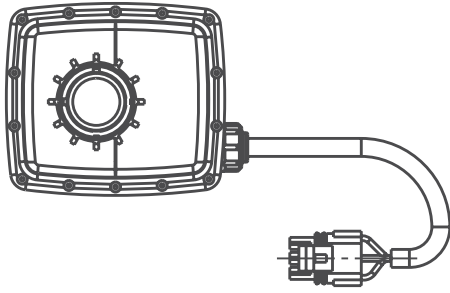
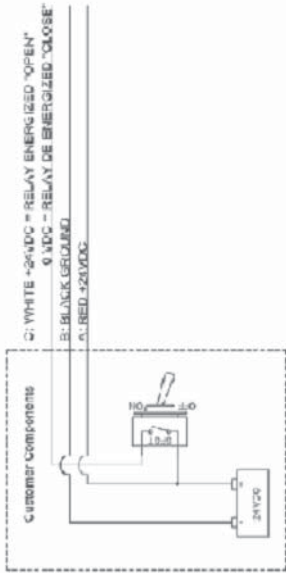
REMOVE SHARP EDGES		INCH [MILLIMETER]		MODEL NAME		MDL CREATED		SHEET	
THIRD ANGLE PROJECTION		OLD PART NO.		5209401		11/13/08		1/2	
MATERIAL NO.		PATTERN NO.		TOLERANCE		W.S. Darley & Co.		ITASCA, IL - CHIPPEWA FALLS, WI	
DO NOT SCALE PRINT		ALL DIMENSIONS IN INCHES UNLESS NOTED		EXCEPT AS NOTED		VALVE - ACTUATED, 12VDC, .50NPT FE		KZCO 84D23-10D20-POI	
THIS DESIGN IS THE PROPERTY OF W.S. DARLEY AND CO. - UNAUTHORIZED REPRODUCTION IS PROHIBITED		KZCO 84D23-10D20-POI		.00 ±.03 .000 ±.010 ANGLES ±1°		DATE 13-Nov-08		SCALE 1/4	
DR: N RJG		CHKD DWS		TRCD		5209401		A	



REVISIONS

LTR	DESCRIPTION	DATE	CHG NO.	APPR'D

60AD WIRING Standard On/Off applications, SPST Switch 24 VDC



REMOVE SHARP EDGES

INCH
[MILLIMETER]



OLD PART NO.

TOLERANCE
EXCEPT
AS NOTED

MODEL NAME 5209402 MDL CREATED 11/13/08 SHEET 1/3 A

MATERIAL DESCRIPTION:
KZCO 84D23-60AD-M

MATERIAL NO.

PATTERN NO.

W.S. Darley & Co.
ITASCA, IL - CHIPPEWA FALLS, WI
VALVE - ACTUATED, 24VDC, .50NPT FE
KZCO 84D23-60AD-M

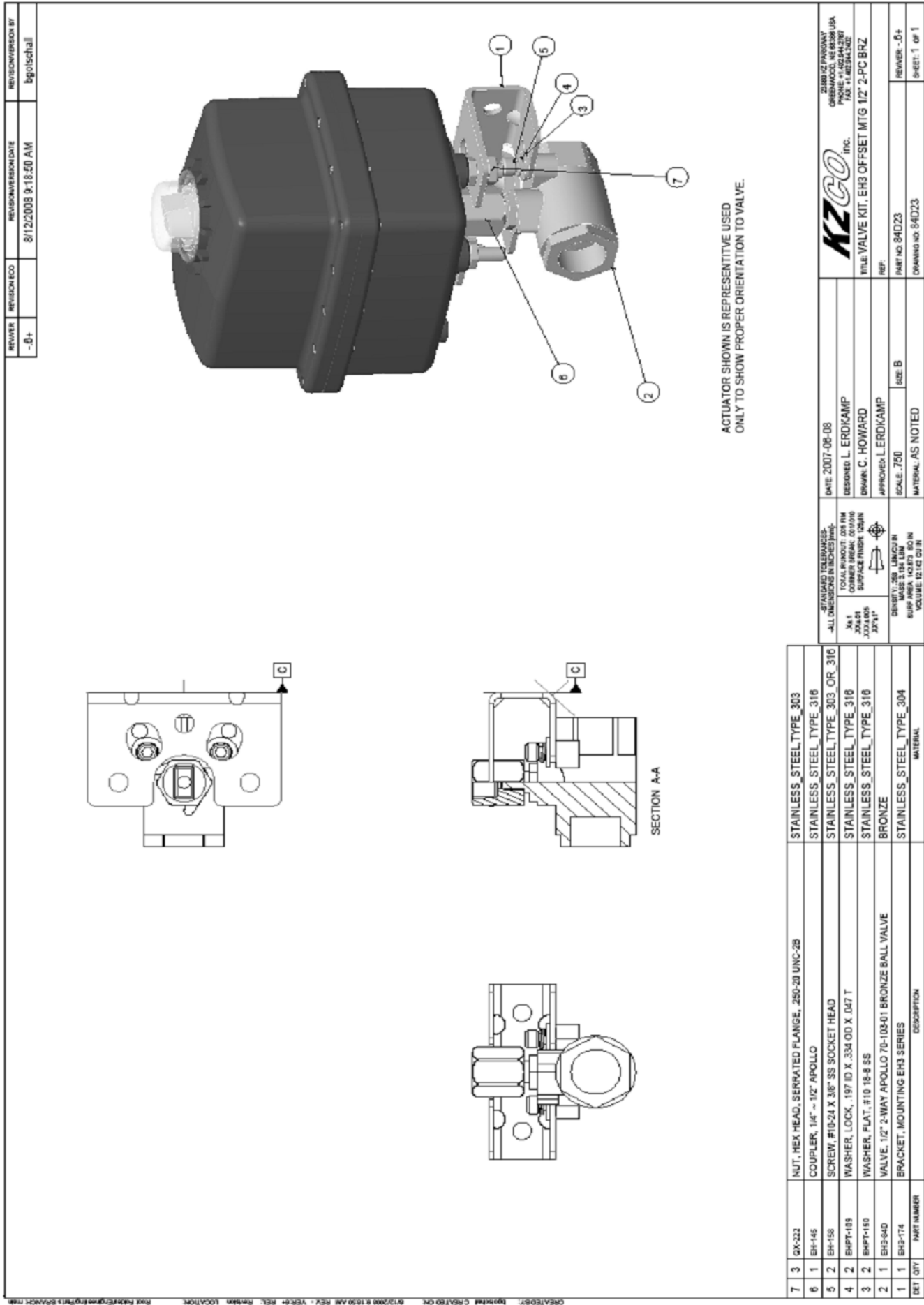
THIS DESIGN IS THE PROPERTY OF
W.S. DARLEY AND CO. - UNAUTHORIZED
REPRODUCTION IS PROHIBITED

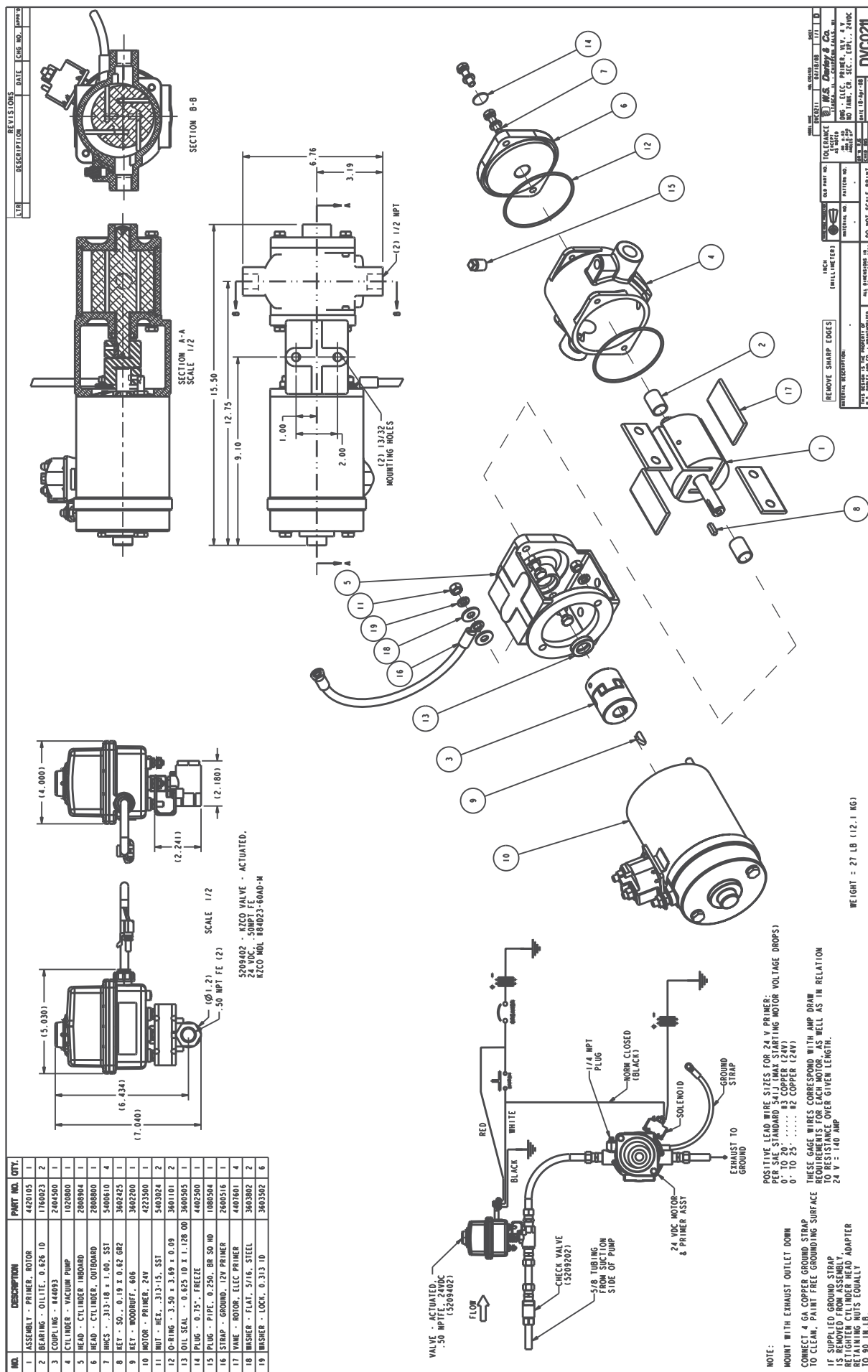
ALL DIMENSIONS IN
INCHES UNLESS NOTED

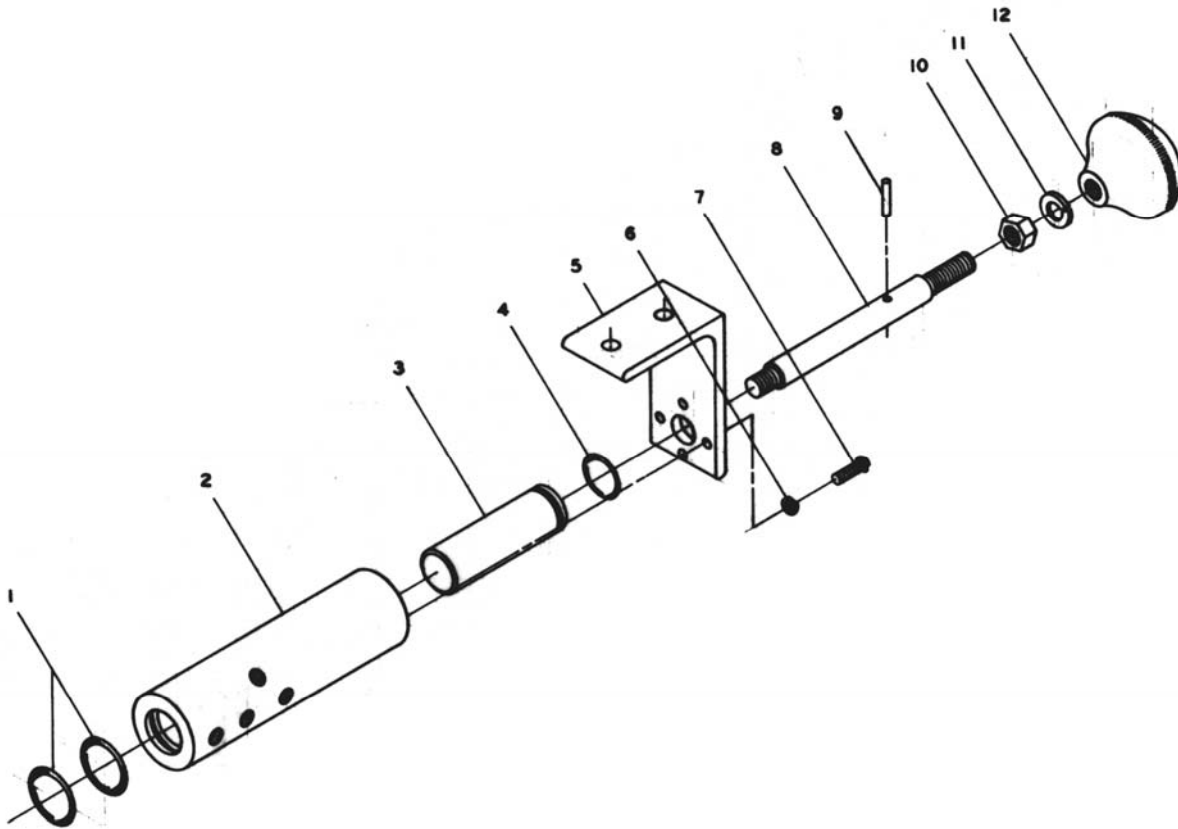
DO NOT SCALE PRINT

DATE 13-Nov-08
SCALE 1/4

5209402







PARTS LIST FOR MULTIPLE DRAIN VALVE

DRAWING NO. DGC0112

Rep. No.	Name of Part	Qty	Rep. No.	Name of Part	Qty
1	O-ring – Body	2	7	Round Head Cap Screw	4
2	Drain Valve Body	1	8	Drain Valve Stem	1
3	Drain Valve Plug	1	9	Pin	1
4	O-ring – Plug	1	10	Hex Nut	1
5	Drain Valve Mounting Bracket	1	11	Lock Washer	1
6	Lock Washer	4	12	Knob	1

BALL VALVE QUARTER TURN - SELF LOCKING

The Darley Ball Valve is a quarter turn, all bronze valve designed for the fire service.

The ball is cast bronze, precision machined *stainless steel ball* for long trouble free service. It is easily serviced in the field.

The lever is self locking and easily adjusted, even under extreme high pressure.

TO DISASSEMBLE AND REPAIR THE BALL VALVE ILLUSTRATION DGC0100

TOOLS REQUIRED:

- 3/16" Allen Wrench
- 1-1/8" Wrench
- 3/4" & 1" Wrench
- Vise Grips or Pliers

1. Remove cap nut (20) and adjusting nut (16).
2. Lever Assembly (11) pulls straight up. Watch for 2 cam balls (12).
3. Unbolt and remove clutch ring (9), clutch sleeve (8), valve stem (7), spring (14), and valve stem washer (15). Check clutch ring (9) and sleeve (8) for scoring or excessive wear. Check o-ring (26). Replace if necessary.
4. Remove nipple (2). Check Quad Ring (25). Replace if necessary.
5. Unscrew ball guide screw (6). Check o-ring (23). Replace if necessary.
6. Remove valve ball (3). Check for scratches, corrosion, and wear. Replace if necessary.
7. Remove seat assembly (4). Check condition of rubber seat. Replace seat assembly if necessary.

REASSEMBLY OF BALL VALVE ILLUSTRATION DGC0100

1. Position ball (3) in body so ball guide screw (6) engages bottom of ball as it is screwed into position.
2. Put valve stem (7) into position. Make certain stem engages slot on top of ball.
3. Slip washer (15), spring (14), and clutch sleeve (8) over the stem. Place clutch ring (9) over the sleeve and secure with the four (4) 1/4" NC x 5/8" socket head cap screws.
4. Set the two cam balls (12) into the V grooves in the clutch sleeve (8) and drop lever assembly over them. Tighten the adjusting nut (16) so that approximately 1/8" play is left at the end of a 6" lever. Over tightening this nut will make the clutch lock inoperative. Lock adjusting nut (16) with cap nut (20). Recheck this adjustment after valve is placed in service.
5. Place seat assembly (4), seat o-ring (5), and quad ring (25) into position.
6. Secure nipple (2) to valve body with eight (8) 1/4" NC x 5/8" socket head cap screws.

**If more information is needed, call W.S. DARLEY & CO. at
Chippewa Falls, WI at 800-634-7812 or 715-726-2650**

REMOTE CONTROL SUCTION RELIEF VALVE DRAWING DGC0115

The suction relief valve bypasses water from the pump suction extension to the ground at a set pressure, preventing excessive rise of supply pressure when relay hose lines are shut off.

Turning pressure setting hand wheel (14) clockwise raises the relief pressure, and counterclockwise lowers it.

The self cleaning fine mesh strainer will prevent entry of solids that could cause the relief valve to malfunction. Open the strainer flush valve to remove small accumulations. This is accomplished by turning the strainer flush valve knob (6) counterclockwise 2 to 3 full turns. Strainer trapped debris will be flushed to the ground. Pump supply pressure should be 50-100 PSI when performing this procedure.

TO SET SUCTION RELIEF VALVE

1. Connect a discharge line from an auxiliary pump to the pump suction containing the suction relief valve. The auxiliary pump must be able to supply a pressure greater than the desired pressure setting of the suction relief valve.
2. Close all other discharge and suction valves.
3. Increase auxiliary pump engine throttle setting until pressure gage indicates the pressure that suction relief valve is open.
4. If suction relief valve opens to bypass excessive pressure, slowly turn hand wheel (14) clockwise until valve closes.
5. If suction relief valve does not open, turn hand wheel (14) counterclockwise until valve opens and begins bypassing water. Continue to turn hand wheel (14) counterclockwise 2 more complete turns. Now slowly turn hand wheel clockwise until valve closes and stops bypassing water.

The suction relief valve will now prevent damage to the pump from a pressure surge (water hammer) which is the result of rapid closing or opening of relay line valves.

Should a higher or lower relief pressure be desired, repeat the above procedure.

CAUTION: With all discharge valves closed, the water in auxiliary pump casing will heat up rapidly. Avoid damage by allowing a very low flow of water to discharge when pump is running.

REMOTE CONTROL SUCTION RELIEF VALVE WITH MECHANICAL SHUTOFF DRAWING DGC0115

MAINTENANCE

Open the relief valve strainer flush valve (6) during every operation at 50-100 PSI supply pressure to insure against foreign material blocking the screen.

The relief valve, pilot unit, and strainer assemblies should be taken apart for inspection and cleaning at least annually, or as often as found necessary to insure trouble free performance.

To disassemble pilot head, first turn hand wheel (14) counterclockwise to remove spring compression. Remove the four 1/4" screws holding regulator spring housing (18). Lift out diaphragm (23) and pilot valve (51) assembly. Clean and make certain 3/32" diameter orifice hole is free of obstruction.

When reassembling pilot head, turn hand wheel (14) a few times clockwise to compress spring before tightening four screws holding spring housing. This will properly center valve seat and diaphragm.

The valve piston (40) and spring (44) chamber should be inspected and cleaned.

Replace diaphragm and o-rings if damaged or deteriorated.

Apply a thin coating of waterproof grease lubricant: to spring housing counterbore that guides pilot valve (51) and ball (52), to end of tension screw (17), and between piston (40) and center post

Self cleaning strainer (63) can be removed for inspection or replacement by alternately turning valve knob (6) and stop nut (7) counterclockwise until stem is free for removal. To avoid discharging water through opening created by stem (62) removal, pump should be completely shut down before stem (62) is removed. Inspect and clean screen (63) if required. Check quad ring (64) for damage or deterioration. Reverse procedure to reassemble valve. Use care when initially inserting screen into body to avoid damaging quad ring (64) or valve seat.

To replace flush valve seat (69), remove stem/screen assembly. Disconnect tubing lines attached to (61) body half, unscrew (61) body half from (70) body half. Replace (69) valve seat. Reverse procedure to reassemble valve.

SUCTION RELIEF VALVE PARTS LIST DRAWING DGC0115

REP NO.	DESCRIPTION	REP NO.	DESCRIPTION
3	Decal - RV Flush	41	Relief Valve Body
4	Panel Nut	42	O-ring Flange
5	Socket Set Screw	43	Spring Centering Plug
6	Flush Valve Knob	44	Spring
7	Stop Nut	45	O-ring Body Flange
11	Panel Valve Nut	46	O-ring Piston
12	Decal - Pressure Hi-Lo	50	O-ring, Bleed Port
13	Trim Plate	51	Pilot Valve
14	Hand wheel	52	Ball
16	Socket Set Screw	53	Pilot Valve Body
17	Spring Tension Screw	61	Body Half, Flush Valve
18	Spring Housing	62	Stem
19	Spring Retainer	63	Screen
20	Regulator Spring	64	Quad Ring
21	Pilot Valve Nut	66	O-ring Valve Seat
22	Housing Pilot Ring	67	Valve Seat Ring
23	Diaphragm	68	O-ring Flush Valve Body
37	O-ring, Relief Valve Head	69	Flush Valve Seat
38	Relief Valve Head	70	Body Half, Flush Valve
40	Relief Valve Piston	84	Flange
		86	Lock Washer Internal

**IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT
CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650**

W.S. DARLEY & CO.

REMOTE CONTROL PRESSURE RELIEF VALVE WITH MECHANICAL SHUTOFF

Refer to Drawing DGC0141

The relief valve bypasses water from the pump discharge manifold to the suction chamber at a set pump pressure, preventing excessive rise of discharge pressure when hose lines are shut off.

Turning pressure setting hand wheel (14) clockwise raises the relief pressure, and counter clockwise lowers it.

The self-cleaning fine mesh strainer will prevent the entry of solids that could cause the relief valve to malfunction. Open the strainer flush valve to remove small accumulations. This is accomplished by turning the strainer flush valve knob (6) counter clockwise 2 to 3 full turns. Strainer trapped debris will be flushed to the ground. Pump supply pressure should be 50-100 PSI when performing this procedure.

TO SET RELIEF VALVE

1. Turn four-way valve OFF.
2. Open at least one discharge valve and increase engine throttle setting until pressure gage indicates the pressure at which relief valve is to open.
3. Turn four-way valve ON.
4. If gage reading drops below pressure set in step 2, turn hand wheel (14) clockwise until pressure returns to set point.
5. If gage reading does not drop, turn hand wheel (14) counter clockwise until pressure drops 5 to 10 PSI below set point. Then slowly turn hand wheel clockwise until pressure returns to pressure set in step 2.

The relief valve will now prevent the discharge pressure from rising above that for which it is set, and requires no further attention.

Should a higher or lower relief pressure be desired, repeat above procedure.

CAUTION

With all discharge valves closed, water in the auxiliary pump casing will heat up rapidly. To avoid possible damage, allow a very small stream of water to discharge when the pump is running.

REMOTE CONTROL PRESSURE RELIEF VALVE WITH MECHANICAL SHUTOFF

MAINTENANCE DRAWING DGC0141

Open the relief valve strainer flush valve (6) during every operation at 50-100 PSI supply pressure to insure foreign material is not blocking the screen.

The 3/32" diameter metering orifice and diaphragm chamber at (21) may be back-flushed if necessary while the pump is delivering water by opening the pilot head drain and placing valve handle (9) midway between ON and OFF position.

The relief valve, pilot unit, and strainer assemblies should be taken apart for inspection and cleaning at least annually, or as often as found necessary to insure trouble free performance.

To disassemble pilot head, first turn hand wheel (14) counter clockwise to remove spring compression. Remove the four 1/4" screws holding regulator spring housing (18). Lift out diaphragm (23) and pilot valve (51) assembly. Clean and make certain 3/32" diameter orifice hole is free of obstruction.

When reassembling pilot head, turn hand wheel (14) a few times clockwise to compress spring before tightening four screws holding spring housing. This will properly center valve seat and diaphragm.

The valve piston (40) and spring (44) chamber should be inspected and cleaned.

Replace diaphragm and o-rings if damaged or deteriorated.

Apply a thin coating of waterproof grease lubricant: to spring housing counterbore that guides the pilot valve (51) and ball (52), to end of tension screw (17), and between piston (40) and center post.

Self-cleaning strainer (63) can be removed for inspection or replacement by alternately turning valve knob (6) and stop nut (7) counter clockwise until stem is free for removal. To avoid discharging water through opening created by stem (62) removal, pump should be completely shut down before stem (62) is removed. Inspect and clean screen (63) if required. Check quad ring (64) for damage or deterioration. Reverse procedure to reassemble valve. Use care when initially inserting screen into body to avoid damaging quad ring (64) or valve seat.

To replace flush valve seat (69), remove stem/screen assembly, disconnect tubing lines attached to (61) body half and unscrew (61) body half from (70) body half. Replace (69) valve seat. Reverse procedure to reassemble valve.

All Darley relief valves can be provided with a micro switch and either one or two pilot lights to indicate when the valve is open or closed.

**IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT
CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650**

W.S. DARLEY & CO.

Relief Valve Alarm Installation Instruction

This Alarm is designed to concentrate audible sound in the operator zone only. For optimum Performance, position alarm sound opening so it is facing the operator at a distance of 24 - 36 inches.

Mount unit in 1.12 diameter panel hole. If panel is thicker than .09 inches, invert nut.

Do not mount with sound opening in an upward position. Do not obstruct opening.

Connect to 12 VDC only.

Two (2) wires are required to complete the circuit. The alarm is sensitive to polarity and will not operate if connected with polarity reversed.

Relief Valve Alarm Installation Instructions

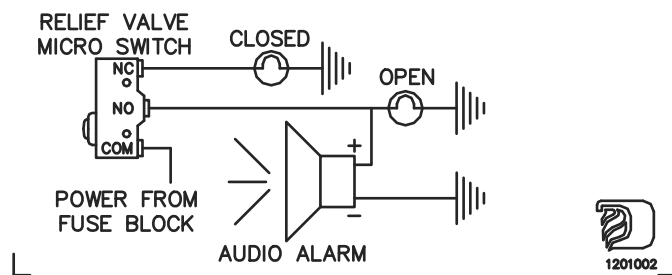
THIS ALARM IS DESIGNED TO CONCENTRATE AUDIBLE SOUND IN THE OPERATOR ZONE ONLY. FOR OPTIMUM PERFORMANCE, POSITION ALARM SOUND OPENING SO IT IS FACING THE OPERATOR AT A DISTANCE OF 24-36 INCHES.

MOUNT UNIT IN 1.12 DIAMETER PANEL HOLE. IF PANEL IS THICKER THAN .09 IN., INVERT NUT.

DO NOT MOUNT WITH SOUND OPENING IN AN UPWARD POSITION. DO NOT OBSTRUCT OPENING.

CONNECT TO 12 VDC ONLY.

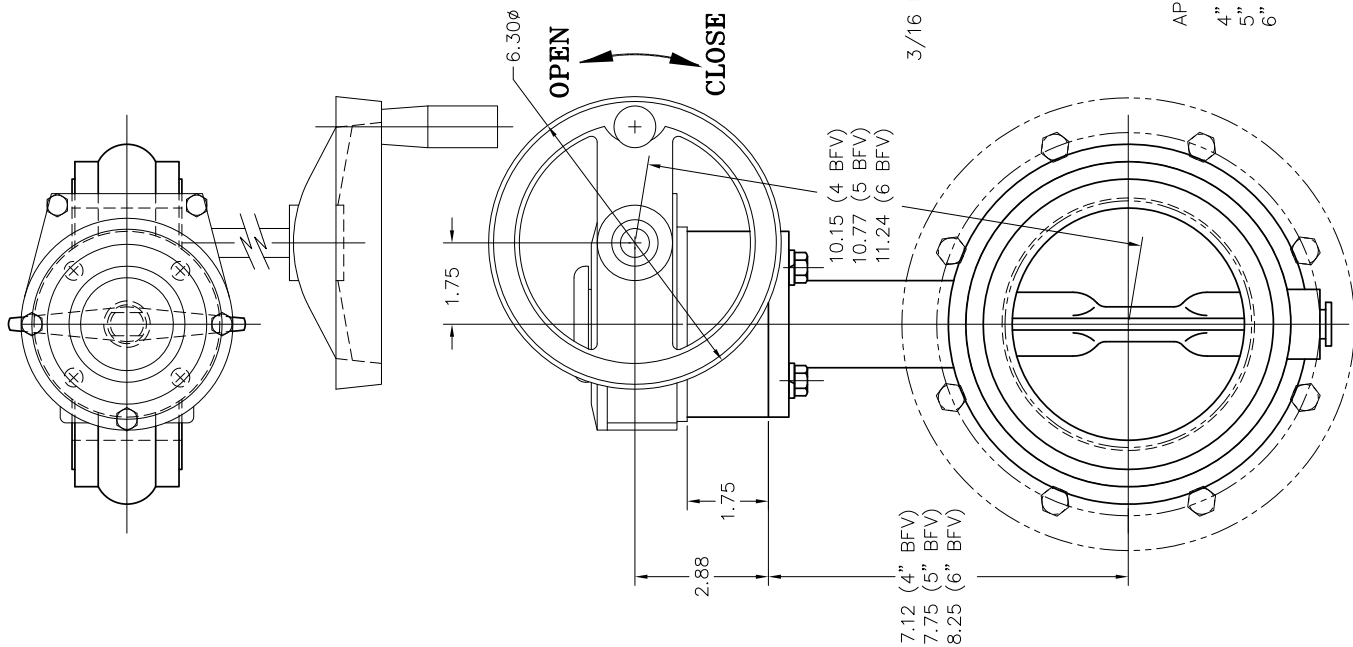
TWO (2) WIRES ARE REQUIRED TO COMPLETE THE CIRCUIT. THE ALARM IS SENSITIVE TO POLARITY AND WILL NOT OPERATE IF CONNECTED WITH POLARITY REVERSED.



**IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT
CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650**



ATTACH WARNING PLATE
(ASSY AM00700) TO
CONTROL PANEL VIA
REV. NO. 96-115 (REF). Ⓢ



APPROXIMATE WEIGHT

4" - 19 LB = 8.6 KG
5" - 22 LB = 10 KG
6" - 27 LB = 12.2 KG

96-115/9/19/96 Ⓢ ADDED PLATE LMH

PART NO.	DATE	CHANGE
W.S. DARLEY & CO.		
MELROSE PARK, IL - CHIPPENVA FALLS, WI		
DWG - BIRFLY VALVE ASSY		
MANUAL CONTROLLED		
DATE	SEP25,91	
SCALE	1/2	



OLD PART NO.
G2420

CHKD	DWS	TRCD

DO NOT SCALE PRINT

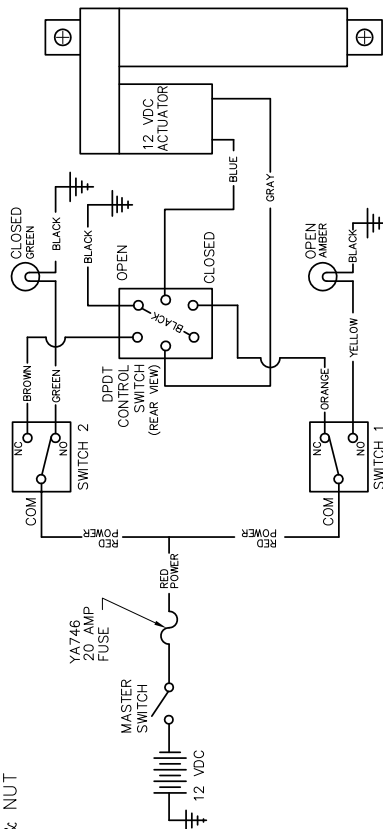
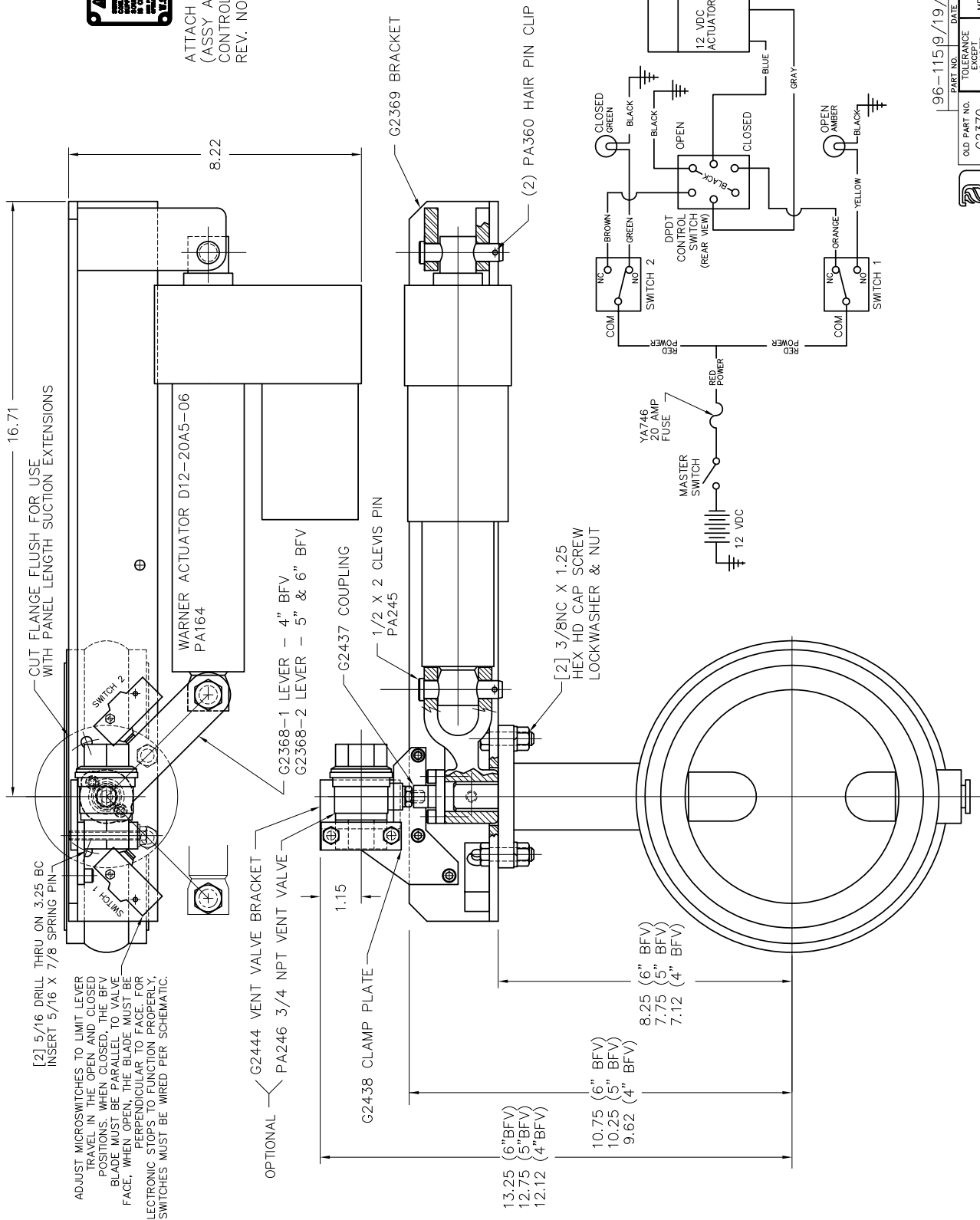
ALL DIMENSIONS IN
INCHES UNLESS NOTED

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W.S. DARLEY & CO.
REPRODUCTION IS PROHIBITED

DGD0104



ATTACH WARNING PLATE
(ASSY AM00700) TO
CONTROL PANEL VIA
REV. NO. 96-115 (REF). ③



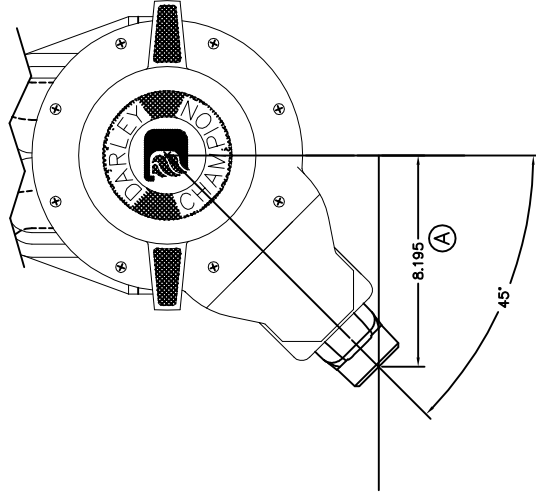
APPROXIMATE WEIGHT = 35 LB = 15.9 KG (6" BFV)
30 LB = 13.6 KG (5" BFV)
27 LB = 12.2 KG (4" BFV)

96-115/19/96 ③ ADDED PLATE LMH	
PART NO.	CHANGE
OLD PART NO.	DATE
G2370	
TOLERANCE EXCEPT AS NOTED	
FRACTIONAL ANGLES 3/16"	
DECIMAL ANGLES 3/16"	
W.S. DARLEY & CO. MELROSE PARK, IL - CHIPPewa FALLS, WI	
DWG - BUTTERFLY VALVE ASSEMBLY ELECTRIC ACTUATED	
DATE NOV20,91	SCALE 1/2
CHKD SETZ	TRGD
DO NOT SCALE PRINT	
ALL DIMENSIONS IN INCHES UNLESS NOTED	
MATERIAL:	
THIS DESIGN IS THE PROPERTY OF W.S. DARLEY & CO. UNAUTHORIZED REPRODUCTION IS PROHIBITED	
DGD0101	

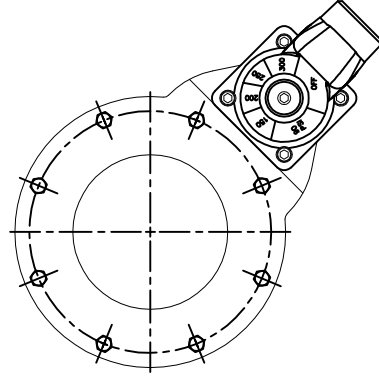
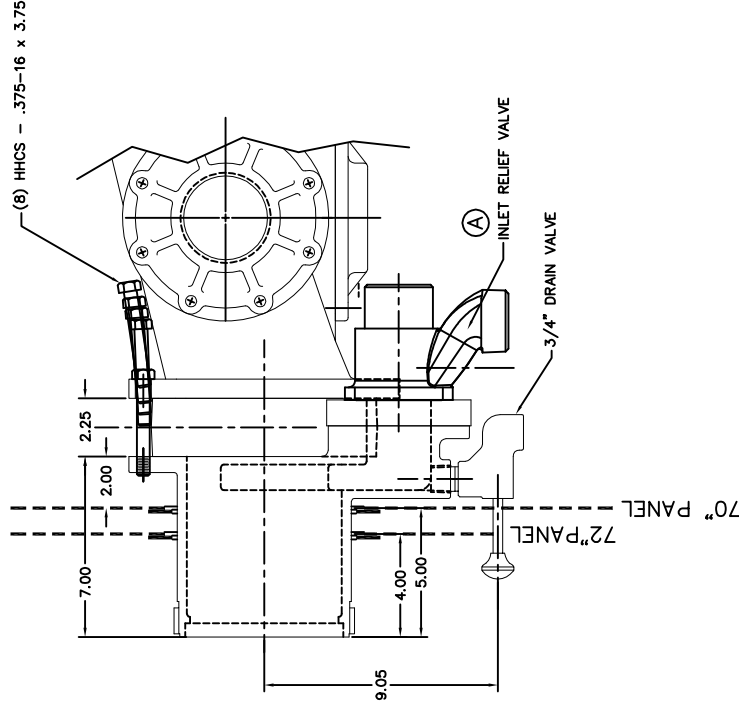
INLET RELIEF VALVE INFORMATION: (A)

ACTUAL PRESSURE RANGE IS 90PSI – 300PSI
THERE MAY BE SOME DIMINISH IN FLOW AT HIGHER
PRESSURE SETTINGS. (SETTINGS BELOW 200 PSI
RECOMMENDED FOR MOST APPLICATIONS).
RELIEF VALVE IS FACTORY SET AT 125 PSI AND
WHEN PRESET AT 125 PSI, THE PRESSURE RELIEF
VALVE SHALL NOT ALLOW A PRESSURE RISE GREATER
THAN 60 PSI AT THE DEVICE INLET WHILE FLOWING A
MINIMUM OF 150 GPM.
THIS VALVE IS NFPA 2009 1901 COMPLIANT PER SECTION
16.6.6.3

ADJUSTMENT INSTRUCTIONS (IF REQUIRED): (A)
ADJUST CENTER HEX COUNTERSUNK HEX HEAD PRESSURE
ADJUSTING BOLT WITH A 1/4" ALLEN WRENCH, 9/16" OR
14 MM SOCKET.
TO SET AT THE DESIRED RELIEF PRESSURE, ADJUST THE ADJUSTING
BOLT HEAD SO THE TOP OF THE BOLT HEAD IS EVEN WITH
THE DESIRED PRESSURE.



NOTE:
SUCTION NIPPLE WILL BE ROTATED
45° OFF CENTER WHEN MOUNTED.



**TYPICAL SUCTION EXTENSION CONFIGURATION
FOR EM, LDM, N, & S PUMP**

EXTENSION AND NIPPLE
ARE THE SAME FOR 70 & 72"
ASSEMBLIES WITH AND
WITHOUT BUTTERFLY VALVES.
IF BFV IS NOT REQUIRED, THEN
KC00300 ASSY W/2.25 THICK SPACER IS USED.
USE 1962503 STAINLESS STEEL PANEL TRIM RING



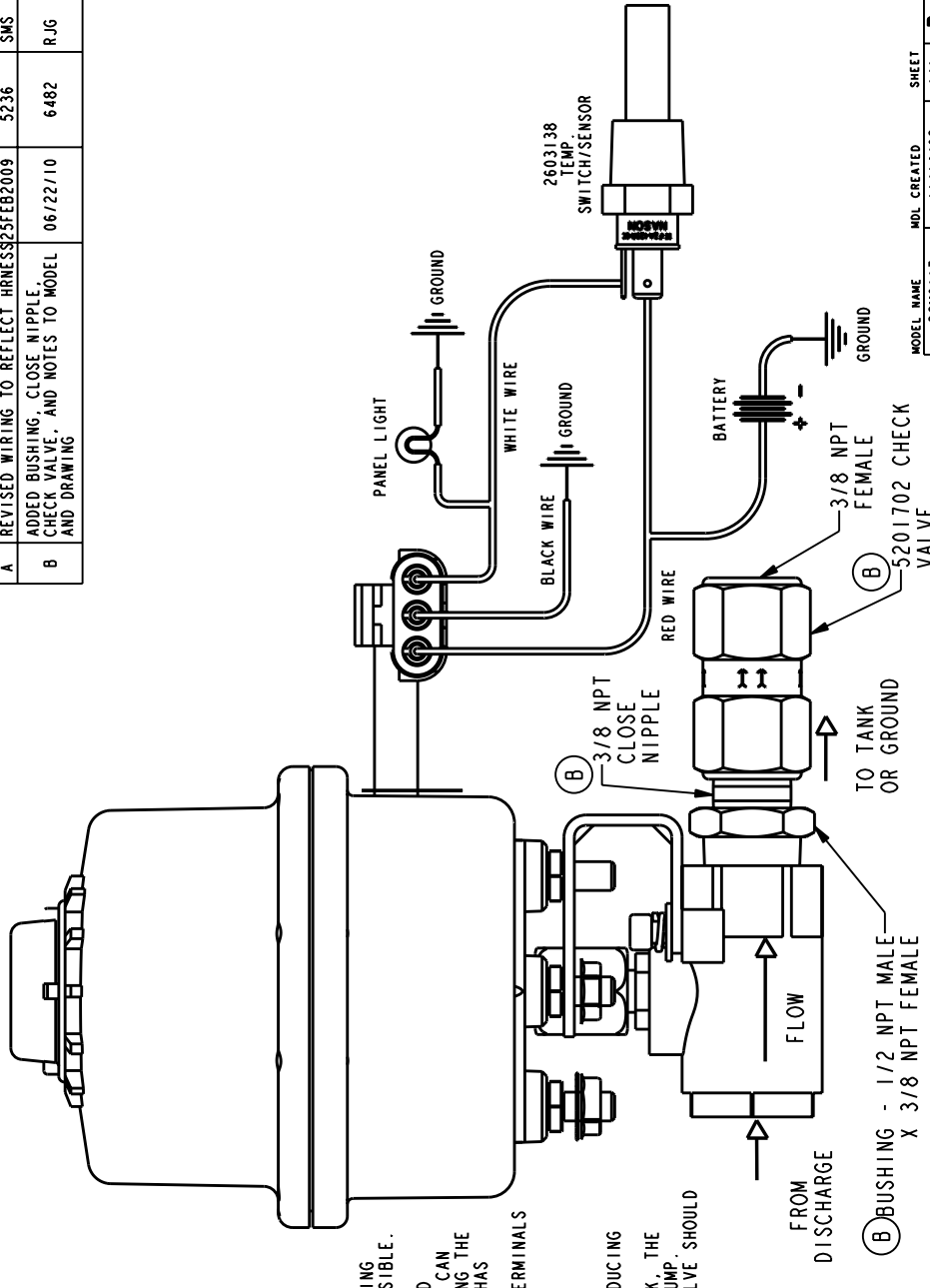
MATERIAL:		DO NOT SCALE PRINT	
THIS DESIGN IS THE PROPERTY OF W.S. DARLEY & CO. UNAUTHORIZED REPRODUCTION IS PROHIBITED		ALL DIMENSIONS IN INCHES UNLESS NOTED	
PATTERN NO.		SCALE 1/4	

A - #9512		07/18/12		UPDATED DRAWING TO REFLECT USE OF 8.20 WAS 5.22, UPDATED INFO AND ADJUSTMENT SECTION	
PART NO.		DATE		ADJUSTMENT SECTION	
TOLERANCE EXCEPT AS NOTED		W.S. DARLEY & CO.		MELROSE PARK, IL - CHIPPENAW FALLS, WI	
FRACTIONAL DECIMAL ANGLES ±1°		INSTR - INLET RV ADJUSTMENT		1200572	
DRN		DATE 03FEB.00		SCALE 1/4	
CHKD S/L		DATE 03FEB.00		SCALE 1/4	
TRGD		DATE 03FEB.00		SCALE 1/4	

REVISIONS				
LTR	DESCRIPTION	DATE	CHG NO.	APPR'D
A	REVISED WIRING TO REFLECT HRNESS25FEB2009	5236		SMS
B	ADDED BUSHING, CLOSE NIPPLE, CHECK VALVE, AND NOTES TO MODEL AND DRAWING	06/22/10	6482	RJG

INSTALLATION INSTRUCTIONS:

- 1) INSTALL TEMPERATURE SWITCH ON SUCTION SIDE OF PUMP, AS NEAR TO IMPELLER AS POSSIBLE, PREFERABLY BETWEEN THE IMPELLER AND THE RELIEF VALVE.
 - 2) INSTALL THE ACTUATOR VALVE ON THE DISCHARGE SIDE OF THE PUMP. ITS LOCATION IS NOT CRITICAL. IT CAN BE MOUNTED DIRECTLY TO THE PUMP OR POSITIONED REMOTELY AND CONNECTED WITH A 3/8 OD MINIMUM LINE.
 - 3) VALVE DOES NOT NEED TO BE MOUNTED VERTICALLY AS SHOWN.
 - 4) *PLUMB THE VALVE OUTLET TO TANK, IF DESIRED, USING 3/8 OD MINIMUM LINE. KEEP ALL LINES AS SHORT AS POSSIBLE.
 - 5) ATTACH ONE LEAD OF VALVE TO THE TERMINAL ON THE TEMPERATURE SWITCH. ATTACH THE OTHER LEAD TO A FUSED VOLTAGE SOURCE. IF DESIRED, THE TEMPERATURE SWITCH CAN ALSO BE USED TO CONTROL A LIGHT ON THE PANEL NOTIFYING THE OPERATOR THAT OVERHEATING IS OCCURRING AND THE UNIT HAS BEEN ACTIVATED.
 - 6) TO TEST THE VALVE: WITH PUMP RUNNING, JUMP THE TERMINALS ON THE TEMPERATURE SWITCH. IF THE VALVE DOES NOT DISCHARGE, IT MAY BE INSTALLED BACKWARDS OR WIRING CONNECTIONS MAY BE INCORRECT.
- * CONNECTION TO TANK MAY CAUSE TANK TO OVERHEAT, REDUCING THE EFFECTIVENESS OF THIS DEVICE.
 - TO PREVENT FREEZING DAMAGE, WHEN CONNECTING TO TANK, THE VALVE SHOULD BE MOUNTED ON THE TANK, BUT ABOVE THE PUMP.
 - IF THE VALVE IS TO DISCHARGE TO ATMOSPHERE, THE VALVE SHOULD BE MOUNTED ON, OR AS CLOSE TO THE PUMP AS POSSIBLE.

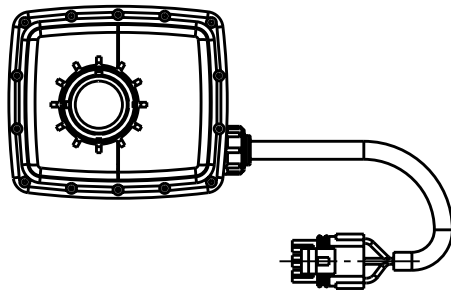
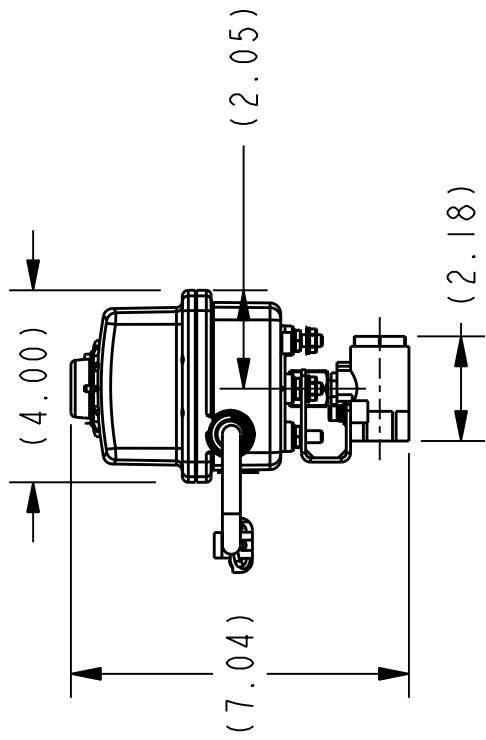
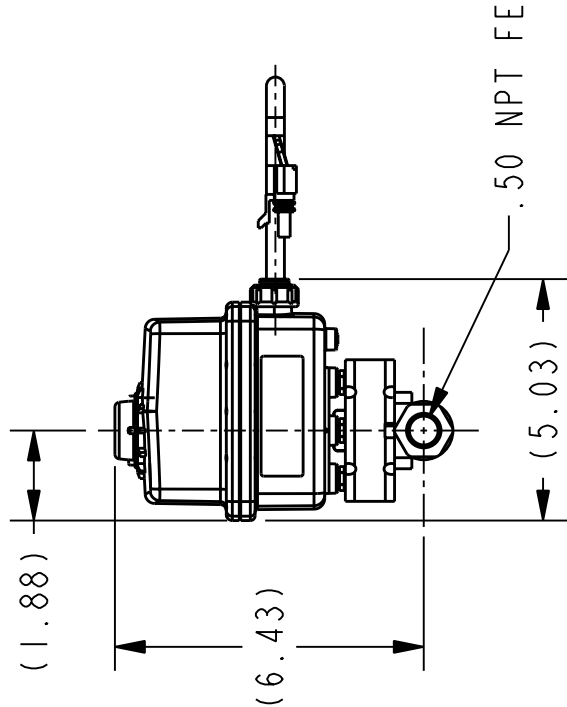
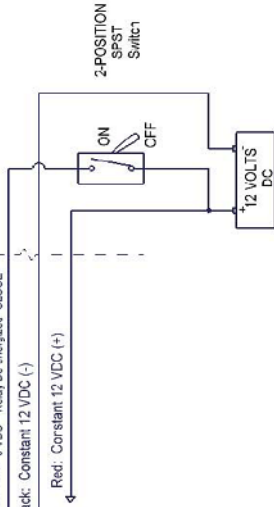


REMOVE SHARP EDGES		INCH (MILLIMETER)	
MATERIAL DESCRIPTION:		ALL DIMENSIONS IN INCHES UNLESS NOTED	
THIS DESIGN IS THE PROPERTY OF W.S. DARLEY AND CO. - UNAUTHORIZED REPRODUCTION IS PROHIBITED		DO NOT SCALE PRINT	
MATERIAL NO.		SCALE	
PATTERN NO.		DATE 14-Nov-08	
TOLERANCE EXCEPT AS NOTED -00 ± 0.03 ANGLES ± 1°		DR'N RJG CHKD DWS TRCD	
OLD PART NO.		1/1	
NEW PART NUMBER		1/1	
MODEL NAME		DGM0117	
MDL CREATED		11/14/08	
SHEET		1/1	
W.S. Darley & Co.		DGM0117	
ITASCA, IL - CHIPPewa FALLS, WI		DWG - THERMAL RELIEF SYSTEM	
		KZCO VALVE, 2603138 TEMP. SENSOR	

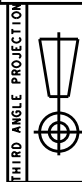
REVISIONS			
LTR	DESCRIPTION	DATE	CHG NO.

SUPPLIED BY KZCO

White: Switched 12VDC (+) - Relay Energized "CLOSE"
 Black: Constant 12VDC (-)
 Red: Constant 12VDC (+)



REMOVE SHARP EDGES



OLD PART NO.

TOLERANCE
EXCEPT
AS NOTED

W.S. Darley & Co.
ITASCA, IL - CHIPPEWA FALLS, WI

VALVE - ACTUATED, 12VDC, .50NPT FE
KZCO 84D23-10D20-P01

DATE 13-Nov-08
SCALE 1/4

DO NOT SCALE PRINT

ALL DIMENSIONS IN
INCHES UNLESS NOTED

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5209401

ANALYST	ANALYST ID	ANALYST NAME	ANALYST ADDRESS	ANALYST CITY	ANALYST STATE	ANALYST ZIP	ANALYST PHONE	ANALYST FAX	ANALYST EMAIL
ANALYST 1	ANALYST 1 ID	ANALYST 1 NAME	ANALYST 1 ADDRESS	ANALYST 1 CITY	ANALYST 1 STATE	ANALYST 1 ZIP	ANALYST 1 PHONE	ANALYST 1 FAX	ANALYST 1 EMAIL
ANALYST 2	ANALYST 2 ID	ANALYST 2 NAME	ANALYST 2 ADDRESS	ANALYST 2 CITY	ANALYST 2 STATE	ANALYST 2 ZIP	ANALYST 2 PHONE	ANALYST 2 FAX	ANALYST 2 EMAIL
ANALYST 3	ANALYST 3 ID	ANALYST 3 NAME	ANALYST 3 ADDRESS	ANALYST 3 CITY	ANALYST 3 STATE	ANALYST 3 ZIP	ANALYST 3 PHONE	ANALYST 3 FAX	ANALYST 3 EMAIL
ANALYST 4	ANALYST 4 ID	ANALYST 4 NAME	ANALYST 4 ADDRESS	ANALYST 4 CITY	ANALYST 4 STATE	ANALYST 4 ZIP	ANALYST 4 PHONE	ANALYST 4 FAX	ANALYST 4 EMAIL
ANALYST 5	ANALYST 5 ID	ANALYST 5 NAME	ANALYST 5 ADDRESS	ANALYST 5 CITY	ANALYST 5 STATE	ANALYST 5 ZIP	ANALYST 5 PHONE	ANALYST 5 FAX	ANALYST 5 EMAIL
ANALYST 6	ANALYST 6 ID	ANALYST 6 NAME	ANALYST 6 ADDRESS	ANALYST 6 CITY	ANALYST 6 STATE	ANALYST 6 ZIP	ANALYST 6 PHONE	ANALYST 6 FAX	ANALYST 6 EMAIL
ANALYST 7	ANALYST 7 ID	ANALYST 7 NAME	ANALYST 7 ADDRESS	ANALYST 7 CITY	ANALYST 7 STATE	ANALYST 7 ZIP	ANALYST 7 PHONE	ANALYST 7 FAX	ANALYST 7 EMAIL
ANALYST 8	ANALYST 8 ID	ANALYST 8 NAME	ANALYST 8 ADDRESS	ANALYST 8 CITY	ANALYST 8 STATE	ANALYST 8 ZIP	ANALYST 8 PHONE	ANALYST 8 FAX	ANALYST 8 EMAIL
ANALYST 9	ANALYST 9 ID	ANALYST 9 NAME	ANALYST 9 ADDRESS	ANALYST 9 CITY	ANALYST 9 STATE	ANALYST 9 ZIP	ANALYST 9 PHONE	ANALYST 9 FAX	ANALYST 9 EMAIL
ANALYST 10	ANALYST 10 ID	ANALYST 10 NAME	ANALYST 10 ADDRESS	ANALYST 10 CITY	ANALYST 10 STATE	ANALYST 10 ZIP	ANALYST 10 PHONE	ANALYST 10 FAX	ANALYST 10 EMAIL



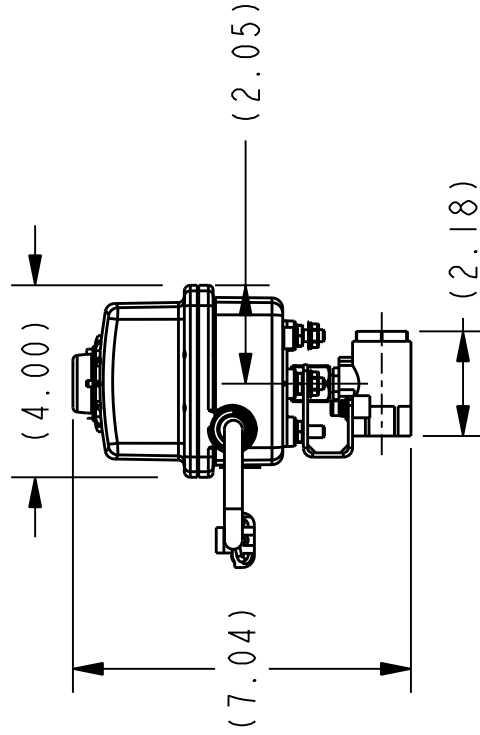
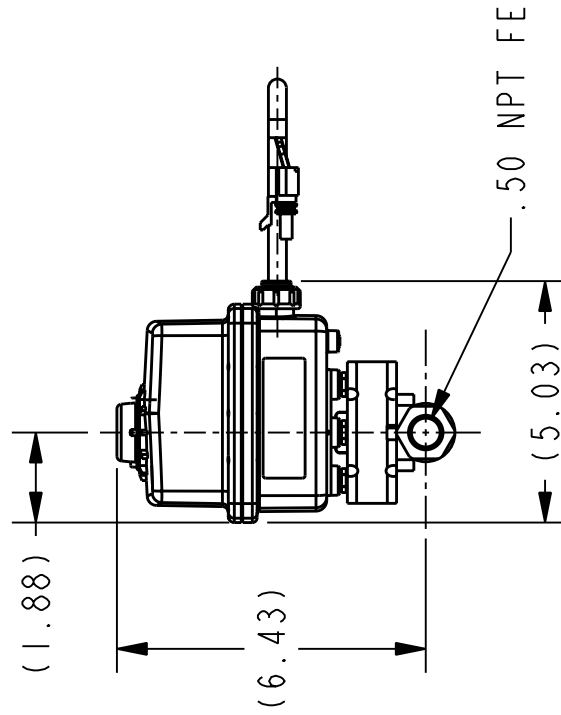
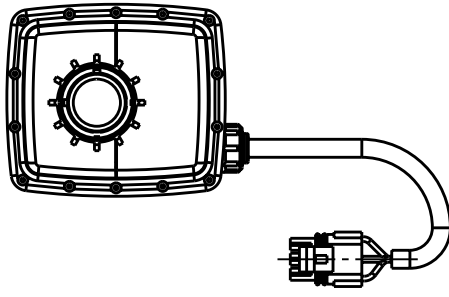
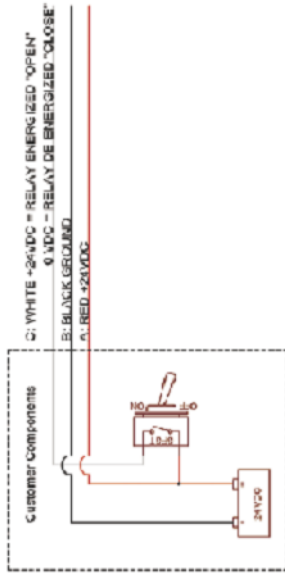
INPUT/OUTPUT
POWER = 1224 W
1 - BLACK - POWER
2 - RED - POWER
3 - WHITE - SIGN
4 - 4mm
OIP SX 4 YON'





SCALE .500

MA520 SET DATA
CONTROL INPUT = 1
FEEDBACK = 0
OPERATION = 0
MOTOR = MC
SPEED CT = 0.8
ZEROSCT = 135
RANGE TOLERANCE
ROTATION = 0032
A CTUATOR RANGE =
DESIGN VOLTAGE = 8
DIP SW = 3
LABEL = K200
OPTIONS = 0
PLUG TYPE = P
TEST PLUG =

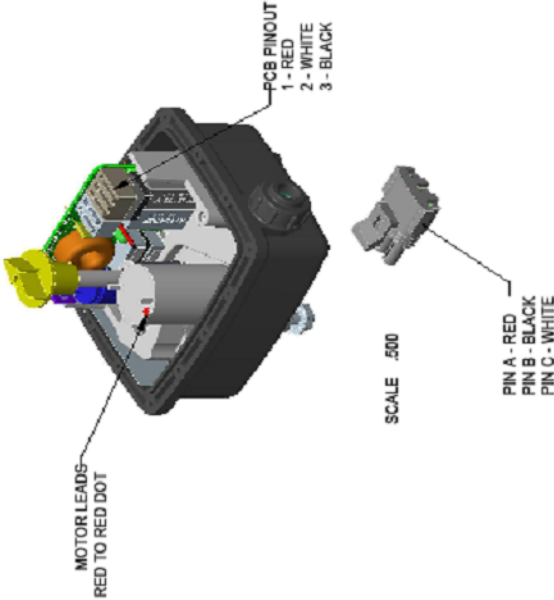
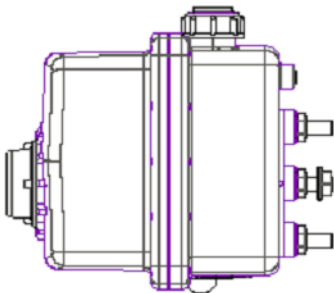
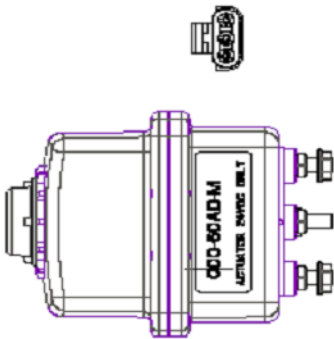
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26		7/15/2008 9:41:57 AM	log attached



REVISIONS				
LTR	DESCRIPTION	DATE	CHG NO.	APPR'D

REMOVE SHARP EDGES						INCH [MILLIMETER]		THIRD ANGLE PROJECTION 		OLD PART NO.		TOLERANCE EXCEPT AS NOTED .00 ± .03 .000 ± .010 ANGLES ± 1°		 W.S. Darley & Co. ITASCA, IL - CHIPPEWA FALLS, WI		MODEL NAME 5209402		MDL CREATED 11/13/08		SHEET 1/3		A	
MATERIAL DESCRIPTION: KZCO 84D23-60AD-M												MATERIAL NO. -		PATTERN NO. -				DATE 13-Nov-08		SCALE 1/4		5209402	
THIS DESIGN IS THE PROPERTY OF W.S. DARLEY AND CO. - UNAUTHORIZED REPRODUCTION IS PROHIBITED						ALL DIMENSIONS IN INCHES UNLESS NOTED						DO NOT SCALE PRINT		DR'N R/G CHKD DWS TRCD									

8	1	EH-502	LABEL, EH ACTUATOR, 2.5" X 1", THERMAL TRANSFER, WHITE	NO. MATERIAL
7	2	QX-222	NUT, HEX HEAD, SERRATED FLANGE, .250-20 UNC-2B	STAINLESS_STEEL_TYPE_303
6	4	EH3-135	STUD, 1/4-20 X 1/2 MTG	STAINLESS_STEEL_TYPE_303
5	1	800018	HARNISS, 60A, 3ST, WP3T, 22.5"	
4	1	EH3-115-Y	FLAG, YELLOW	POLYPROPYLENE_TALC_20
3	1	800468	PCB ASSY, EH3 24VDC 60AD/60DF	
2	1	EH3-110	GEAR MOTOR, EH3 SERIES M1 18 RPM @ 12VDC	
1	1	485-0005	CASE KIT, EH3 IM DOME NYLON M1 & N1 MTR	
DET	QTY	PART NUMBER	DESCRIPTION	MATERIAL

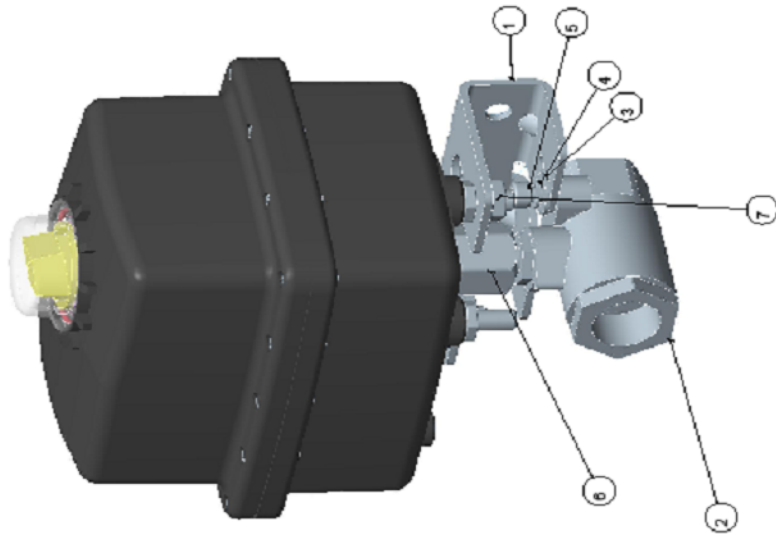
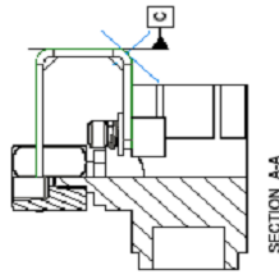


MASRO SET DATA
CONTROL INPUT = 1
FEEDBACK = 0
OPERATION = 0
MOTOR = M
SPEED CT = 0.8
ZEROSSET = 135
RANGE TOLERANCE = 1
ROTATION = CC091
ACTUATOR RANGE = 0.0010.00
DESIGN VOLTAGE = D
DIP SX =
LABEL = KZCO
OPTIONS = 0
PLUG TYPE =
TEST PLUG =

REVISION	REVISION ECO	REVISION/REVISION DATE	REVISION/REVISION BY
1.1	1	3/28/2008 1:49:38 PM	wfeddie

DESIGNED BY: L. EROKAMP DRAWN BY: W. FEDDE APPROVED BY: A. KOTTAS	DATE: 2007-04-10	KZCO inc. 330000 PARKWAY GREENSBORO, NC 27409 PHONE: 336.402.9443 FAX: 336.402.9443
DESIGN: 247, 18MCM IN SUPPLY: 18MCM IN VOLUME: 12.000, 0.1 IN	SCALE: .500	TITLE: ACTUATOR, EH3, 8S @24VDC
SIZE: B	PART NO: 000-60AD-M	REVISION: 1.1
MATERIAL:	DRAWING NO: 000-60AD-M	SHEET: 1 of 1

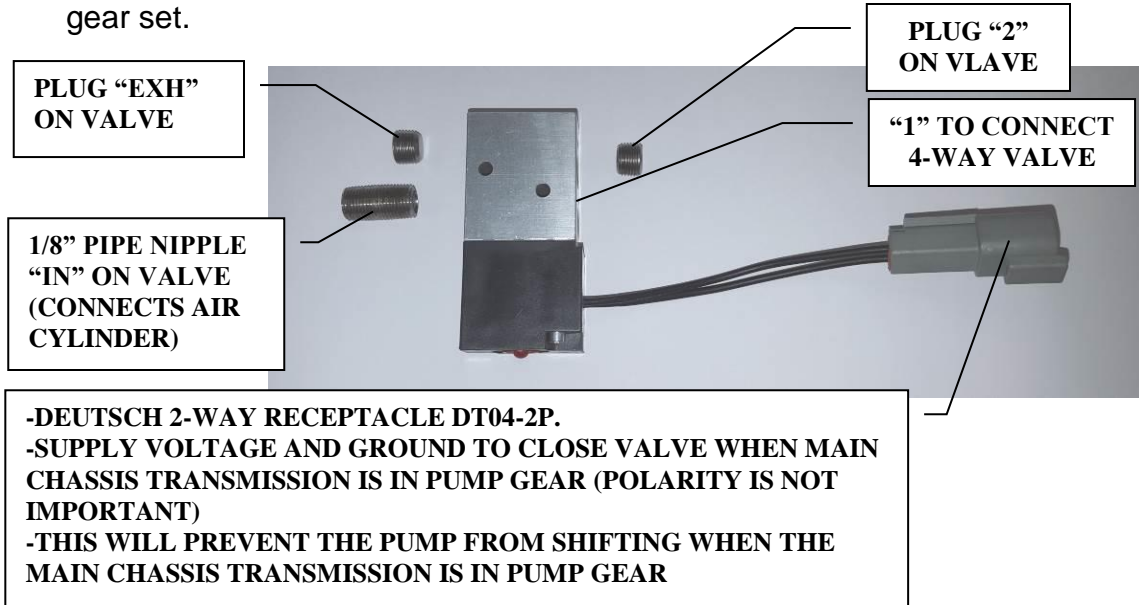
STANDARD TOLERANCES - ALL DIMENSIONS IN INCHES UNLESS SHOWN OTHERWISE X.X1 TOTAL HOLE DIA. MAX X.X2 DIA. OF HOLE X.X3 DIA. OF HOLE X.X4 DIA. OF HOLE		DATE: 007-08-08 DESIGNED: L. ERD/KAMP CHECKED: L. ERD/KAMP DRAWN: C. HOWARD APPROVED: L. ERD/KAMP		2389112 PAROVARY GREENWOOD, NE 68309 USA PHONE: 402.342.3421 FAX: 402.342.3421 KZCO INC.	
X.X5 TOTAL HOLE DIA. MAX X.X6 DIA. OF HOLE X.X7 DIA. OF HOLE X.X8 DIA. OF HOLE		TITLE: VALVE KIT, EH3 OFFSET MTG 12" 2-PC BRZ REF:		PART NO: 84D23 REWORK: -6+ SHEET: 1 of 1	
X.X9 TOTAL HOLE DIA. MAX X.X10 DIA. OF HOLE X.X11 DIA. OF HOLE X.X12 DIA. OF HOLE		SCALE: 750 BASE: 1.04 LEM SUB: 12.42 LEM VOLUME: 12.42 LEM		DRAWING NO: 84D23	



ACTUATOR SHOWN IS REPRESENTATIVE USED ONLY TO SHOW PROPER ORIENTATION TO VALVE.

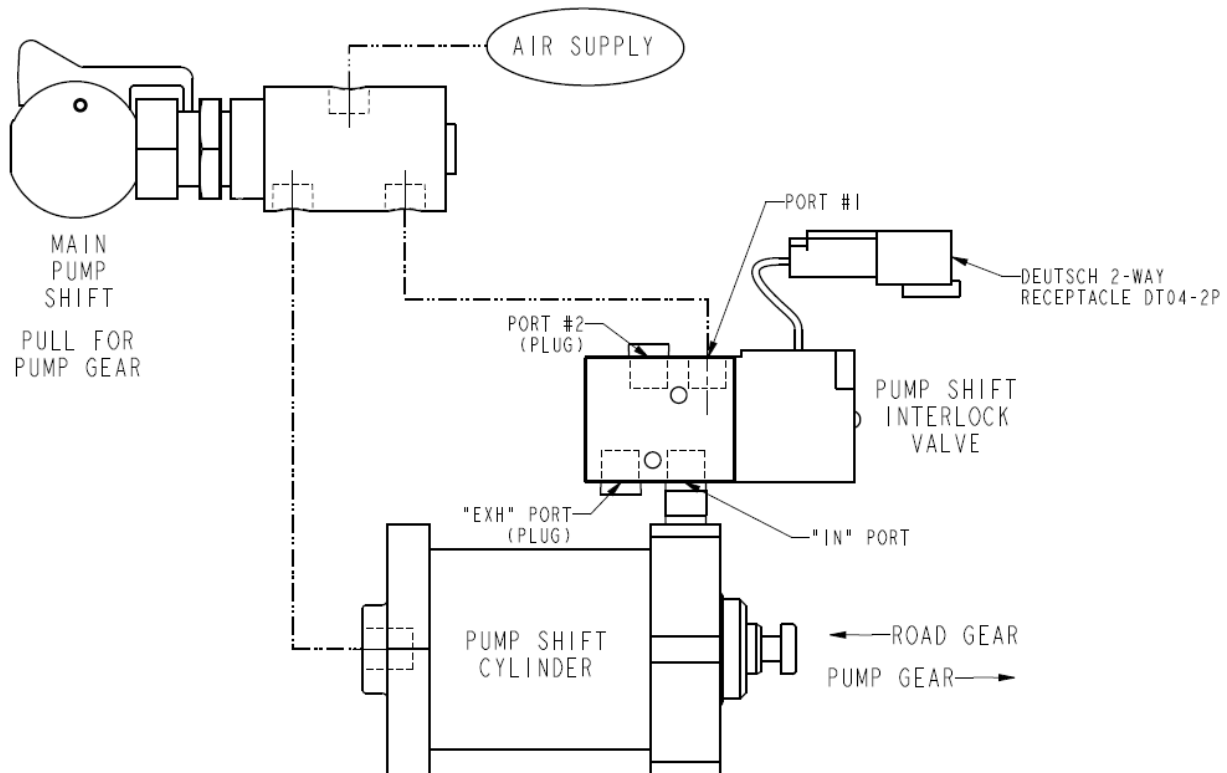
SHIFT SAFETY INTERLOCK SCHEMATIC MID-SHIP GEAR CASE

The safety interlock prevents the main pump shift from actuating while the automatic truck transmission is in drive gear. This eliminates clashing and possible damage of the pump gear set.



Assembly Notes:

- 1) Use Loctite 565 PST or equivalent pipe sealant on tapered pipe joints. (DO NOT USE TEFLON TAPE)
- 2) DO NOT over tighten fittings, doing so may damage thread.



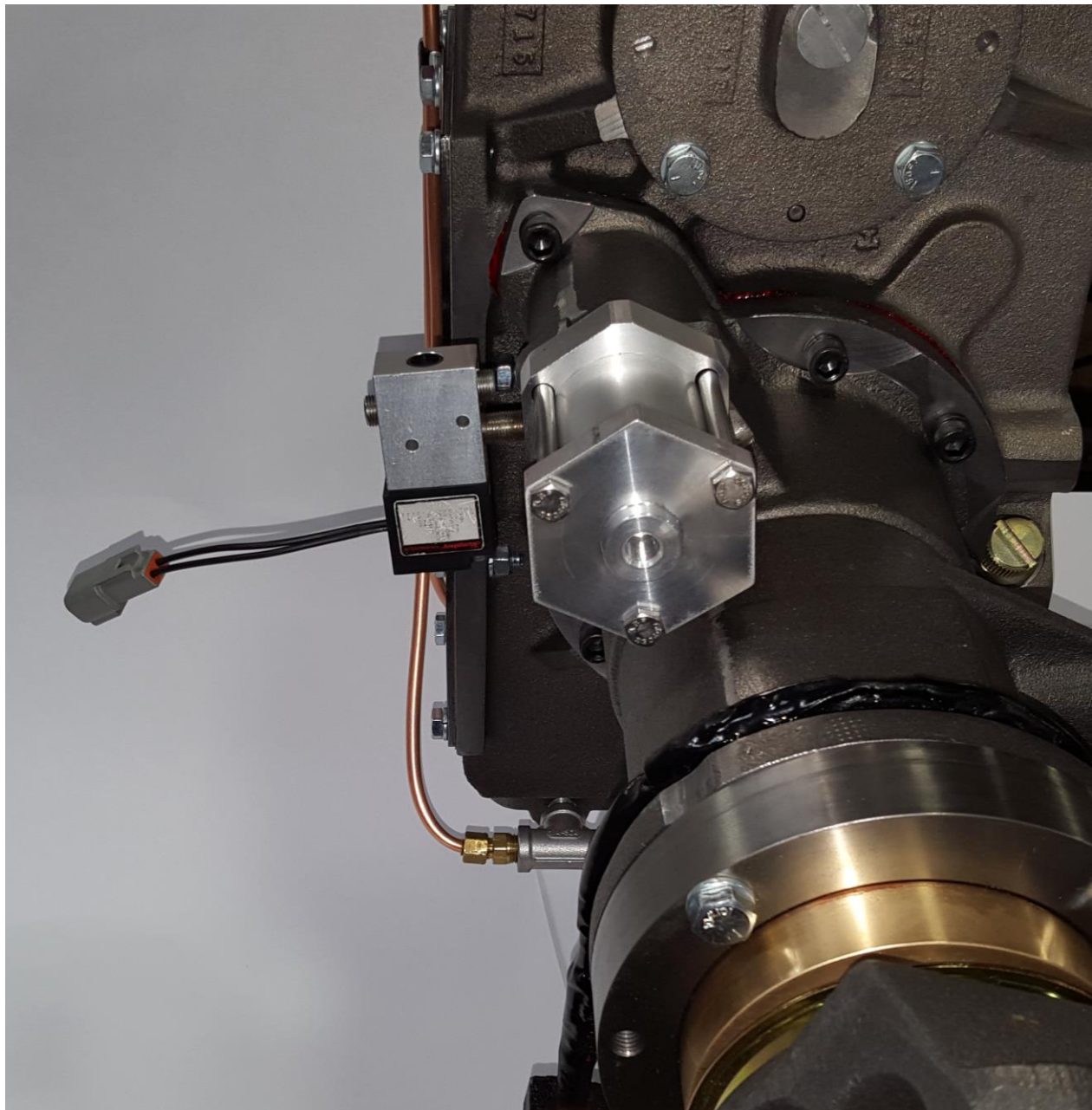
Prepared by: TED
Approved by: DJF

Date: Dec 07, 2015

Revision Date: Dec 07, 2015 - TED

Rev. #: 0
Page: 1 of 1

1205562.doc



SECTION 2

Air Compressor System Components, Operation, Maintenance

Description - Air Compressor System

A Gardner-Denver Tamrotor rotary screw air compressor provides compressed air for the Darley EMBC AutoCAFS II Compressed Air Foam System.

Rotary screw air compressors are widely used in industrial, transportation, and construction applications where compactness, high efficiency, smooth operation, and reliability are paramount.

The compressor air end is driven via the fire pump impeller shaft through a high performance, Gates Poly Chain drive belt. Compressor engagement is controlled by an electric multi-plate clutch system (1) providing hot shift capability. The air end and drive system components are rated to provide up to 220 CFM airflow at 125 psi.

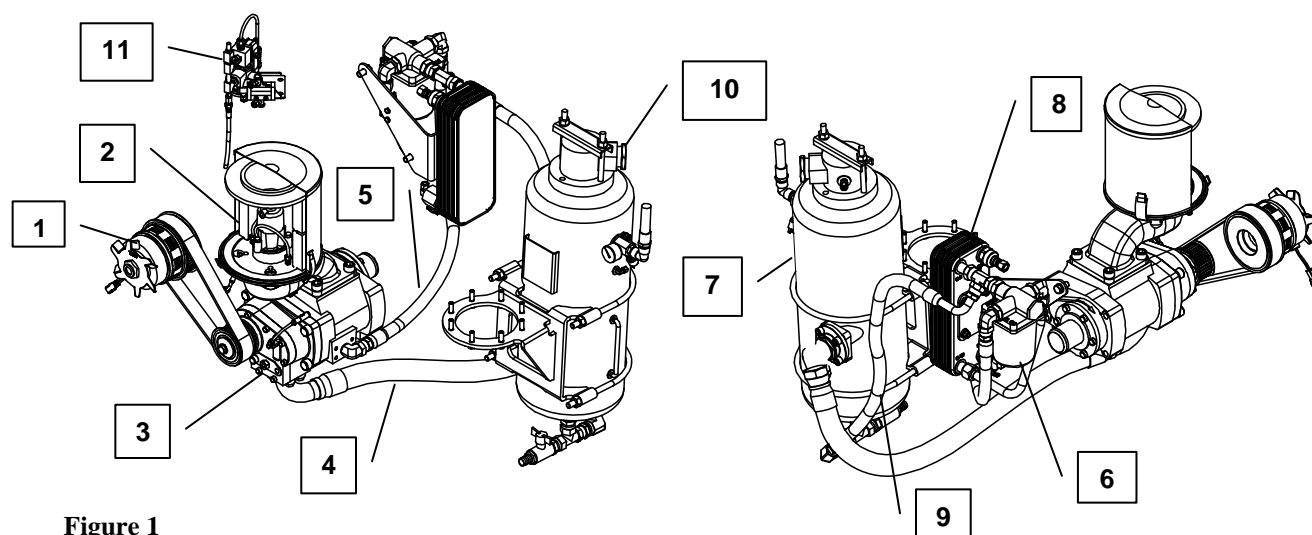


Figure 1

Referring to figures 1 and 2, the compressor system operates as follows: Air is drawn in through the filtered inlet-modulating valve (2) that also functions as a non-return valve during shut down. From the inlet valve, air enters the air end (3) where pressurization occurs. Cooling and lubricating oil is continuously injected into the rotor housing through hydraulic supply line (5). The pressurized air/oil mixture discharged from the air end flows through a hydraulic hose (4) into the oil receiver/separator tank (7) where oil is removed from the pressurized air.

Oil removal is a two-step process. Most of the oil is removed by the centrifugal effect of the cyclone in the lower part of the receiver (7). The remaining oil is removed by two coalescing elements located in the upper region of the separator tank (7). The oil removed by the separator elements is then returned to the air end via oil return line (13). An orifice in this return line restricts air circulation back to the air end. Clean air is then discharged through valve port (10).

From the oil separator tank (7), hot oil flow through hose connection (9) is led through oil cooler (8) to cool down the screw unit. The oil circuit includes a thermostat (15) in the filter head that bypasses the cooler when the oil is cold.

Oil circulation is forced, and is maintained by the pressure difference between the receiver and the screw unit. To keep oil in circulation under all operating conditions, discharge port (10) includes a minimum pressure check valve (12). This valve prevents the receiver pressure from dropping below 45 psi, thus assuring continuous oil flow through the system.

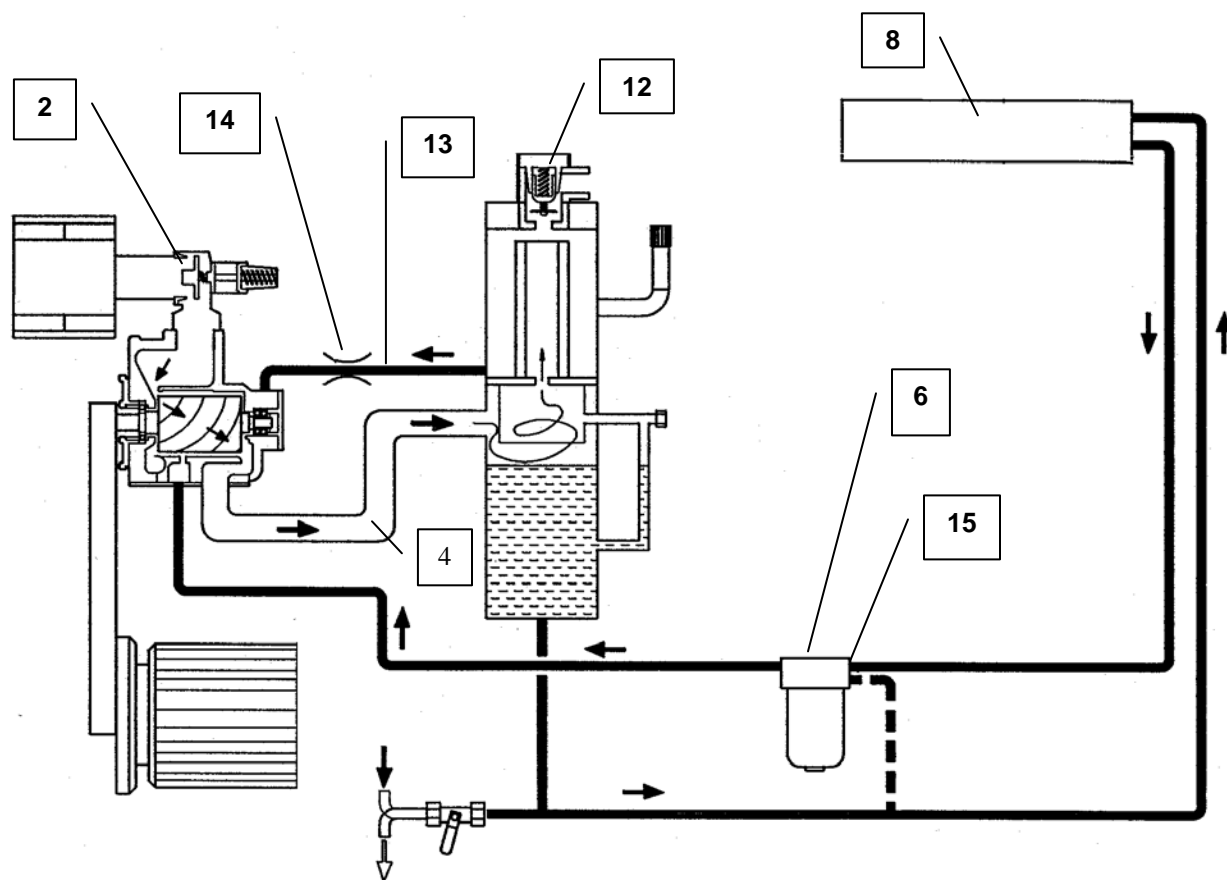


Figure 2

AutoCAFS II pressure balance valve assembly (11) includes a pressure balancing system and a system blow down valve.

Refer to drawing DCS0502 for review of control system schematic.

Control Sensitivity Adjustment

Should the needle valve need adjustment, use the following as a guide. Start by closing the valve (4) completely. Then open it approximately 3 turns. Operate the unit at around 125 PSI, begin by flowing about one third the capacity of the air compressor. At this flow rate, the air inlet modulator valve will open to bring in air and then close as air pressure builds. The goal is to set the needle valve at a position where pressure fluctuations are minimized. If the red needle on the pressure gauge is fluctuating more than 20 PSI above or below the water pressure, then the needle valve should be adjusted out or counter-clockwise. As the pressures come closer to balancing, less flow meter fluctuation should also be noticed. **Note: Some pressure modulation is normal and required for the system to auto-balance while delivering CAFS. Expect pressure variation to range from 5-20 psi.**

Compressor System Pressure Limiting Valve

A pressure-limiting valve (6) is incorporated in the control airline to limit maximum air pressure to a preset value. This valve is factory preset to and should be maintained at 150 psi. As such, the compressor control system will maintain a balance between water and air up to 150 psi.

Pressure Limiting Valve Adjustment

Engage pump and compressor using prescribed methods. Initiate water flow through the pump to assure circulation through the heat exchanger, maintain a 50-100 gpm flow rate. Increase pump pressure to approximately 175 psi. Adjust air pressure manual adjustment valve (6) clockwise to increase pressure, setting the air pressure (red needle) to 150 psi.

To test setting, open an airflow valve on a CAFS discharge until air pressure drops, and then close it again. The pressure should quickly build back up to the maximum governed pressure as set by the manual pressure valve.

Important Note: Choose a discharge that will safely discharge plain air to atmosphere such as deck gun. ***Do not discharge air into a preconnected bed of lay flat hose.***



CAUTION:

- Do not over speed compressor - Input RPM should not exceed that required to produce rated air flow of 220 cfm at 150 psi maximum pressure.
- Disengage air compressor when service testing or performing UL test on CAFS equipped vehicle.

System Blow Down (Depressurization)

After compressor shutdown, system pressure is bled off to guard against overloading drive components at startup. If the receiver assembly is not

depressurized on shut down, oil will flood the compressor filling the area above the screws. Oil trapped above the screws will then cause a hydraulic lockup when compressor rotation rapidly accelerates during startup. A hydraulic lockup of this type can induce extreme loads on the power train.

A blow down valve (5) is included in the system to automatically relieve system pressure at shutdown. System blow down valve (5) is a basic 2-way pilot operated pneumatic shuttle valve. When the compressor is operating, pilot pressure for shuttle valve port 'I', being connected to the inlet side of the compressor, is sensing a vacuum; ports 'R' and 'P' do not communicate.

At shutdown, inlet valve (3) closes, acting as a check valve. At the same time, inlet side of the compressor is pressurized from the receiver tank via the 1 ½" discharge line. Pilot port 'I' is in turn pressurized, shifting valve spool and connecting port 'R' to port 'P'. Receiver pressure is thus vented to atmosphere inside the filter housing (3). Allow a 1-minute minimum time period between compressor shutdown and restart for system blow-down.

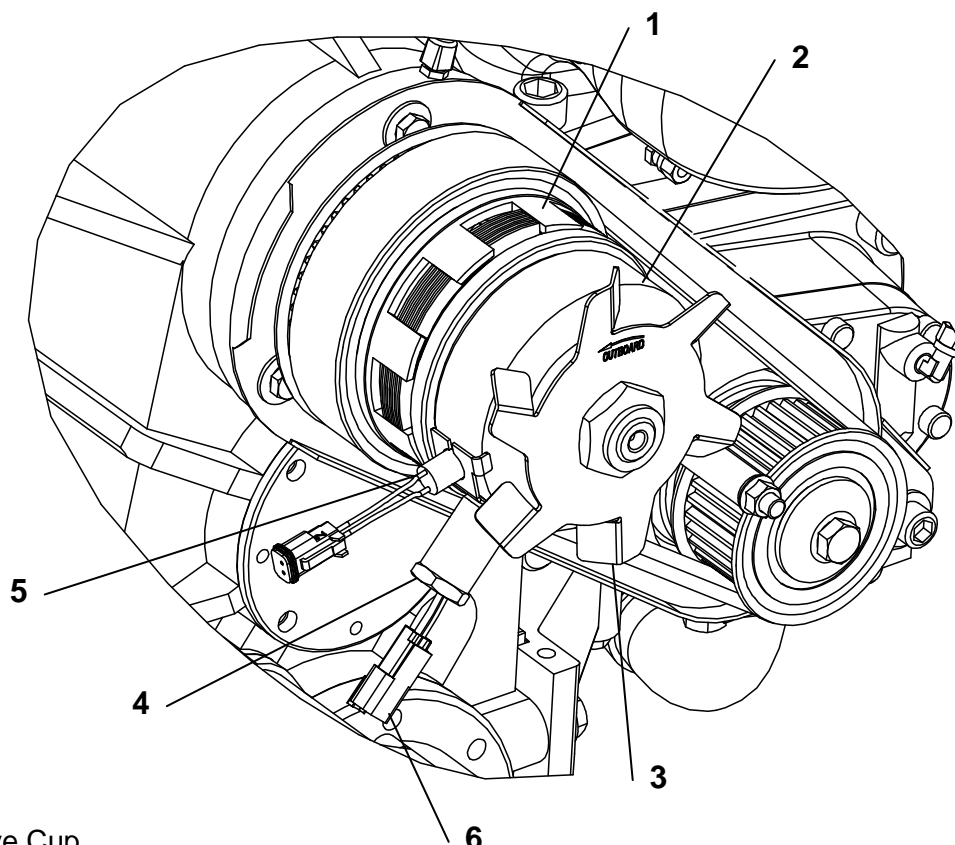
A separator tank pressure switch prohibits clutch engagement if tank pressure is above 10 psi thus assuring system blow-down before restart. *Always reduce engine rpm to 900 rpm or lower when switching the compressor engagement from DISENGAGE to ENGAGE.*



CAUTION: *Avoid immediate restart of compressor after shutdown. Allow a 1-minute minimum time period between compressor shutdown and restart for system blow-down.*

Compressor Clutch Assembly

The compressor air end is driven via the fire pump impeller shaft through a high performance, Gates Poly Chain drive belt. Compressor engagement is controlled by an electric multi-plate clutch system providing hot shift capability. Chassis electrical power is utilized to provide engagement of the clutch.



1. Drive Cup
2. Clutch
3. Cooling Fan
4. Clutch Anti-rotation Bolt
5. Temperature Sensor
6. Electric Clutch Power Connection

12 VDC $\pm 10\%$ must be supplied to the clutch (2) for proper performance. If supplied voltage is too low, then the clamping force on the clutch discs may not be adequate to carry compressor torque loads at full capacity. This will result in clutch slippage and consequent overheating. Power is supplied to the clutch through the AutoCAFS Commander control module. Refer to Section 3 of this manual for further details.

Compressor Engagement RPM

The compressor may be engaged before or after the pump is engaged, however, do not engage compressor when engine is turning faster than 900 rpm. Engine rpm must be reduced to 900 rpm or lower before engagement. The AutoCAFS Commander module will only allow compressor engagement at engine speeds below 900 rpm. The Commander will display 'RPM >900' when engagement is requested with rpm higher than 900 rpm. Refer to Section 3 of this manual for further details on the AutoCAFS Commander control module.

Compressor Disengage RPM

The compressor can be switched off (DISENGAGED) at any time or input speed.



CAUTION: *Avoid immediate restart of compressor after shutdown. Allow a 1-minute minimum time period between compressor shutdown and restart for system blow-down.*

Maximum Compressor RPM

Air pressure will match water pressure up to 150 PSI if pump input speed is adequate to maintain flow rate setting. Note: Do not exceed 175-PSI pump pressure while compressor is engaged. Maximum air pressure has been factory preset to 150 PSI.

To avoid compressor over-speed, the commander will display a warning message 'OVERSPD' when engine rpm approaches maximum allowable compressor speed. The Commander is by default programmed to provide a visual speed warning at $3650 \div \text{pump ratio}$. As an example, if the pump has a 2.44:1 ratio, over-speed warning would be at 1500 engine rpm.

If the engine rpm continues to increase to the maximum allowable compressor rpm, $4500 \div \text{pump ratio}$, the Commander will automatically disengage the compressor. As an example, if the pump has a 2.44:1 ratio, over-speed disengagement would be at 1844 engine rpm.

If the compressor is disengaged due to over-speed, engine rpm must be reduced to 900 rpm and the compressor system must blow-down before re-engagement can occur. Refer to Section 3 of this manual for further details on the AutoCAFS Commander control module.



CAUTION:

- Do not over speed compressor - Input RPM should not exceed that required to produce rated air flow of 220 cfm at 150 psi maximum pressure.
- Disengage air compressor when service testing or performing UL test on CAFS equipped vehicle.

System Temperature Sensors

The AutoCAFS Commander incorporates two thermal sensors.

A transmission overheat warning will be displayed on the Commander if temperature sensor (5) on the clutch rises above its limit. The Commander display will alternately flash 'SHUTDOWN' – 'TRANS HOT'. The Commander will automatically disengage the compressor clutch if an over heat condition occurs.

A second thermal sensor is attached to the compressor air end with a digital display on the AutoCAFS Commander. This sensor is incorporated into the compressor engagement system to avoid compressor over heating that may result in premature bearing failure, scored housing or rotor seizure. If compressor temperature rises above normal operating temperature to 212°F, the Commander will flash a warning 'COMP HOT' and the compressor temperature will be displayed. *If temperature warning is indicated, shut down the compressor as soon as practical.* The compressor can be switched off (DISENGAGED) at any time or input speed.

If compressor temperature is allowed to increase to 240°F, the AutoCAFS Commander will automatically disengage the compressor. At this time the Commander will alternately display 'SHUTDOWN' – 'COMP HOT' along with the actual compressor temperature.

Check for adequate water flow through heat exchanger. Check for adequate oil level in separator tank. See trouble-shooting guide for further options. *Do not restart compressor until source of problem is determined and rectified.*



WARNING: *If compressor temperature continues to rise to 240°F, the compressor will be automatically disengaged.*

Compressor Maintenance

	Daily or After Use	25 Hr	6 Mo	100 Hr	12 Mo	2000 Hr	24 Mo
Check Oil Level	X ¹	X	X				
Check Air Filter		X ²					
Change Oil/Filter				X	X		
Replace Air Filter				X ³	X		
Check Safety Valve					X		
Inspect Hoses and Fittings						X	X
Inspect Drive Belt						X	X
Replace Oil Separator Elements						X	X

- 1) Check oil in stopped compressor (wait until air and oil are separated)
- 2) Check air filter more frequently under adverse/dusty operating conditions.
- 3) As conditions dictate

The air filter is the most important filter in the system; if it is kept clean the other filters will also stay cleaner. Always use a new filter element; **DO NOT** blow out element with compressed air and reuse.

Compressor Oil:

It is recommended that a circulation oil (hydraulic oil) or synthetic lubricating oil per the following specifications be used.

Mineral Oil:

Use compressor oil specially made for screw compressors, including antioxidants and rust, foaming, and wearing preventative components.

Synthetic Lubricant:

Use compressor oil specially made for screw compressors, including antioxidants and rust, foaming, and wearing preventative components.

Viscosity:

- Maximum 500mm²/s (centistokes) at startup temperature
- Minimum 7mm²/s at running temperature (185°F)

Flash point:

- Minimum 360°F

Under normal conditions, the above requirements are fulfilled using and ISO VG 32 oil.

Examples:

Phillips 66 MAGNUS OIL ISO VG 32 (mineral oil) or

Phillips 66 SYNDUSTRIAL E Compressor Oil 32 (synthetic)

Approximate Capacity - 12 to 16 Qt.

Compressor Oil Filter: Part No. 1122802, (1) req'd

Compressor Separator Cartridge: Part No. 1122702, (2) req'd

Air Filter Element: Part No. 1122601, (2) req'd

NOTE:

Refer to pump and apparatus manual (Section 1) for maintenance requirements of the main pump and components.

Refer to proportioner manual (Section 4) for maintenance requirements on the foam proportioner system.

Oil Change



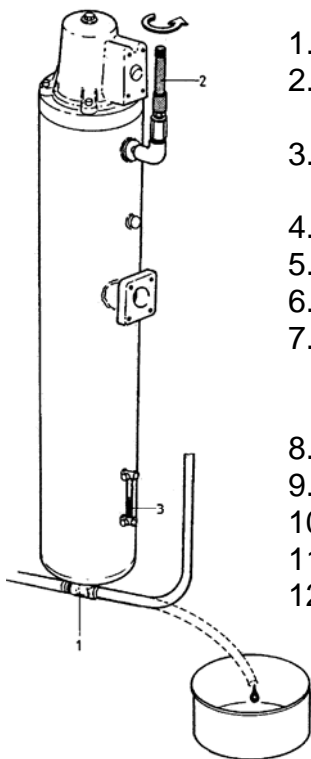
WARNING:

- Oil is hot when compressor is first shut down (185°F); allow the compressor system to cool before starting maintenance work.
- Dispose of the used oil according to regulations on waste oil.
- Do not open the oil drain valve if receiver is pressurized. Open safety valve 4-5 turns before opening oil/fill valve.



CAUTION:

- Use recommended oil types only; do not mix different oil types.



1. Run the compressor to warm up the oil to approximately 100°F.
2. Stop the compressor and check that the receiver is not under pressure. After stopping, blow down empties the compressor; wait approx. 2 minutes.
3. Secure apparatus so it cannot be started while maintenance is being performed.
4. Open safety valve (2) 4-5 turns.
5. Open drain/fill valve (1) and let oil run into suitable container.
6. Close drain/fill valve (1). Drain and clean fill hose.
7. Confirm correct oil type. Using a filtered funnel, fill receiver tank to mark on oil level indicator (3). Use care to assure oil system is kept clean and free of contamination.
8. Close safety valve (2).
9. Replace oil filter.
10. Run compressor for 1 minute.
11. Stop compressor.
12. Allow air and oil to separate; recheck oil level.

**DRAIN
VALVE**



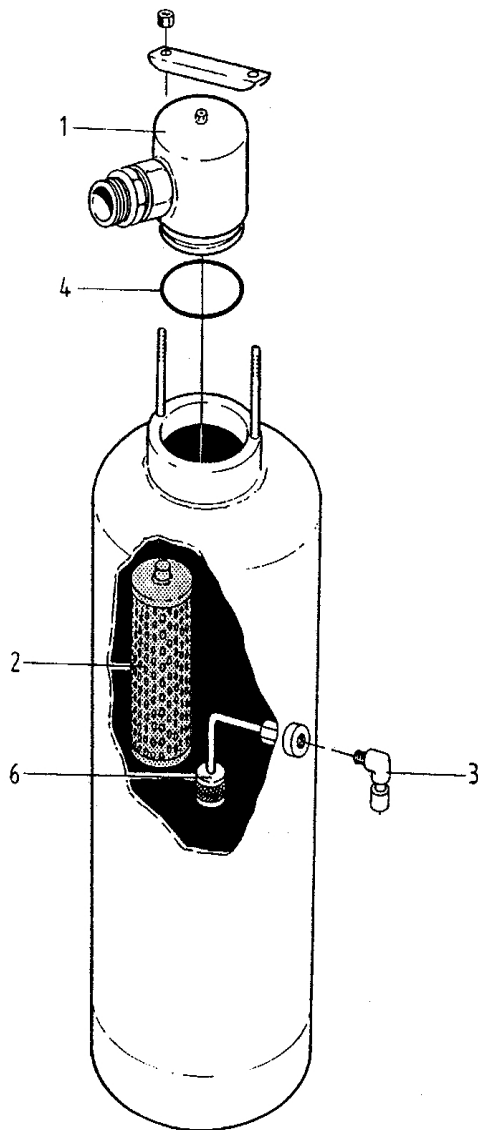
FILL PORT

Replacing Oil Separator Element



WARNING:

- Allow the compressor to cool down before starting maintenance work.
- Dispose of the used separator element according to regulations on toxic waste.



REMOVAL

1. Stop the compressor and check that the receiver is not pressurized. After shutdown, allow 2 minutes for system blow down.
2. Make sure system cannot be started while maintenance is being performed.
3. Remove output valve (1).
4. Remove the separator elements (2) by removing the two SHCS that retain the elements.

INSTALLING

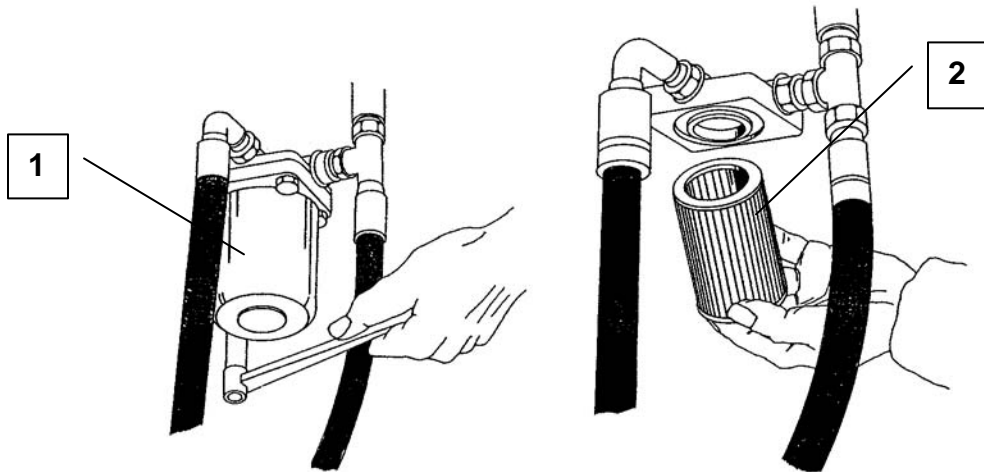
1. Carefully clean the sealing surfaces on the receiver and output valve (1).
2. Clean the .7mm orifice in the oil return line which is located in the fitting sleeve at the compressor (see DWG DCS0500)
3. Clean the return oil screen filter (6) (inside the receiver) by blowing in pressurized air through fitting (3).
4. Install the new separator elements (2) in place. Secure with two SHCS.
5. Check the condition of the sealing of the output plate.
6. Inspect seal (4), replace if damaged.
7. Install the valve assembly (1).
8. Tighten retaining nuts alternately and evenly.

Replacing Oil Filter



WARNING:

- Allow the compressor to cool down before starting maintenance work.
- Dispose of the used filter element according to regulations on toxic waste



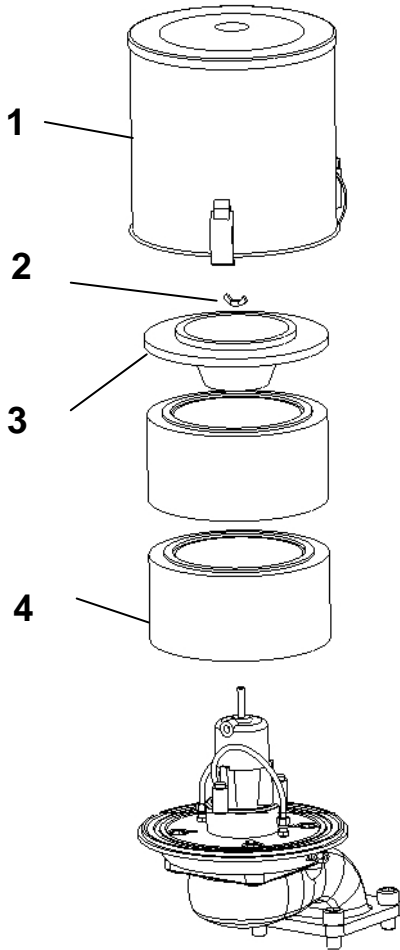
1. Stop the compressor and check that the receiver is not pressurized. After shutdown, allow 2 minutes for system blow down.
2. Make sure system cannot be started while maintenance is being performed.
3. Remove the filter housing cover (1) and take out the old filter (2).
4. Install a new filter element (2).
5. Inspect cover o-ring and replace if required.
6. Replace cover.
7. Tighten the cover mounting screws alternately and evenly.

Replacing Air Filter Elements



WARNING:

- Allow the compressor to cool down before starting maintenance work.



REMOVAL

1. Toggle three retaining clips and remove filter housing (1).
2. Remove wing nut (2) and retaining plate (3).
3. Remove and discard the filter elements (4).

INSTALLING

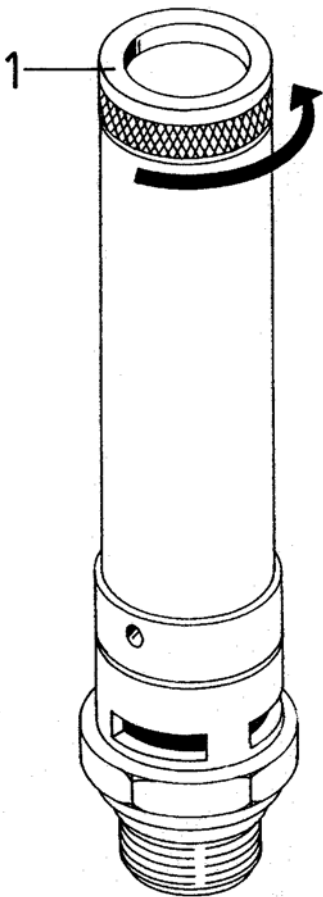
1. Carefully clean and inspect the sealing surfaces and housing components.
2. Install two new filter elements (4).
3. Assemble retainer plate (3) and wing nut (2).
4. Replace filter housing (1) and fasten three retainer clips.

Testing Safety Valve



WARNING:

- Oil is hot when compressor is first shut down (185°F); allow the compressor system to cool before starting maintenance work.
- All adjusting and repair work on the safety valve must be left to a qualified mechanic (observe local regulations)
- *Never operate the compressor system with a malfunctioning, modified, plugged, or missing air safety valve.*



The receiver tank safety valve provides for pressure relief should the control system malfunction. The valve is factory preset at 200 psi and is non-adjustable.

The operation of the valve can be confirmed by turning the safety valve cap (1) counterclockwise 1-2 turns while the receiver is pressurized. Air should be released as the valve is opened. Close valve.

Opening (safety blow-off) pressure of the valve must be tested with the valve removed from the receiver tank and connected to test air supply.

Belt Adjustment and Replacement



WARNING:

- Stop the compressor and check that the receiver is not pressurized. After shutdown, allow 2 minutes for system blow down.
- Allow the compressor to cool down before starting maintenance work.
- Make sure system cannot be started while maintenance is being performed

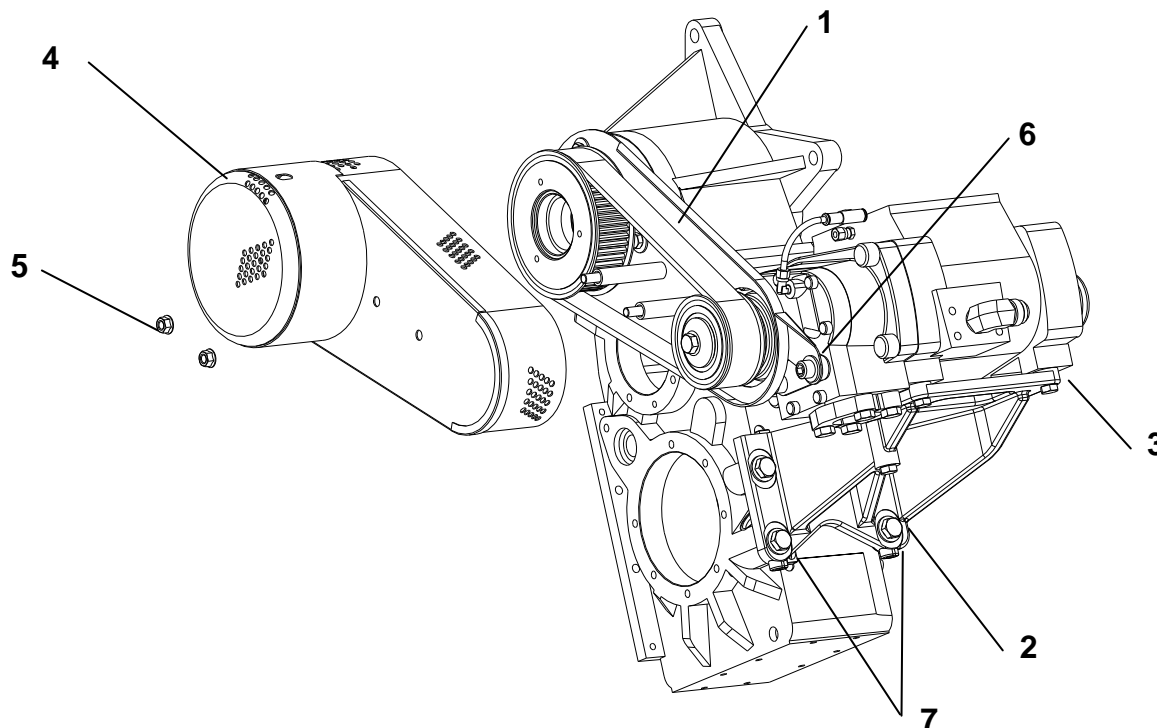


Figure 3

A high performance, poly chain, toothed belt drives the compressor. The belt is constructed using a combination of a chemical resistant elastomeric compound and Kevlar tensile cords that provide for virtually no elongation.

The belt has been properly tensioned on assembly. Under normal circumstances, the belt is maintenance free and will last for years of service. Should adjustment or replacement become necessary, use the following steps as a guide.

In addition to figure 3, please refer to drawings DLC1003, DCM0501 and DCM0700.

Belt Inspection and Adjustment

1. Remove belt cover retaining nuts (5). Move and temporarily secure pressure balance valve assembly so it is out of the way.
2. Remove clutch electrical connection, anti-rotation post (55) and temperature sensor wire connection (Ref. Dwg DLC1003).
3. Remove belt cover (4).
4. Inspect belt (1) for wear. Note that it is normal for a small amount of dust to accumulate around the belt housing as the belt breaks in.
5. Check for proper belt tension. A 22-pound force applied in the middle of the belt span should deflect the belt approximately 3/16 (.19) inches.
6. Should belt tension adjustment be required:
 - a. Loosen belt cover bracket mounting bolt (6) and four compressor bracket bolts (2).
 - b. Install 3/8-16NC jackscrews in compressor bracket at location (7).
 - c. Apply pressure to jackscrews until proper belt tension is achieved.
 - d. Tighten four compressor bracket bolts (2), torque to 50 ft lb.
 - e. Tighten belt cover bracket mounting bolt (6).
 - f. Remove jackscrews.
7. Replace belt cover (4) feeding temperature sensor wire through cover opening. Perforated edges of the cover should be positioned inside cover bracket flanges.
8. Secure belt cover with two retaining nuts (5).
9. Position temperature sensor and bracket (14) on clutch threaded wire inlet fitting. Apply 2-3 drops of Loctite 243 to male threads of wire inlet fitting. Slide anti-rotation post (55) over clutch wire connection and clamp bracket (14) in place. (Ref. Dwg DLC1003).

Belt Replacement

1. Remove belt cover retaining nuts (5). Move and temporarily secure pressure balance valve assembly so it is out of the way.
2. Remove clutch electrical connection, anti-rotation post (55) and temperature sensor wire connection (Ref. Dwg DLC1003).
3. Remove belt cover (4).
4. Loosen four compressor bracket bolts (2).
5. Remove belt cover bracket mounting bolt (6).
6. Remove four compressor bolts (3).
7. Without removing compressor drive sprocket from inside belt cover bracket clearance hole, lift and twist compressor assembly so that belt can be slipped over sprocket and removed.
8. Reverse procedure and slip new belt over sprockets.
9. Replace and tighten four compressor-mounting bolts (5).
10. Rotate and inspect the belt to confirm it has been seated properly.
11. Install 3/8-16NC jackscrews in compressor bracket at location (7).
12. Apply pressure to jackscrews until proper belt tension is achieved.
13. Tighten four compressor bracket bolts (2), torque to 50 ft lb.
14. Rotate belt by hand and recheck tension.
15. Tighten belt cover bracket mounting bolt (6).

16. Remove jackscrews (7).
17. Replace belt cover (4) feeding temperature sensor wire through cover opening.
Perforated edges of the cover should be positioned inside cover bracket flanges.
18. Reposition balance pressure valve assembly over cover mounting bolts and secure with two retaining nuts (5).
19. Position temperature sensor and bracket (14) on clutch threaded wire inlet fitting.
Apply 2-3 drops of Loctite 243 to male threads of wire inlet fitting. Slide anti-rotation post (55) over clutch wire connection and clamp bracket (14) in place. (Ref. Dwg DLC1003).

REVISIONS			
LTR	DESCRIPTION	DATE	CHG NO.
APPROD			
REPLACED SUCTION HEADS WITH 2810110			
A	ADDED COVER ASSEMBLY	16JUN11	7433
EAS2			
B	ADDED SPACERS - PART NO. 3301201	16JUN11	7433
EAS2			

OPTIONAL FOAM MANIFOLD AND VALVES
SEE MANIFOLD SELECTION GUIDE
FOR DISCHARGE EXTENSION AVAILABILITY
AND MANIFOLD OPTIONS

BELT COVER/BACKET
REF DWG DCM0501
ALLOW 2.50 INCH REAR CLEARANCE
FOR BELT/CLUTCH MAINTENANCE

BELT DRIVE CLUTCH
AND TRANSMISSION ASSEMBLY
REF DWG DLC1001

PRESSURE CONTROL ASSEMBLY
REF DWG DCM1001, DCS0502

AIR INLET VALVE/FILTER ASSEMBLY
REF DWG DCM0103

AIR END ASSEMBLY
REF DWG DCM0801

COMPRESSOR HEAT
EXCHANGER/FILTER ASSEMBLY
REF DWG DCM0106

STANDARD LDM PUMP FORWARD
CONFIGURATION

COMPRESSOR MOUNTING BRACKET
REF DWG DCM0700

TRANSMISSION COOLING PLATE ASSEMBLY
REF DWG DGC0218

OIL LEVEL
SIGHT GAGE

OIL SEPARATOR TANK
REF DWG DCM0801

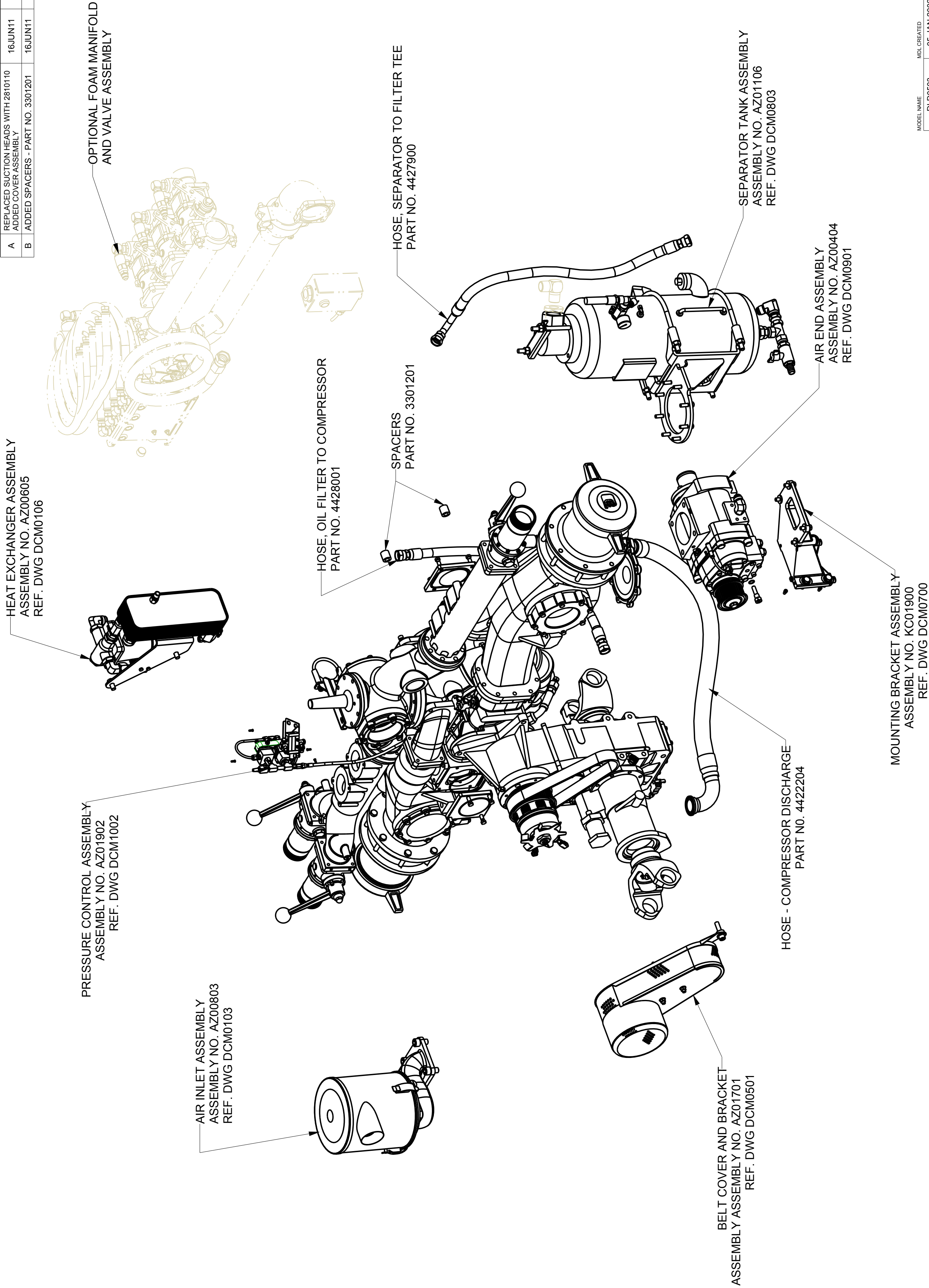
COMPRESSOR OIL FILL
AND DRAIN PORT/VALVE

FRONT VIEW

REAR VIEW

MODEL NAME		REV. CREATED		SHEET	
DLD0503		05-JAN-2002		1/3	
TOLERANCE		OLD PART NO.		D	
AS NOTED		PATTERNING		1/3	
0.005		MATERIAL NO.		DO NOT SCALE PRINT	
0.005		DATE		05-Dec-02	
0.005		DWG		SCALE	
0.005		TRCD		1/8	
REMOVE SHARP EDGES		ALL DIMENSIONS IN INCHES UNLESS NOTED		DLD0503	
MATERIAL DESCRIPTION:		THIS DESIGN IS THE PROPERTY OF W.S. DARTLEY & CO. UNLESS OTHERWISE NOTED. REPRODUCTION IS PROHIBITED.			
INCH [MILLIMETER]		DO NOT SCALE PRINT			
W.S. Dartley & Co.		DO NOT SCALE PRINT			
ITASCA, IL - CHIPPEWA FALLS, WI					
DWG - LDMBC, ELECT CLUTCH					
DIMENSIONAL, MAN. FORWARD					

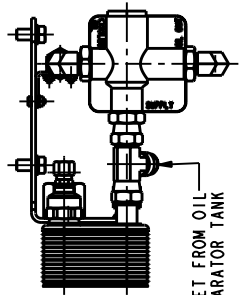
REVISIONS				
LTR	DESCRIPTION	DATE	CHG NO.	APPR'D
A	REPLACED SUCTION HEADS WITH 2810110 ADDED COVER ASSEMBLY	16JUN11	7433	EAS2
B	ADDED SPACERS - PART NO. 3301201	16JUN11	7433	EAS2



MODEL NAME		MDL CREATED		SHEET	
DLD0503		05-JAN-2002		2/3	
TOLERANCE		W.S. Darley & Co.		DATE	
EXCEPT		ITASCA, IL - CHIPPEWA FALLS, WI		11-Oct-00	
AS NOTED		DWG - LDMBC, ELECT CLUTCH		SCALE	
DIMENSIONS		DIMENSIONAL, MAN. FORWARD		1/8	
DRN. DWS		DO NOT SCALE PRINT		DLD0503	
CHKD. -		ALL DIMENSIONS IN INCHES UNLESS NOTED		DATE	
TRCD		THIS DESIGN IS THE PROPERTY OF W.S. DARLEY AND CO. - UNAUTHORIZED REPRODUCTION IS PROHIBITED		11-Oct-00	

NO.	DESCRIPTION	PART NO.	QTY.
1	BRACKET - HEAT EXCH	4016500	1
2	BRACKET - SUPPORT, FILTER	4017700	1
3	DRAINCOCK - 0.250 NPTM, 9HC BR	5203600	1
4	FITTING - .75BSPPMx.75JICFE SWIV	3501535	1
5	FITTING - .75BSPPMx.75JICM, 90	3501537	2
6	FITTING-TEE, SWIVEL NUT RUN	3501534	1
7	FTG - 1.00 NPTF DRAIN TEE	3503005	1
8	GROMMET - .38 ID x .63 MTG OD	4401725	4
9	HEAT EXCH - LDMBC BRAZED PLATE	4215102	1
10	HHCS - .250-20 x 0.63, GR5	5400001	2
11	HHCS - .500-13 x 1.00, GR5	5400065	2
12	NUT - FLANGED, M8-1.25, TOPLOC	5403603	4
13	NUT - FLANGED TOP LOCK	5403504	4
14	OIL FILTER - W/THERMOSTAT	4420902	1
15	REDUCER - PIPE, 1.00 X 0.38	1080214	1
16	TUBE - HEAT EXCHANGER	4427803	1
17	TUBE FITTING - EL, .38 x .25	3500511	1
18	TUBE FITTING - STR, .50 x .38	3500031	1
19	U-BOLT - .25"	4406801	1
20	WASHER - FLAT, 0.31 ID, SST	3603812	4
21	WASHER - LOCK, 0.500 ID	3603505	2

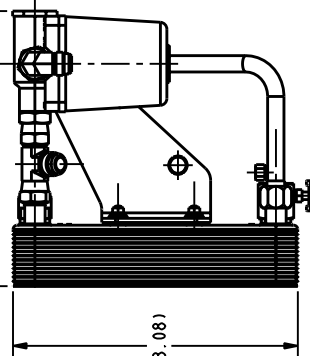
COOLING WATER RETURN TO PUMP
SUCTION - Ø .50 TUBE
NO DIPS IN LINE



OIL INLET FROM OIL
SEPARATOR TANK

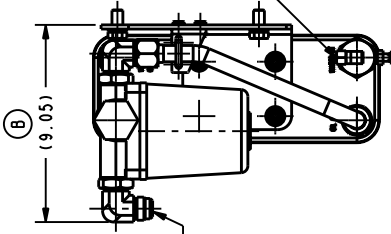
(12.63)

(9.05)



OIL OUTLET TO
AIR END

(13.08)

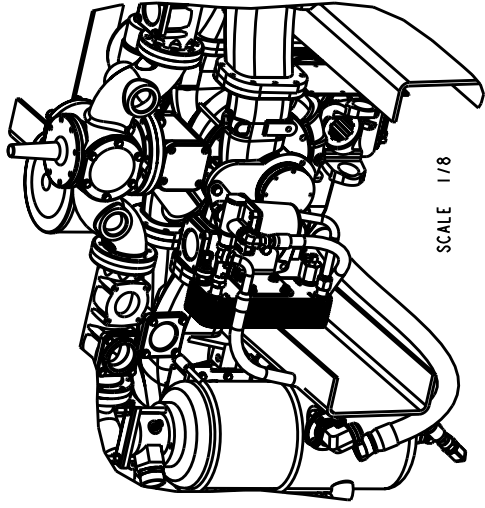


COOLING WATER INLET
Ø .38 FROM SCREEN
PROTECTED PUMP DISCHARGE
PORT

POSITION DRAIN PORT DOWN
CONNECT TO MULTIPLE DRAIN VALVE

SCALE 1/4

REV.	DESCRIPTION	DATE	ENG. NO.	APP'D
A	12.63 WAS 11.80	18JUL01	2001-181	DMS
B	9.05 WAS 8.13	18JUL01	2001-181	DMS
C	3403603 WAS 3403468	18JUL01	2001-181	DMS
D	3500507 WAS 3500031	25AUG05	2005-250	DMS
E	3500031 WAS 3500507	10MAR09	5238	AAN

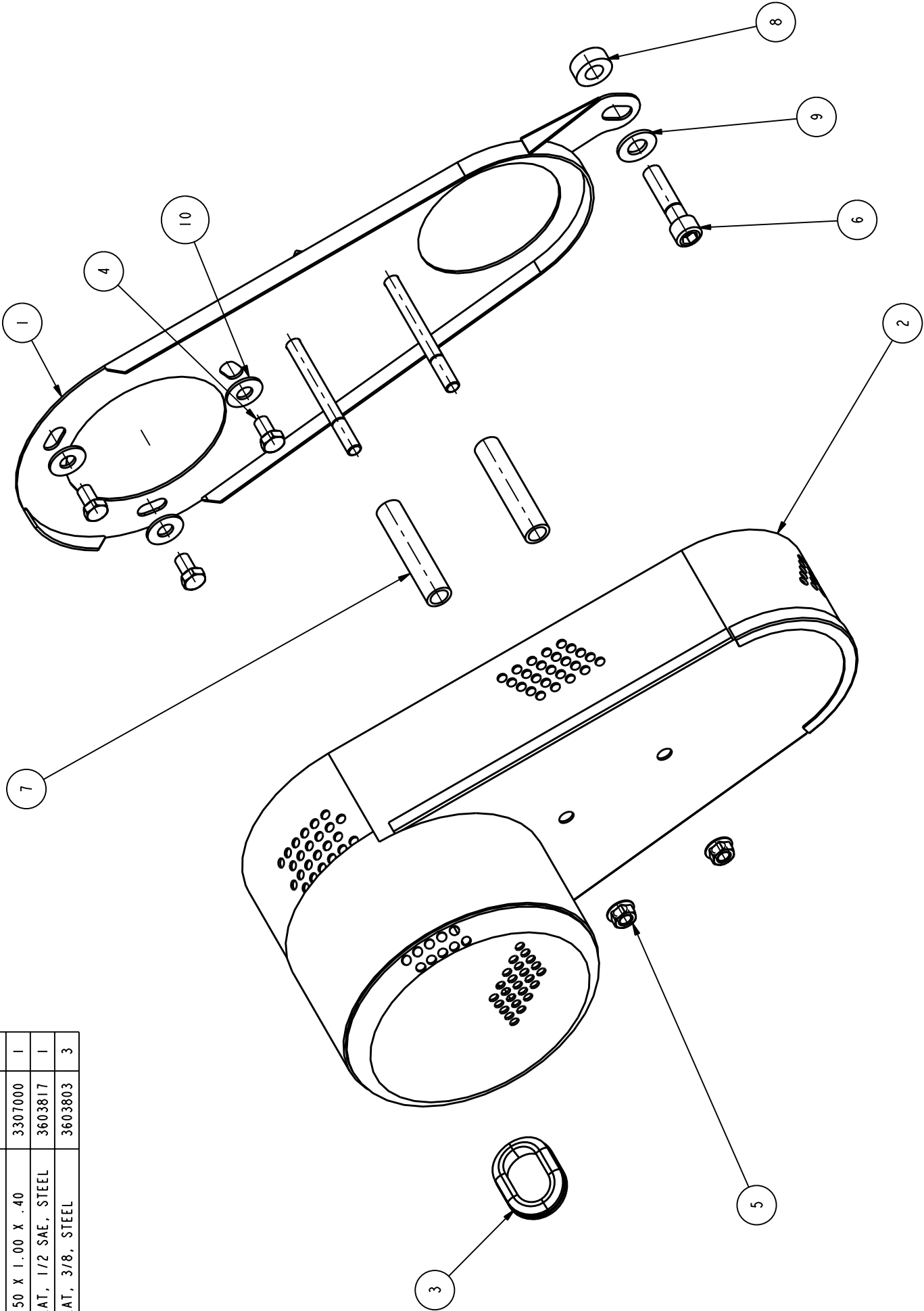


SCALE 1/8

DRAWING NUMBER A200605		REV. CREATED 23-APR-2001		SHEET 1/1	
TOLERANCE AS NOTED EXCEPT UNLESS SHOWN OTHERWISE FRACTIONS 25 & 50 DECIMALS .01		MATERIAL NO.		PATTERN NO.	
INCH (MILLIMETER)		REMOVE SHARP EDGES		DO NOT SCALE PRINT	
MATERIAL DESCRIPTION: THIS DESIGN IS THE PROPERTY OF W.S. DARLEY AND CO. UNAUTHORIZED REPRODUCTION IS PROHIBITED		ALL DIMENSIONS IN INCHES UNLESS NOTED		DATE 04-MAY-01 SCALE 5/16	
W.S. Darley & Co. 11360-111 CUTLERMAN FALLS, MI. DWG - ASSEMBLY, HEAT EXCH. PUMP MOTOR, A200605		DCM0106			

NO.	DESCRIPTION	PART NO.	QTY.	REVISIONS			
				LTR	DESCRIPTION	DATE	CHG NO.
							APPR'D

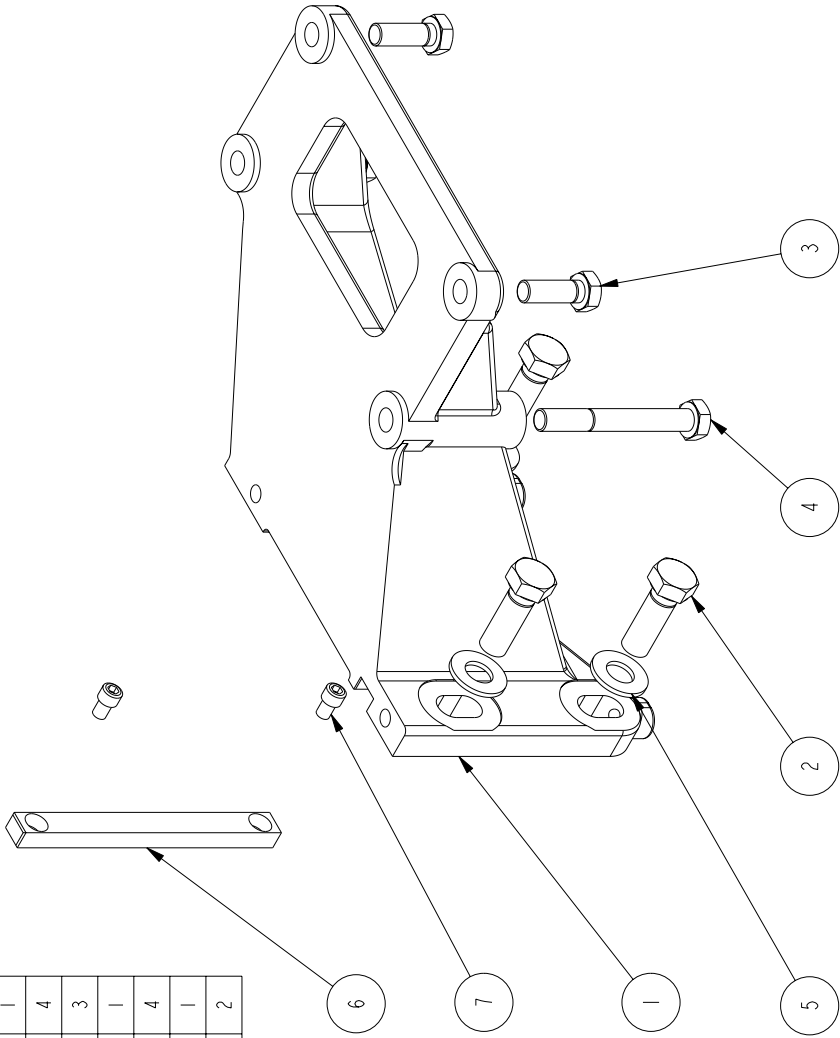
1	BRACKET - BELT COVER	4027900	1
2	COVER - BELT, ELECTRIC CLUTCH	2505401	1
3	GROMMET - 1.50 ID X 1.75 MTG OD	4401726	1
4	HHCS - .375-16 x 0.63, GR5	5400033	3
5	NUT - FLANGED TOP LOCK	5403468	2
6	SHCS - M12-1.75 x 50MM, GR8.8	5401211	1
7	SPACER - 0.49 x 0.68 x 3.18	3306705	2
8	SPACER - 0.50 X 1.00 X .40	3307000	1
9	WASHER - FLAT, 1/2 SAE, STEEL	3603817	1
10	WASHER - FLAT, 3/8, STEEL	3603803	3



MODEL NAME		DWG. CREATED		SHEET	
A201701		14-JUNE-2002		171	
TOLERANCE		W.S. Darley & Co.		C	
EXCEPT		CHITPEWA FALLS, WI - MELROSE PARK, IL			
AS NOTED		DWG - ASSEMBLY DETAIL, CAFS			
.00 ± .03		BELT COVER/BRKET, ELECT CLUTCH			
ANGLES ± 1°		DATE 14-Jun-02			
OR "N" DIMS		SCALE		1/2	
CHMD TCD		DO NOT SCALE PRINT		DCM0501	
TRCD					
REMOVE SHARP EDGES		INCH			
		(MILLIMETER)			
MATERIAL DESCRIPTION:		ALL DIMENSIONS IN			
		INCHES UNLESS NOTED			
		THIS DESIGN IS THE PROPERTY OF			
		W.S. DARLEY AND CO. - UNAUTHORIZED			
		REPRODUCTION IS PROHIBITED			

NO.	DESCRIPTION	PART NO.	QTY.
1	BRACKET - COMPRESSOR MOUNT	4027800	1
2	HHCS - .500-13 x 1.50, GR5	5400067	4
3	HHCS - M10-1.50 x 30MM, GR8.8	5400309	3
4	HHCS - M10-1.50 x 80MM, GR8.8	5400305	1
5	WASHER - FLAT, 1/2 SAE, STEEL	3603817	4
6	KEY - 0.38 x 0.50 x 5.50	3602311	1
7	SHCS - .250-20 x 0.38, GR8	5401000	2

REVISIONS			
LTR	DESCRIPTION	DATE	CHG NO. APPR'D



LOCTITE 243 OR EQUIV
TORQUE 50-55 FT-LB

LOCTITE 243 OR EQUIV
TORQUE 30-35 FT-LB

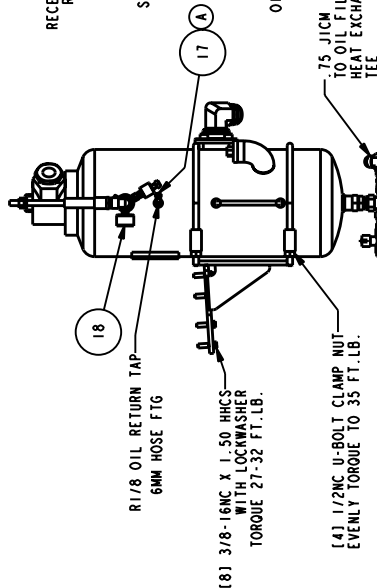
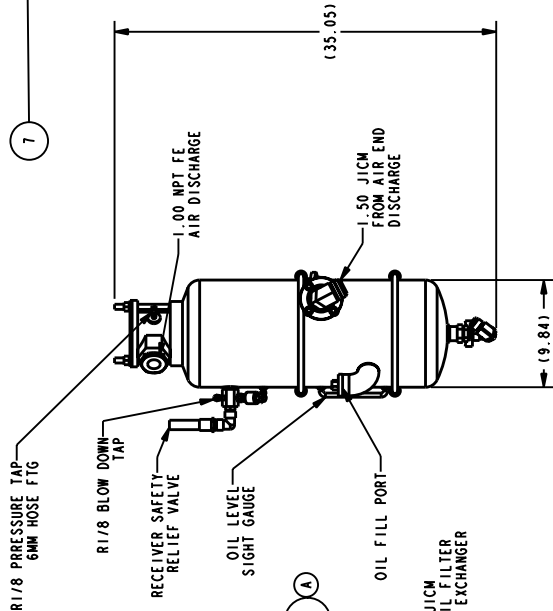
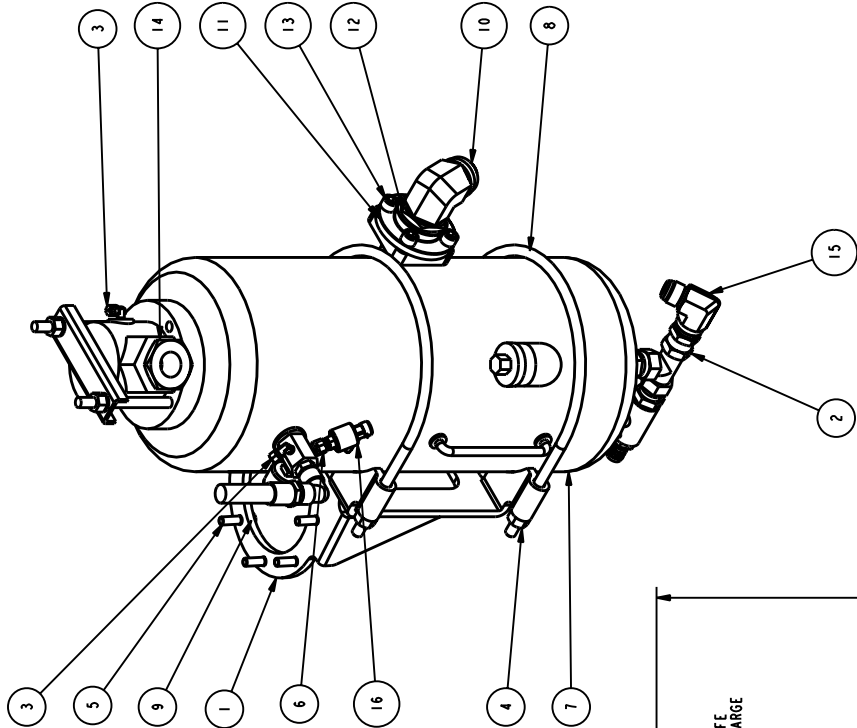
LOCTITE 243 OR EQUIV
TORQUE 30-35 FT-LB

REMOVE SHARP EDGES	INCH [MILLIMETER]	VIEW ANGLE PROJECTOR	OLD PART NO.	TOLERANCE EXCEPT AS NOTED - .00 ± .03 ANGLES ±1°	MODEL NAME KC01900	MDL CREATED JUNE 14, 1999	SHEET 1/1
MATERIAL DESCRIPTION: -			PATTERN NO.		W.S. Darley & Co. MELROSE PARK, IL - CHIPPEWA FALLS, WI		
THIS DESIGN IS THE PROPERTY OF W.S. DARLEY AND CO. - UNAUTHORIZED REPRODUCTION IS PROHIBITED	ALL DIMENSIONS IN INCHES UNLESS NOTED	DO NOT SCALE PRINT			DWG - ASSEMBLY, MTG BRACKET LDMC/TAMROTOR		
					DATE 14-JUN-99 SCALE 1/2		
					DCM0700		

REVISIONS			
LTR	DESCRIPTION	DATE	CHK. NO.
A	ADDED BARB FITTINGS	03/11/08	2008-0800 R JG

NO.	DESCRIPTION	PART NO.	QTY.
1	BRACKET - OIL RECEIVER TANK	4028102	1
2	FITTING - .75NPTM x .75SPPS SWV	3501550	1
3	FITTING - .12 BSPT x 6MM HOSE, 90	3503000	3
4	NUT - HEX, .500-13, HEAVY, GR2	5403018	4
5	HHCS - .375-16 x 1.50, GR5	5400038	8
6	FITTING - .12 NPT M x .25NPTF	3501556	1
7	TANK - OIL RECEIVER	4218600	1
8	U-BOLT - TANK MOUNTING	4406805	2
9	WASHER - LOCK, 0.375 ID	3603503	8
10	FITTING - 1.50 JICM x 1.50 NPS, 90	3501543	1
11	FLANGE - RECEIVER TANK INLET	1925300	1
12	O-RING - 2.38 x 2.56 x 0.09	3601123	1
13	SHCS - M12-1.75 x 30MM, GR8.8	5401213	4
14	ADAPTER - 1.5 BSPTM x 1.0 NPTF	1602301	1
15	FITTING - .75 JICM x .75 NPTF, 90	3501551	1
16	TRANSDUCER - PRESSURE, 300 PSI	4224500	1
17	FTG - SLEEVE, 4MM x 6MM TU	3503507	3
18	GAGE - AIR PRESS, 0-16 BAR	2603093	1

* REF PART 4224500 IS A COMPONENT OF 2603400 CONTROL



SCALE 1/8

MAX. PRESSURE: 205 PSI
OIL VOLUME: 12.7 QTS.
APPROXIMATE WEIGHT: 140 LBS w/OIL

REMOVE SHARP EDGES		INCH (MILLIMETER)		TOLERANCE		DATE 15-FEB-02		SCALE 1/4	
MATERIAL DESCRIPTION:		MATERIAL NO.		PATTERN NO.		DO NOT SCALE PRINT		TCD	
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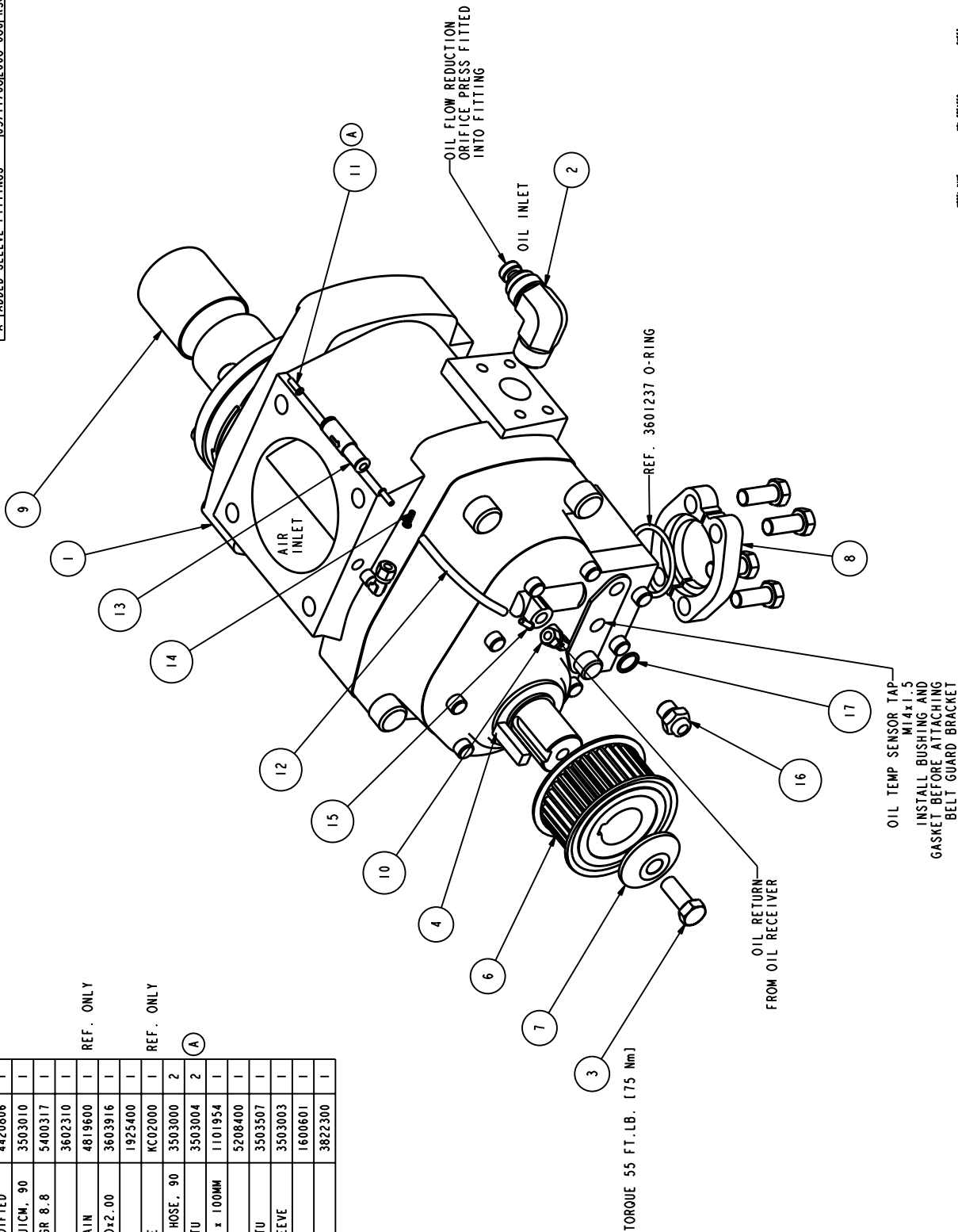
REVISIONS			
LTR	DESCRIPTION	DATE	CHKG NO.
A	ADDED SLEEVE FITTINGS	03/11/08	008-080 R.J.G

NO.	DESCRIPTION	PART NO.	QTY.
1	COMPRESSOR - 225 CFM, MODIFIED	4420806	1
2	FITTING - .75BSPTM x .75JICM, 90	3503010	1
3	HHCS - M12-1.75 x 30MM, GR 8.8	5400317	1
4	KEY - 10MM x 8MM x 36MM	3602310	1
6	SPROCKET - GATES POLY CHAIN	4819600	1
7	WASHER - SPECIAL, 0.53 IDx2.00	3603916	1
8	FLANGE ASSEMBLY	1925400	1
9	SLEEVE - SHAFT PROTECTIVE	KC02000	1
10	FITTING - .12 BSPT X 6MM HOSE, 90	3503000	2
11	SLEEVE - FTG, 4MM X 6MM TU	3503004	2
12	TUBE - OIL RETURN, 6x4MM x 100MM	1101954	1
13	VALVE - CHECK, 6MM TUBE	5208400	1
14	FTG - SLEEVE, 4MM x 6MM TU	3503507	1
15	ORIFICE - 0.7MM X 4MM SLEEVE	3503003	1
16	BUSHING - THERMO SENSOR	1600601	1
17	GASKET - BUSHING, COPPER	3822300	1

REF. ONLY

REF. ONLY

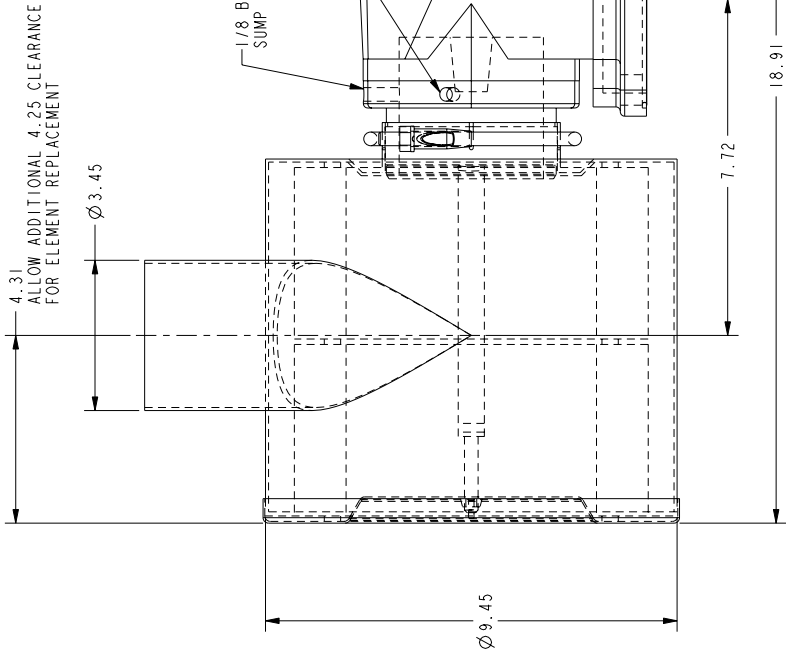
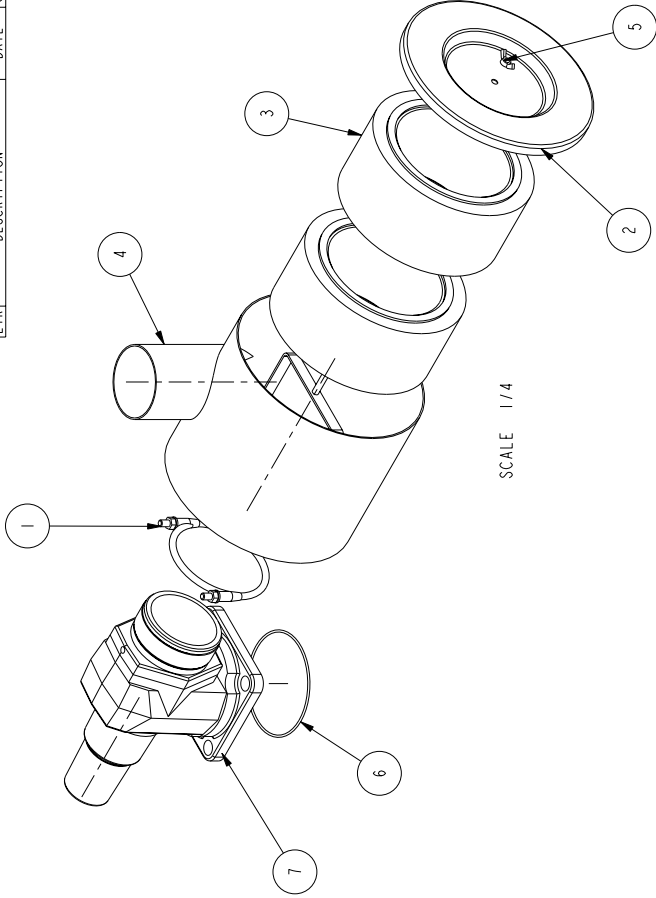
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REMOVE SHARP EDGES		INCH (MILLIMETER)		TOLERANCE		QCT PART NO.		DRAWING SYMBOL		SHEET	
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				PATTERN NO.						1/2	
				MATERIAL NO.						1/2	
				DO NOT SCALE		PRINT				SCALE	
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NO.	DESCRIPTION	PART NO.	QTY.
1	U-BOLT - AIR FILTER CLAMP	4406806	1
2	COVER - AIR FILTER	1122503	1
3	ELEMENT - AIR CLEANER	1122601	2
4	HOUSING - AIR CLEANER, HORIZ.	1122502	1
5	NUT - WING, M8-1.25	5403500	1
6	O-RING - 4.75 x 4.94 x 0.09	3601114	1
7	VALVE - INLET, HORIZONTAL	5204802	1

REVISIONS			
LTR	DESCRIPTION	DATE	CHG NO.



NOTES:

1] POSITIVE CONTROL AIR INLET VALVE - SEE DCS0501 SCHEMATIC FOR CONTROL LINE CONNECTION DETAILS

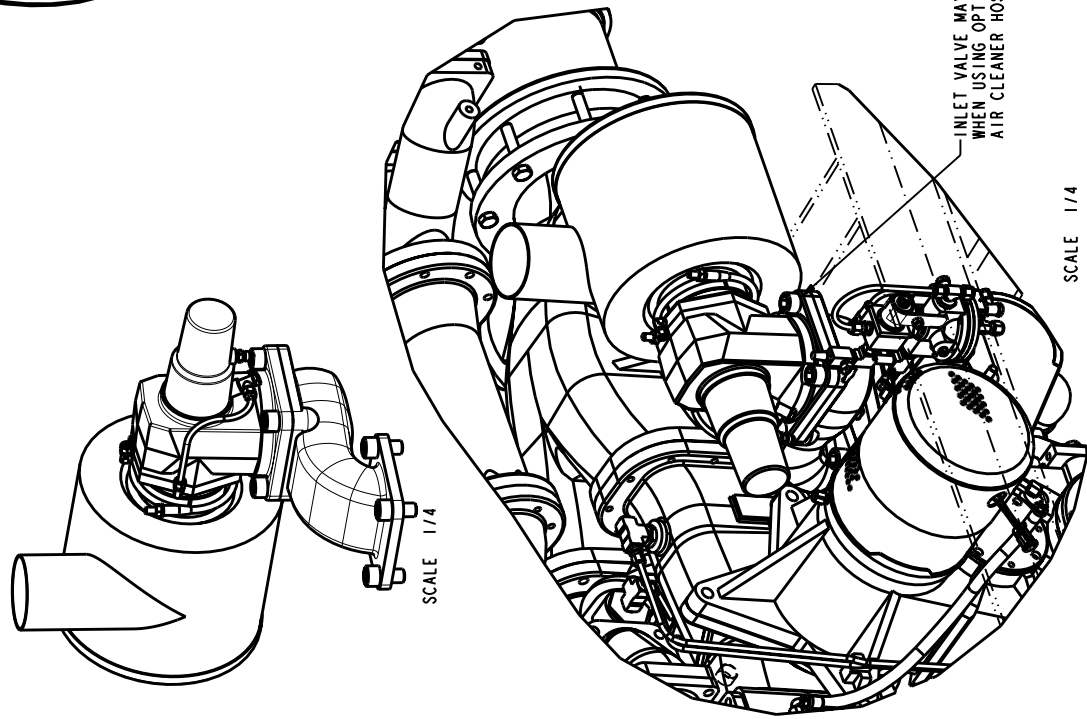
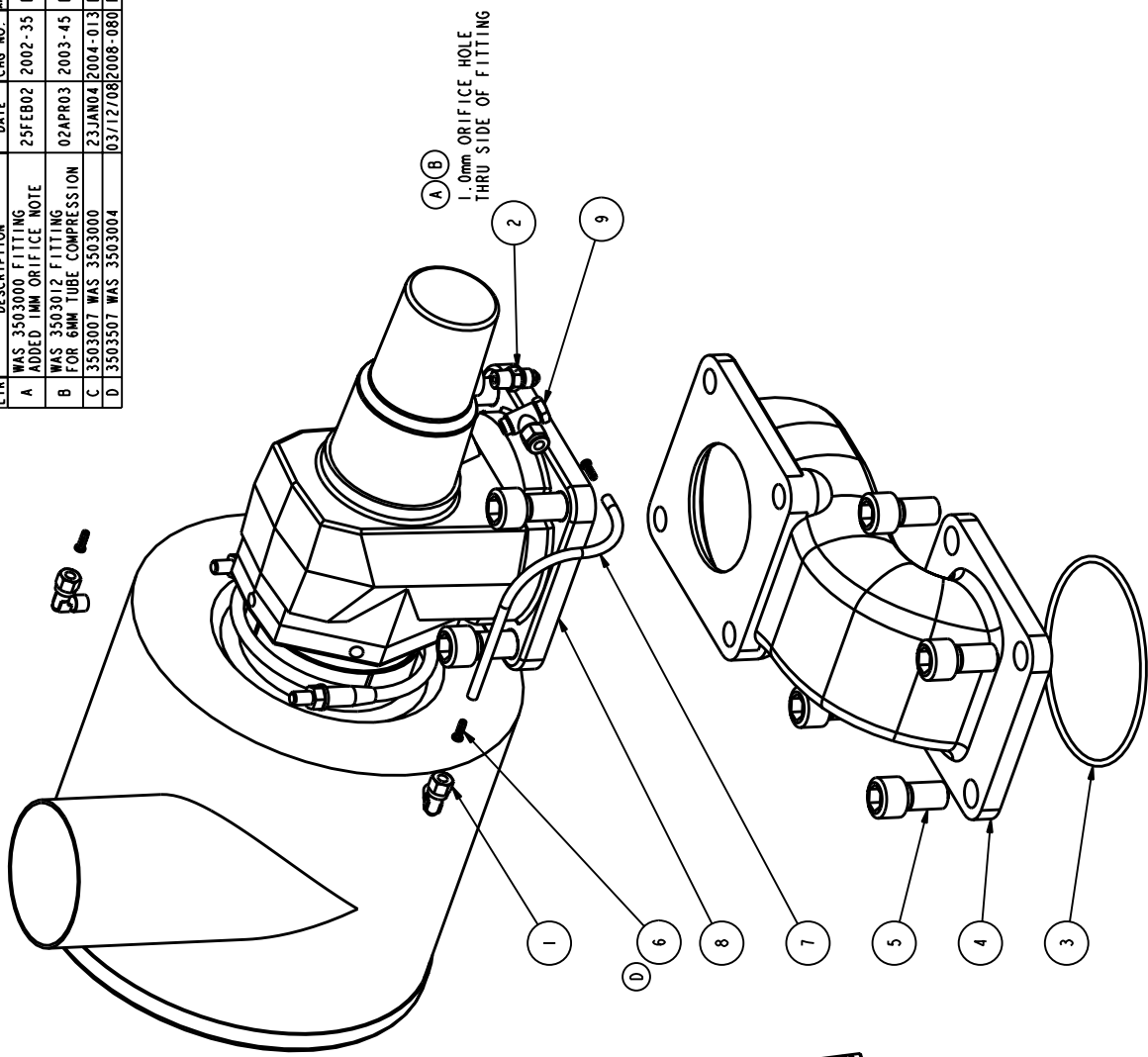
2] REF ASSEMBLY AZ00704 (DWG DCM1400) FOR REMOTE AIR CLEANER MOUNTING COMPONENTS

OPTIONAL HORIZONTAL INLET VALVE

REMOVE SHARP EDGES		INCH [MILLIMETER]	ENTER UNIT INDICATOR	OLD PART NO.	TOLERANCE EXCEPT AS NOTED ANGLES ± 0.01 RADIUS ± 0.01	MODEL NAME S204803	REV. CREATED 06-DEC-2000	SHEET 1/1	C
MATERIAL DESCRIPTION:		MATERIAL NO.	PATTERN NO.	W.S. Darley & Co. CHATELAIN FALLS, WI MILWAUKEE PARK, IL		VALVE - ASSY, INLET, E12, HORIZ.		DATE 21-DEC-00	
THIS DESIGN IS THE PROPERTY OF W.S. DARLEY AND CO. - UNAUTHORIZED REPRODUCTION IS PROHIBITED		ALL DIMENSIONS IN INCHES UNLESS NOTED	DO NOT SCALE	PRINT	DR 'N DMS CHD TED	SCALE		5204803	

NO.	DESCRIPTION	PART NO.	QTY.
1	FITTING - .12 BSPT X 6MM HOSE, 90	3503000	2
2	FITTING - .12 BSPT X .25 JICM	3501563	1
3	O-RING - 4.62 x 4.88 x 0.12	3601219	1
4	OFFSET - CAFS, AIR INLET	3106700	1
5	SHCS - M16-2 x 30MM, GR8.8	5401212	8
6	FTG - SLEEVE, 4MM x 6MM TU	3503507	3
7	TUBE - CONTROL, 6x4x200MM	1101957	1
8	VALVE - ASSY, INLET, E12, HORIZ.	5204803	1
9	FTG - 6MM TU x .12 BSPP, BANJO	3503007	1

LTR	DESCRIPTION	DATE	CHG NO.	APPR'D
A	WAS 3503000 FITTING ADDED 1MM ORIFICE NOTE	25FEB02	2002-35	DWS
B	WAS 3503012 FITTING FOR 6MM TUBE COMPRESSION	02APR03	2003-45	DWS
C	3503007 WAS 3503000	23JAN04	2004-013	DWS
D	3503507 WAS 3503004	03/12/08	2008-080	RJG



OPTIONAL HORIZONTAL INLET VALVE

MODEL NAME A100805		DESIGNED BY W.S. Darby & Co.		DATE CREATED 06-DEC-2000		SHEET 1/1	
TOLERANCE AS NOTED EXCEPT DIMS & DIA UNLESS STP		OLD PART NO. 11362-11		NEW PART NO. 11362-11		MATERIAL 303 STAINLESS STEEL	
REMOVE SHARP EDGES		PATTERN NO. 11362-11		DO NOT SCALE PRINT YES		SCALE 1/2	
MATERIAL DESCRIPTION: THIS DESIGN IS THE PROPERTY OF W.S. DARBY & CO. UNAUTHORIZED REPRODUCTION IS PROHIBITED							
DATE 15-DEC-00 DWS - ASST. HOR. INLET VALVE W/OFFSET							
DCMI300							

SECTION 3

AutoCAFS Commander

Operation and Installation Reference

AutoCAFS Commander

Compressed Air Foam System Control Module Operation and Installation



WWW.DARLEY.COM

Corporate Office:

2000 Anson Drive
Melrose Park, Illinois 60160-1087
800-323-0244, Fax (708) 345-8993

CAFS Applications:

920 Kurth Rd.
Chippewa Falls, WI. 54729
800-527-0068, Fax (715) 726-2648

Pump Manufacturing:

1051 Palmer St.
Chippewa Falls, WI. 54729
800-6347812, Fax (715) 726-2656

Prepared by: EAS
Approved by:
Revised by:

Description:

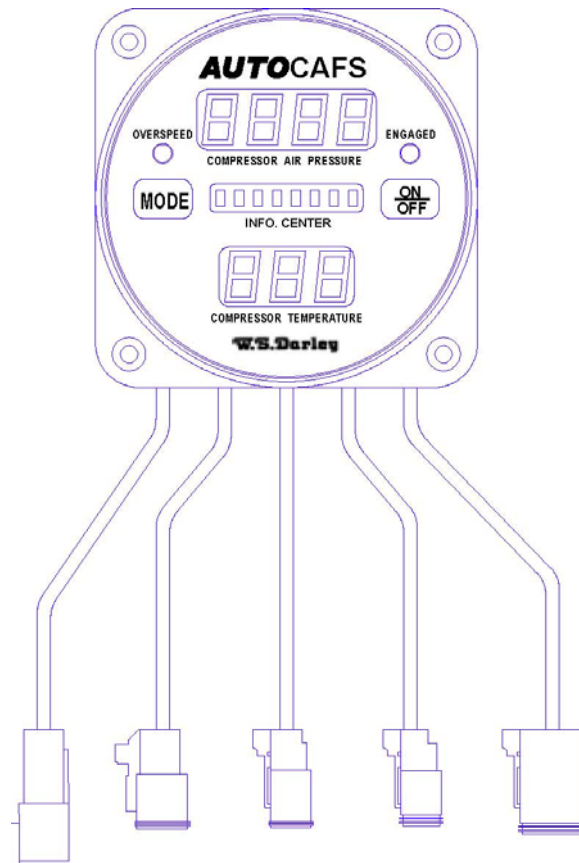
The Darley AutoCAFS Commander control is a programmed logic controller designed to simplify and safe-guard the start-up and operation of Darley compressed air foam systems.

The AutoCAFS Commander can be incorporated to monitor and control compressor operation on the Darley AutoCAFS II LDMBC midship CAFS system as well as PTO driven CAFS compressors driven via an electric hot shift type PTO.

The Commander continuously monitors system input speeds, pressures, and temperatures. By comparing these values to predetermined acceptable values, the Commander will allow compressor engagement if speeds, pressures and temperatures are within limits. Once the compressor has been engaged, the Commander monitors and displays compressor system temperature and pressure. If these values exceed a preset value, the Commander display exhibits a warning. If temperatures or speeds continue to increase to a higher preset value, the Commander will then automatically disengage the compressor.

Please review the following documentation for complete feature description, operation instructions and installation reference.

Darley AutoCAFS Commander



The AutoCAFS Commander system:

The system consists of the following components

1. The control unit
2. Air pressure sensor - 0-300 psi
3. Extension cables - 5 cables supplied
 - a. power cable
 - b. data bus cable
 - c. electric clutch cable
 - d. air pressure sensor cable
 - e. I/O signal and audible warning cable
4. Temperature sender
5. Warning buzzer

Features:

A) Power:

12V, Option for 24V

B) Programmable data using the MODE and ON/OFF buttons:

- Select '°F' or °C for compressor oil temperature reading310
- Select air pressure reading to be 'PSI', kPA, or AR.....311
- Select pump ratio to be '2.44' or 2.67.....312
- Set the maximum engine RPM for engagement - default is **900** RPM.....313
- Set new pump RPM for overspeed warning (other than default -**3650**).....314
- Set new pump RPM for automatic disengagement (default - **4500**).....315
- Set the compressor temperature overheat warning (default = **212 °F**).....316
- Set the overheat cut-out temperature (default = **240 °F**)..... 317
- Select system to turn ON automatically when Interlock is engaged, default = **OFF**...321

C) Display:

1. Compressor Air Pressure reading - 0-300 psi (0-2000 kPA, 0-20.0 Bar)
2. Compressor Oil Temperature reading - 0-250°F (0-120°C)
3. Engine RPM - 0-3000 RPM
4. Airflow in SCFM
5. Compressor operating hours - 0.1 hour increment up to 9999.9 hours
6. ON/OFF LED
7. OVERSPEED LED

D) Engine speed signal

Either from alternator pulse count or J1939 data bus. Default setting is J1939 data bus.

E) Transmission temperature

Thermostat with a single pole open contact.

F) Air pressure signal

From pressure transducer, 0-300 psi

G) Warnings

1. "HI RPM"
2. "COMP. HOT"
3. "BLOWDOWN"
4. "HI PRESS"
5. "OVERSPD"
6. "SHUTDOWN" - "COMP. HOT"
7. "SHUTDOWN" - "LO FOAM"
8. "SHUTDOWN" - "TRAN HOT"
9. "RPM >900"

H) Operating buttons:

- a. ON/OFF button
- b. MODE button

I) Compressor operating hours

The timer is enabled each time the compressor is engaged. An internal memory will keep track of the total operating hours.

1. CONTROL UNIT

The control unit is the 'brain' for the AutoCAFS Commander system. It performs all the controls and also allows control only when all the necessary conditions are met. It also monitors the system and alerts the operator of any system faults or failures. There are several display windows and buttons on the control unit:

i) Compressor air pressure window - this is a 4 digits LED window. It will display the air pressure from 0 to 300 psi. (pressure in kPA and Bar will be displayed when selected)

ii) Compressor oil temperature window - this is a 3 digits LED window. It will display the compressor oil temperature from 0 to 250 degrees Fahrenheit. (temperature in Celsius will be displayed if selected)

iii) An information display window -8 characters alphanumeric display. This window will display the engine RPM, compressor operating hours, airflow, and also any faults or warnings occurred during the operation.

iv) ON/OFF button - Turn the compressor ON and OFF. In order to turn the compressor on, the ON/ OFF button has to be pressed and held for 2 seconds. The green LED above the button will come on to indicate that the compressor is ON. This green LED will only come on if all the conditions are met and an electrical signal has been sent to engage the clutch. Press and hold the ON/OFF button for 2 seconds to turn off the system.

v) MODE button - the MODE button allows the operator to view the engine RPM, airflow, and compressor hours. Other information can be added in the future.

2. Pressure sensor

The pressure sensor is used to detect the air pressure in the compressor. It has a pressure range of 0-300 psi.

3. Extension cables

- a. power cable: 5' long with 3 pins Deutsch connector
- b. data bus cable: 12' long with 2 pin Packard connector
- c. electric clutch cable: 12' long with 2 pin Deutsch connector
- d. air pressure sensor cable: 12' long with 4 pin Deutsch connector
- e. I/O signal and audible warning cable: 8 pin Deutsch connector with 10' cable for transmission thermostat, 14' cable with 3-pin Deutsch for compressor temperature sensor, and 4 - 8" long pigtails

4. Compressor temperature sensor

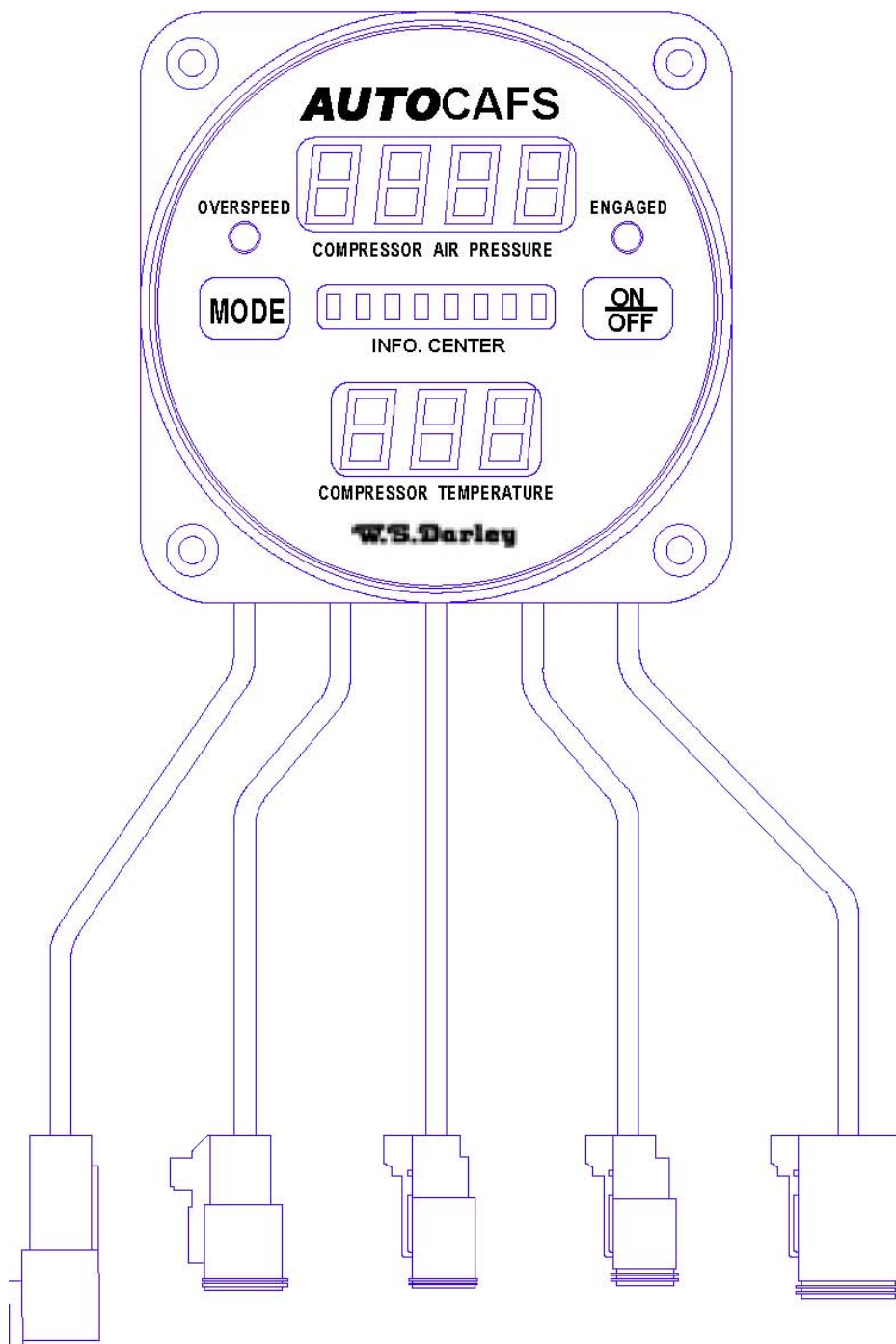
The temperature sensor supplied will be 1/8 NPT with a temperature range of 0°F to 250°F

5. Warning buzzer

6. Transmission thermostat

Operations:

1. RPM must be 900 RPM or less to engage the compressor
2. Pressure must be less than 10 psi in order to engage
3. AUTO ON feature - the system will turn ON automatically when the Interlock is engaged and conditions (1) and (2) above are met. The system can be turned off with the ON/OFF switch.
4. Automatic disengagement when RPM reaches 4500/pump ratio
5. Overspeed warning when the engine RPM exceeds 3650/pump ratio. The warning LED will go off when the engine RPM drops to 3600/pump ratio
6. Oil temperature overheat warning at 212°F (default)
7. Compressor high temperature shutdown. Disengage the compressor at 240°F (default)
8. Audible warning when the foam level is low. The compressor is also disengaged when the foam level in the tank is low.
9. Display messages when compressor engagement is not allowed
10. Display messages for any system fault:
 - i. E3 - "NO RPM" - no RPM signal detected
 - ii. E5 - "NO PRESS" - no pressure transducer detected
 - iii. E10 - "NO TEMP" - no oil temperature sensor detected
11. Audible warning active when:
 - i. RPM overspeed
 - ii. Compressor oil temperature – overheat
 - iii. Transmission temperature – overheat
 - iv. Foam in tank is too low



Power	Data Bus	Electric Clutch	Pressure sensor	I/O signals - 8 pin
- 12 volt	- J1939 (+)	- 12V		- 12 VDC
- Ground	- J1939 (-)	- Ground	- 4 pin	- Ground
- Interlock	- Shield			-Temperature signal
				-Transmission thermostat
				- Audible warning
				- Low foam level warning
				-Airflow (4-20 mA)
				-Airflow (4-20 mA)

DISPLAY:

1. Compressor Air Pressure reading: Using 0-300 psi sensor, units of measure selectable

- a) 0-300 psi
- b) 0-2000 kPA
- c) 0-20 Bar

2. Compressor oil temperature reading:

- a) 0-250°F or
- b) 0-120°C

c. Dot matrix display:

Engine RPM - default display

RPM 1450

Airflow in SCFM

AIR 65

Compressor operating hours

HR. 1154

SWITCHES:

1. ON/OFF

- a. Active only when the 'INTERLOCK' is on

- b. Press and hold for 2 seconds to turn **ON** the air compressor
- c. Press and hold for 2 seconds to turn **OFF** the air compressor

2. MODE

- a. Toggle the information between engine RPM, airflow, and air compressor operating hours
- b. Use to get into the programming mode

Operations:

1. Turn compressor ON

- 'INTERLOCK' is on - the system turns ON when initial start up conditions (c and d) are met.
- OR ON/OFF button is pressed and held for 2 seconds
- Air pressure is < 10 psi
- Compressor oil temperature is $< 212^{\circ}\text{F}$ (100°C)
- Turn 'Engaged' LED on when the compressor is engaged. (After all conditions are met)

2. Shut down compressor if:

- Engine RPM $> 4500/\text{pump ratio}$. (E.g. $4500/2.44 = 1844$ RPM)
- Compressor temperature $> 240^{\circ}\text{F}$ (115°C)
- Low foam level (input signal)
- High transmission temperature (input signal)

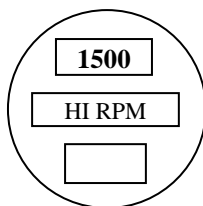
3. System faults:

- E3 - no RPM data
- E5 - no pressure transducer detected
- E10 - no oil temperature sensor detected

4. System Warnings:

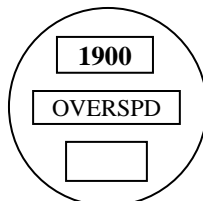
a. "HI RPM" - $4500/\text{pump ratio} > \text{RPM} \geq 3650/\text{pump ratio}$

Flash "HI RPM" and 1500

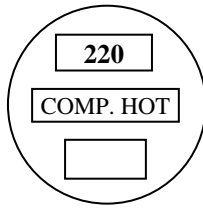


b. "OVERSPD" - $\text{RPM} > 4500/\text{pump ratio}$

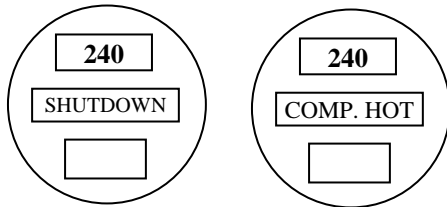
Flash "OVERSPD" and 1900



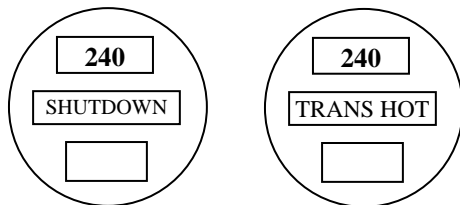
- c. **“COMP. HOT” - 212°F (100) < Oil temperature < 240°F (115)**
Flash “COMP. HOT” and 220



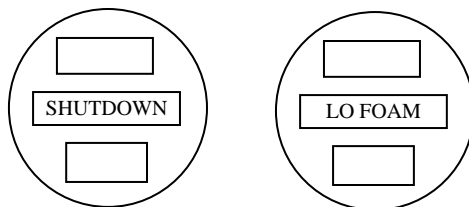
- d. **“SHUTDOWN”, “COMP. HOT” - Oil temperature > 240°F, Flash**



- e. **“SHUTDOWN”, “TRAN HOT” - From transmission temp. overheat input, Flash**

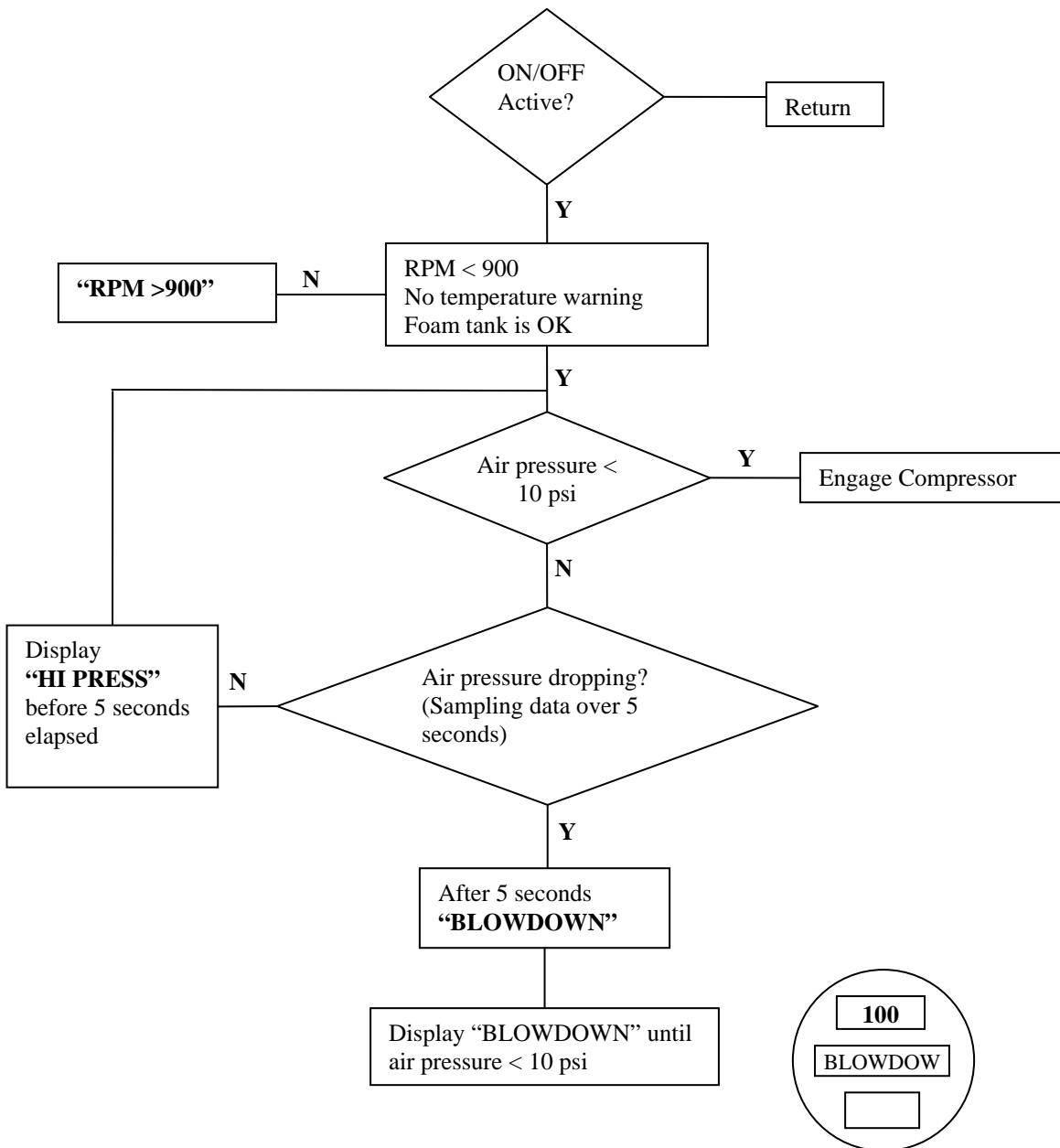


- f. **“SHUTDOWN”, “LO FOAM” - From Foam tank input, Flash**



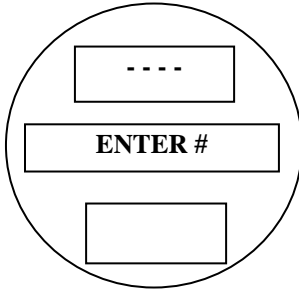
g. "BLOWDOWN" – Flash

- I. When compressor pressure is > 10 psi
- II. When an operator is trying to turn the compressor on



Codes

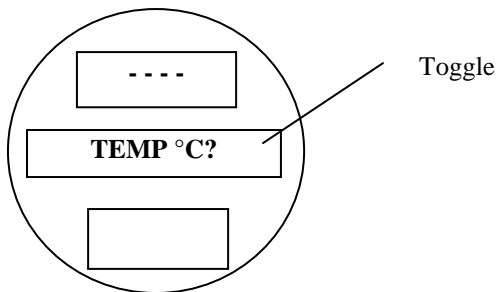
- I. Press and hold "MODE" for 3 seconds to enter the data entry mode.



- II. Press "MODE" and then "ON/OFF" to enter code
- III. Use "MODE" to select the digit and "ON/OFF" to change the number
- IV. Press and hold both "MODE" and "ON/OFF" for 3 seconds to exit

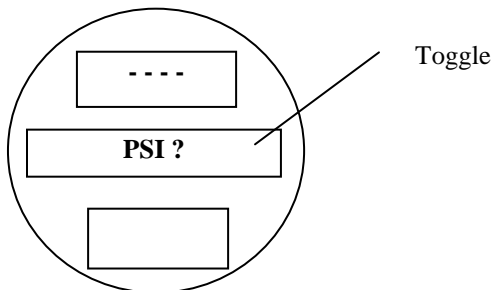
1. Select °F or °C for compressor oil temperature reading - default to °F

CODE - 310



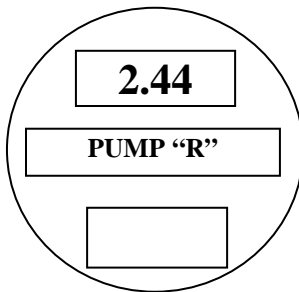
2. Select pressure to be in PSI, kPA, BAR - default to "PSI"

CODE - 311



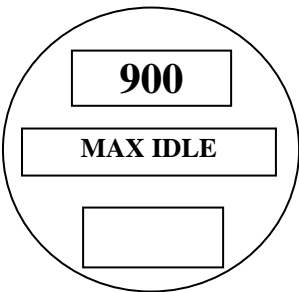
3. Select pump ratio – default to 2.44

CODE - 312



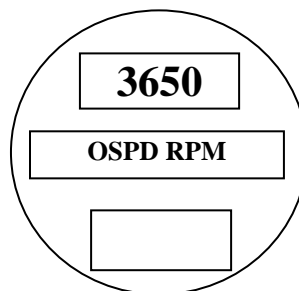
4. Set the maximum idle RPM allowed for engagement – default = 900

CODE - 313



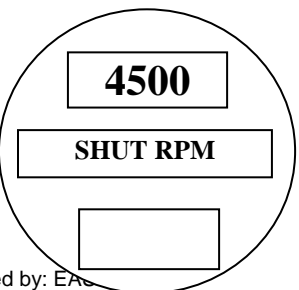
5. Set new pump RPM for overspeed warning – default = 3650

CODE – 314



6. Set new pump RPM for automatic compressor disengagement – default = 4500

CODE – 315



7. Set the compressor temperature overheat warning – default = 212
(100°C)

CODE – 316

A circular display with three horizontal rectangular boxes. The top box contains the number "212". The middle box contains the text "°F WARN". The bottom box is empty.

8. Set the compressor overheat shut down temperature – default = 240 (115°C)

CODE – 317

A circular display with three horizontal rectangular boxes. The top box contains the number "240". The middle box contains the text "°F SHUT". The bottom box is empty.

9. Select system to turn ON automatically when Interlock is engaged

CODE - 321

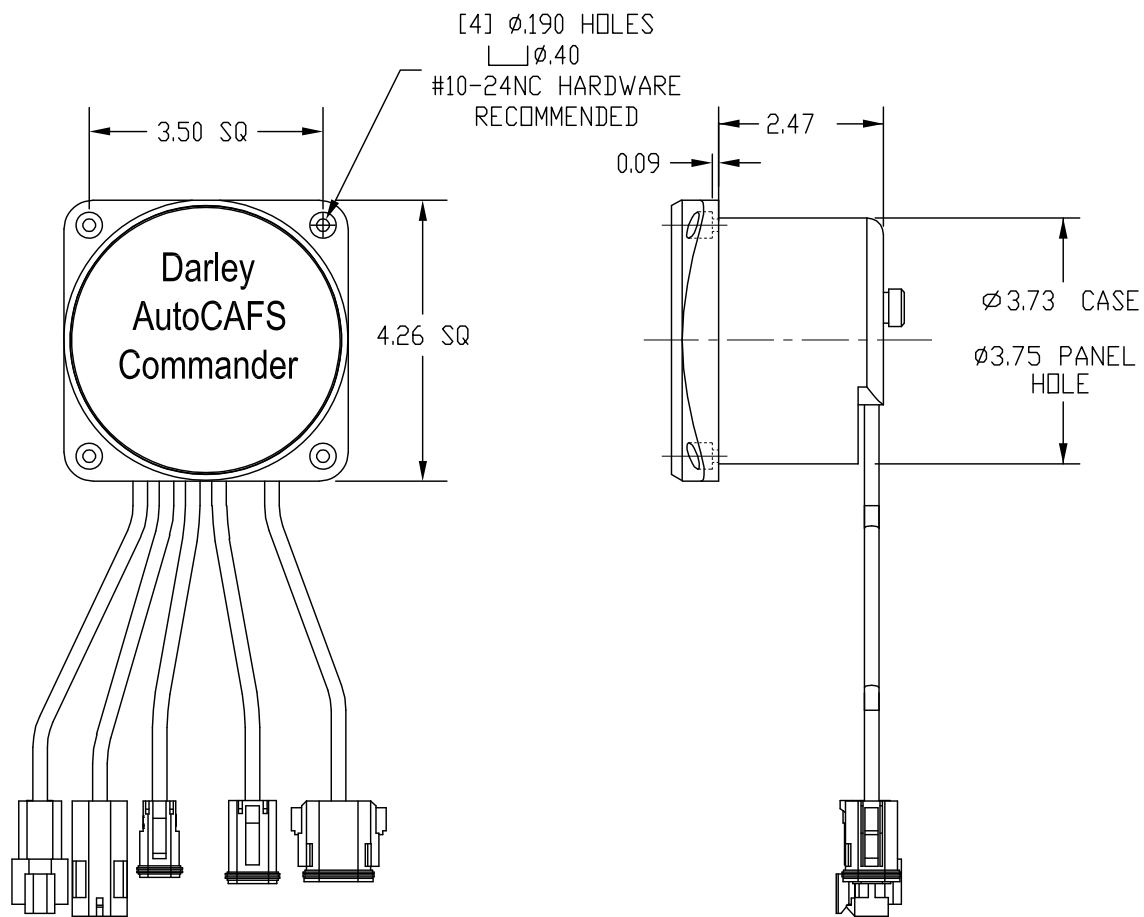
A circular display with three horizontal rectangular boxes. The top box is empty. The middle box contains the text "OFF?". The bottom box is empty.

INSTALLATION

Install Control Module

Note: The control module should be mounted on the pump control panel.

1. Measure and mark mounting location for control module panel cutout and mounting screw holes. Make sure there is clearance behind the panel for the module and cables before cutting holes. Refer to the following diagram for layout and dimensions.
2. Cut out a 3.75 inch (95.25 mm) diameter hole and drill four holes for mounting screws.
3. Place control module in position and secure with four screws (#10-24NC mounting hardware is recommended).



Install Pressure Transducer

The air pressure transducer is mounted to a port on the air/oil separator tank below the main discharge pressure check valve. To correctly read air system pressure during operation as well as during system blow-down, the transducer must be connected to a port located before the system minimum pressure discharge check valve.

1. Mount the transducer in a 1/4-18 NPT threaded air pressure port. A 1/8 BSPP male x 1/4 NPT female adapter is required for attachment to the LDMBC separator tank.
Caution: Do not use the main body that houses the electronics to tighten the pressure transducer. Damage to the transducer may occur.
2. Tighten the transducer with a wrench on the lower hex fitting.
3. Connect the pressure transducer cable from the control module to the pressure transducer.

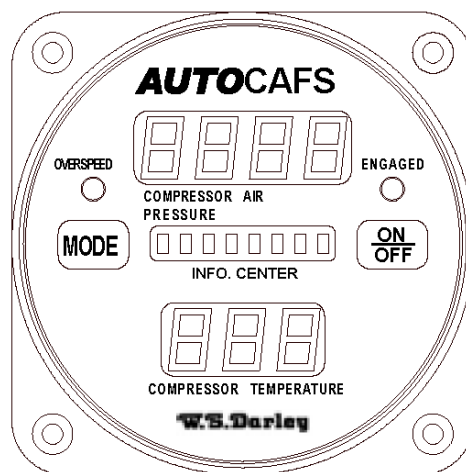


Air Receiver Tank

Air system minimum
pressure discharge check
valve

Pressure Transducer port location.
Note: 1/8 BSPP male x 1/4 NPT female
adapter required.

- Connector Key**
- A** 4-Pin Deutsch Plug
 - B** 3-Pin Deutsch Recep.
 - C** 2-Pin Deutsch Plug
 - D** 3-Pin Deutsch Plug
 - E** 3-Pin Packard
 - F** 8-Pin Deutsch Plug



- Engine Control Cable Connector**
- | Pin/Wire | Description |
|----------|-----------------------------|
| 1/Red | +12v for oil temp sensor |
| 2/Black | Gnd for oil temp sensor |
| 3/White | Signal from oil temp sensor |
| 4/Green | Transmission thermostat |
| 5/Blue | Buzzer (active - ground) |
| 6/Yellow | Low foam level signal |
| 7/Orange | Airflow(4-20 mA) signal |
| 8/Brown | Airflow(4-20mA) return |

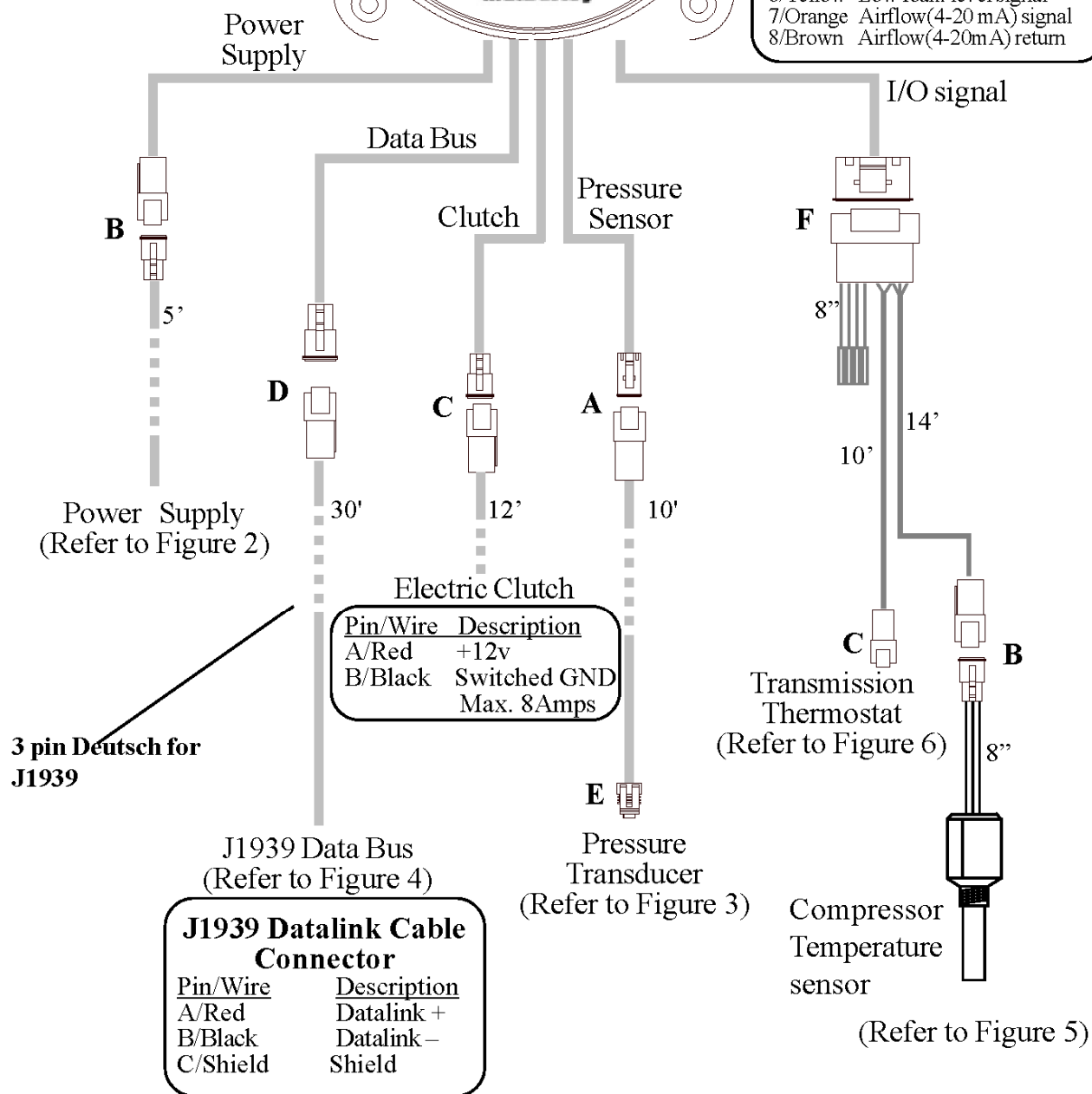


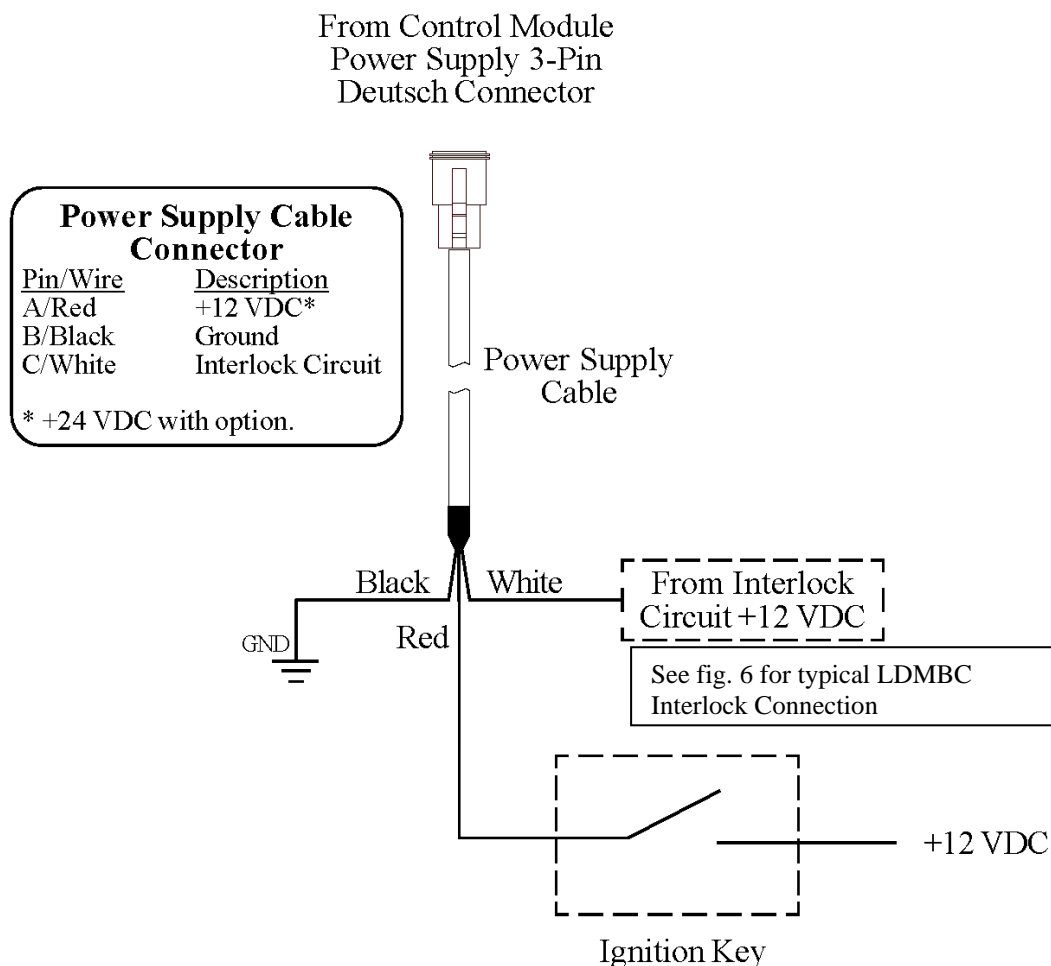
Figure 1. WS Darley AutoCAFS module wiring

WIRING

The following figures include the schematics, wiring diagrams, block diagrams, and cables for the AutoCAFS

Note: If optional 24 VDC unit is installed references to +12 VDC will be +24 VDC.

Power



Note: The interlock circuit will ensure that specific safety conditions are met before the compressor becomes operational. The interlock circuit may include relays, switches, and/or indicator lights for the following conditions:

- Parking Brake On
- PTO Engaged
- Transmission In Drive/Neutral

Figure 2. Power Supply Wiring

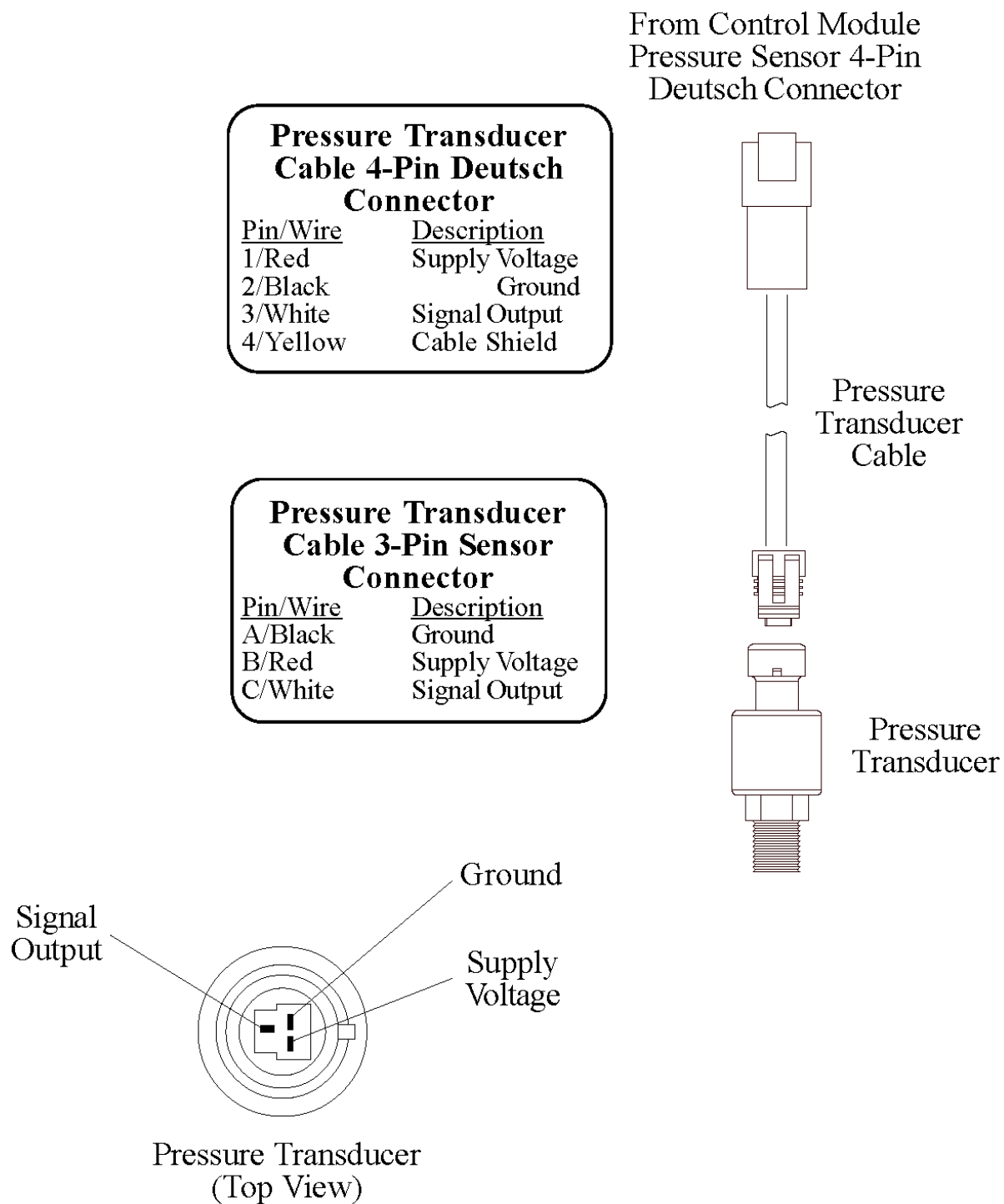


Figure 3. Pressure Transducer Wiring

J1939 Data Bus

Typical 9-pin Deutsch diagnostic connector.

Pin C-J1939 Datalink Positive

Pin D-J1939 Datalink Negative,

Pin E - Shield

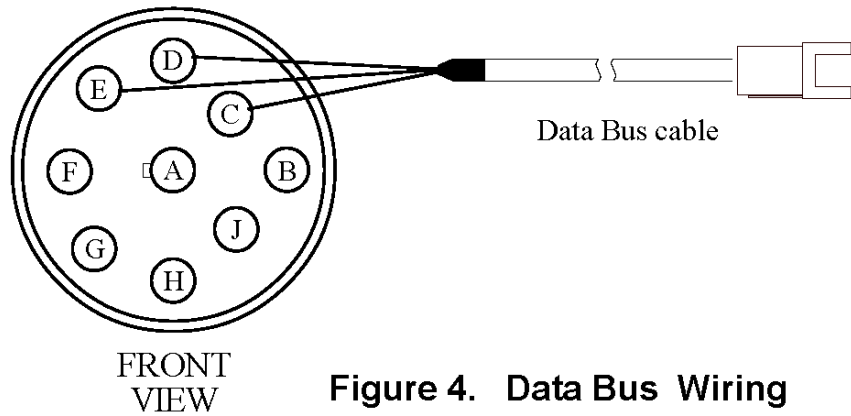
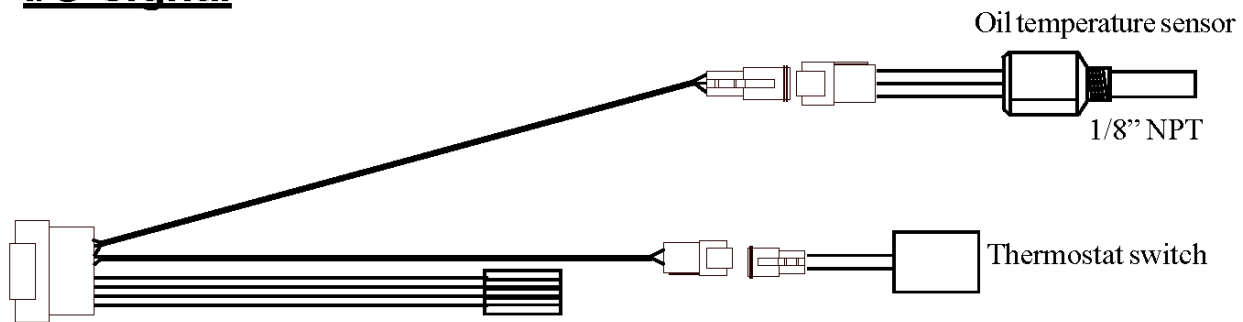


Figure 4. Data Bus Wiring

I/O signal



Engine Control Cable Connector

Pin/Wire	Description
1/Red	+12V for oil temp. sensor
2/Black	Gnd for oil temp. sensor
3/White	Signal from oil temp. sensor
4/Green	Transmission thermostat
5/Blue	Buzzer(active - ground)
6/Yellow	Low foam level signal
7/Orange	Air flow(4-20 mA) signal
8/Brown	Air flow(4-20 mA) return

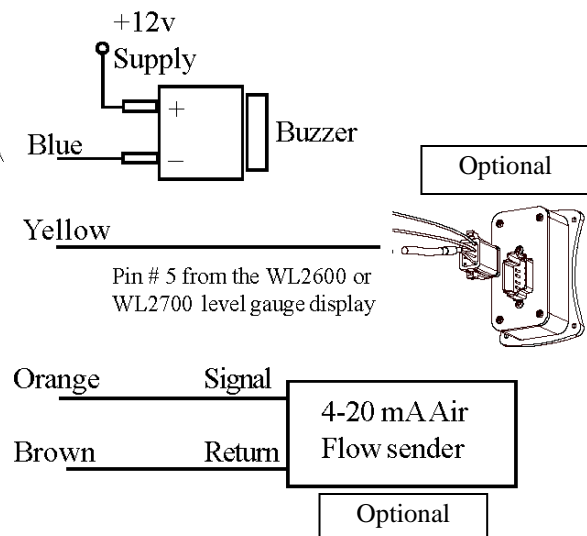


Figure 5. Other Input/output wiring

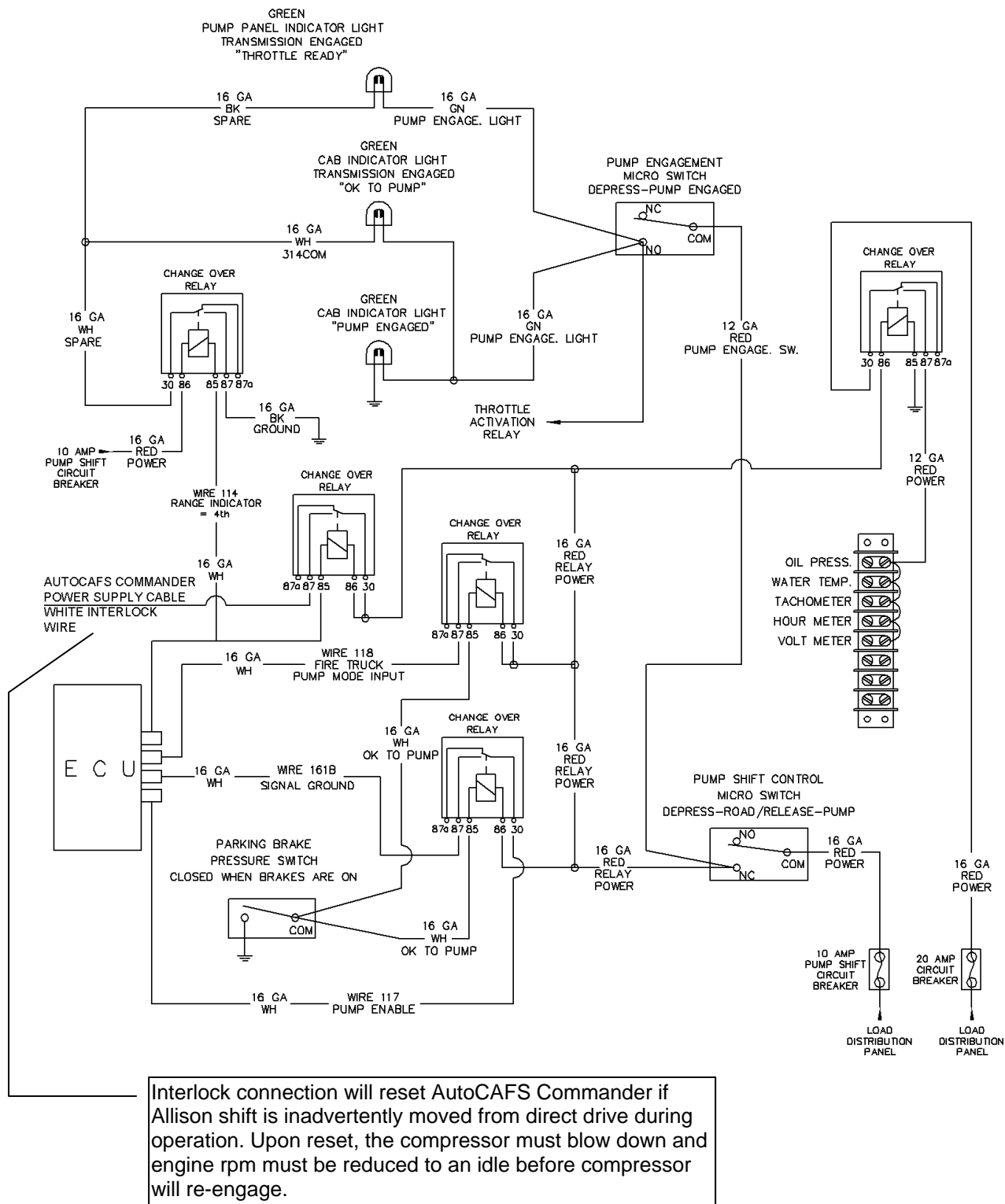


Figure 6. Typical Interlock Wiring

Program Access Mode

When in the program access mode the digital display will show operator inputs, program options, and error codes. To gain access to the program features a three digit program code must be entered. Review the Program Code Descriptions or refer to Table

1. Program Code Quick Reference for the proper three digit code.

Note: There is a timeout feature that will return the program to normal operation in three seconds if input is not detected at the buttons.

Select Program Access Mode

Press the MODE button and hold it until the display shows four dashes. The program access mode is ready for a code number to be input. (Refer to Figure 7.)

Enter Program Code Number

Note: There is a time out feature that will return the program to normal operation in three seconds if input is not detected at the buttons.

1. Select the Program Access Mode (four dashes are shown in the display).
2. Press the ON/OFF button. The display will show the number 100 and the first digit 1 will flash. Each time the ON/OFF button is pressed the number will scroll up by 1. Set the first digit to the number desired.
3. Press the MODE button. The second digit shown in the display will flash. Each time the MODE button is pressed the number will scroll up by 1. Set the second digit to the number desired.
4. Press the ON/OFF button. The third digit shown in the display will flash. Each time the ON/OFF button is pressed the number will scroll up by 1. Set the third digit to the number desired.

When a valid three digit program code is entered the display will show a program value or an option. If an invalid code is entered the display will show an error code.

Note: When a valid code has been entered and the display shows a programed value or an option, the timeout feature is disabled.

Change Values or Options

Press the MODE button to select the digit that is to be changed. The digit will flash. Press the ON/OFF button to change the digit or the option choice.

Exit Program Access Mode

Press both the MODE and then ON/OFF buttons and hold until four dashes are shown in the display. Release the buttons and enter a new code or after 3 seconds the program will timeout and return to normal operation.

TABLE – 1

Code Number	Settings	Default value
3-1-1	To select °F or °C	°F
3-1-0	To select PSI, kPA, Bar	PSI
3-1-2	To set pump ratio	2.44
3-1-3	Set max. engine RPM for engagement	900 engine RPM
3-1-4	High pump RPM for warning only	3650 pump RPM
3-1-5	High pump RPM for disengagement	4500 pump RPM
3-1-6	To set Oil Temp. warning only	212°F
3-1-7	To set Oil Temp. for disengagement	240°F
3-2-1	To set 'Auto ON' function	OFF

SECTION 4

Foam Proportioner

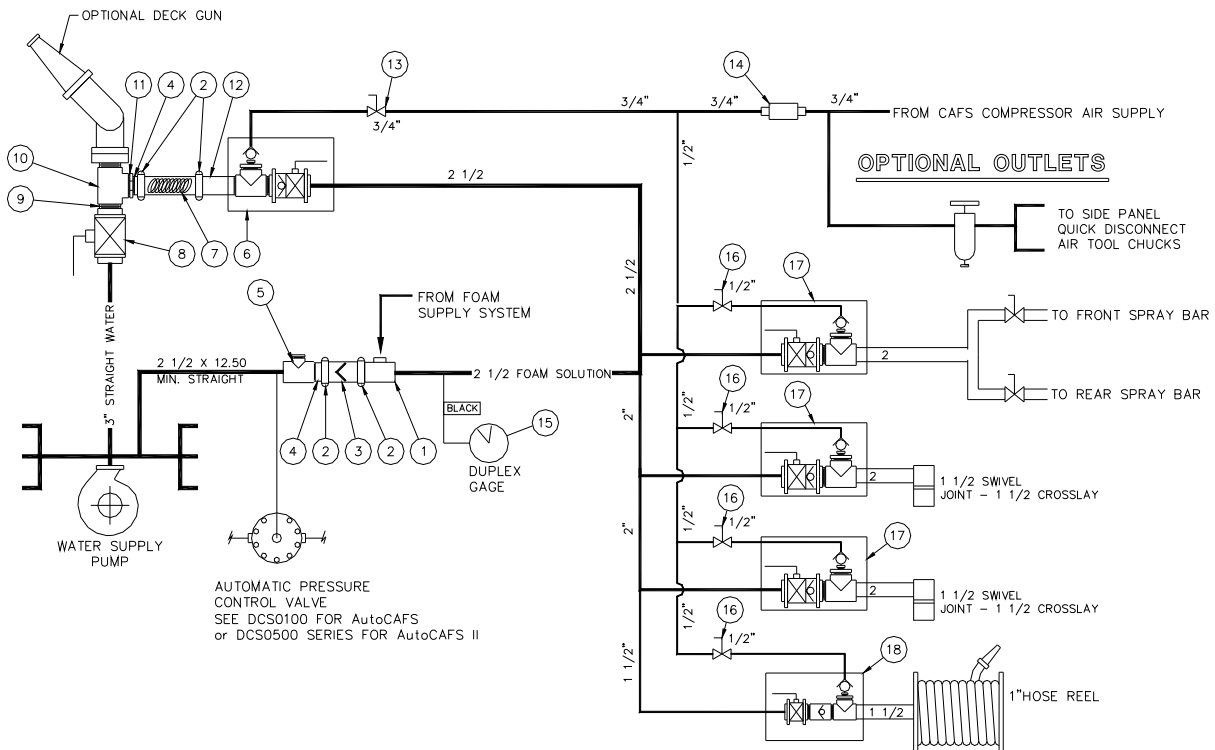
(Insert Foam Proportioner Manual Here)

The following text is a generic description of the operating procedures for a FoamPro Model 2001 foam proportioner. Please refer to the manual supplied with your apparatus for specific operating instructions for your unit.

This apparatus has been fitted with a compressed air foam system. In addition to the main UL pump, there are two basic subsystems that comprise a compressed air foam system on an apparatus. Number one is the addition of a foam concentrate proportioner to inject foam concentrate into the discharge side of the water pump. Number two is the addition of an air compressor system to supply compressed air for foam making. Operation of the apparatus with only the foam concentrate proportioner functioning will result in the apparatus functioning as a conventional foam equipped unit. Various nozzles and devices may be used to create and discharge foam. Operation of the apparatus with proportioner and air compressor engaged will result in the engine being capable of creating compressed air foams. Compressed air foams are generally applied through smooth bore type nozzle devices.

The air compressor has a rated capacity of 220 cfm (cubic feet per minute). It attains this capacity at approximately 1500 engine rpm. The air compressor is driven by an auxiliary gear case mounted directly to the pump split shaft gear case. The pump and compressor gear ratios are matched to provide approximately 500 GPM @ 125 PSI water flow while simultaneously providing 220 CFM @ 125 PSI air flow. It is important to remember that during operations from a pressurized hydrant source, engine RPM will be slower; therefore compressor output will be reduced. If high compressor flows are required, operate from draft or from the booster tank. Engine RPM will then be high enough to assure adequate compressor performance. Another option is to turn on the discharge relief valve, set it for the desired pressure, and throttle pump up to the necessary RPM for maximum compressor output.

The benefits of compressed air foam use are variable, but they are directly proportionate to the knowledge of the user. Please read and understand this operations manual before operating the unit.



Example of Typical Compressed Air Foam Schematic

FoamPro Electronic Foam Proportioner

This unit is equipped with a FoamPro 2001 automatic, electronic, discharge side, foam proportioning system.

The foam proportioner is a built in, fully self contained, flow meter based, direct injection system.

There are five basic units that make up the system.

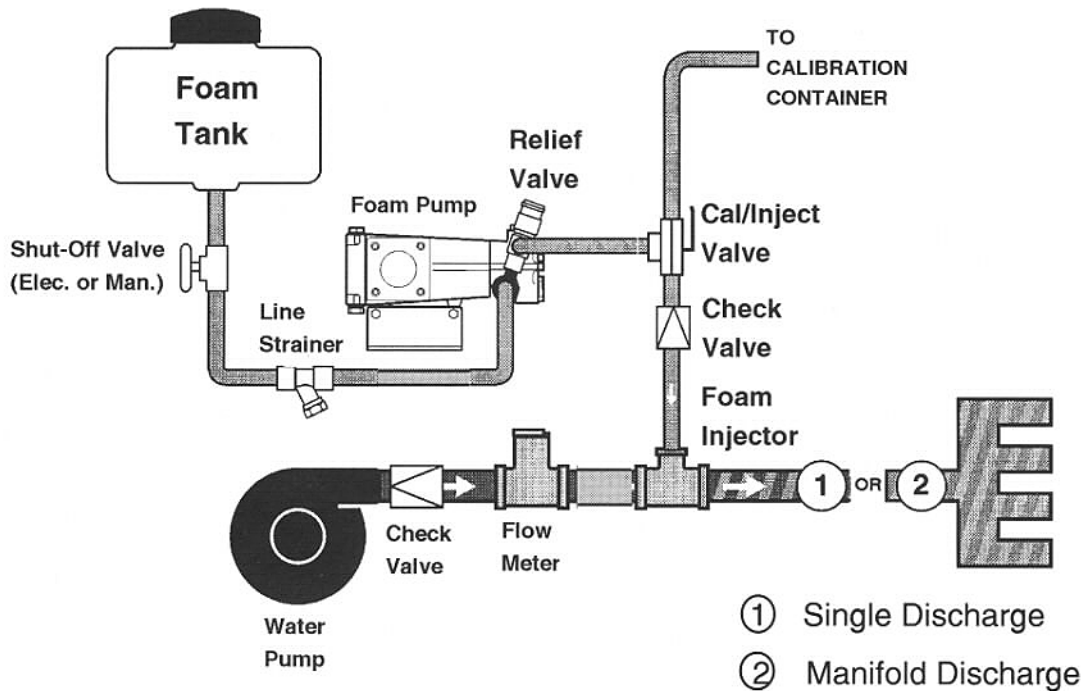
They are: the injection pump, motor, paddle-wheel type flowmeter, injection fitting, and the panel mounted, digital, push button, control module unit.

An optional three way "Foam Supply Valve" may be installed behind an access door on the side pump panel. It has four (4) basic functions. 1) On Position - To allow foam to travel from the foam tank to the FoamPro pump, 2) Off Position - to shutoff the foam tank for cleaning of the strainer, 3) to serve as an overboard pickup hose, and 4) Drain Position - to drain the foam tank using the overboard pick-up/drain hose.

To utilize the overboard pickup hose the hose must first be primed. Step 1) insert hose into pail of foam, 2) Next turn cal/inject valve on FoamPro discharge fitting to calibrate/flush position. Run FoamPro pump in "Simulated Flow" mode to prime. See Hypro manual for instructions. Switch cal/inject valve to inject.

The unit operates by sensing water flow. The Paddle wheel flowmeter sends a signal to the control unit displaying this flow. If the unit is turned on, the microprocessor control sends a signal to the injector motor to begin injecting foam concentrate into the plumbing based on the percentage set at the control module.

FoamPro 2001 Basic System Layout

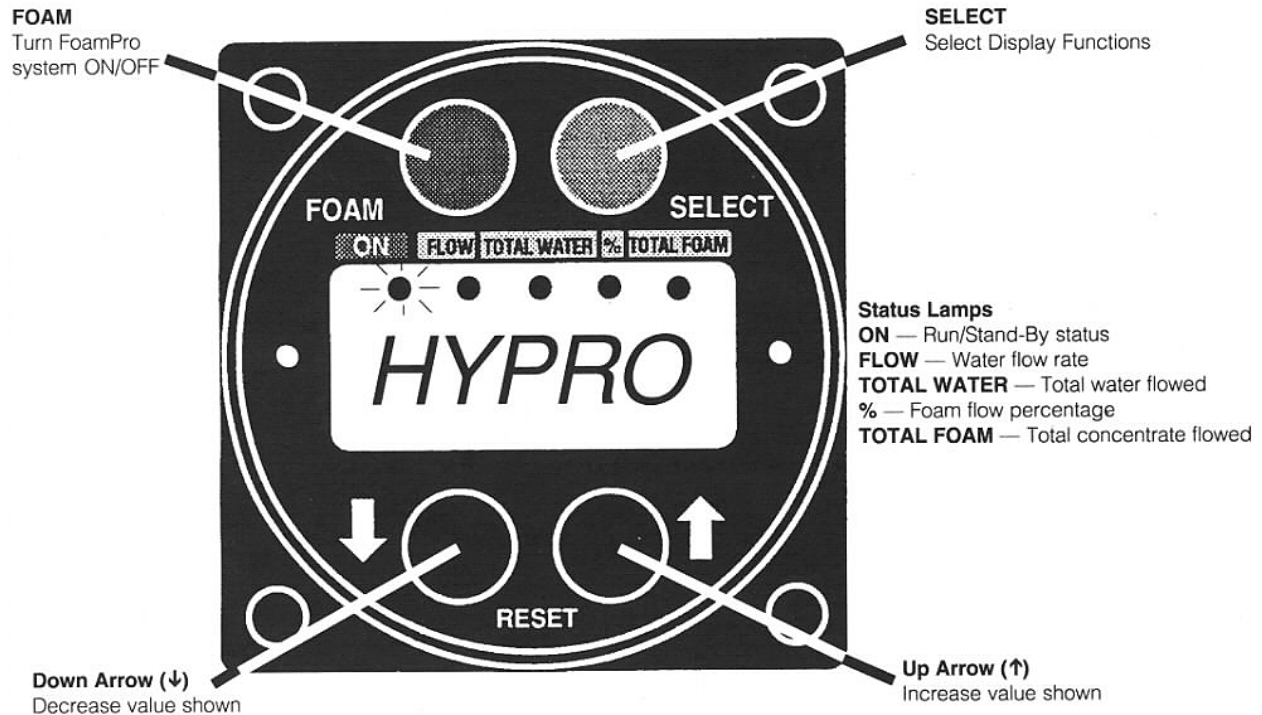


This system allows for continuous operation without interruption of foam concentrate flow. If the level of foam in the supply tank is reaching empty, a low concentrate (LO CON) warning will flash on the display. The tank then must be refilled within two minutes or the unit will automatically shut down to avoid doing damage to the injector pump. If the unit has shut down a no concentrate (NO CON) message will be displayed. The foam percentage to water ratio is adjustable from 0.1% to 9.9% in .1% increments. Weather affects viscosity of the concentrates and therefore, the ratio can be adjusted to user's choosing.

The micro-processor based, panel mounted control unit can perform multiple functions. It performs the basic function of turning the unit on or off. It also has two buttons with up/down arrows to adjust the injection percentage ratio of the foam to water. These buttons also play a part in the initial set up of the units calibration. With the selector button in the upper right hand corner of the unit, four functions can be accomplished.

Selector Button Functions

- 1)**Flow Mode**: Displays present water flow out any of the CAFS discharges even if the foam system is not turned on.
- 2)**Total Water Mode**: Displays total water flowed since the unit began to flow water.
- 3)**Percentage (%) Mode**: Displays the present ratio that foam will be injected at, if the unit was turned on.
- 4)**Total Foam Mode**: Displays the total amount of foam, rounded off to the nearest gallon, injected since the unit was last turned on.



The following chart gives the approximate water treatment capacities and relative flow times for various foam concentration settings. Chart is based on a water flow rate of 120 GPM and a single tank capacity of 30 gallons.

<u>Meter Setting</u>	<u>US Gallons Treated</u>	<u>Flow Time - 30 Gal Tank</u>
0.1%	30000	250 min.
0.2%	15000	125 min.
0.3%	10000	83.3 min.
0.5%	6000	50 min.
1.0%	3000	25 min.
3.0%	1000	8.3 min.

TO GET FOAM:

- 1) Push the red on/off button.
- 2) The foam percentage default is set at 0.3%, adjust if desired.

TO FLUSH SYSTEM:

- 1) Turn off the foam system by pushing the red on/off button. The red light below the button will go off.
- 2) Flow water out of the foam discharge for 2 minutes.

To drain unit of water when in freezing weather, turn dual tank selector switch, if so equipped, to flush(center)position, and open all pump drains. Refer to Hypro 2001 installation/operators manual for other specific operation or maintenance information.

SECTION 5

Operation of Apparatus Compressed Air Foam System

SECTION 5 – Operation of Apparatus Compressed Air Foam System

This apparatus has been fitted with a compressed air foam system. In addition to the main UL pump, there are two basic subsystems that comprise a compressed air foam system on an apparatus. Number one is the addition of a foam concentrate proportioner to inject foam concentrate into the water on the discharge side of the water pump. Number two is the addition of an air compressor system to supply compressed air for generating foam.

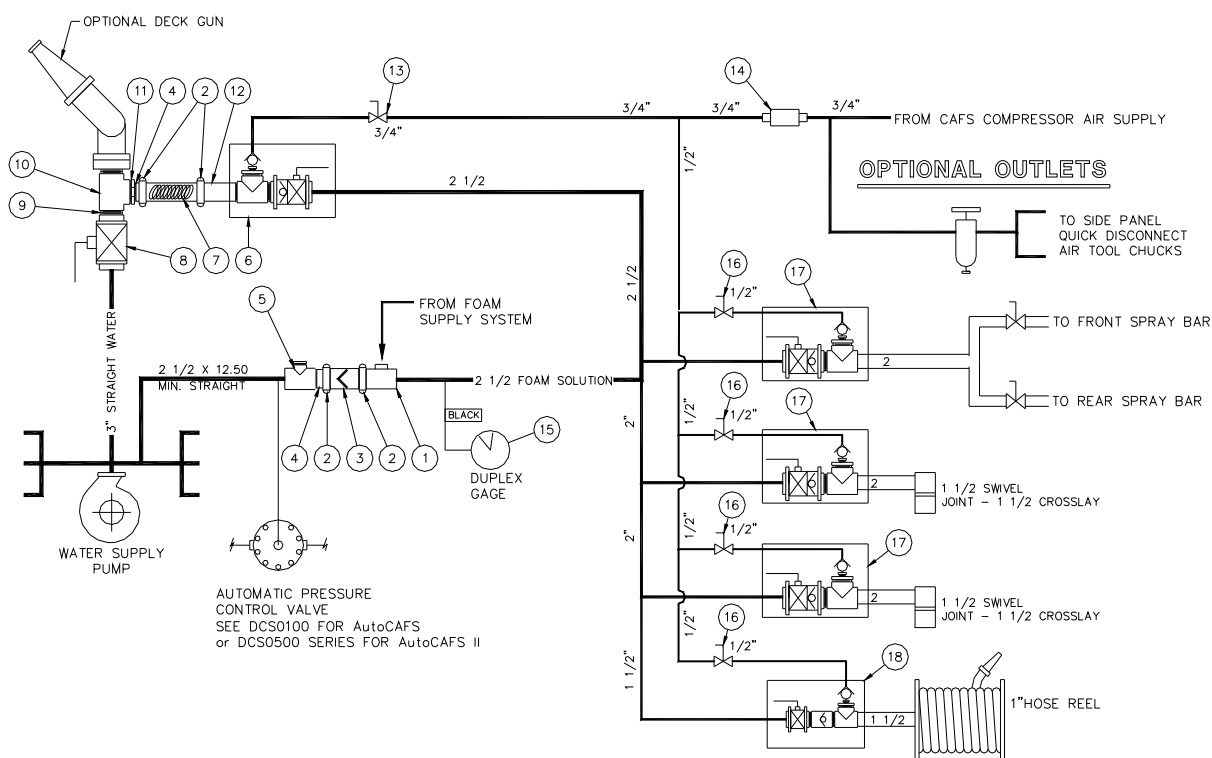
Operation of the apparatus with only the foam concentrate proportioner functioning will result in the apparatus functioning as a conventional foam equipped unit. Various nozzles and devices may be used to create and discharge foam.

Operation of the apparatus with proportioner and air compressor engaged will result in the engine being capable of creating compressed air foam. Compressed air foam is generally applied through smooth bore devices.

It is important to remember that during operations from a pressurized hydrant source, engine RPM will be slower causing the compressor output to be reduced as well. If high airflow is required, operate from draft or from the booster tank. Engine RPM will then be high enough to ensure adequate compressor performance. Another option is to turn on the discharge relief valve, set it for the desired pressure, and throttle pump up to the necessary RPM for maximum compressor output.

The benefits of compressed air use are variable, and are directly proportional to the knowledge of the user. Please read and understand the operations manuals before operating the unit.

Typical Compressed Air Foam Schematic



The following chart gives the approximate water treatment capacities and relative flow times for various foam concentration settings. This chart is based on a water flow rate of **120 GPM** and a single foam concentrate tank capacity of **30 gallons**.

<u>Meter Setting</u>	<u>US Gallons Treated</u>	<u>Flow Time - 30 Gal Tank</u>
0.1%	30000	250 min.
0.2%	15000	125 min.
0.3% standard	10000	83.3 min.
0.5%	6000	50 min.
1.0%	3000	25 min.
3.0%	1000	8.3 min.

TO START FOAM FLOW:

- 1) Push red on/off button. (Hypro FoamPro 2001 & 2002 only)
- 2) The foam percentage default is set at 0.3%; adjust as desired.

TO FLUSH SYSTEM:

- 1) Turn off the foam system.
- 2) Flow water out of the foam discharge for 2 minutes.

To drain water from unit during freezing weather, turn dual tank selector switch to flush (center) position (as applicable), and open all pump drains.

Compressed Air Foam System Operation

- 1) Referring to PUMP Shifting Procedures detailed in Section I, shift water pump to ENGAGED position.
- 2) Engage the air compressor by pressing and holding the AutoCAFS Commander ON/OFF button down for 2 seconds. Note: The compressor can be switched on before or after the pump is engaged, however, do not engage compressor when engine is turning faster than 900 rpm. Reduce engine rpm before engagement. An interlock has been implemented to limit engagement rpm to 900 rpm.
- 3) Establish water flow in main pump. Open tank to pump valve and tank refill valve slightly to provide water circulation through pump.



CAUTION: *The air compressor is cooled by water supplied by the fire pump and circulated through a water/oil heat exchanger. Water circulation must be established before or immediately following compressor engagement to assure proper cooling. Also, if water is continually circulated back to tank, cooling water will be heated. Operating with a continuously refreshed water supply eliminates this concern.*

Note: *A temperature sensor is incorporated into the AutoCAFS Commander control module to avoid compressor over heating which may result in rotor seizure. If compressor temperature rises above normal operating temperature*

to 212°F, a warning, 'COMP HOT' will flash on the Commander display panel. If temperature warning is indicated, shut down the compressor as soon as practical. The compressor can be switched off (DISENGAGED) at any time or input speed. Check for adequate water flow through heat exchanger. Check for adequate oil level in separator tank.



WARNING: *If compressor temperature continues to rise to 240°F, the compressor will be automatically disengaged.*

- 4) Turn on the foam proportioning system. When a FoamPro 2001 or 2002 is enabled, a red indicator light will be on steady. Light will flash as foam is injected. If a FoamPro 1601 system is used then upon turning the system "on" the red low foam indicator light will flash once to inform that the system is enabled.



CAUTION:

- Do not over speed compressor - Input RPM should not exceed that required to produce rated air flow of 220 cfm at 150 psi maximum pressure.
- Disengage air compressor when service testing or performing UL test on CAFS equipped vehicle.

Automatic Balanced Air Pressure Control

Air pressure will match water pressure up to 150 PSI if pump input speed is adequate to maintain flow rate setting. Note: Do not exceed 175-PSI pump pressure while compressor is engaged. Maximum air pressure has been factory preset to 150 PSI. (To avoid compressor over-speed, the AutoCAFS Commander control is programmed to provide a visual speed warning at $3650 \div$ pump ratio. Additionally the Commander is programmed to disengage the compressor at an input speed of $4500 \div$ pump ratio.)

NOTE: Oil Separator Tank Safety Relief Valve - 200 psi.

- 5) Increase engine speed to the desired operating pressure using the throttle or governor control provided. Common CAFS operating pressures range from 100 - 150 PSI. NFPA standard recommends 125 PSI.
- 6) Slowly open the CAFS discharge valve that is desired. Open completely to first fill the hose with foam solution. Then close the valve to approximately 1/3 open.
- 7) Open the accompanying airflow valve approximately 50% full open or turn the toggle switch "ON" to activate the preset airflow to the desired CAFS discharge.
- 8) Monitor the water and air flow rates on the flow meters and adjust to desired ratio. A one to one mix is a good ratio to start with. That is for example: 40 GPM to 40 CFM. If a higher water flow is used then the foam will be wetter. If a higher airflow is used then the foam will be dryer. Many operating guideline variables exist. A variety of standard operating procedures may be necessary to meet different incident objectives. For example: a drier (shaving cream type foam) will be necessary to provide exposure protection. It can be achieved by using a low flow rate of water (25gpm) and a higher flow rate of air (40 cfm). To achieve a large fire knockdown, higher flow rates of water (60 gpm) will be more desirable. At water flow rates over 50 gpm, airflow rates should be used at about an equal one to one ratio for best results.

<i>Foam Type</i>	<i>Hose size</i>	<i>Foam Solution GPM</i>	<i>Air Flow CFM</i>
Very Dry - Fluffy	1"	10 GPM	25 CFM
Dry to Medium	1"	20 GPM	20 CFM
Medium to Wet	1"	25 GPM	10 CFM
Very Dry - Fluffy	1-1/2" or 1-3/4"	15 GPM	60 CFM
Dry	1-1/2" or 1-3/4"	20 GPM	60 CFM
Medium	1-1/2" or 1-3/4"	40 GPM	60 CFM
Wet	1-1/2" or 1-3/4"	60 GPM	60 CFM
Very Wet	1-1/2" or 1-3/4"	70 GPM	50 CFM
Dry	2-1/2"	50 GPM	100 CFM
Medium	2-1/2"	80 GPM	100 CFM
Wet	2-1/2"	120 GPM	100 CFM

The above rates are based upon having a large ball shutoff and a large smooth bore tip approximately equal to the hose size. Fog nozzle tips will almost always limit flow rates, and usually reduce the flow of air. Dry foam types are next to impossible to achieve with fog nozzles. High gallonage fog nozzles do work very well for interior attack if solution flow gpm is high from 50-70 gpm and airflow rates are moderate 40-60 CFM.

- 9) Monitor the booster tank level and temperature during prolonged operation from tank only.
- 10) Monitor compressor temperature. Normal operating temperature is 170°F-185°F. If compressor temperature rises above normal operating temperature to 212°F, the Commander display will flash 'COMP HOT'. If temperature warning is indicated, shut down the compressor as soon as practical. The compressor can be switched off (DISENGAGED) at any time or input speed.



WARNING: *If compressor temperature continues to rise to 240°F, the compressor will be automatically disengaged.*

Steps for Shutdown

- 1) Close air valves.
- 2) Reduce pressure to idling condition.
- 3) Flush foam system per instructions.
- 4) If desired, use air to expel water from hose lines during freezing weather.
- 5) Disengage compressor.



CAUTION: *Avoid immediate restart of compressor after shutdown. Allow a 1-minute minimum time period between compressor shutdown and restart for system blow-down.*

Compressed Air for Air Tool Usage

- 1) Using standard shifting procedures shift the compressor and fire pump to the 'ENGAGED' position. **NOTE:** Water pump must be engaged and running to utilize air compressor for operating air tools.
- 2) Establish water flow in main pump. Open tank to pump valve and tank refill valve slightly to provide water circulation through pump.
- 3) Air pressure for operating air tools is automatically balanced with the water pump pressure. Maximum 150 PSI.

NOTE: Output capacity of air compressor is determined by pump RPM. Higher RPM's may be required to flow desired output if high flow rates are necessary.

- 4) Monitor airflow and pressure. Increase engine speed if necessary to supply needed air volume.
- 5) Monitor the booster tank temperature during prolonged operation from tank only.
REMEMBER: The air compressor lubrication system is water cooled by main water pump. If water is continually circulated back to tank, cooling water will be warmed.
- 6) Monitor compressor temperature. Normal operating temperature is 170°F-185°F. If compressor temperature rises above normal operating temperature to 212°F, the Commander display will flash 'COMP HOT'. If temperature warning is indicated, shut down the compressor as soon as practical. If air end temperature continues to rise to 240°F, the compressor will be automatically disengaged. Check for adequate water flow through heat exchanger. Check for adequate oil level in separator tank.

Important reminder: The air compressor can be disengaged (shifted out of gear) at any time if the need arises.



CAUTION: *Engaging of compressor must be done only when pump input shaft is less than 900 rpm.*

Usable Hose and Flow Rate Combinations

A proportioner setting of .3% is usually adequate for making compressed air foam in hose lines. Setting the proportioner for a lesser percentage will yield "wetter" appearing foam. Setting the proportioner to a higher percentage will yield "drier" appearing foam. Setting the proportioner too low (below .2%) may result in pulsation (water slugs) in the hose. This is due to not having enough concentrate in solution to form foam in the hose.

Much has been made over the ability of compressed air systems to create foam of shaving cream consistency. This foam is very stable and possesses a long drain time. However, the firefighter must make sure that this type of foam will release enough water to suppress fire if it is used in a direct attack. This "shaving cream" foam usually is only suited to defensive operations involving barrier, or fuel pre-treatment operations.



WARNING: *A compressed air foam hose possesses a pneumatic character in its performance due to the presence of the compressed air. This effect reveals itself most visibly in the surge of product at the time the hose is opened. This is a release of stored energy due to the compressibility of the foam in the hose. This effect may be detrimental if the firefighter is not prepared for the energy release. For this reason, valves must be opened slowly to dissipate the energy in a controlled manner.*

Hose Lays

Hose Diameter	Water GPM	Air CFM	Tip	Pressure	Hose Length
1"	20	20	3/4"	125-150	>200'
1"	15	15	1/2"	125-150	>400'
1 1/2"	30-40	30-40	1"	110-150	>800'
1 1/2"	50-60	50-60	1.25"	110-150	>400'
1 3/4"	30-40	30-40	1"	110-150	>1400'
1 3/4"	50-60	50-60	1.25"	110-150	>700'

On short hose lays (less than 200') of 1 3/4" hose the operator may establish flows of up to 70 gpm water and 60 cfm air. This is a very effective initial attack flow for structural fires.

The figures above are based on making mid range foam in terms of "wetness" and drain time. Using a smaller tip will yield wetter foam with some increase in reach. Using a larger tip will yield drier foam with an accompanying decrease in reach.

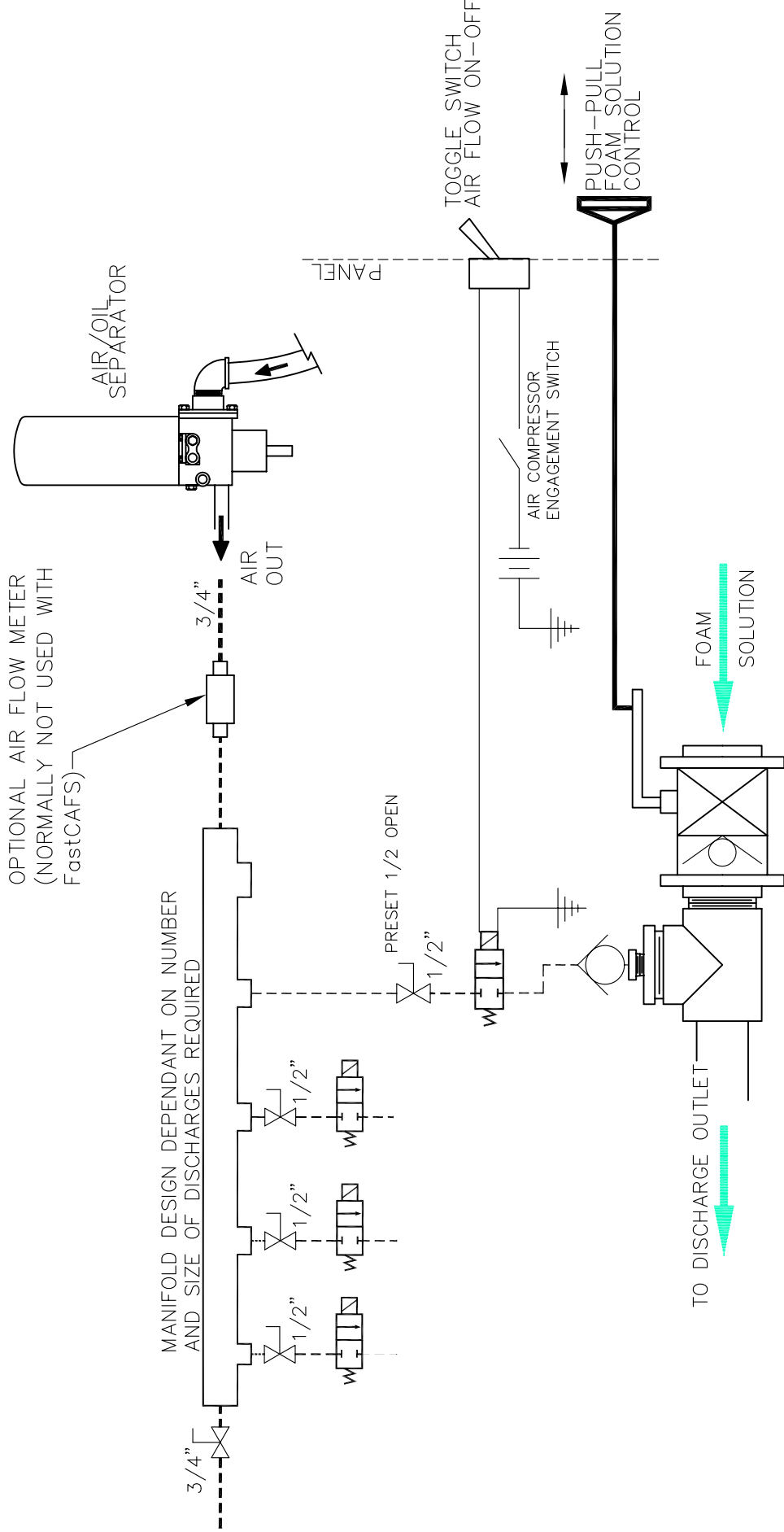
The foam concentrates designed for use on class B fires will work well with a compressed air foam system. The primary benefit of compressed air over nozzle aspiration lies in the extended drain times that compressed air foams exhibit and the increased discharge distance.

The drain time is usually measured as a "quarter drain" time. This is the time that it takes for foam to have 25% of the water drain from the bubble structure. Some aspirated foams have a quarter drain time as fast as two minutes. Compressed air foam made with the same concentrate ratio may have a quarter drain time of up to fifteen minutes. A long quarter drain time is very important on incidents involving un-ignited fuel, where water run-off from tactical operations is a problem.

A long quarter drain time is also desirable during many operations involving class A foam. Defensive operations involving exposure protection of fire line construction are two primary tactics that utilize the long quarter drain time of compressed air foam. The long quarter drain time allows the firefighter to position water on the subject fuel for an extended period of time. This characteristic coupled with the active fuel-wetting characteristic of class A foam makes a very good fire barrier.

NOTES





- 1) MANIFOLD AND VALVES MOUNTED BEHIND PANEL
- 2) PRESET AIR VALVES 1/2 OPEN
- 3) EACH SOLENOID VALVE CONTROLLED BY INDIVIDUAL PANEL MOUNTED SWITCH
- 4) AIR PRESSURE CONTROL VALVE SYSTEM IS MOUNTED BEHIND PANEL PRESET TO DESIRED OPERATING PRESSURE

THIRD ANGLE PROJECTION

INCH
[MILLIMETER]

MATERIAL DESCRIPTION: —		MATERIAL NO. —	PATTERN NO. —	OLD PART NO. —	TOLERANCE EXCEPT AS NOTED .00 ±.03 .000 ±.010 ANGLES ±1°		W.S. DARLEY & CO. MELROSE PARK, IL — CHIPPEWA FALLS, WI			
					INSTR — FastCAFS ASSEMBLY —					
THIS DESIGN IS THE PROPERTY OF W.S. DARLEY AND CO. — UNAUTHORIZED REPRODUCTION IS PROHIBITED		ALL DIMENSIONS IN INCHES UNLESS NOTED		DO NOT SCALE PRINT		DR'N	DWS	DATE	APR20,99	1205510
						CHKD	—	SCALE	1/16	
						TRCD				

W.S. DARLEY & CO.
MELROSE PARK, IL — CHIPPEWA FALLS, WI

INSTR — FastCAFS ASSEMBLY

REVISIONS							
REP#	NAME OF PART	QTY	LTR	DESCRIPTION	DATE	CHG NO.	APPR'D
1	BALL VALVE	1					
2	CHECK VALVE	1					
3	GASKET	1					
4	HEX HD CAP SCREW	4					
5	CLOSE NIPPLE	1					
6	TEE	1					
7	BUSHING	1					
8	CHECK VALVE	1					

NOTES:

POSITION CHECK VALVE #2 BETWEEN VALVE FLANGE AND VALVE BODY, INCLUDING GASKET ON FLANGE SIDE

TIGHTEN #4 CAP SCREWS EQUALLY IN A CROSSING PATTERN

TORQUE TO 25-30 FT.LBS.

DO NOT MOUNT AT AN ATTITUDE THAT WILL TRAP WATER ABOVE CHECK VALVE #2

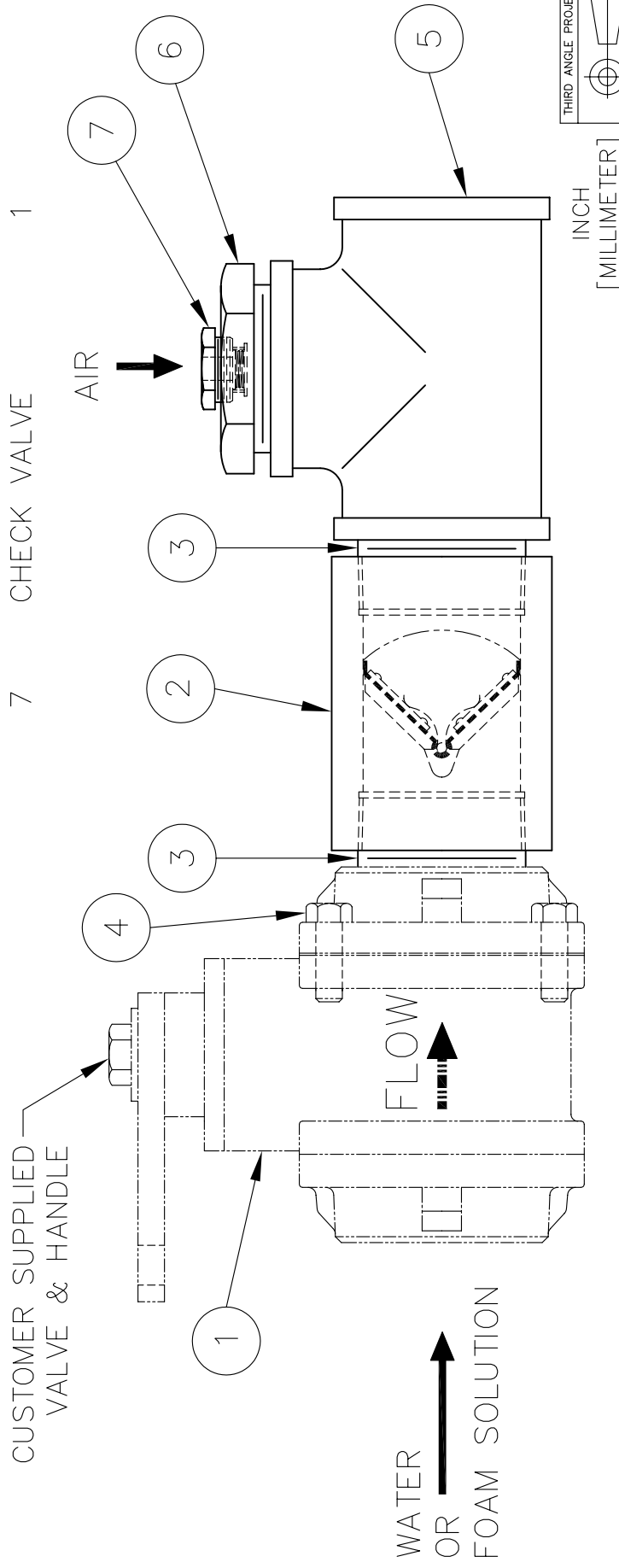
PROVIDE DRAIN TAPS WHERE REQUIRED

MATERIAL DESCRIPTION:		REF. AZ00115 AZ00116		OLD NO. C152	TOLERANCE EXCEPT AS NOTED FRAC DIM ±.01 .00 ±.01 .000 ±.005 ANGLES ±1°		W.S. DARLEY & CO. MELROSE PARK, IL - CHIPPEWA FALLS, WI
			MATERIAL NO.	PATTERN NO.	DR'N DWS CHKD TRCD		2" & 2 1/2" CHECK VALVE ASSEMBLY
THIS DESIGN IS THE PROPERTY OF W.S. DARLEY AND CO. - UNAUTHORIZED REPRODUCTION IS PROHIBITED		ALL DIMENSIONS IN INCHES UNLESS NOTED		DO NOT SCALE PRINT		DATE MAR10,94 SCALE 1/4	
						DCM0301	


REVISIONS

- 1) POSITION CHECK VALVE #2 BETWEEN VALVE FLANGE AND TEE.


<u>REP #</u>	<u>NAME OF PART</u>	<u>QTY</u>
1	BALL VALVE	1
2	CHECK VALVE	1
3	CLOSE NIPPLE	2
4	HEX HD CAP SCREW	4
5	PIPE TEE	1
6	BUSHING	1
7	CHECK VALVE	1



INCH
 [MILLIMETER]



THIRD ANGLE PROJECTION

REF. — AZ00128		OLD PART NO. —	 W.S. DARLEY & CO. MELROSE PARK, IL — CHIPPEWA FALLS, WI	
MATERIAL DESCRIPTION: —	MATERIAL NO. —	PATTERN NO. —	TOLERANCE EXCEPT AS NOTED .00 ±.03 .000 ±.010 ANGLES ±1°	
	DO NOT SCALE PRINT			
THIS DESIGN IS THE PROPERTY OF W.S. DARLEY AND CO. — UNAUTHORIZED REPRODUCTION IS PROHIBITED	ALL DIMENSIONS IN INCHES UNLESS NOTED	4" CHECK VALVE ASSEMBLY —		DATE 2/12/99 SCALE 1/4
				DCM0302

AutoCAFS II TROUBLESHOOTING GUIDE

SYMPTOM	POSSIBLE CAUSES	CORRECTIVE ACTION
Air compressor will not engage	<ul style="list-style-type: none"> • Pump input rpm is too fast • Separator tank pressurized • Circuit Breaker/Fuse open • Faulty/loose control connections • Compressor over heated • Clutch coil failure • AutoCAFS Commander failure 	<ul style="list-style-type: none"> • Reduce throttle setting • Allow time for blow-down • Check blow-down valve and pressure sensor switch • Reset – diagnose and correct cause • Inspect and repair • Find cause and correct • Replace clutch • Replace
Air Compressor will not make any air pressure or air pressure is too low	<ul style="list-style-type: none"> • Air compressor is not engaged • Air compressor pressure limiting valve - set too low • RPM of engine too low to support the flow of air being discharged 	<ul style="list-style-type: none"> • Engage air compressor using proper shifting procedures • Adjust pressure setting raise the air pressure (red needle) to 150 psi • Increase engine RPM - relief valve may need to be used to hold pump pressure within range
Air pressure too low to run air tools from idle through 1200 rpm range	<ul style="list-style-type: none"> • Independent air tool regulator set too low • Pump rpm too low, water/air pressure low 	<ul style="list-style-type: none"> • Raise regulator pressure by pulling up on knob and turning clockwise • Increase engine rpm
CAFS over speed indicated 'HI RPM'	<ul style="list-style-type: none"> • Engine RPM too fast 	<ul style="list-style-type: none"> • Reduce throttle setting to normal operating speed.

Compressor Overheating 'COMP HOT'	<ul style="list-style-type: none"> Oil temperature has exceeded the recommended maximum operating temperature of approx. 212 °F Water pump has not been circulating water and has overheated Oil/Water heat exchanger has water supply blocked, either the supply line, return line, or the heat exchanger body has a blockage Thermo-valve in oil filter mounting block has failed or has become obstructed Oil level low Oil filter blocked 	<ul style="list-style-type: none"> Disengage air compressor, if fire fighting - return to conventional water or foam solution fire fighting practices Circulate fresh water through the water pump so that the heat exchanger receives cool water flow through it to cool the oil. Remove the water return line on the suction side of the pump, try to locate the source of the blockage, remove obstruction Remove obstruction from thermo-valve or replace oil filter mounting block/thermo-valve housing WSD # 4420902 Add oil Replace
Transmission High Temp Shutdown 'TRANS HOT'	<ul style="list-style-type: none"> Pump transmission lubricant level incorrect Pump transmission bearing failure Compressor Clutch slipping <ul style="list-style-type: none"> Clutch voltage low, must be 12 VDC \pm10% Clutch disc contamination Clutch disc wear Compressor clutch dragging when disengaged 	<ul style="list-style-type: none"> Inspect and correct Rebuild transmission Check voltage and correct Inspect and clean clutch discs Replace clutch discs Inspect and replace clutch disc springs
Compressor Automatically Disengages	<ul style="list-style-type: none"> Compressor overheated above 240°F Input RPM too fast Transmission Overheated 	<ul style="list-style-type: none"> See above Disengage compressor switch <ul style="list-style-type: none"> Reduce throttle setting to idle Wait 1 minute for system blow down Engage compressor switch See above

Air pressure continually rises and cannot be controlled - without opening an air flow valve to dump excess pressure	<ul style="list-style-type: none"> • Check for loose connection in ¼" air pressure control line to the air inlet valve • Plugged control line or orifice 	<ul style="list-style-type: none"> • Tighten all hose fittings and air pressure control line connections • Clean lines and orifices • NOTE: lowering control pressure into inlet valve will result in air compressor building pressure
Hose line is erratic, jumping all over, hard to hang onto the line	<ul style="list-style-type: none"> • Condition known as "Slug flow" Created by lack of foam solution or too low of % - water and air do not mix without foam added 	<ul style="list-style-type: none"> • Eliminate airflow in line until foam concentrate can be introduced at the proper rate of 0.3%. Some foam concentrates may require special consideration or attention. (i.e. higher %)
Foam is too dry, - Can't soak into anything or absorb much heat	<ul style="list-style-type: none"> • Ratio of air to water is too high or a very long hose line is being used • Foam percentage is too high 	<ul style="list-style-type: none"> • Increase water flow or decrease air flow, or slightly close nozzle • Lower the percentage using the gray down ↓ arrow button
Foam is too wet and runny, - Not making shaving cream type foam	<ul style="list-style-type: none"> • Ratio of water to air is too high • Foam percentage is too low • Incorrect Nozzle on hose line, fog nozzles break up bubbles • Kink in hose or too short of run of hose (100 ft minimum) 	<ul style="list-style-type: none"> • Reduce water flow/increase air flow • Be sure proportioner is set at least 0.3% and use good foam • Nozzle must be full flow with a large smooth bore tip • Straighten out kink in hose or add lengths to the hose line
Insufficient air output	<ul style="list-style-type: none"> • Air filter dirty • Oil separator blocked • Intake valve faulty • Manual pressure valve faulty or incorrectly set • Faulty balance valve • RPM is too low 	<ul style="list-style-type: none"> • Replace • Replace • Inspect & repair • Inspect & reset • Inspect, clean, repair • Increase RPM

Receiver tank safety valve blows at 200 psi	<ul style="list-style-type: none"> • Air pressure control valve is set too high 	<ul style="list-style-type: none"> • Adjust control valve (see section 2)
Valve blows at less than 200 psi	<ul style="list-style-type: none"> • Safety valve is defective 	<ul style="list-style-type: none"> • Replace safety valve
Oil consumption high Oil is coming out of hand lines / air tool chucks	<ul style="list-style-type: none"> • Oil level in hydraulic oil reservoir tank is too high • Oil/Air Separator cartridge has become clogged or defective • Too much condensation in the oil • Oil return line or orifice clogged • Wrong type oil • Oil leak 	<ul style="list-style-type: none"> • Check and adjust oil level with compressor off • Replace Separator cartridge • Inspect oil, drain and replace • Clean • Drain all components and hoses, replace oil with correct type • Repair
Air compressor surges which raises and lowers rpm, pressure, and also cfm flow of air	While air is flowing, the air inlet modulator valve opens and closes to keep air pressure in the proper range. This is most noticeable in the 20 - 80 cfm range.	This is a normal and required occurrence. The surging should be at a rate fast enough to keep air pressure from falling too low. Adjustment to the surge cycle can be accomplished by turning the needle valve on pressure control assembly
Compressor drive overloaded at startup (system still pressurized at startup)	<ul style="list-style-type: none"> • System blow down has not been completed • Faulty blow down valve • Faulty separator tank pressure sensing valve • Intake valve leaking or open 	<ul style="list-style-type: none"> • Allow at least 1 minute from shutdown to startup for system blow down • Replace • Replace • Inspect and Repair

FOAM PROPORTIONER SECTION

Foam pump runs but produces no foam flow	Foam pump is not primed	See foam pump priming (page 20 in FoamPro manual)
Foam pump loses prime, makes a chattering noise	<ul style="list-style-type: none"> • Air leak in suction hose or fittings • Suction line is blocked or kinked • Clogged suction strainer 	<ul style="list-style-type: none"> • Fix leaks • Remove suction hose and check for loose lining. Inspect for blockage or remove kinks • Clean strainer
No characters in the digital display	<ul style="list-style-type: none"> • The main power switch on the motor is not turned on • Cables are defective or installed improperly 	<ul style="list-style-type: none"> • Turn on power toggle switch • Inspect cables and secure connections or replace if defective

System is powered up and the foam on/off switch has been pressed but the foam pump will not run	<ul style="list-style-type: none"> No water is flowing in any of the foam discharges Flow meter is obstructed or defective Foam tank level sensor is sending low foam level signal Control cable is defective 	<ul style="list-style-type: none"> Flow water out a desired foam discharge Remove obstruction or replace flow meter Fill foam tank if low or repair level sensor if it is incorrectly operating Check connections or replace cable
LO. CON appears in the digital display	<ul style="list-style-type: none"> Foam concentrate level in tank is low Low level tank sensor is incorrectly sensing low level 	<ul style="list-style-type: none"> Refill concentrate tank with the proper foam type Repair or replace level sensor
NO. CON appears in the digital display This automatically happens 2 minutes after LO.CON appears in display	<ul style="list-style-type: none"> Foam concentrate level in tank is empty Tank level sensor is incorrectly sensing empty tank level 	<ul style="list-style-type: none"> Refill concentrate tank with the proper foam type Repair or replace level sensor
Foam pump runs full speed when main power circuit is turned on	<ul style="list-style-type: none"> Poor ground either to motor driver or mounting bracket Bad motor driver box 	<ul style="list-style-type: none"> Make sure screws are tight and that good ground is maintained Replace motor driver box
Display shows a ?	<ul style="list-style-type: none"> Flow meter is sensing water flow, but the rate is too low for precise proportioning 	<ul style="list-style-type: none"> This is common at start up and shut down of water flow. Check flow meter or flow more water.
System returns to standby mode or HYPRO appears in display momentarily while pumping	<ul style="list-style-type: none"> Insufficient power supply Current resistance in wiring circuits 	<ul style="list-style-type: none"> Inspect and correct power and ground connections and wiring Make sure a minimum 8 AWG wire is used to install to battery

Class A Foam References

The National Wildfire Coordinating Group (NWCG) has sponsored the publication of the following items produced by the NWCG Working Teams. Copies of each of these items may be ordered from the National Interagency Fire Center (NIFC). To order, mail or fax a purchase order or requisition to:

National Interagency Fire Center
ATTN.: Supply
3905 Vista Avenue
Boise, Idaho 83705

FAX 208-387-5573

Orders must be from agencies or organizations, not private individuals. Use the "NFES" number for the item(s) you are ordering. Do not send money, checks, or money orders with the order. Phone orders are not accepted. You will be billed the cost of the item(s) after the items are sent. Orders from other than Federal wild land fire agencies or State land protection agencies will receive an 18% surcharge on the bill. Transportation charge, other than mail, will also appear on the bill. Questions regarding ordering procedures can be addressed to the NIFC Supply Office, 208-387-5542. Questions regarding billing procedures can be addressed to NIFC Finance Office, 208-387-5533.

PLEASE NOTE THAT THE NIFC FIRE CACHE PERFORMS INVENTORY DURING THE MONTH OF JANUARY. ORDERS ARE NOT PROCESSED DURING INVENTORY. ORDERS RECEIVED DURING THIS INVENTORY PERIOD ARE DATE STAMPED AND PROCESSED IN THE ORDER THEY WERE RECEIVED.

ESTIMATED PRICES ARE SHOWN FOR SOME OF THE ITEMS. ACTUAL PRICES WILL NOT BE KNOWN UNTIL ITEMS HAVE BEEN RECEIVED. ACTUAL COSTS WILL BE CHARGED WHEN FILLING ORDERS.

PLEASE INSURE THAT ALL ORDERS HAVE CORRECT NFES #'S FOR THE ITEMS BEING ORDERED.

INTRODUCTION TO CLASS A FOAM, 1989

13:00 minute videotape, VHS size only

NFES 2073

First of a videotape series dealing with foam use. This tape is a brief introduction to class A Foam technology covering foam chemistry, foam generating equipment, and examples of foam application. PMS 445-1.

THE PROPERTIES OF FOAM, 1993

15:00 minute videotape, VHS size only

NFES 2219

Second in a videotape series about class A foam. Explains how class A foam enhances the abilities of water to extinguish fire and to prevent fuel ignition. Basic foam concepts including drain time, expansion and foam type are explained. This revised 1993 version differs from the original 1992 videotape only in the way "foam types" are categorized. The original 1992 version described foam types as "foam solution, fluid, dripping and dry." The 1993 revision of the video describes foam types as "foam solution, wet, fluid and dry." PMS 445-2.

CLASS A FOAM PROPORTIONERS, 1992

23:10 minute videotape, VHS size only

NFES 2245

Third in a videotape series about class A foam. Explains how common foam proportioners devices, which add a measured amount of foam concentrate to a known volume of water, work. Advantages and disadvantages are presented. PMS 445-3.

ASPIRATING NOZZLES, 1992

10:13 minute videotape, VHS size only

NFES 2272

Fourth in a videotape series about class A foam, the difference between low and medium expansion nozzles, and appropriate uses for each nozzle. PMS 445-4

COMPRESSED AIR FOAM SYSTEMS, 1993

20:00 minute videotape, VHS size only

NFES 2161

Fifth in a videotape series about class A foam. Describes equipment, including water pumps, air compressors, drive mechanisms, and nozzles, used to generate compressed air foam. Presents rules of thumb for simple and reliable foam productions. Explains procedures for safe operation. Compares compressed air foam to air-aspirated foam. Presents advantages and disadvantages of the system.

FOAM VS. FIRE, PRIMER, 1992

NFES 2270

This 9-page publication covers the basics of using class A foams and discusses their adaptability to present application equipment. First is a series of three "Foam vs. Fire" publications. PMS 446-2.

FOAM VS FIRE, CLASS A FOAM FOR WILD LAND FIRES, 1993

NFES 2246

This 28-page publication explains how to get the most fire fighting punch from water by converting water to class A. foam. Discusses how and why foam works. Explains drain time, expansion ratio, foam type, proportioning, aspirating nozzles and compressed air foam systems. Also discusses application for direct attack, indirect attack, mop up, structure protection, and safety considerations. Slightly revised from 1992 edition to clarify foam types and descriptions. Second in a series of three "Foam vs. Fire" publications. PMS 446-1.

For those who would like a list of training materials and other publications available from NIFC, please order:

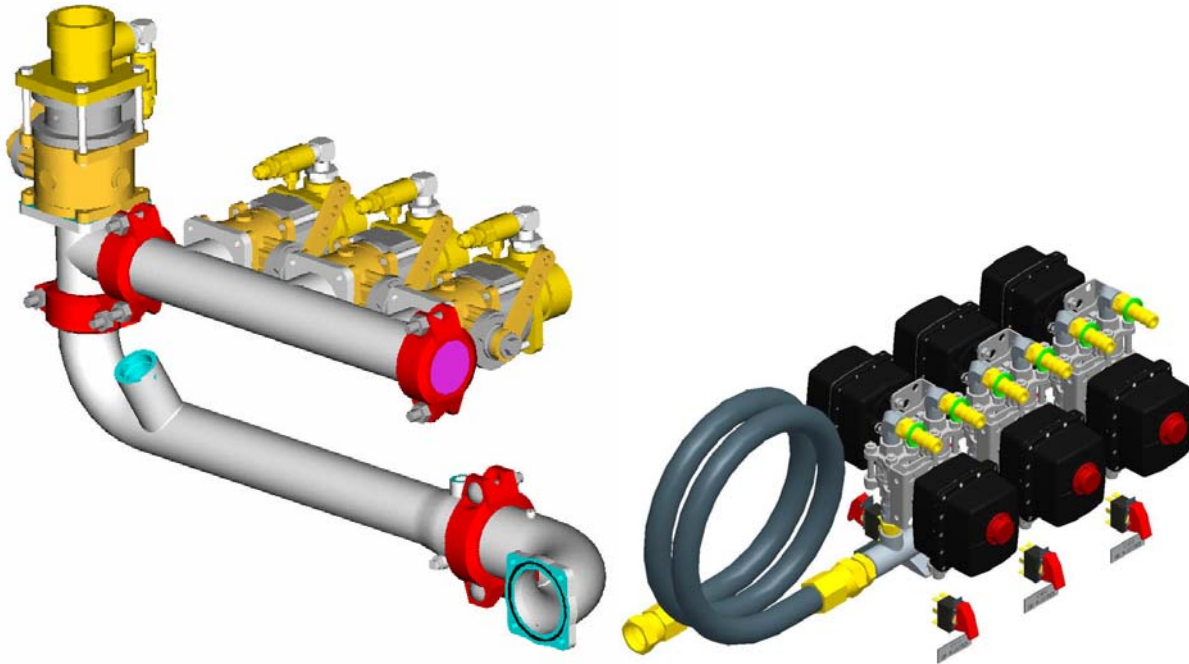
NFES 3362 1994 NWCG NFES Publications Catalog
(Available April 1, 1994)

APPENDIX

- ***FOAM MANIFOLD PARTS AND CONFIGURATION***
- ***ELECTRIC CLUTCH MAINTENANCE AND REPAIR GUIDE***
- ***AutoCAFSII TEST REFERENCE GUIDE***
- ***DETAILED SPECIFICATIONS***

FOAM MANIFOLD CONFIGURATION and PARTS GUIDE

**Including Darley AutoCAFS AIR (Air Injection Regulator)
Air distribution manifold assembly**



Introduction:

The Darley CAFS manifold system is a complete package offering all the components required to equip your apparatus with injection ports and check valves for foam concentrate and compressed air injection. For foam only applications, a complete manifold assembly may be configured to include a foam concentrate back flow check valve, foam concentrate injection port, FoamPro flow meter port, and a number of 2" and 2 1/2" outlet ports. The addition of CAFS discharge valve assemblies and air discharge distribution components complete the system for compressed air foam operation. All components are integrated with a stainless steel modular piping system.

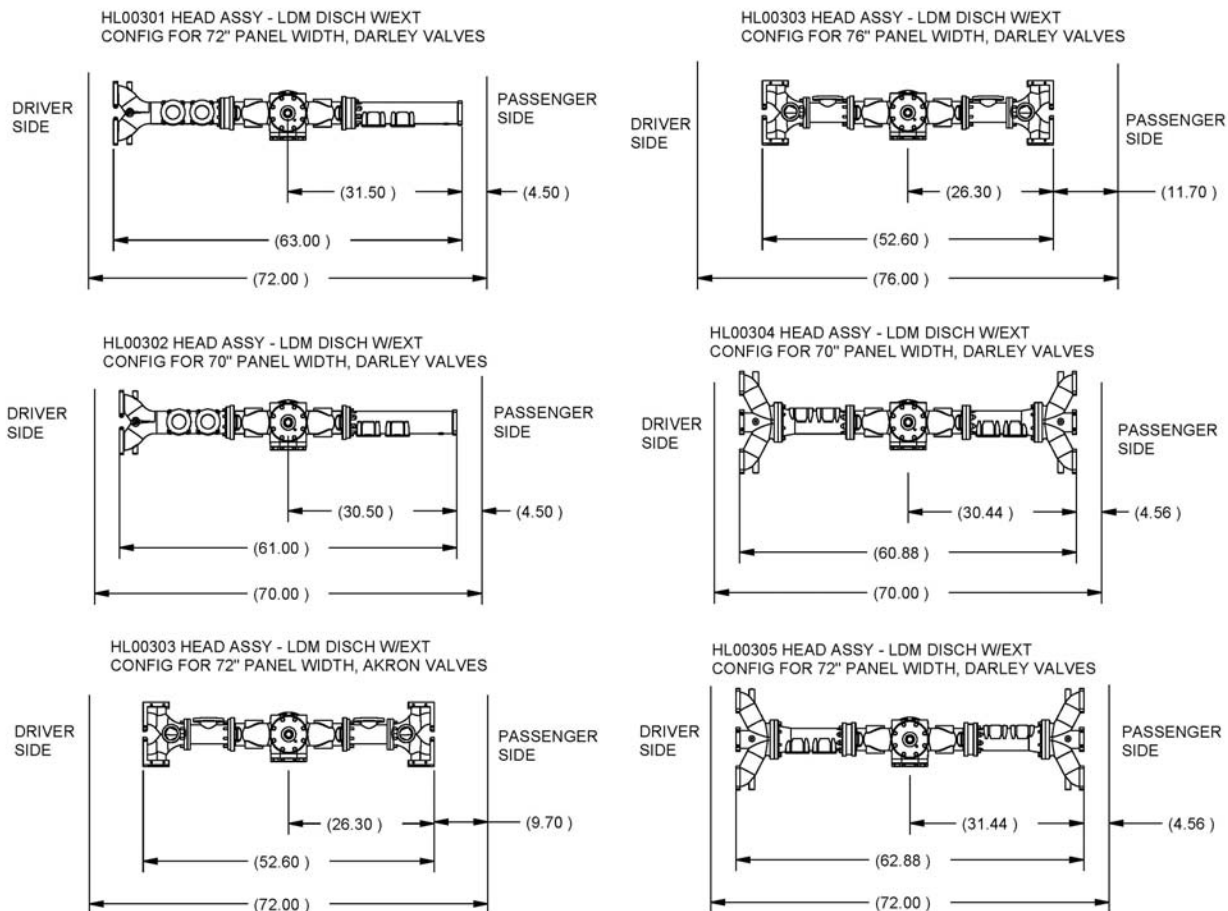
Although designed for direct mounting to the LDMBC pump, the foam manifold is readily adaptable to all CAFS installations.

In addition, a *FastCAFS* air distribution valve assembly is available for complete "motorized" electric valve control of individual air injection ports. This actuated electric valve provides for smooth air injection flow rate resulting in smooth, no-shock operation. The *FastCAFS* air distribution assembly incorporates a stainless steel manifold with 6 ports and (1) 12VDC "motorized" electric valve for each CAFS discharge valve assembly. An optional air flow meter may be mounted directly to the outlet port of the oil separator tank for air flow measurement and air valve calibration. A 72" primary air hose is provided for connection of the distribution manifold to the air flow meter, providing flexible remote mounting in the pump compartment.

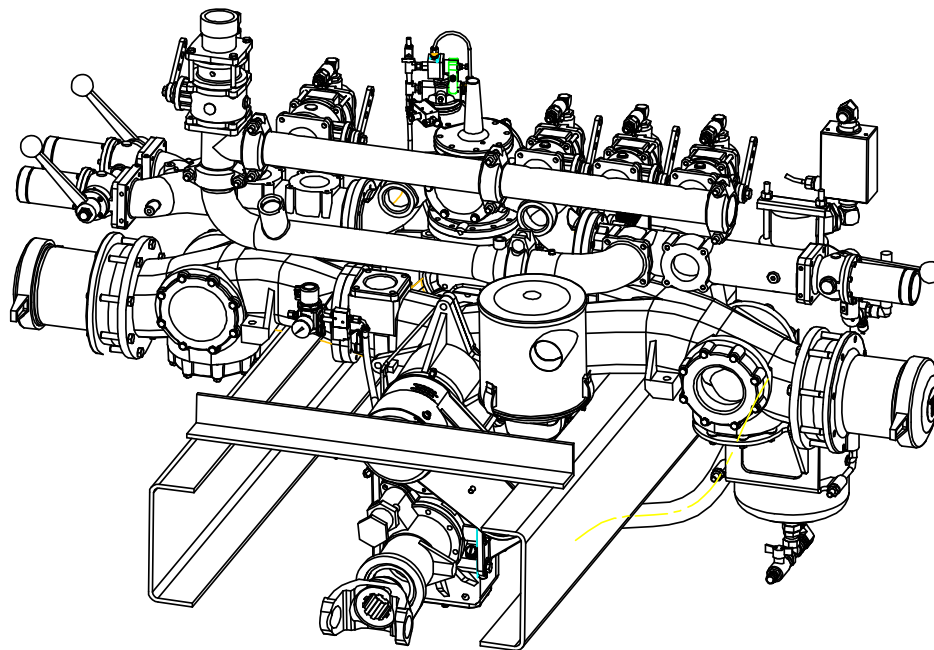
Six variations in the primary discharge extension assembly are available for attachment of the foam manifold system. Each of these assemblies is specifically designed to provide an attachment point in either the forward or rearward direction for the foam manifold while providing for the most widely used panel discharge valve configuration.

Please review the following assemblies noting the body width, type of panel discharge valve and number of panel discharge valves. With this information as a guide, select desired HL003__ series Head Assembly.

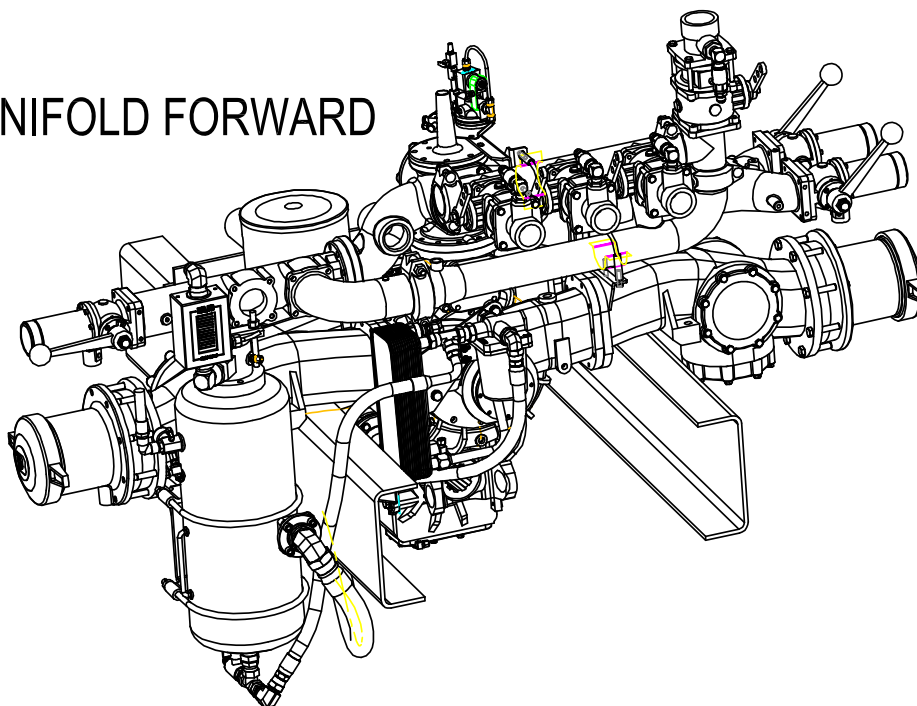
DISCHARGE EXTENSION OPTIONS FOR FOAM MANIFOLD ATTACHMENT



R MANIFOLD REAR



MANIFOLD FORWARD

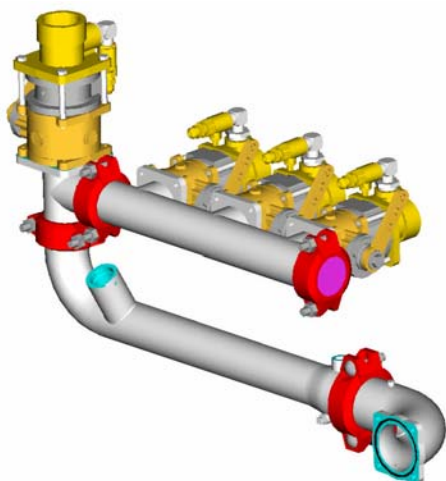


Three basic options are available for manifold configuration.

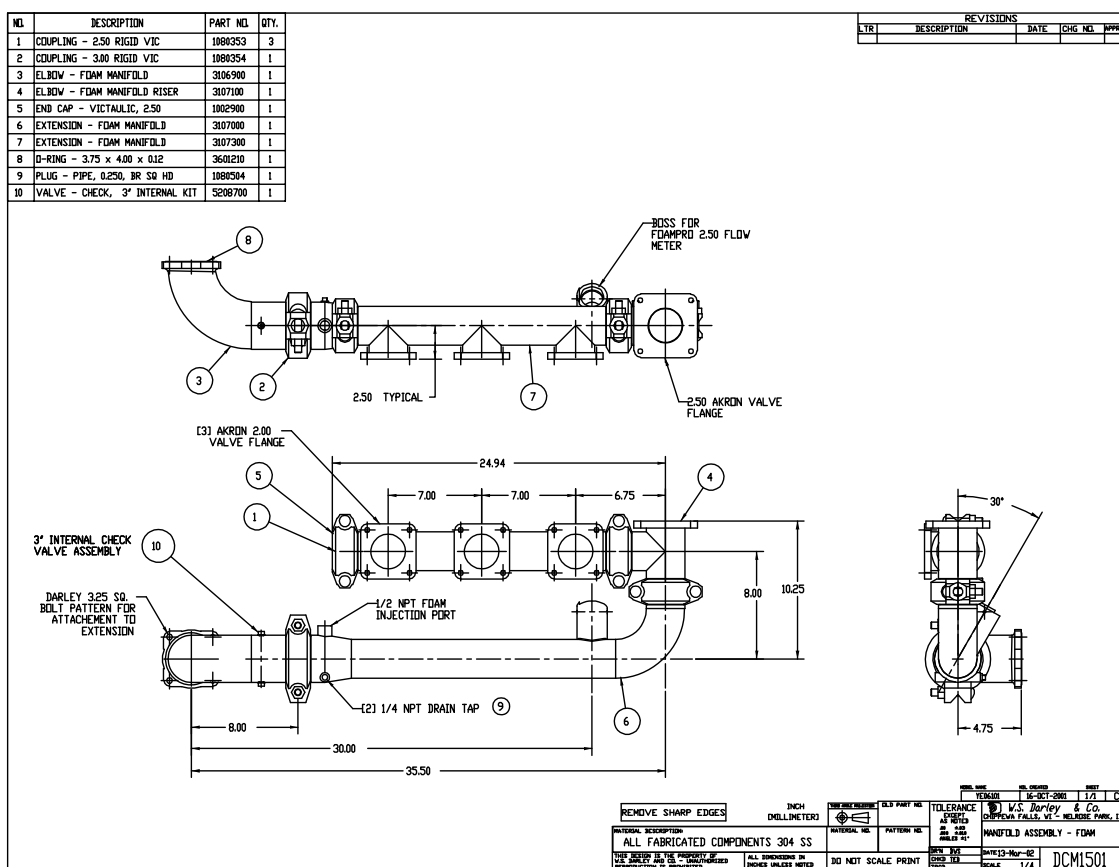
Option **00** – Drawing DCM1500, (2) Flange for 2 ½” CAFS Valve, (3) Flange for 2” CAFS Valve – Extensions may be rotated to position valves in a horizontal or vertical orientation.

Option **01** – **Standard OEM Configuration** - Drawing DCM1501, (1) 2 ½” flange for CAFS Valve, (3) Flange for 2” CAFS Valve – Extensions may be rotated to position valves in a horizontal or vertical orientation. Additional (3) outlet manifold extension can be optionally added for expansion.

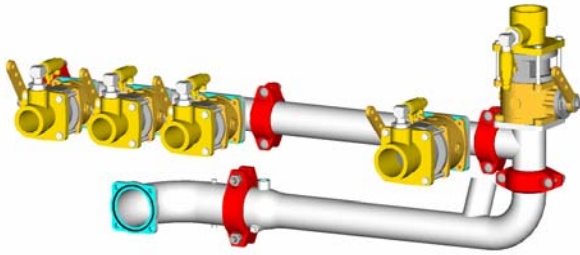
Option **02** – Drawing DCM1502, (2) 2 ½” victaulic rolled groove, (3) 2” victaulic rolled groove.



Option 01 - OEM Standard



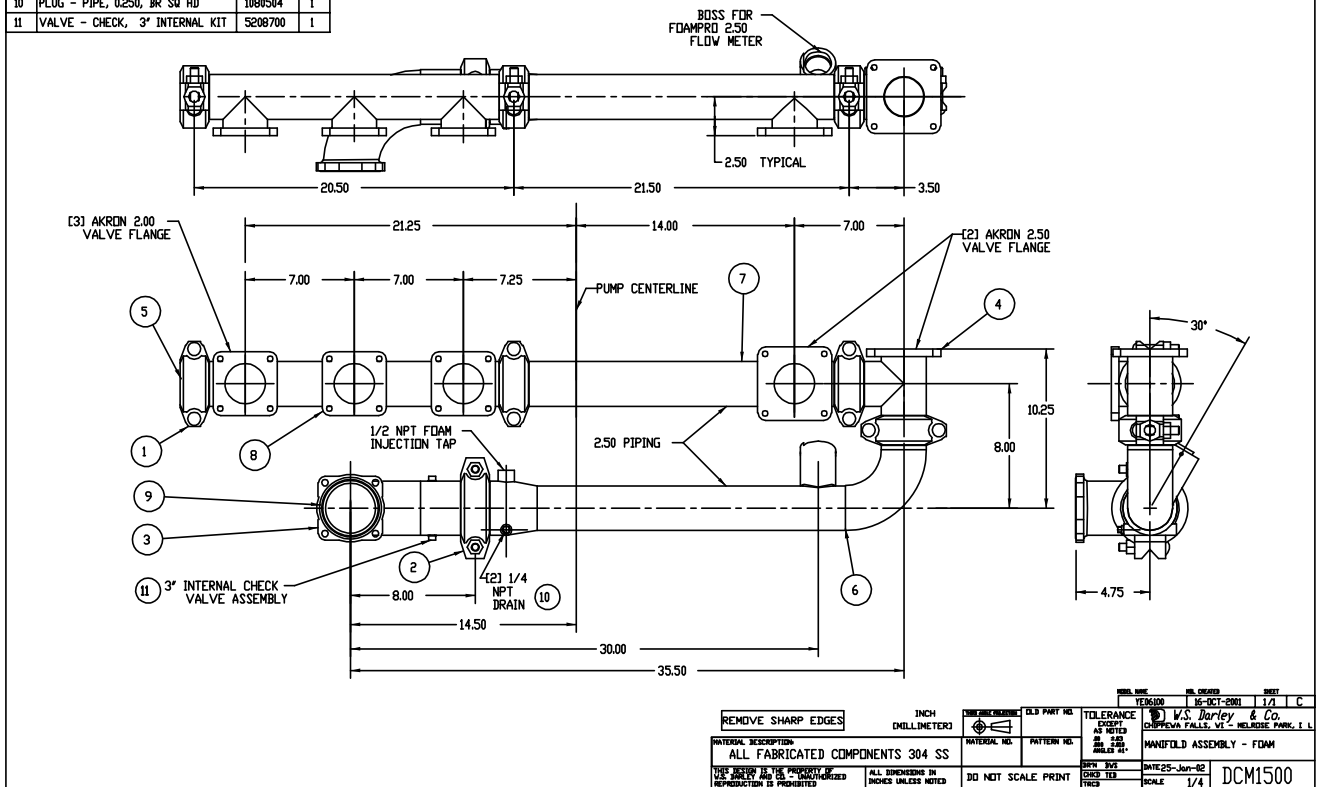
Prepared by: DWS
Approved by: TMC
Revised by: DWS

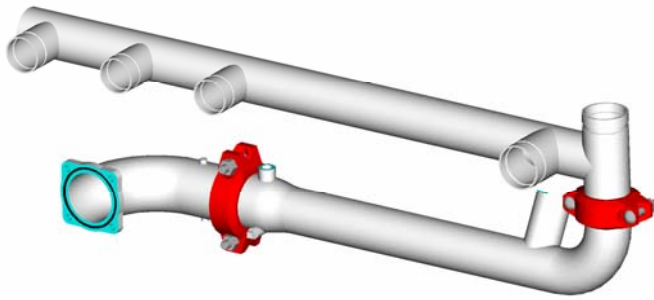


Option 00

NO.	DESCRIPTION	PART NO.	QTY.
1	COUPLING - 2.50 RIGID VIC	1080353	4
2	COUPLING - 3.00 RIGID VIC	1080354	1
3	ELBOW - FOAM MANIFOLD	3106900	1
4	ELBOW - FOAM MANIFOLD RISER	3107100	1
5	END CAP - VICTAULIC, 2.50	1002900	1
6	EXTENSION - FOAM MANIFOLD	3107000	1
7	EXTENSION - FOAM MANIFOLD	3107200	1
8	EXTENSION - FOAM MANIFOLD	3107300	1
9	O-RING - 3.75 x 4.00 x 0.12	3601210	1
10	PLUG - PIPE, 0.250, BR SQ HD	1080504	1
11	VALVE - CHECK, 3" INTERNAL KIT	5208700	1

REVISIONS			
REV	DESCRIPTION	DATE	CHG NO.

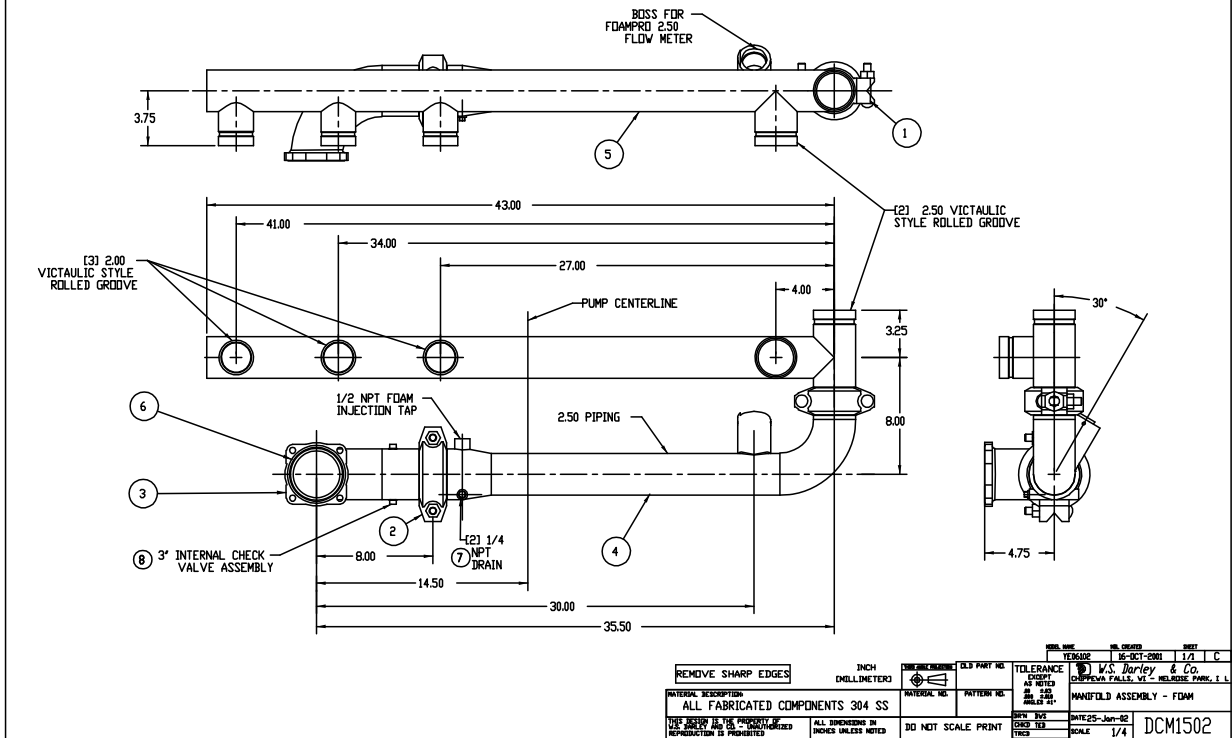




Option 02

NO.	DESCRIPTION	PART NO.	QTY.
1	COUPLING - 2.50 RIGID VIC	1080353	1
2	COUPLING - 3.00 RIGID VIC	1080354	1
3	ELBOW - FOAM MANIFOLD	3106900	1
4	EXTENSION - FOAM MANIFOLD	3107000	1
5	EXTENSION - FOAM MANIFOLD	3107500	1
6	O-RING - 3.75 x 4.00 x 0.12	3601210	1
7	PLUG - PIPE, 0.250, BR SQ HD	1080504	1
8	VALVE - CHECK, 3" INTERNAL KIT	5200700	1

REVISIONS			
TR	DESCRIPTION	DATE	CHG. NO.

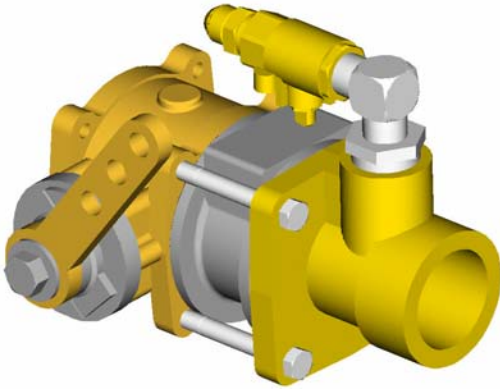


This manifold assembly would be used in applications where it is desired to run extension piping from the manifold to individual flow meters (not supplied). OEM supplied Akron x victaulic flange is required to attach CAFS valve assembly to manifold.

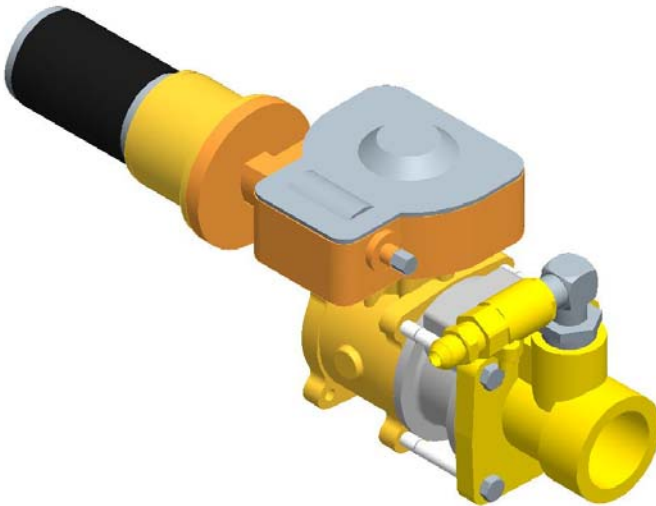
CAFS Discharge Valve Assemblies:

Darley CAFS discharge valve assemblies incorporate a foam solution check valve, air inlet check valve, and an adjustable air inlet flow control valve. The CAFS valve assemblies are available in 2" and 2 ½" sizes. Use a 2" valve assembly for crosslays and preconnects. A 2 ½" CAFS valve assembly is applicable to 2 ½" discharge lines or deck guns. For dimensional information on the 2" valve assembly, part number AZ02500, refer to drawing DCM0303. For the 2 ½" assembly, part number AZ02600, refer to drawing DCM0304.

The standard OEM Manifold, DCM1501, is configured for (1) 2 ½" and (3) 2" CAFS valve assemblies. If additional valves are required, please indicate total number of valves desired on the Option-Pricing form.

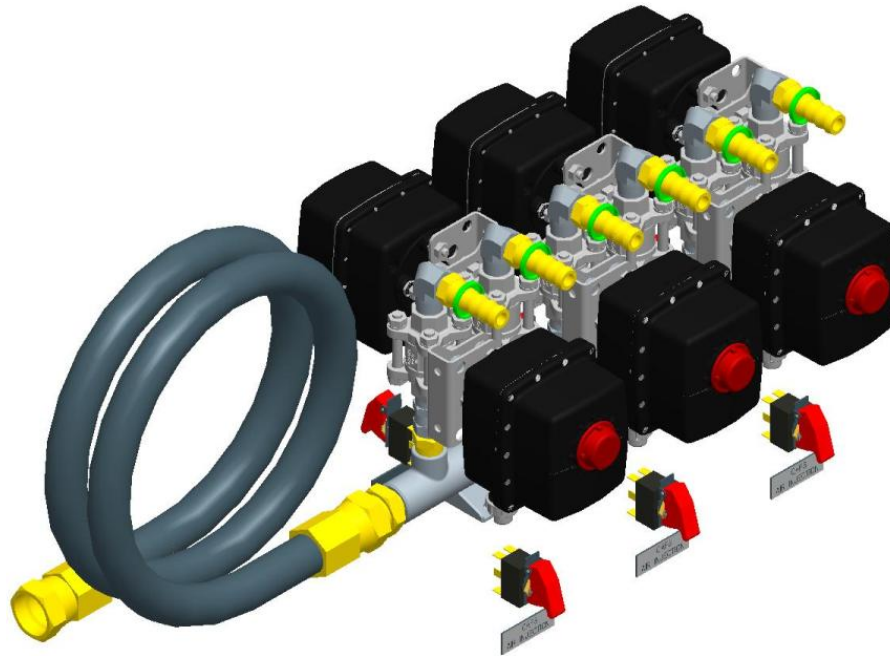


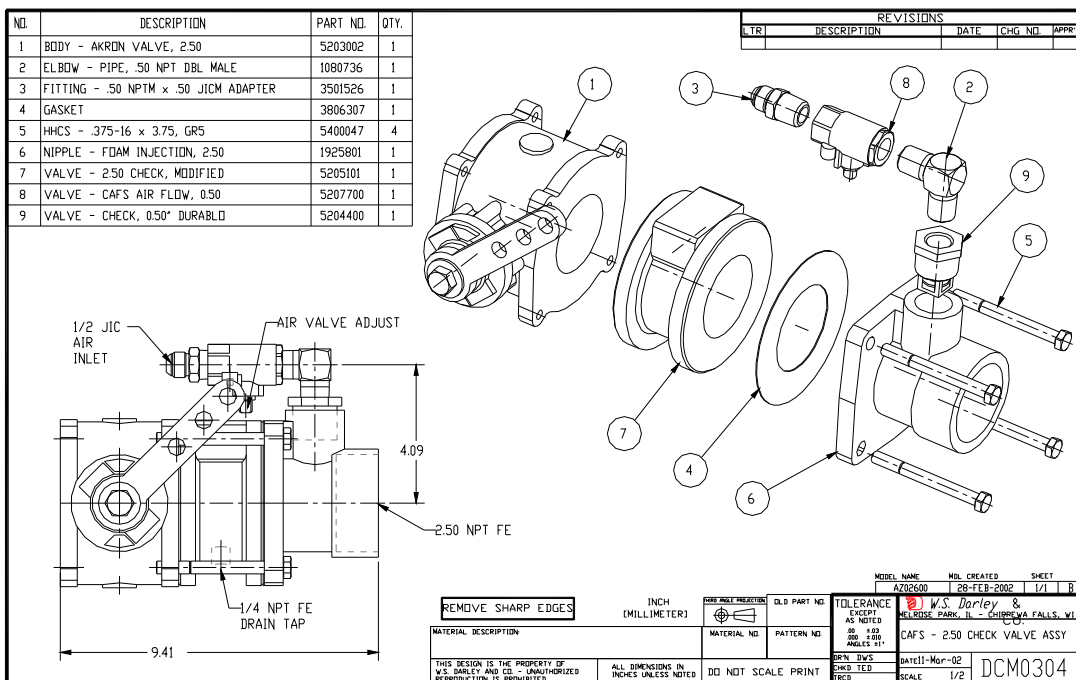
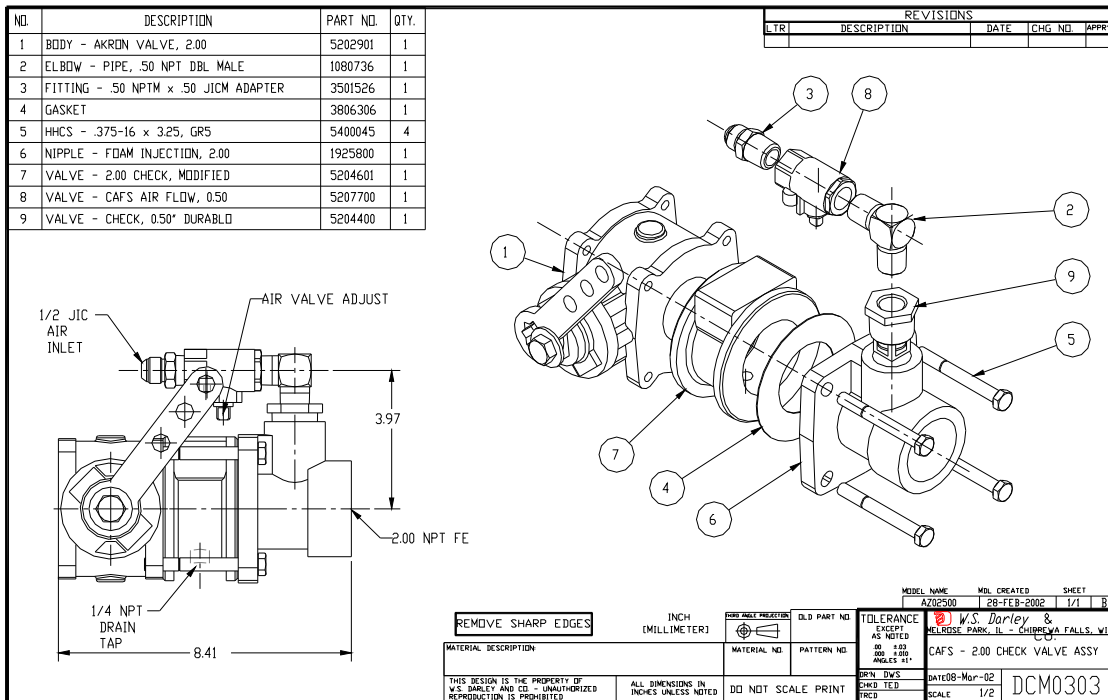
Optionally, the CAFS discharge valve assembly is available complete with an integrated electric valve actuator and panel control. For dimensional information on the actuated 2" valve assembly, part number AZ02700, refer to drawing DCM0305. For the 2 ½" assembly, part number AZ02800, refer to drawing DCM0306.



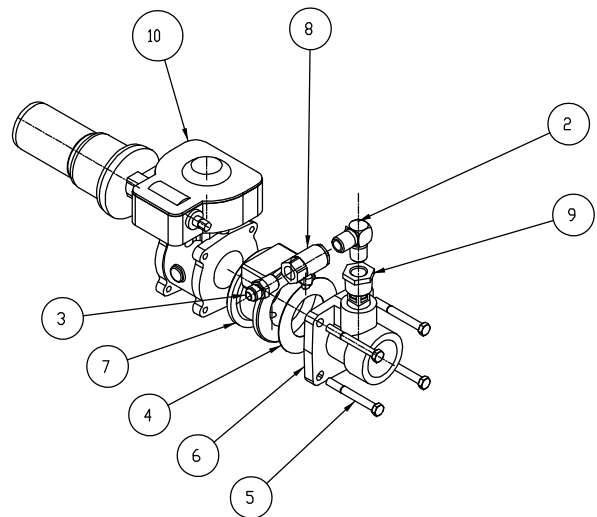
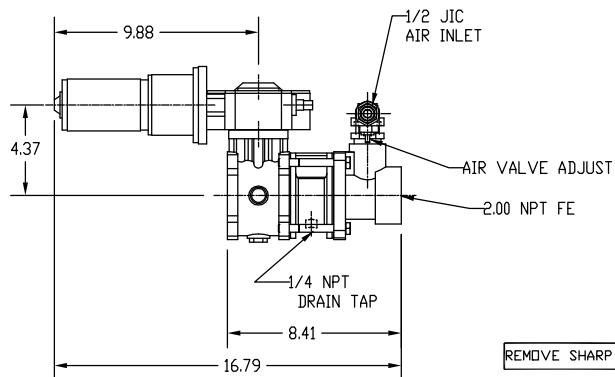
CAFS Air Distribution Valve Assemblies:

Darley AutoCAFS **AIR** (Air Injection Regulator) air distribution manifold assembly, AZ02302, is available for complete “motorized” valve control of individual air injection ports. The AutoCAFS **AIR** distribution assembly incorporates a stainless steel manifold with 6 ports for the application (1) 12VDC air distribution actuated valve for each CAFS discharge valve assembly. Optionally, assembly AZ02303 includes an air flow meter which is mounted directly to the outlet port of the compressor system oil separator tank for air flow measurement and air valve calibration. A 72” primary air hose is provided for connection of the distribution manifold to the air flow meter, providing flexible remote mounting in the pump compartment. Please refer to DCM1701 for dimensional information.





NO.	DESCRIPTION	PART NO.	QTY.
2	ELBOW - PIPE, .50 NPT DBL MALE	1080736	1
3	FITTING - .50 NPTM x .50 JICM ADAPTER	3501526	1
4	GASKET	3806306	1
5	HHCS - .375-16 x 3.25, GR5	5400045	4
6	NIPPLE - FOAM INJECTION, 2.00	1925800	1
7	VALVE - 2.00 CHECK, MODIFIED	5204601	1
8	VALVE - CAFS AIR FLOW, 0.50	5207700	1
9	VALVE - CHECK, 0.50" DURABLO	5204400	1
10	VALVE - AKRON BALL, 2.00", SUB-ASSY	5202902	1



REMOVE SHARP EDGES

INCH
(MILLIMETER)



OLD PART NO.

TOLERANCE
EXCEPT
AS NOTED

00 .003
000 .0005
ANGLES .01°

DATE 29-AUG-04

SCALE 1/4

MATERIAL DESCRIPTION:
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ALL DIMENSIONS IN
INCHES UNLESS NOTED

DO NOT SCALE PRINT

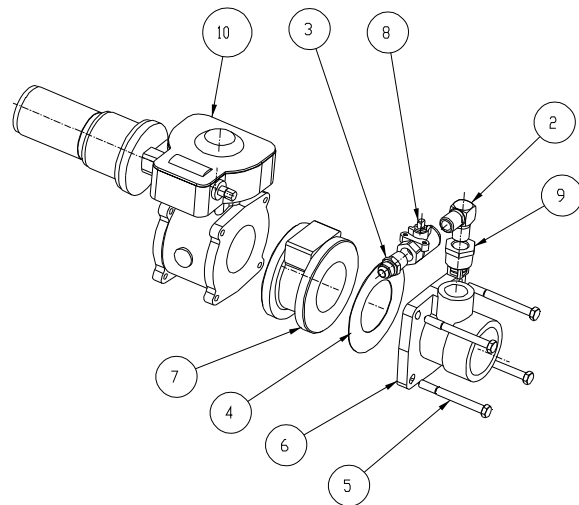
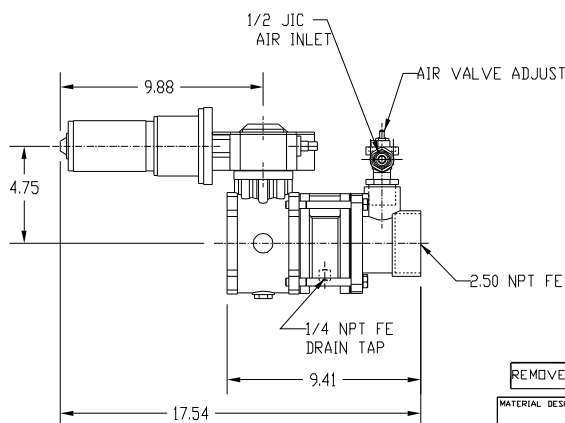
DRN DWS
CHKD JSS
TRCD

DATE 29-AUG-04

SCALE 1/4

DCM0305

NO.	DESCRIPTION	PART NO.	QTY.
2	ELBOW - PIPE, .50 NPT DBL MALE	1080736	1
3	FITTING - .50 NPTM x .50 JICM ADAPTER	3501526	1
4	GASKET	3806307	1
5	HHCS - .375-16 x 3.75, GR5	5400047	4
6	NIPPLE - FOAM INJECTION, 2.50	1925801	1
7	VALVE - 2.50 CHECK, MODIFIED	5205101	1
8	VALVE - CAFS AIR FLOW, 0.50	5207700	1
9	VALVE - CHECK, 0.50" DURABLO	5204400	1
10	VALVE - AKRON BALL, 2.50, SUB-ASSY	5203003	1



REMOVE SHARP EDGES

INCH
(MILLIMETER)



OLD PART NO.

TOLERANCE
EXCEPT
AS NOTED

00 .003
000 .0005
ANGLES .01°

DATE 29-AUG-04

SCALE 1/4

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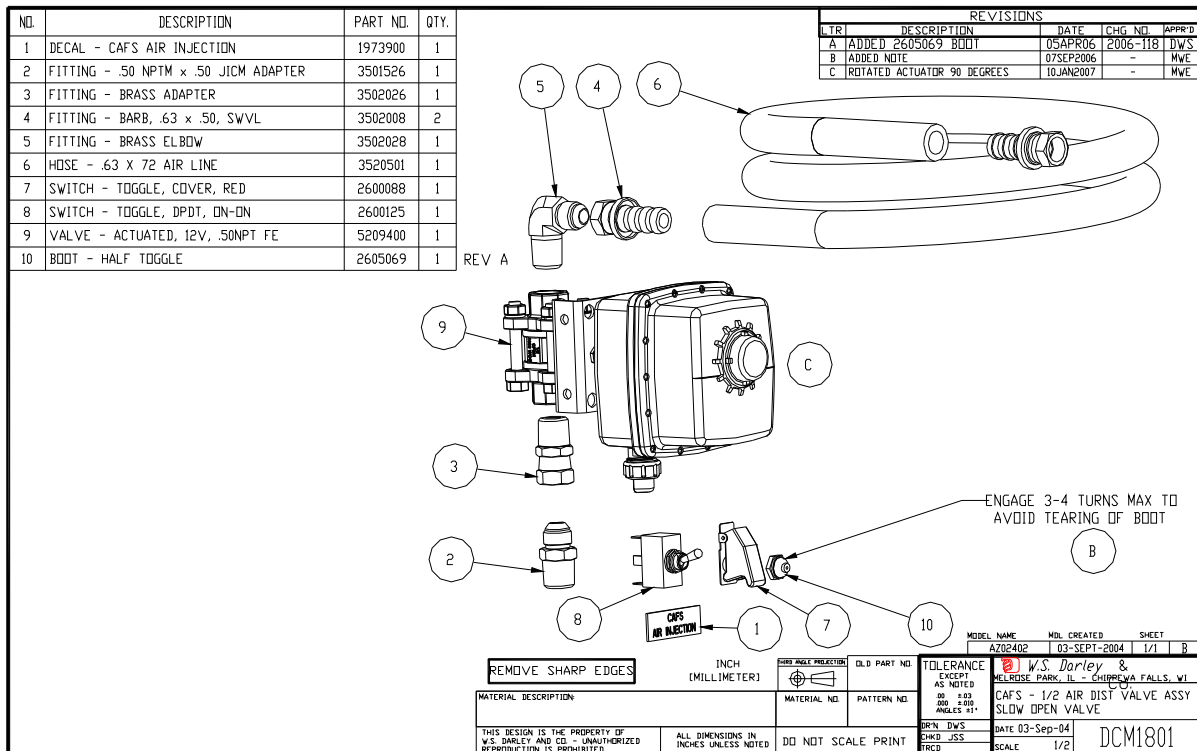
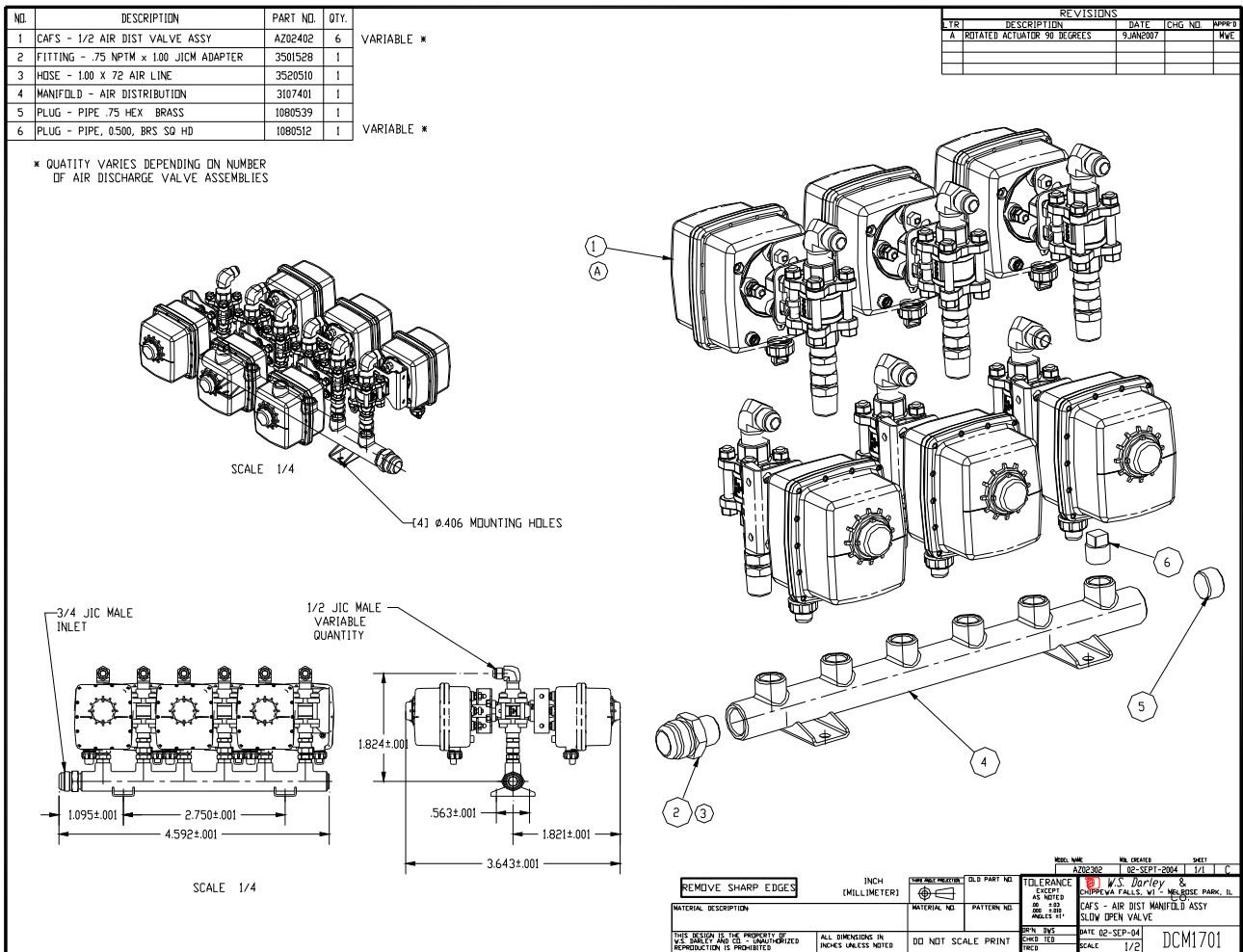
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CHKD JSS
TRCD

DATE 29-AUG-04

SCALE 1/4

DCM0306

Prepared by: DWS
Approved by: TMC
Revised by: DWS



Prepared by: DWS
Approved by: TMC
Revised by: DWS

ELECTRIC CLUTCH MAINTENANCE AND REPAIR GUIDE

CARLYLE JOHNSON MAXITORQ®

MODEL EMA ELECTRIC MULTIPLE DISC CLUTCH

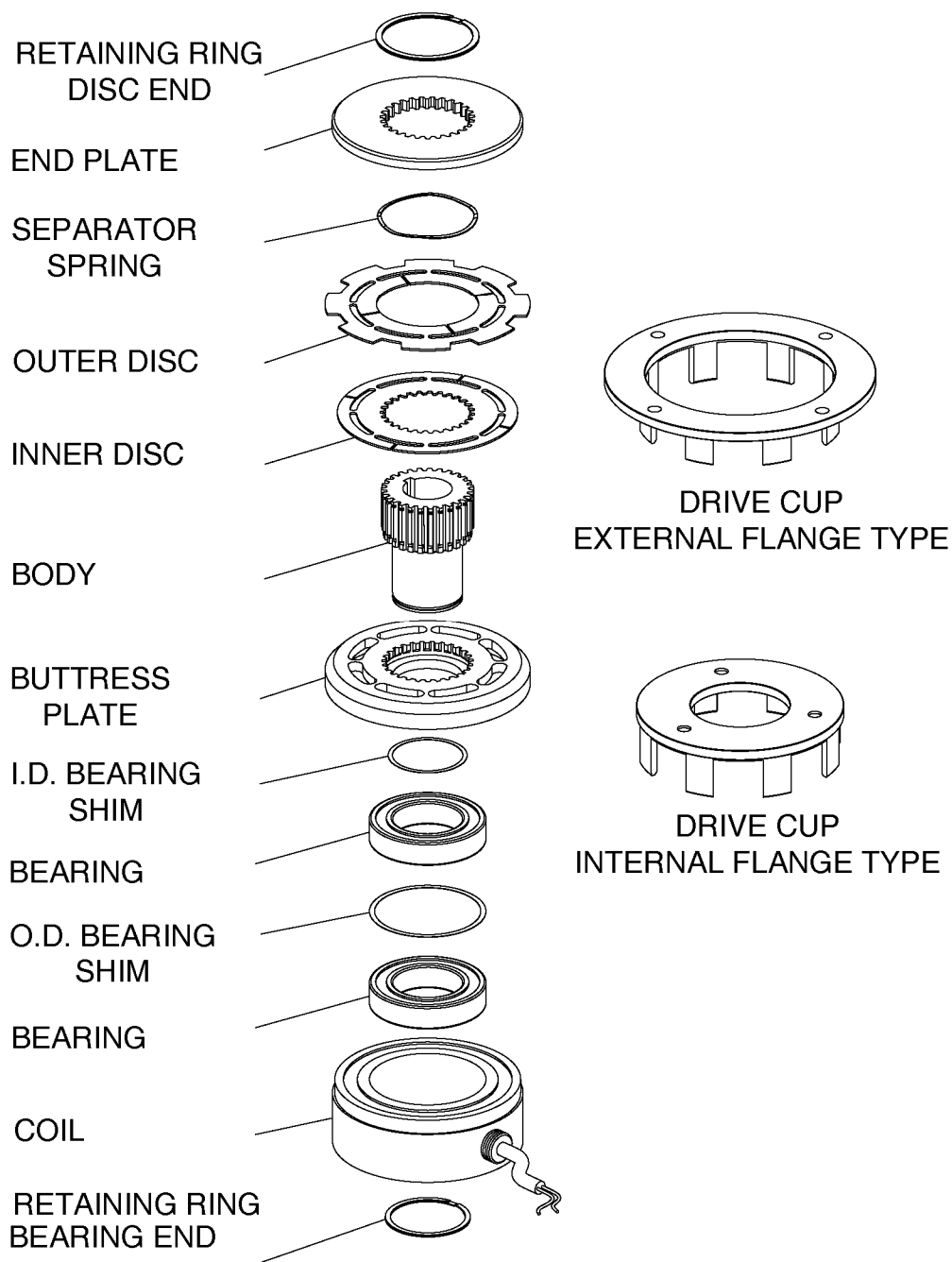
MAINTENANCE, REPAIR, TROUBLESHOOTING MANUAL



SAFETY WARNING

Always disconnect power and air and lock out / tag out machine before performing service or removing or reinstalling your clutch . On-machine measurements must be performed with power and air disconnected.
Where voltage readings are required, electrical meters must be attached with power and air disconnected.

EXPLODED PARTS DIAGRAM



ROUTINE MAINTENANCE

Preventive Maintenance

Maxitorq® multiple disc clutches need little or no maintenance in normal use.

Discs on clutches run dry may be washed in kerosene to remove any foreign material and restore clutch performance.

When a clutch is operated in oil, the oil may eventually break down along the friction surfaces. Over time, the hardened surfaces will wear. Discs should be visually inspected from time to time to make sure warping and galling have not occurred. If any such wear is observed, disc replacement is necessary.

Replacement of Clutch Discs

Always replace discs as a set. Do not mix old and new discs on a clutch.

Although springs may be reused if they are still serviceable, frequently they lose their hardness, and clutch performance - particularly disengagement and neutral drag - become a problem. We recommend purchasing a complete disc/spring set to restore like-new performance.

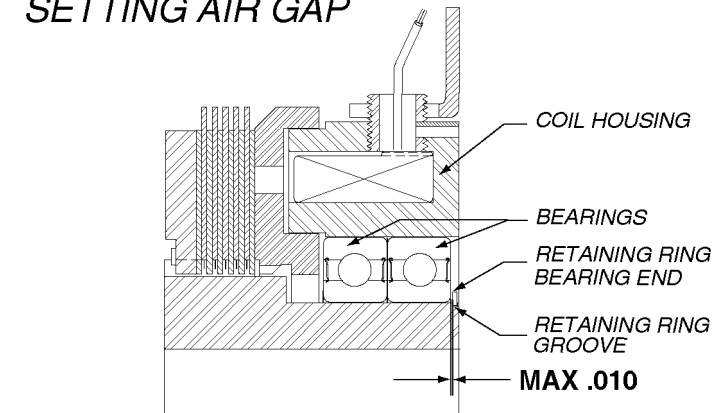
Follow Disassembly Instructions 1 through 3 ONLY. Assemble following Reassembly Instructions 10 through 13 ONLY.

Replacement of Bearings or Coil Housing Assembly

Follow Disassembly Instructions 1 through 8. Assemble following Reassembly Instructions 1 through 13.

Always replace **Ball Bearings** if replacing **Coil Housing Assembly**. The shim between the bearings may be reused if it is not damaged.

SETTING AIR GAP



GENERAL TROUBLESHOOTING ISSUES

- 1 Check for worn parts. Obviously damaged or worn parts must be replaced to insure correct clutch operation. Determine whether this wear is due to normal operation over a long period of use, or improper installation, maintenance, or clutch contamination. Replacing worn parts will provide only temporary improvement if a more fundamental problem is present and goes undetected.
- 2 Check alignment of clutch and drive cup. Review the paragraph on Alignment . Improper alignment will asymmetrically load the clutch and drive cup support bearings, causing premature wear, and possibly interfering with clutch operation.
- 3 Check for contamination of clutch discs. Discs in dry applications may become contaminated with oils from adjacent bearings or other external sources, which will prevent the clutch from transmitting full torque. Slippage is frequently caused by such contamination. Clean the discs as outlined under Contamination.
- 4 Check clearances and air gap. Check that the relationship of the drive cup to the Buttress Plate meets our specifications as outlined under Alignment . Also verify that when the clutch is disengaged, it will rotate freely with no binding or interference. Check the topic Air Gap .
- 5 Check fuses and electrical power. If the circuit is fused, check that the fuse is good. If fuse is being replaced, be sure the proper type fuse is installed in accordance with the equipment manufacturer's specifications. If no specifications are available, see the section on Fuse for recommended fuse application.

If the fuse is OK, or if no fuse is in the circuit, verify that power is reaching the clutch. To operate correctly, the clutch must receive voltage within 10% of the nominal rated voltage of the clutch coil.
- 6 Check for missing or damaged parts. If the clutch has been subjected to repair, removal, and reinstallation, check to see if the clutch has been reassembled correctly. Review the parts diagram included in this manual and replace or repair any damaged or missing parts.

SPECIFIC TROUBLESHOOTING ISSUES

Clutch Fails to Engage / No Torque Transmitted when Power Applied

Check the following items with the clutch installed:

- 1 Alignment
- 2 Fuse
- 3 Coil
- 4 Electrical Connection
- 5 Drive Cup Engagement

Remove the clutch to check the following:

- 6 Contamination
- 7 Drive Cup Wear
- 8 Disc Wear
- 9 Air Gap

Clutch “Slips”/Only Partial Torque Transmitted when Power Applied

Check the following items with the clutch installed:

- 1 Alignment
- 2 Clutch Voltage

Remove the clutch to check the following:

- 3 Contamination
- 4 Drive Cup Wear
- 5 Disc Wear
- 6 Air Gap
- 7 Springs

SPECIFIC TROUBLESHOOTING ISSUES

“Neutral Drag”/Clutch Transmits Torque when Disengaged

NOTE: A small amount of torque is transmitted in the neutral “disengaged” position. This is normal with multi-disc clutches. At very low speeds, up to 2% of the static torque may be transmitted. At high neutral speeds, this value will fall to 1% or less. If significant torque transmission is evident when the clutch is disengaged, the clutch should be repaired.

Check the following items with the clutch installed:

- 1 Alignment
- 2 Residual Magnetism

Remove the clutch to check the following:

- 3 Contamination
- 4 Drive Cup Wear
- 5 Air Gap
- 6 Springs

MAINTENANCE/REPAIR PROCEDURES

Air Gap

The permanent air gap between the stationary Coil Housing and the Buttress Plate is established by machined tolerances when manufactured. It should not be disturbed. However disassembly and reassembly of the clutch requires verification that the original clearances are restored.

The air gap should be verified before Loctite is applied to the I.D. of the Ball Bearings for permanent assembly.

Check the clearance between the Retaining Ring Groove on the bearing end of the clutch body, and the lower Ball Bearing. See the diagram at the bottom of Page 3. The maximum clearance allowed is .010". If greater, the clutch has not been reassembled correctly. Install additional I.D. Bearing Shims to correct. Up to 4 may be used. If installation of 4 shims does not reduce the clearance to < .010", clutch is assembled incorrectly or worn beyond repair. Reverify assembly. If no problem can be located and the proper clearance cannot be restored, contact the factory for assistance.

After achieving proper air gap clearance, clutch must turn freely when disengaged, with no interference.

Drive Cup Engagement

Drive cup must fully engage all outer discs. Adjust alignment of drive cup if necessary or repair/replace external components or clutch mounting to correct any deficiency.

Drive Cup Alignment

Clutch and Drive Cup must be concentric within .005 T.I.R. Misalignment may be caused by improper clutch mounting; improper mounting or support of Drive Cup; worn bearings supporting Drive Cup; improperly installed or missing anti-rotation strap; or if rigid conduit is used in providing electrical service to clutch, it may be distorting the alignment of the clutch.

Clearance between Drive Cup fingers and Buttress Plate must be approximately 1/16" around the entire circumference. This dimension must be uniform around the circumference of the Drive Cup.

Drive Cup Wear

Improper alignment, support, worn bearings, or extreme service may eventually wear "grooves" into the fingers of the Drive Cup. This will interfere with the compression and separation of discs, preventing proper engagement/disengagement of the clutch. If any such wear is evident, replace the Drive Cup, and if needed, its supporting mechanism. Any further damage to clutch discs may require disc replacement. Verify alignment after reassembly.

MAINTENANCE/REPAIR PROCEDURES

Fuse

If the circuit is fused, check the fuse condition. If the fuse is blown, replace with the same type/rating as specified by the equipment manufacturer.

If no specifications are given, use a fuse which will tolerate an inrush current approximating 135% of the nominal rating of the clutch coil.

NOTE: Always follow the manufacturer's recommendation for fuse replacement. The fuse protects upstream equipment in the machinery, not the clutch. Use the table on Page 12 ONLY if no manufacturer's instructions are given.

Coil

Check the coil resistance for open or shorted condition. Follow the table on Page 12 for nominal resistance. Coil leads must be disconnected from power source before taking resistance reading. Shorted or open coils must be replaced.

The Coil Housing Assembly includes the coil and its housing. The coil is encapsulated in epoxy resin, and must only be repaired by the factory. A complete replacement Coil Housing Assembly may be purchased, or the failed assembly may be returned to the factory for coil replacement.

Clutch Voltage

Attach a voltmeter to the clutch with the power OFF. When power is applied to the clutch, it must be $\pm 10\%$ of the nominal voltage rating of the coil. If sufficient power is not being applied to the clutch, full engagement and full torque transmission will not take place. Repair or replace power supply to assure good clutch actuation.

Disc Contamination

Disc contamination of clutches run dry may be caused by oils from external sources or other debris. Discs may be flushed with kerosene to remove oils or other contaminants, and restore normal operation.

Bearings in the vicinity of the clutch – for example used to support Drive Cup – should be adequately shielded to prevent clutch disc contamination. Clutches run in oil must not contain extreme pressure additives. We recommend ATF oils such as Dexron II for this application.

MAINTENANCE/REPAIR PROCEDURES

Disc Wear

After extended use, clutch discs will wear to the point where replacement is necessary. In a dry application, if normal operation is not restored to a slipping clutch with kerosene flushing, then disc replacement is necessary.

In oil-bathed applications, oil will eventually break down along the friction surfaces. Over time, the hardened surfaces become worn to a point where warping or galling occurs. This damage can be clearly seen by checking the disc surfaces, and requires prompt disc replacement to maintain good clutch performance.

Always replace discs and springs as a set. The factory can supply disc/spring kits.

Separator Springs

If the clutch transmits excess torque when in neutral, separator springs may be worn or bent. Springs should be replaced under these circumstances.

Proper spring performance is achieved when discs are uniformly spaced in the disengaged position.

Contact the factory to purchase replacement springs. It is a good idea to replace the discs at the same time, to restore the clutch to like-new performance.

Residual Magnetism

Occasionally, after installation of a new or rebuilt clutch, the clutch may build up residual magnetism after the first few cycles, and fail to disengage properly when power is removed. This condition can be easily overcome by reversing the power leads to the coil, energizing the clutch momentarily, then restoring the leads to their original polarity. The clutch should now fully engage when power is applied, and fully disengage when power is removed from the coil.

Electrical Connections

Check that all electrical connections are properly made. There is no polarity to the clutch leads - either one may be considered positive (+).

CLUTCH DISASSEMBLY

1. Place the clutch on a workbench with the **Coil Housing** on the bottom.
2. Remove the **Retaining Ring, Disc End** from the top of the Body.
3. Remove the **End Plate, Discs**, and **Separator Springs**.
4. Turn the clutch over, and support the assembly on the **Buttress Plate**.
5. Remove the **Retaining Ring, Bearing End** from the **Body** where it protrudes through the **Coil Housing**.
6. Press the **Body** out from the **Coil Housing**. This will allow removal of the I.D. Bearing Shim(s) and the **Buttress Plate** from the **Body**.
7. To remove the **Ball Bearings**, turn the **Coil Housing** over so that the epoxy resin side is facing DOWN.
8. Support the **Coil Housing** - DO NOT ALLOW PRESSURE TO BE APPLIED TO THE EPOXY RESIN AREA OF THE HOUSING - and press out the **Ball Bearings** and **O.D. Bearing Shim**. Because pressure must be applied to the inner race when pressing out bearings, **Ball Bearings** may not be reused once removed.

CLUTCH REASSEMBLY

1. Support the **Coil Housing** so that the epoxy resin side is facing up.
2. Apply Loctite 271 sparingly to the O.D. of the first **Ball Bearing**, and press into the **Coil Housing**. **MAKE SURE THE BEARING IS FULLY SEATED. WARNING!! DO NOT PUT THE TWO BEARINGS TOGETHER WITH A SHIM BETWEEN THEM AND ATTEMPT TO PRESS THEM IN AS A SINGLE UNIT.** The **Ball Bearings** will not seat squarely using this procedure, and the clutch will be damaged in use.
3. Insert a single **O.D. Shim** on top of the first **Ball Bearing**
4. Apply Loctite 271 sparingly to the O.D. of the second **Ball Bearing**, and press into the **Coil Housing**. **MAKE SURE THE BEARING IS FULLY SEATED.**
5. Assemble the **Buttress Plate** to the **Body**, and place the same number of **I.D. Bearing Shims** on the smooth end of the body as were removed during disassembly. The shim I.D. is the same size as the I.D. of the **Bearing**.
6. Press the **Body** into the **Coil Housing**.
7. Turn the clutch over and observe the relationship of the retaining ring groove on the **Body** to the lower **Ball Bearing**. Make sure there will be less than .010" clearance between the **Retaining Ring, Bearing End** when installed, and the **Bearing**. If the clearance exceeds .010", press the **Body** out and install additional **I.D. Bearing Shims** to reduce the clearance to less than .010". Up to four (4) shims may be installed if necessary (If more than four shims are required, the clutch may be improperly assembled. Check the assembly procedures carefully, and if necessary, contact the factory for assistance).
8. When the proper clearance has been achieved, press out the **Body**, and apply Loctite 271 sparingly to the I.D. of the **Ball Bearings**. Press the **Body** back into the **Coil Housing** for permanent assembly.
9. Install the **Retaining Ring, Bearing End** on the **Body**.
10. Install an **Inner Disc**. **Inner Discs** differ from **Outer Discs** in that they have smaller O.D. and have a toothed I.D. to fit over the spline on the **Body**.
11. Install an **Outer Disc**, with a **Separator Spring** in the center.
12. Continue installing the discs, alternating between **Inner Discs** and **Outer Discs** until all discs and springs have been installed. Most (but not all) standard clutches have five **Outer Discs**.

Install the **End Plate**, and the **Retaining Ring, Disc End**. Clutch assembly is now complete. The clutch body should turn freely without any binding or interference.

CLUTCH ELECTRICAL CHARACTERISTICS

Clutch Model	EMA 0475 – 12VDC (Darley)
Coil Power (watts)	58
Current Draw, (amps)	4.8
Fuse Size (amps)	10
Coil Resistance (ohms)	10

Clutch Model	24v DC Coil				100v DC Coil			
	Coil	Current	Fuse	Coil	Coil	Current	Fuse	Coil
	Power (watts)	Draw (amps)	Size (amps)	Resistance (ohms)	Power (watts)	Draw (amps)	Size (amps)	Resistance (ohms)
EMA 0265	48	2.0	2 1/2	12	33	0.3	1	299
EMA 0325	48	2.0	2 1/2	12	49	0.5	1	204
EMA 0375	40	1.7	2 1/2	15	41	0.4	1	244
EMA 0425	41	1.7	2 1/2	14	48	0.5	1	208
EMA 0475	58	2.4	3	10	62	0.6	1	161
EMA 0625	58	2.4	3	10	64	0.6	1	157
EMA 0800	68	2.8	4	9	78	0.8	1 1/4	128
EMA 0950	96	4.0	5	6	97	1.0	1 1/2	103
EMA 1150	89	3.7	5	7	115	1.1	1 1/2	87

NOTE: Always follow equipment manufacturer's recommendation on fuse type/size.
 Use the above chart only if fuse size is not specified.
 Use a fuse which will tolerate an inrush current of 135% of nominal rating.



Carlyle Johnson Machine Co. L.L.C.

291 Boston Turnpike Bolton, CT

860-643-1531 Toll Free 888-MAXITORQ

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 Approved by: TMC
 Revised by: DWS

101

Rev. #: D
 Date: 7/12/05
 Rev Date: 1/10/07
 1200638.doc

AutoCAFS II - TESTING REFERENCE GUIDE

General: This reference guide is based on the Darley Model LDMBC midship, Enduro 12 TS, AutoCAFS II package with an electric compressor clutch. This compressor system has a 220 CFM rating at 125 PSI and provides for automatic air pressure balancing.

INSTRUCTIONS:

1. PREPARE COMPRESSOR FOR RUNNING TEST:

- 1.1. The test technician should have available for reference and must be familiar with the LDMBC 'OPERATION INSTRUCTION' manual, part number 1200597.
- 1.2. The first step, before testing the system, is to make sure that all equipment has been properly installed. All air compressor related components should be clean and free of obstructions before installation. Follow installation instructions and component layout diagrams and verify correct installation.
- 1.3. Once this has been established be sure to have the correct oil type and proper oil level in both the compressor system and the water pump gear case.
 - 1.3.1. Air Compressor - 32 Weight (Hydraulic air compressor oil) Ex: Phillips Magnus oil ISO VG 32 RX mineral oil (refer to Operation Instructions for further specifications) Fill the oil tank reservoir to the high level mark on the sight tube. Depending on total length of hoses, more oil may need to be added after unit begins running and oil circulates throughout the system. **WARNING:** Oil separator tank is pressurized after compressor shutdown until system blow down is complete. Allow 2 minutes for system pressure blow down before opening oil fill valve. **Caution:** Do not overfill the oil reservoir. If the oil level is above the maximum level, the separator elements may not be able to handle the flow of air/oil mixture being supplied. The result may be an oily air discharge mist when the airflow valve is opened.
- 1.4. The water pump transmission requires SAE 80W - 90 GL4/GL5 gear lube oil filled to the proper level on the dipstick.

2. Additional Testing Equipment

- 2.1. A calibrated 250 CFM air flow meter with valve must be installed on the air compressor reservoir outlet. A calibrated, 300-PSI gauge should be installed on the air flow meter.
- 3. Prime and prepare pump for water discharge. Begin rotating the pump with engine at idle speed. RPM should be as low as possible when air compressor is engaged. Turn on the air compressor by pressing and holding the "On/Off" button on the AutoCAFS Commander for two seconds. The light next to the "On/Off" button will be illuminated. The compressor will now begin to build air pressure. Air pressure is controlled by pump pressure.**
- 3.1. Observe the following precautions:
 - 3.1.1. RPM should be as low as possible when air compressor is engaged.
Warning: Never engage the air compressor at over 1000 RPM.
 - 3.1.2. Allow 1 minute between compressor stop and start for system blow down.

3.1.3. Once compressor is engaged, do not exceed 3650/pump ratio input rpm.
Example: $3650/2.44 = 1500$ rpm for 2.44 pump ratio.

3.1.4. During all tests it is important to keep cool fresh water circulating through the water pump. Water flow from the pump is used to cool the air compressor oil using a brazed plate type heat exchanger.

3.2. Check to see that there are no oil or air leaks in the air compressor system. Do not proceed until all leaks are repaired. Check oil separator tank oil level. Shutdown compressor and adjust oil level as required.

3.3. Begin by raising the water pump pressure slowly to 75 PSI. The water pump should have a discharge valve gated slightly open to circulate water. Once again, check to see that all fittings are tight and no leaks are found. A leak can cause a malfunction in air pressure adjustment and control. Commonly leaks will cause air pressure to rise higher than attempted settings.

There are seven steps to perform to test the air compressor system.

- Ensure that all AutoCAFS Commander settings are correct
- Pump Ratio 2.44 standard, 2.67 optional
- Pressure Limiting Valve Adjustment Setting
- Control Pressure Sensitivity adjustment
- Maximum Air flow test
- Temperature test
- Blow-down test
- Speed Calibration test

4. Pressure Limiting Valve Adjustment Setting – Reference drawing DCM1002

4.1. This test is designed to set the maximum air pressure limiting control to 150 PSI. The water pump pressure determines how much pressure the air compressor will produce.

4.2. The water pump pressure must be increased to at least 160 PSI.

4.3. Open the airflow valve slightly. Flow approximately 40-50 CFM.

4.4. The pressure-limiting valve, normally bracket mounted to the pump discharge head, has a threaded adjustment screw with a locking nut to hold the setting in place. First loosen the lock nut. The threaded bolt must be turned in clockwise to raise the governed pressure and counter-clockwise to lower pressure. Adjust the air pressure to a governed maximum factory setting of 150 PSI. Tighten the lock nut.

4.5. Close the airflow valve to verify that the compressor stays at 150 PSI. Re-open the valve to flow over 100 CFM then close it again. Verify that the air compressor stays at 150 PSI.

5. Control Pressure Sensitivity adjustment – Reference drawing DCM1002

- 5.1.** This test is designed to set the sensitivity of the air pressure balancing system.
- 5.2.** It is important to properly adjust the needle valve which is bracket mounted to the pump discharge head. The needle valve's function is to dampen the control pressure bleed off line, reducing modulator sensitivity. As a result, the inlet valve will respond slower to pressure change thus reducing modulator pulsation. If it is set too far in or closed (clockwise rotation) no pressure modulation will take place. If it is open too far (counter-clockwise rotation) pressure fluctuations will go unnoticed and pressure spikes are then unavoidable. For example: When a CAFS discharge is closed at the nozzle, pressure may build until the pressure relief valve releases at 200 PSI. Should the needle valve need adjustment, use the following as a guide. Start by closing the valve (4) completely. Then open it approximately 3 turns. Operate the unit at around 125 PSI; begin by flowing about one third the capacity of the air compressor. At this flow rate, the air inlet modulator valve will open to bring in air and then close as air pressure builds. The goal is to set the needle valve at a position where pressure fluctuations are minimized. If the air pressure gauge is fluctuating more than 20 PSI above or below the water pressure, then the needle valve should be adjusted out or counter-clockwise. As the pressures come closer to balancing, less flow meter fluctuation should also be noticed. Note: Some pressure modulation is normal and required for the system to auto-balance while delivering CAFS. Expect pressure variation to range from 5-20 PSI. Pressure fluctuations should be pulsing at the beat of at least one per second but no more than 20 in a ten second period.

6. Air Flow Test

The Air Flow test is performed to verify that the system can flow at least 220 CFM of air at 125 PSI.

- 6.1.** Note: the water pump will commonly need be flowing at least 500 – 1000 GPM to keep both the air compressor and the water pump both operating at 125 PSI.
- 6.2.** To test the airflow capability of the unit start by running the pump at approximately 140 PSI. This will be approximately 3200/pump ratio rpm, (1300 RPM for 2.44:1 ratio). The desired goal is to try to find the lowest RPM required flowing 220 CFM and at least 450 GPM all while both the air compressor and the water pump are at 125 PSI. Commonly water flow is required to be higher than 450 GPM due to the necessary reduction of the pressure of the pump. Open at least two of the pumps 2-1/2" discharges until the water pump pressure is 125 PSI. Begin to flow air by opening airflow valve until the air pressure begins to drop below 125 PSI. Slowly close air discharge valve raising air pressure back to 125. This will be the maximum airflow of the unit at this RPM. If the airflow is not at least 220 CFM, higher rpm may be needed to attain this rating.
- 6.3.** Record air and water flow and pressure along with input rpm and power requirement on the pump test sheet.

7. Operating Temperature test

The operating temperature test can be performed during the maximum airflow test.

- 7.1. To test the operating temperature of this system you will need to operate the air compressor at over 150 CFM at 70-75 degrees F ambient temperature for at least 10 minutes, to check that the thermostatic valve in the oil filter assembly is working properly. The lubricating/cooling oil in the oil reservoir, if 170 degrees F or less, travels through the oil filter before going to the air compressor. If the oil temperature is higher than 170 degrees F a thermostatic valve in the oil filter mounting block will redirect the flow of oil through the heat exchanger before entering the opposite side of oil filter housing and then onto the air compressor.
- 7.2. The oil temperature should not exceed 170 degrees F by more than 20 degrees or it may indicate a limited flow, or high temperature water flow through the heat exchanger. There is a high temperature overheat warning message for both the air compressor and the clutch on the AutoCAFS Commander.
- 7.3. Observe and Record compressor temperature on test sheet.
- 7.4. Using infrared temperature sensor, measure clutch temperature. Normal temperature range is 110-125°F. The upper limit is 135°F.
- 7.5. Using infrared sensor measure outboard pump bearing cap. Normal temperature range is 180-200°F. The upper limit is 210°F.
- 7.6. Observe fan to confirm airflow direction is from rear screen opening forward over clutch and belt.
- 7.7. If compressor temperature rises above normal operating temperature to 212°F, the Compressor COMP HOT warning will flash on the Commander display. If temperature warning is indicated, shut down the compressor as soon as practical by depressing the "On/Off" button for 2 seconds. The compressor can be switched off (DISENGAGED) at any time or input speed. Check for adequate water flow through heat exchanger. Check for adequate oil level in separator tank. See trouble-shooting guide for further options.



WARNING: *If compressor temperature continues to rise to 240°F, the compressor will be automatically disengaged.*

8. System Blow-Down

- 8.1. After compressor shutdown, system pressure is bled off to guard against overloading drive components at startup. If the receiver assembly is not depressurized on shut down, oil will flood the compressor filling the area above the screws. Oil trapped above the screws will then cause a hydraulic lockup when compressor rotation rapidly accelerates during startup. A hydraulic lockup of this type can induce extreme loads on the power train. Refer to Instruction manual for detailed explanation of blow down process.
- 8.2. After compressor clutch disengagement, observe test gage mounted on the separator tank below the safety relief valve. Record time it takes for system pressure to bleed down to 0 psi. System should blow down in 1 minute or less.

- 8.3. Confirm that the clutch exhibits minimal drag when disengaged. With the pump idling and the clutch disengaged, observe that the compressor belt is not moving.

9. Speed Protection

- 9.1. Speed control is included in the compressor engagement circuit. The AutoCAFS Commander has been calibrated by the apparatus manufacturer to allow compressor engagement only at engine speeds below 900 rpm and allows for a maximum compressor operating speed. RPM signal for the speed control needs to be wired to the data bus. Refer to Section 3 of this manual for further details on the AutoCAFS Commander control module.
- 9.2. To verify complete system; switch off compressor ENGAGED switch, raise engine RPM above 900 RPM, switch compressor on; the clutch connector should not be energized. Reduce engine throttle to below 900 RPM; clutch connector should be energized. Raise RPM to above high limit and power to connector should be de-energize. Reduce RPM and power should remain off until RPM is below the low limit set point and blow down is achieved. Adjust if required using the AutoCAFS Commander programming sheet as a guide.
- 9.3. Shut down pump and engine.



CAUTION: *Avoid immediate restart of compressor after shutdown. Allow a 1-minute minimum time period between compressor shutdown and restart for system blow-down.*

10. If the unit being tested has performed as stated and conforms to the test requirements then the system is ready for delivery.
11. After shutdown, thoroughly drain water from compressor heat exchanger and feed lines.
12. Visually inspect belt for adjustment and tracking. Belt adjustment can be checked by pushing a 1/8-diameter rod through the cover perforations on the middle of the belt span with. As a guide, a 22-pound force in the middle of the belt span should deflect the belt approximately 3/16 inch.
13. Confirm that all control tubes are bundled and wire tied in a neat and orderly fashion.

Detailed Specifications

When preparing specifications for a new Compressed Air Foam equipped apparatus, use the following technical specifications to assure that your apparatus will be equipped with the most advanced CAFS system available; Darley AutoCAFS II.

SINGLE STAGE FIRE PUMP - CAFS COMPATIBLE

The pump shall be a Darley LDMBC single stage fire pump, capable of a water flow rating from 1000 to 1750 GPM.

Power to drive the pump shall be provided by the same engine used to propel the apparatus. The pump shall be midship mounted and designed to operate through an integral transmission, including a means for power selectivity to the driving axle or to the fire pump.

The pump casing shall be a fine grain cast iron alloy, vertically split, with a minimum 30,000 PSI tensile strength and bronze fitted.

The pump shall contain a cored heating jacket feature that, if selected, can be connected into the vehicle antifreeze system to protect the pump from freezing in cold climates.

The impeller shall be a high strength bronze alloy of mixed flow design, accurately balanced and splined to the pump shaft for precision fit and durability. The impeller shall feature a double suction inlet design with opposed volute cutwaters to minimize radial thrust.

The seal rings shall be renewable, double labyrinth, wrap around bronze type.

The pump shaft shall be precision ground stainless steel with long wearing chromium oxide hard coating under the packing glands. The shaft shall be splined to receive broached impeller hubs, for greater resistance to wear, torsional vibration, and torque imposed by engine.

A stuffing box shall be provided and shall be of the plunger injection style, utilizing a plastallic composite packing equalizing pressure around the shaft. Packing shall be renewed by removing the plunger, inserting the packing, and reinstalling the plunger. This packing design shall be provided to minimize friction, heat generation, and apparatus down time. This feature is designed to allow replacement and/or adjustment of packing within a 15-minute time period.

Due to the advantages of the above packing feature, rope or braid type packing gland designs are not acceptable.

The bearings provided shall be heavy duty, deep groove, radial type ball bearings. They shall be oversized for extended life. The bearings shall be protected at all openings from road dirt and water splash with oil seals and water slingers.

The transmission case shall be heavy-duty cast iron alloy with adequate oil reserve capacity for low operating temperatures. The transmission case shall contain a magnetic drain plug for draining the gearcase oil and a dipstick for checking and filling the level of the gear case through its opening. The transmission shall also allow the use of an external heat exchanger for increased cooling under extreme conditions.

The pump driveshaft shall be precision ground, heat-treated alloy steel, with a minimum 2-1/2" x 10" splined ends. Gears shall be helical design, and shall be precision cut for quiet operation and extended life. The gears shall be cut from high strength alloy steel, carburized, heat-treated and ground. The gear face shall be 2 3/8" minimum.

The gearshift shall be a heat-treated alloy steel splined spur gear to engage either the pump drive gear or the truck drive shaft gear. The pump and apparatus manufacturer's Engineering Department shall select the gear ratio of the pump.

Due to the advantages of the above gear and drive feature, chain drive and designs requiring additional lubrication are not acceptable.

A discharge manifold, as supplied as part of the pump by the pump manufacturer, shall include a discharge check valve assembly to allow priming of the pump from draft with discharges open and caps off. No exception.

Due to the importance of the above discharge manifold and check valve assembly, intended to be included with the overall pump design, there shall be no exception allowed to this requirement.

Discharge outlets shall have extensions with companion flange openings to allow ease of service. Two ports shall be provided on a pump panel for testing of vacuum and pressure readings. A weather resistant Performance Data Plate shall be installed on a pump panel.

The pump priming system, heat exchanger system, discharge and suction valves, relief valves, pump shift, and master drain shall be as detailed elsewhere in these specifications.

Two (2) manuals covering the fire pump, pump transmission and selected options of the fire pump shall be provided with the apparatus.

CAFS COMPATIBLE

The pump transmission shall be designed to accommodate an integrated, air compressor mounting bracket. This bracket shall be installed to properly align a rotary screw air compressor with an external sprocket driven by the pump transmission.

The air compressor shall be driven using a Gates “Poly Chain GT” belt drive system.

The air compressor drive sprocket shall be supplied with an electric multi plate, industrial clutch providing engagement at idle and disengagement at any rpm. The AutoCAFS shall be supplied with the “Commander” control and instrumentation system. The AutoCAFS Commander display shall include a digital air pressure, oil temperature, and RPM display. A mode button can be used to display air flow (if so equipped), and total air compressor system hours. The system shall incorporate an automatic, high CAFS oil temperature shut down to avoid damage to the rotary screw air compressor. The system also provides electronic protection to prohibit air compressor engagement if engine rpm is higher than recommended and also features blow down protection.



AutoCAFS - COMPRESSED AIR FOAM SYSTEM

Description: This fire apparatus shall be equipped with a high-energy, automatic compressed air foam system (AutoCAFS).

Ratings: The fire pump and air compressor shall be sized to provide at least 220 CFM (cubic feet per minute) of compressed air while simultaneously flowing at least 440 GPM (gallons per minute) of water flow. The pressure of the system shall be set at 125 PSI for the duration of this test as outlined in the NFPA document 1906. This rating is consistent with the NFPA recommendation that the water pump shall discharge two gallons of water for every one CFM of compressed air discharge. Fire pumps with UL ratings in excess of 1000 GPM commonly flow near capacity while simultaneously operating the air compressor at full output.

Components: The air compressor shall be a high quality, industrial rated, modulating, continuous duty, and of rotary screw design. The air compressor shall be mechanically driven by the main pump and shall be so designed as to provide optimum performance at 70% of rated engine RPM. Air compressor drive train shall provide a means to engage and disengage the air compressor as required.

The air compressor system shall include a pressurized oil lubrication system, oil reservoir with receiver/separators element, oil filter, inlet air filter, and modulating air inlet control. The air compressor shall be provided with a pressure control system to automatically balance air pressure to water pressure. The air compressor inlet valve shall open and close to provide the airflow desired while maintaining the air system pressure to water pump pressure to within 5 PSI differential. This balancing system is essential for safe operation of a compressed air foam system.

The air compressor lubrication system shall require cooling water to be supplied from the fire pump through a brazed plate type heat exchanger to cool the air compressor oil. The water flow to this oil cooler shall be supplied using a flushed strainer system to ensure a consistent flow of cooling water. The oil temperature shall be thermostatically controlled to remain at a consistent operating temperature within a range from 170° F to 180° F.

Panel Mounted Controls: The air compressor system shall have mounted on the operators control panel an AutoCAFS Commander control that displays all pertinent CAFS information including Air compressor pressure, CAFS oil temperature and system RPM. The Commander also provides the necessary temperature warning and shut down, RPM protection, and blow-down related safety systems.

Gauges and controls shall be positioned and clearly marked so as to provide simple and easy operation.

Each of the components of this Automatic Compressed Air Foam System - (air compressor, drive system, foam proportioner, control and instrumentation system) shall be sized, driven, and installed to produce a well operating and reliable CAFS unit.

The compressed air foam system (AutoCAFS) shall be completely installed and tested by the fire pump manufacturing facility before delivery. No exceptions.