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### IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650

# This manual is for DARLEY FIRE PUMP:

Model: HD100 Pump Serial Number:

# **Description of Pump Type**

The **Type HD100** pump is a high speed, single stage, centrifugal pump. The pump impeller shaft is driven via a helical tooth, two gear speed increasing gear box.

The pump assembly is direct mounted to a standard SAE 'B' PTO adapter flange and is driven by the 13T, involute spline, PTO output shaft. The PTO clutch system is utilized to engage and disengage the pump.

# **OPERATION AND MAINTENANCE OF TYPE HD100 FIRE PUMP**

### Lubrication

Keep the gear case filled with oil to the level of the oil level plug, which is marked on the pump gear case.

Check the oil level every 25 hours or every three months, and change it every 50 hours or 6 months.

Use 80W/90 gear lube oil - not grease - in the pump transmission.

### **CAUTION:** Do not overfill.

# **Operation of Pump**

The pump is directly coupled to the PTO through a step up gear box. Prepare the pump for operation before engaging the pump drive PTO. Prime the pump system, and then engage the pump.

### Never run the pump dry except momentarily and at low speeds.

Never engage the pump at high RPM.

Never run the pump at high speed at any time unless it is discharging water.

# WARNING: Do not use this pump for hose testing.



230         230           SF         172           SAE         'B'           SAE         'B'           SAE         'B'           172         172           SAE         'B'           SAE         'B'           SAE         'B'           ON INTIM, 9KC BR         520           IO, 12 DP         271           I.00, 12 S         360           I.125, 0.05         360           I.55, 0.09         360           I.56 X         0.09      <	500601       500601       721702       721702       721702       720103       720103       720103       720103       720103       720101       520120       520120       500000       6000001       6000001       6000001       6000001       6000001       6000001       6000001       6000001       6000001       6000001       6000001       6000001       6000001       6000001       6000001       6000000       6000000       601000       601120       601124       601124       601124       601124	
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DESCRIPTION	BEARING CAP	BEARING CAP	DEAKING-DALL	BEARING-BALL, 207SF REARING-RALL 207SF	CAP - BEARING	COMP FLANGE - 1350/1410, 1.38"	FHMS250-20 x 1.00, BR	FLANGE - 2 BSP, O-RING, CDI	GEAR - DRIVE, 12DP, 91, LH	GEAR - PINION, 19T, RH	HEAD - INBOARD, H100	HHCS313-18 x 0.75, GR5	HHCS313-18 x 1.00, GR5	HHCS313-18 x 1.75, GR5	HHCS375-16 x 1.00, GR5	HOUSING - MECHANICAL SEAL, H100	MPELLER - H100, MECH SEAL	KEY - SQ., U.12 X 1.5U GK2	MECHANICAL SEAL - 1.00" KSPAH	NUT - CASTLE, .375-24, 303	NUT - FLANGE, 7/8-14	NUT - HEX, .375-16, GR2	O-RING - 1.00 x 1.12 x 0.06	O-RING - 2.88 x 3.06 X 0.09	O-RING - 3.00 x 3.12 x 0.06	O-RING - 3.00 x 3.19 X 0.09	O-RING - 3.12 x 3.38 x 0.12	O-RING - 3.50 × 3.69 × 0.09	O-RING - 6 00 x 6 25 x 0 12	OIL SEAL - 1.125 ID X 1.750 OD	OIL SEAL - 1.625 ID X 2.000 OD	PIN - COTTER, .094 x 1.00, SST	PLUG - PIPE 1/4 BSP	PLUG - PIPE, 0.125, ZN SOC HD	PLUG - PIPE, 0.375, MAG SQ HD	PLUG - PIPE, 0.375, ZN SOC HD	PLUG - PIPE. 1/8 BSP	PLUG - TACH SHAFT DI LIG - TACH 760-20 ALLIM	PUMP CASING H100	SEAL RING - HH100	SEAL RING - KSPAH	SHAFT - DRIVE, JMP LOCK ON	SHAFT - IMPELLER, H100	SLEEVE - KSPAH, EXTERNAL	SLEEVE - KSPAH, INTERNAL	SPACER - 1.00 X 1.38 X 1.15	SPACER - 1.38 x 1.69 x 0.44 SPACED - 1.63 × 1.27 × 75	SPACER - 1.03 X 1.37 X .73 STHD - 0 375-16 X 1 500 GB5	TRANS - GEARCASE ASSY, J	TUBE FITTING - STR, .31 x .25	VENT - GEARCASE	WASHER - 0.39 X 0.94 X .12 303
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~	BEARING CAP	2300601	~
7	BEARING CAP	2301008	~
e	BEARING-BALL	1721702	~
4	BEARING-BALL, 207SF	1720103	2
5	BEARING-BALL, 307SF	1721903	~
9	BRACKET - HYD MOTOR, JMP/JMH	4013101	~
2	ENCODER - TACHOMETER	4220200	~
ω	HHCS500-13 x 1.50, GR5	5400067	4
0	FHMS250-20 x 1.00, BR	5402405	4
0	FLANGE - 2 BSP, O-RING, CDI	1920414	~
	GEAR - DRIVE, H100, 12 DP	2713203	~
12	GEAR - PINION, 19T, RH	2713404	~
13	HEAD - INBOARD, H100	2805603	~
4	HHCS313-18 x 0.75, GR5	5400018	9
15	HHCS313-18 x 1.00, GR5	5400020	4
9	KEY - SQ., 0.19 x 1.50, GR2	3602428	~
2	HHCS375-16 × 1.00, GR5	5400036	4
<u>8</u> 0	HOUSING - MECHANICAL SEAL, H100	1846101 2008004	~ ~
2 0	KEY - SQ 0.12 X 1.50 GR2	3602423	
2	O-RING - 4.25 x 4.50 X 0.12	3601239	·   <del>.</del>
22	MECHANICAL SEAL - 1.00" KSPAH	1846100	-
23	NUT - CASTLE, .375-24, 303	5403431	-
4	OIL SEAL - 1.875 ID X 2.627 OD	3600530	~
25	NUT - HEX, .375-16, GR2	5403002	16
50	O-RING - 1.00 x 1.12 x 0.06	3601016	7
27	O-RING - 2.88 x 3.06 X 0.09	3601126	~
8	O-RING - 3.00 x 3.12 x 0.06	3601011	~
00	O-RING - 3.00 x 3.19 X 0.09	3601125	
2 2	0-KING - 3. IZ X 3.30 X 0. IZ 0-DING - 3. 50 × 3. 60 Y 0.00	36011200	
- 2	U-KING - 3.30 X 3.09 X U.U9	3001124	
2 22	KING - KE I AINEK, 5160-137 O-RING - 6 00 x 6 25 x 0 12	3600421 3601226	
8 2	OIL SEAL - 1.125 ID X 1.750 OD	3600559	
35	SHAFT - DRIVE, HH100	5005301	-   -
36	PIN - COTTER, .094 x 1.00, SST	3605202_B	~
37	PLUG - PIPE 1/4 BSP	1080554	5
88	PLUG - PIPE, 0.125, ZN SOC HD	1080537	4
33	PLUG - PIPE, 0.375, MAG SQ HD	1080536	~ .
	PLUG - PIPE, U.3/5, ZN SUC HD DI LC DIDE 1/8 DSD	1080541	4 c
+ 6	TRANS - GFARCASE ASSY .I		ч <del>.</del>
; 4	PUMP CASING H100	2054908	
15	SEAL RING - HH100	3406001	-
46	SEAL RING - KSPAH	3407401	~
48	SHAFT - IMPELLER, H100	5016906	-
6†	SLEEVE - KSPAH, EXTERNAL	1886801	~
00	SLEEVE - KSPAH, INTERNAL	1886800	~
20	SPACER - 1.00 X 1.38 X 1.15	3306200	~
22	SPACER - 1.38 x 1.69 x 0.44	3301704	-
4	STUD - 0.375-16 X 1.500, GR5	3606203	16
90	TUBE FITTING - STR, .31 x .25	3500010	~
22	VENT - GEARCASE	4401800	~ .
β	WASHER - 0.39 X 0.94 X .12 303	3603303	~





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PART NO.	2300601	2301008	1721702	1720103	1721903	4013101	4220200	5400067	5402405	1920414	2713203	2713404	2805603	5400018	5400020	3602428	5400036	1846101	2908004	3602423	3601239	1846100	5403431	3600530	5403002	3601016	3601126	3601011	3601125	3601200	3601124	3600421	3601226	3600559	5005301	3605202_B	1080554	1080537	1080536	1080541	TG00300	2054908	3406001	3407401	5016906	1886801	1886800	3306200	3301704	3606203	3500010	4401800	3603303		3	
DESCRIPTION	BEARING CAP	BEARING CAP	BEARING-BALL	BEARING-BALL, 207SF	BEARING-BALL, 307SF	BRACKET - HYD MOTOR, JMP/JMH	ENCODER - TACHOMETER	HHCS500-13 x 1.50, GR5	FHMS250-20 x 1.00, BR	FLANGE - 2 BSP, O-RING, CDI	GEAR - DRIVE, H100, 12 DP	GEAR - PINION, 19T, RH	HEAD - INBOARD, H100	HHCS313-18 x 0.75, GR5	HHCS313-18 x 1.00, GR5	KEY - SQ., 0.19 x 1.50, GR2	HHCS375-16 x 1.00, GR5	HOUSING - MECHANICAL SEAL, H100	IMPELLER - H100, MECH SEAL	KEY - SQ., 0.12 X 1.50 GR2	O-RING - 4.25 x 4.50 X 0.12	MECHANICAL SEAL - 1.00" KSPAH	NUT - CASTLE, .375-24, 303	OIL SEAL - 1.875 ID X 2.627 OD	NUT - HEX, .375-16, GR2	O-RING - 1.00 x 1.12 x 0.06	O-RING - 2.88 x 3.06 X 0.09	O-RING - 3.00 x 3.12 x 0.06	O-RING - 3.00 x 3.19 X 0.09	O-RING - 3.12 x 3.38 x 0.12	O-RING - 3.50 x 3.69 X 0.09	RING - RETAINER, 5160-137	O-RING - 6.00 x 6.25 x 0.12	OIL SEAL - 1.125 ID X 1.750 OD	SHAFT - DRIVE, HH100	PIN - COTTER, .094 x 1.00, SST	PLUG - PIPE 1/4 BSP	PLUG - PIPE, 0.125, ZN SOC HU	PLUG - PIPE, 0.375, MAG SQ HD	PLUG - PIPE, 0.3/5, ZN SOC HU	TRANS - GEARCASE ASSY. J	PUMP CASING H100	SEAL RING - HH100	SEAL RING - KSPAH	SHAFT - IMPELLER, H100	SLEEVE - KSPAH, EXTERNAL	SLEEVE - KSPAH, INTERNAL	SPACER - 1.00 X 1.38 X 1.15	SPACER - 1.38 x 1.69 x 0.44	STUD - 0.375-16 X 1.500, GR5	TUBE FITTING - STR, .31 × .25	VENT - GEARCASE	WASHER - 0.39 X 0.94 X .12 303	8		
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NOTE: 1) O-RINGS (26) ARE SUPPLIED WITH MECHANICAL SEAL

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# **Mechanical Shaft Seal**

This pump assembly incorporates high quality mechanical shaft seal(s) separating the pump housing components from atmosphere. Depending on the pump design, there may be one or two seals on each impeller shaft.

The seal size, design type, component materials, and housing configuration have been specifically designed for this pump application and rated operating parameters.

# **Mechanical Seal Basics**

A mechanical seal is a device that houses two highly polished components (known as faces). One face rotates, the other is stationary. A secondary elastomer bellows seals the primary ring to the shaft. An oring or cup seal seals the mating ring in the housing. The polished seal faces of the primary and mating rings are pressed together by a spring mechanism to provide adequate force to affect a seal. The force acting between the seal faces increases in direct proportion to product pressure.

The elastomer bellows seal utilized in this pump has the following design features:

- Mechanical drive of the primary seal ring. The drive band's notch design eliminates overstressing the elastomer sealing bellows.
- Bellows design provides automatic compensation for shaft endplay, run out, and primary ring wear.
- Seal face contact pressure is controlled by a single, non-clogging coil spring. This coil spring has been custom welded per Darley specifications to eliminate high-speed spring distortion.

The seal housing is designed and ported to provide optimal water flow and pressure assuring proper cooling and flushing of the seal components.

![](_page_8_Figure_10.jpeg)

Prepared by: DWS Approved by: MCR Revised by: RJG Rev.: A Date:09/25/2001 1200583.doc Revision Date: 02/07/12

### **Operation and Maintenance**

When operated within rated operating conditions of this pump, these seals will provide trouble free service for extended periods.

Properly selected and applied mechanical shaft seals are leak free and require no adjustment. Should the seal area develop a leak, investigate the cause as soon as possible. Seal failure, leakage, may be the result of; worn seal faces, leaking bellows, or damaged o-rings. These failures may be attributed to bearing failure, impeller blockage, impeller imbalance, seal housing contamination, operating beyond pump design rating, or dry running,

Mechanical shaft seal design relies on the sealed media, in this case, water, to cool and lubricate the sealing surfaces. Therefore, extended dry operation may cause overheating and scoring or damage to the sealing surfaces, resulting in excessive leakage or a much shortened seal life.

To maximize seal life, minimize operation at pump pressures higher than pump rating. While operating at pressures beyond rating will not immediately damage the seal, it will increase sealing surface wear rate.

# **CAUTION:** DO NOT RUN THE PUMP DRY EXCEPT MOMENTARILY AND AT LOW SPEEDS

**CAUTION:** DO NOT USE THIS PUMP FOR HOSE TESTING

**CAUTION:** THE MECHANICAL SEAL SHOULD NOT BE RUN DRY, WHILE THE PUMP IS NOT ENTRAINED WITH WATER, FOR A PERIOD LONGER THAN 2 MINUTES. FAILURE TO FOLLOW THIS RECOMMENDATION WILL LEAD TO PREMATURE WEAR AND FAILURE OF YOUR MECHANICAL SHAFT SEAL.

Prepared by: DWS Approved by: MCR Revised by: RJG Rev.: A Date:09/25/2001 1200583.doc Revision Date: 02/07/12

![](_page_10_Picture_0.jpeg)

# INSTALLATION OF MECHANICAL FACE SEAL WITH O'RING

### SPECIAL HANDLING

Study the engineering layout before installing the seal. This shaft seal is a precision product and should be handled and treated with care. Take special care to prevent scratches on the lapped faces of the primary and mating ring. Provide a very clean work area where the assembly will take place. Clean hands prior to assembly.

### **INSTRUCTION STEPS:**

#### Instructions for Installing a Mechanical Shaft Seal

1. Inspect mating ring pocket in seal housing ensuring it is clean, free of chips, and nick free, to provide a proper sealing surface. Isopropyl alcohol may be used to clean the surfaces if required.

![](_page_10_Picture_7.jpeg)

SEAL HOUSING

2. Inspect the pump shaft surface under the bellows, ensuring it is clean and nick free to provide a proper sealing surface. Isopropyl alcohol may be used to clean surface if required.

![](_page_10_Figure_10.jpeg)

3. Lightly lubricate the o-ring on the mating ring with a single drop of P-80 water soluble rubber lubricant (do not over lubricate) and push it into the cavity using the recommended installation tool or other suitable plastic tube free of contaminants, firmly seating the mating ring square.

Note: The polished face of the mating ring must face out – away from the pump's gear case. Try to not touch the polished sealing face with your fingers; the oils from your fingerprint can cause the seal to leak. Remove any P-80 from the sealing face after installation.

![](_page_10_Figure_13.jpeg)

4. Clean the mating ring surface with isopropyl alcohol to remove any fingerprints and any other contaminants left on mating ring.

Prepared by: AAN Approved by: TED Revised by: TED (19July2010) Note: Steps 5 - 9 need to all be completed with in 15 minutes or less.

- 5. Apply a small drop of P-80 rubber lubricant or water-soluble lubricant (not soapy water) to the inside diameter of the bellows assembly allowing it to be pushed easily into position.
- 6. Clean the polished sealing face of the primary ring with a clean lint free rag with isopropyl alcohol to remove all fingerprints and other contaminants.
- Slide a seal save, similar to X6550, over the shaft splines to ensure that the seal is not damaged during installation. Place the primary ring and lubricated bellows assembly (without the spring) on the shaft, using a proper pusher - push the assembly into position so that the seal surfaces are in contact. Remove the seal save from the shaft.

![](_page_11_Figure_4.jpeg)

- 8. Put the spring in place, seated tight against the spring retainer on the primary ring. Note: Some springs may be slightly tapered, so one end fits the seal better than the other. The end of the spring that best fits the seal should go towards the seal to ensure even spring pressure all the way around.
- 9. Slide impeller onto impeller shaft, engage the spring into the groove of the impeller hub and install impeller washer, impeller nut, and stainless steel cotter key.

![](_page_11_Figure_7.jpeg)

\*\* Reference pump configuration for individual mechanical seal instructions. \*\* Reference pump assembly drawings and pump assembly tips for further assembly.

Note: If the seal leaks slightly after assembly, it may be necessary to run the pump for approximately 30 minutes at 50-60 psi to rinse out excess lubricant and other contaminants.

Once a mechanical seal has been installed, it is recommended that it not be reused.

If further information is needed, call **DARLEY** in Chippewa Falls, WI. at 800-634-7812 or 715-726-2650

The approximate

size of a drop should

be between the sizes

of these two circles.

# WARNING: DO NOT USE THIS PUMP FOR HOSE TESTING

### **OPERATING THE ENGINE**

After the pump has been primed, the engine speed should be increased gradually -- never jerk throttle wide open. Likewise, the engine speed should be decreased gradually when shutting down.

Watch the pump pressure gage and open throttle only enough to give the desired pressure. The pressure may rise high enough to burst the discharge hose, when using small nozzles, if the engine is given full throttle (except pumps equipped with pressure regulators set for desired pressure).

Never run engine at high speeds except when pump is primed and ready to discharge water.

### **COOLING THE ENGINE**

NFPA 1901 requires that a supplementary heat exchanger cooling system be provided. On most models, this heat exchanger is an integral part of the pump, and the installation of two hoses from the engine cooling system to the pump is all that is required.

On some models an external heat exchanger must be used. In that case two hoses from the engine cooling system and two lines from the pump will run to the heat exchanger.

The cooling line should not be opened until pressure develops in the pump, and pump should never be operated under heavy loads prolonged without an adequate supply of cooling water flowing.

Coolant temperatures should never be allowed to exceed 200° F while pumping and 180° F is usually taken as a safe operating temperature.

Always shut off cooling line when through pumping.

# SUCTION STRAINERS

A large suction strainer, which will prevent the passage of a body larger than the pump impeller ports, must always be used on the free end of the suction line when pumping from draft.

The small hydrant strainer must always be inserted in the suction manifold of pump, when pumping from hydrants and at all other times except when maximum capacity is required from draft.

Failure to use a strainer at all times when pumping will cause serious trouble by clogging the pump because, even in water mains, foreign matter is invariably present, and will be drawn into pump by the high velocity of the water entering.

### SUCTION LINE

The suction line of a fire pump can be the source of more operating difficulties than all the rest of the pump when working with a suction lift. Faults in the suction line which cause trouble in operation are as follows:

AIR LEAKS -- A small amount of air, expanding in the vacuum of the suction line, displaces a considerable volume of water which subtracts from the capacity that the pump is able to deliver, makes the priming difficult or causes pump to lose its prime. Therefore, it is absolutely essential to keep the suction line and the suction side of pump casing air tight at all time when drafting water.

Air leakage into pump while operating is usually indicated by a rattling sound in pump casing, miniature explosions in stream issuing from the nozzle, or by losing of prime when operating at very low capacities.

The usual cause of leaky suction lines is carelessness in handling of suction hose. Bruising of hose threads by bumping against hard surfaces or sand in the coupling often prevents tightening of the joints up against the gaskets. The hose gaskets are often defective and are sometimes lost without being noticed by the operator.

INSUFFICIENT SUBMERGENCE -- The free end of suction hose must be submerged to a sufficient depth to prevent the entrance of air that may be sucked down from the surface of the water to a considerable depth when operating at large capacities.

Entrance of air into suction lines in this manner is indicated by a small whirlpool, or vortex, on the surface of the water over the end of the hose.

Prepared by: EAP Approved by: MCR Rev. #: 2 Date: 1/29/07 1200509 A minimum submergence of 4 times the hose diameter to the upper holes in suction strainer is recommended where full capacity of pump is required. Where sufficient submergence is not possible, a board or sheet of metal laid over end of suction line will keep air from entering.

SUCTION LINE ENTRANCE TOO CLOSE TO BOTTOM -- If the end of suction line is laid on the bottom of the source of supply, a part of the suction opening will be shut off; and if the bottom is soft the hose will suck itself down into the earth, closing more of the opening and loosening sand and mud to be carried into the pump.

The suction entrance should be suspended a foot or more above the bottom, or if this is not possible, it should be laid on a board or piece of sheet metal. A rope tied to the suction strainer is a convenient means of holding it off the bottom.

OBSTRUCTION OF SUCTION STRAINER BY FOREIGN MATTER -- The high velocity of water entering the suction line will carry loose foreign bodies in against the strainer from a considerable distance. Therefore, all weeds and refuse should be removed from close proximity of the suction entrance.

SUCTION LINE TOO SMALL OR TOO LONG -- The flow of water into the pump is opposed by the frictional resistance in the suction line. This friction loss must be added to the height of the pump above the water (static lift) to determine the "total lift" of the pump. When all of the vacuum in the pump (atmospheric pressure) is consumed in raising water through this total life, then the limit of capacity has been reached. This capacity can be increased only by decreasing total lift. If the static lift cannot be reduced, then the friction loss must be reduced by using a shorter or larger suction hose.

The rated capacity of the pump is guaranteed for a static lift of 10 feet, with 20 feet of recommended suction hose at sea level. To increase the capacity without reducing the static lift, or to increase lift without sacrificing capacity, requires larger suction hose.

An excessively long suction line is a handicap to any pump, for besides reducing capacity through the added friction lose, it retards priming and it produces a detrimental effect known as "cavitation". This means a separation of the water column in the pump suction, or void spaces, produced by the inertia of the heavy mass of water in the line resisting sudden change in the velocity when the pump starts to deliver or when discharge valves are opened or closed. This phenomenon reduces capacity further, and usually sets up a vibratory motion and "water hammer" as the water surges in and out of the void spaces.

When operating with a long suction line, the driving engine should be accelerated gradually, the discharge gates opened gradually, and the capacities of the pump should be held down to within the range of smooth performance.

AIR TRAP IN SUCTION LINE -- If the suction line is laid so that part of it is higher than any other part that is nearer to the pump, as when hose is laid over a high bridge rail, an air trap is formed at the highest part of the hose from which the air cannot be sucked out by the primer. This trapped air is expanded and carried into the pump with the first rush of water causing the pump to immediately lose its prime.

If suction line cannot be laid so that it slopes all the way from pump to water, it can still be primed easily by simply allowing the primer to continue to function until all the trapped air in the hose has been carried into the pump and picked up by the primer.

### **TESTING FOR AIR LEAKS**

Tests for leakage should be made with the suction hose attached and capped, discharge gate open, and all other openings closed tightly.

Run electric priming pump with primer shut-off valve open, until maximum vacuum is shown on the gage. The vacuum should hold for several minutes before satisfactory performance of pump can be expected.

If excessive leakage of air occurs, the source of leaks can be located by shutting off primer motor, with vacuum at its highest point, and listening for the hiss of air.

In the absence of a vacuum gage, the vacuum in pump may be judged by closing suction opening with the flat of hand or a rubber pad.

Water or air pressure may be applied to pump casing to test for air leakage if more convenient. DO NOT pressurize with air beyond 10 PSI

### SOURCE OF WATER SUPPLY

Water may be drafted from a pond, lake, stream, cistern, stock tank, or well; but whatever the source, the static lift must not exceed 20 feet from the center of the pump to the surface of the water and a lift not exceeding 10 feet is recommended. The source of supply should be reasonably clear and free from foreign matter. It is recommended that all water holes, which may be needed for fire protection, be deepened if necessary and kept free from weeds and refuse. In many fire protection areas, cisterns or reservoirs are built and allowed to fill up with rain water to be used in emergencies.

### PUMPING IN COLD WEATHER

The first insurance against cold weather trouble is to keep fire apparatus stored in heated quarters. All water must be eliminated from pump casing and primer line between periods of operations.

When setting up for pumping, unnecessary delays should be avoided by having thoroughly trained pump operators. Be sure that primer and booster lines are kept closed until ready for use. Having discharge lines ready so that pump may be started as soon as it have become primed. Do not stop flow of water through the pump until ready to drain and return to the station.

Engine Coolant from the engine circulated through the heater jacket in pump casing prevents all ordinary freezing troubles.

### WHEN FINISHED PUMPING

Drain water out of pump casing immediately. (Drain valve is located at lowest point in pump casing, and accessible from underneath chassis.)

Don't forget to close all drain cocks after all water has been drained out. Trouble in priming will follow on the next run if this is forgotten.

Shut off cooling line to make pump ready for priming again.

If pump transmission is equipped with a transmission cooler it must be drained also. If the master drain is located below the cooler outlets it can be connected to the master drain, if not, two separate drains must be connected to the transmission cooler. Failure to drain transmission cooler will result in water in the gearcase if water in the cooling coil freezes.

If pump is equipped with an external heat exchanger, drain heat exchanger using gravity and vacuum drain on all trucks as follows: Close all open lines and drain cocks. Open cooler valve and open air vent at top or drain cock at bottom of heat exchanger depending on model. With the pump air-tight open primer with engine running for about a minute and then close primer. Drain pump of water which was deposited when heat exchanger and lines were being drained.

Pump not often used for fire service should be inspected and run periodically to ensure that they will be in readiness for an emergency.

### PUMPING SALT WATER

The pump should be flushed out with fresh water immediately after pumping salt water to prevent excessive rusting. (Except pumps which are built of special material to resist the corrosive action of the brine.)

When measuring sea water with a Pitot Gage, capacities shown in Table No. 2 should be discounted approximately  $1 \frac{1}{2\%}$  to determine the correct capacity.

A centrifugal pump will show 2 1/2% higher pressure and require 2 1/2% more power when handling sea water than when handling fresh water if operated at the same speed and capacity.

# **TESTING OF EQUIPMENT FOR PRACTICE**

It frequently happens that operators of fire apparatus, who are not thoroughly familiar with its operations, become confused under the stress of emergency and neglect some little detail that may cause trouble or delay in getting the equipment into operation. Therefore, we urge that practice tests be conducted repeatedly until operators are thoroughly trained. More than one person in the department should be a competent operator.

Practice should include pumping from low lifts, high lifts with short and long suction lines, with suction line elevated to form an air trap, and from hydrants, at large and small capacities.

It is well, also, to note the effects of air leaks in hose, insufficient submergence and restriction of suction line. (Suction line can be restricted by placing a can or other strong closure around the suction strainer).

NEVER BREAK OR RESTRICT SUCTION OR ALLOW AIR TO ENTER SUCTION LINE WHILE ENGINE IS OPERATING WITH THROTTLE OPEN. This will release the load and allow engine to run away.

Do not allow personnel to hold a large nozzle while working at high pressures for serious accidents may result if hose breaks loose.

### **MEASURING PUMP PERFORMANCE**

Pump performance is measured by the quantity of water it can deliver per minute against a certain pressure called "Total Head" or "Net Pump Pressure", as it is usually termed in fire pump testing.

The net pump pressure is the sum of the pump discharge pressure, as shown on the pressure gage with which the pump is regularly equipped, and the total suction lift converted to equivalent pounds per square inch. If pump is operating from a hydrant, the net pump pressure is the discharge pressure less the incoming pressure from hydrant measured at the suction entrance of pump.

Capacity of fire pump is measured in gallons per minute. The usual method of measurement is to determine the pressure of the jet of water leaving a given size of nozzle by means of a "Pitot Gage" from which the capacity is computed mathematically.

A Pitot Gage consists of a small tube adapted to a point directly into the hose nozzle from the center of the issuing stream, the other end of the tube being connected to an accurate pressure gage.

The nozzle jet drives straight into the Pitot tube and converts the velocity of the jet to pressure which is an accurate measure of velocity of the water as it leaves the nozzle. The tip of the Pitot tube should be one-half the diameter of the nozzle away from nozzle tip while taking reading. Table No. 2 gives nozzle capacities for various Pitot Gage readings.

If a Pitot gage is not available approximate pump capacities can be determined by reference to Table No.3

### **ACCEPTANCE TESTS**

Acceptance tests require continuous tests of three hours duration: 2 hours at 100% rated capacity and 150 PSI net pump pressure; one-half hour at 70% capacity and 200 PSI; one-half hour at 50% capacity and 250 PSI; and a spurt test at 100% capacity and 165 PSI.

Table No. 1 shows recommended set-ups and gage readings for rating tests.

To adjust nozzle pressure for the correct capacity, while maintaining the correct pump pressure, it is necessary to make simultaneous adjustments of engine throttle and the discharge gate valve, partially closing the latter until just the right discharge resistance is built up.

### ENGINES

An Underwriter fire pump imposes heavy loads on the engine that drives it, often absorbing all of the power the engine is capable of delivering at full throttle. Continuous pumping gives the engine no time to rest. Therefore, a new engine and pump unit must be thoroughly broken-in before it is required to deliver prolonged maximum pump performance.

We recommend a minimum break in period of 20 hours at light pumping loads, with occasional spurt tests and interruptions. Temperature and lubrication should be checked during this period.

Engine manufacturers' power ratings usually show maximum performance of a selected, factory adjusted engine, operating without fan, generator, muffler or other accessories, and corrected for "ideal" conditions, i.e. sea level barometer (29.92" of mercury) 60°F and high humidity. Therefore, the actual power delivered by an average truck mounted engine is considerably lower than the manufacturers' rating, and allowances must be made in predicting pump performance.

### EFFECTS OF ATMOSPHERIC CONDITIONS ON ENGINE AND PUMP PERFORMANCE

Each one inch of drop in Barometric pressure or each 1000 feet of elevation of the pumping site reduces engine power approximately 3 1/2% for engines not equipped with a turbo charger.

Each  $12^{\circ}$  rise in temperature above  $60^{\circ}$  F of carburetor intake air reduces engine power approximately 1%.

Lowering of humidity reduces power slightly.

Each one inch drop in Barometric pressure or each 1000 feet of elevation reduces the maximum possible static lift of a pump approximately one foot.

Temperature of the water supply affects the attainable suction lift of a pump. The effect is slight at low water temperatures but becomes increasingly detrimental as the temperature rises.

A  $10^{\circ}$  rise from  $70^{\circ}$ F will subtract about 1/2 foot from the maximum attainable suction lift, while an equal rise from  $100^{\circ}$ F will reduce the lift at least 1 1/2 feet.

Temperature is an important consideration when pumping from a test pit where the water is heated by recirculation.

### IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650

### DEFINITIONS

- HEAD OF WATER -- vertical depth of water measured in feet or in pressure per unit or area. In hydraulics, head always represents pressure and it is expressed interchangeably in feet of water or pounds per square inch and sometimes in inches of depth of mercury.
- STATIC HEAD -- the pressure that is exerted by a stationary column of water of a given height or depth.
- TOTAL HEAD OR TOTAL DYNAMIC HEAD -- the maximum height above the source of supply to which the pump would elevate the water plus all the resistance to flow in the pipe or hose line.
- DISCHARGE HEAD -- the pressure measured at the discharge outlet of a pump.
- SUCTION HEAD -- the positive pressure measured at the suction entrance of a pump (when pumping from an elevated tank or hydrant).
- VELOCITY HEAD -- the equivalent pressure represented by fluid in motion as measured by means of a Pitot Gage.
- STATIC LIFT -- the vertical height of the center of the pump above the source of supply (when pump from draft).
- TOTAL SUCTION LIFT -- the static lift plus the friction in suction line plus entrance losses.
- NET PUMP PRESSURE -- the total dynamic head of the pump.
- EFFECTIVE NOZZLE PRESSURE -- the pump discharge pressure minus hose friction plus or minus the difference in elevation above or below pump.
- WATER HORSEPOWER the theoretical power required to deliver a given quantity of water per minute against a given head.
- BRAKE HORSEPOWER -- Actual power as delivered by a motor or engine to a driven machine.
- PUMP EFFICIENCY -- The quotient of the water horsepower divided by brake horsepower required to produce it.
- WATER HAMMER -- a series of shock waves produced in a pipeline or pump by a sudden change in water velocity. A sudden change in flow velocity can result from rapid closure of valves. A pressure wave is set up which travels back and forth in the water column at extremely high speed producing rapid vibrations that may be violent and destructive if the water column is long.
- THE MAXIMUM THEORETICAL LIFT of a pump is 34 feet, which is the pressure of the atmosphere at sea level. The maximum practical total lift at sea level is 20 to 25 feet (depending on the type and condition of the pump) and this decreases with drops in barometric pressure.

### **OPERATING CHARACTERISTICS OF PUMPS**

- CENTRIFUGAL PUMPS: A centrifugal pump develops pressure by centrifugal force of the liquid rotating in the impeller wheel. The pressure developed depends upon the peripheral speed of the impeller (increasing as the square of the speed) and it remains fairly constant over a wide range of capacities up to the maximum output of the pump, if speed remains constant.
- If the discharge outlet of a centrifugal pump is entirely shut off, with speed kept constant, there is a small rise in pressure, the water churns in the pump casing and the power drops to a low value. If the discharge is opened wide, with little resistance to flow the pressure drops while the capacity and power both increase to their maximum.
- A centrifugal pump is an extremely simple mechanism mechanically, but rather complex hydraulically; in that many factors enter into the design of the impeller and water ways which will affect the pump's efficiency.
- DISPLACEMENT PUMPS: Rotary and piston pumps are termed "Positive Displacement" pumps because each revolution displaces or discharge (theoretically) an exact amount of liquid, regardless of the resistance. The capacity is, therefore, proportional to the number of revolutions of the pump per minute and independent of the discharge pressure except as it is reduced by "slip" (leakage past the pistons or rotors). For a given speed the power is directly proportional to the head. If the discharge is completely shut off, the pressure, power, and torque climb indefinitely until the drive power is stalled or breakage occurs.
- Slip is the greatest factor affecting efficiency of a displacement pump, and this factor is greatly influenced by the condition of and wears on the working parts.

### IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650

# **CONVERSION FACTORS**

One pound per square inch	=	2.31 feet of water
	=	2.04 inches of mercury
	=	27.7 inches of water
One foot of water	=	0.43 pounds per square inch
One inch of mercury	=	1.13 feet of water
	=	0.49 pounds per square inch
One cubic foot of water	=	62.4 pounds
	=	7.5 gallons
One gallon of water	=	231 cubic inches
-	=	0.13 cubic feet
	=	8.34 pounds
	=	3.8 liters
One Imperial Gallon	=	1.2 U.S. gallons
Atmospheric Pressure (Sea Level)	=	14.8 pounds per square inch
	=	29.9 inches of mercury
	=	34 feet of water

# W.S. DARLEY & CO.

# **OPERATING INSTRUCTIONS - ELECTRIC PRIMING PUMP**

The Darley electric primer will develop up to 25 in. Hg. in an air tight pumping system.

- The Primer is activated by a combination spring return on-off valve and electric switch. Pulling the valve out opens the valve and closes the electrical circuit to start the motor.
- Before the pump can be primed, booster line valves, drain valves, cooling line valve, and all other openings into the pump must be closed and absolutely air tight. The discharge side of the pump is sealed by a check valve, therefore the main discharge valves need not be closed.

When operating from draft, suction hose connections must be tight and free of air leaks.

Make certain the suction hose strainer is properly submerged and free of foreign material.

The main pump drive should remain disengaged until priming is complete to prevent possible damage to impeller seal rings by running "dry".

Pull the primer shutoff valve all the way out to start priming and hold open until water discharges from primer pump exhaust port. Push valve all the way in to shut off primer motor and seal tight.

# **CAUTION:** FOR PRIMING UP TO 10' OF LIFT:

If water does not discharge from the primer exhaust within about 30 seconds (45 seconds with 2 – 20' lengths of hose) stop the primer pump, check for air leaks and make sure primer pump is receiving lubricant from its reservoir, if one is present. MAX PRIMER OPERATION TIME = 90 seconds every 5 minutes. DO NOT EXCEED 90 SECONDS OF PRIMER OPERATION.

# **<u>CAUTION:</u>** FOR PRIMING 10' OF LIFT AND HIGHER:

If water does not discharge from the primer exhaust within 90 seconds stop the primer pump, check for air leaks and make sure primer pump is receiving lubricant from its reservoir, if one is present. DO NOT EXCEED 90 SECONDS OF PRIMER OPERATION.

**CAUTION:** The primer pump and motor will begin to generate heat as soon as operation begins. Extended run times (up to 90 seconds) and repeating priming cycles consecutively or within short time periods may lead to premature failure of the primer pump assembly: such failures include but are not limited to: overheating of the motor, seizure of the rotor, and cracking of primer vanes. To avoid this, after your first priming attempt, thoroughly inspect the pump system for air leaks, check that the primer is

Prepared by: EAP Revised by: TED Approved by: TED

### receiving lubricant from its reservoir if such is present, and resolve the issue before attempting re-prime.

Engage "Pump" shift to start pumping water.

When pumping from hydrants, the primer is not needed and must be kept closed.

It may be necessary to use the primer momentarily when pumping from a booster tank when the suction head is insufficient to force all the air out of the pump.

### LUBRICATING SYSTEM - ELECTRIC PRIMING PUMPS WITH FLUID RESERVOIR

- The electric motor rotary van primer pump creates a high vacuum by continuous lubrication of rotor and vanes. Therefore the primer lubricant supply tanks (4 quarts) should be kept full at all times. Recommended primer system lubricant is Darley PRIME GREEN. PRIME GREEN is an environmentally safe, non-toxic, biodegradable lubricant. Its use assures proper primer vane lubricant while minimizing environmental effects.
- After the main pump is drained, run the primer motor to drain primer lines and re-lubricate the primer pump.
- The vent hole on the lubricant tank cap should be kept open at all times to prevent siphoning lubricant from the tank after the pump is stopped. Do not increase the size of the hole.

Locate the lubricant tank where it may be conveniently inspected and filled.

Should water appear in the lubricant supply tank, the primer valve is leaking. Check and replace valve plug seal o-ring if necessary.

# ELECTRIC PRIMING PUMPS WITHOUT FLUID RESERVOIR

- The fluidless electric-motor rotary-vane primer pump creates a high vacuum by using a special material for the vanes and an initial factory applied lubricant film. This film must be present in order for the primer to operate properly and to provide maximum life for the primer components.
- This film should not wash away completely if the pump is used to pump clean water. If the priming pump is disassembled for any reason, all internal surfaces of the housing and end caps must be cleaned and coated completely with Dow Corning #111 Silicone valve lubricant prior to operating the primer. If a degradation of performance is noticed, performance may be restored by re-applying the film in this manner. It is recommended to service the primer annually to clean and re-apply the silicone film to the inside of the primer housing and end caps. Do not apply grease to the rotor slots, or the sides of the vanes.

After the main pump is drained, run the primer motor to drain primer lines.

![](_page_22_Figure_1.jpeg)

![](_page_23_Figure_0.jpeg)

![](_page_23_Figure_1.jpeg)

# W.S. DARLEY & CO.

### **OPERATING INSTRUCTIONS - ELECTRIC PRIMING PUMP**

### PUSH BUTTON ELECTRIC ACTUATED VALVE

The Darley electric primer will develop up to 25 in. Hg. in an air tight pumping system.

- The Primer is activated by a push button, 0.8 second cycle time, electric-actuated valve. This valve has three wires: ground (black), +12 or +24 VDC constant power (red), and +12 or +24 VDC energizing power (white). The valve has two internal micro-switches that cut the power to the valve when it is either fully closed or fully open. The push button is a simple SPST switch, that bypasses the red (constant power) wire with the white wire and energizes the valve to the open position when it is pushed; and cuts power to the white wire and resumes power to the red wire, closing the valve when it is released. When the button is pushed, the circuit for the primer motor is also completed, priming will begin instantaneously, and likewise when the button is released, the primer motor will shut off instantaneously.
- Before the pump can be primed, booster line valves, drain valves, cooling line valve, and all other openings into the pump must be closed and absolutely air tight. The discharge side of the pump is sealed by a check valve; therefore the main discharge valves need not be closed.

When operating from draft, suction hose connections must be tight and free of air leaks.

Make certain the suction hose strainer is properly submerged and free of foreign material.

- The main pump drive should remain disengaged until priming is complete to prevent possible damage to impeller seal rings by running "dry".
- Push the primer/valve activation button, located on your control panel, and hold until water discharges from primer pump exhaust port.

If water does not discharge from primer exhaust within about 30 seconds (45 seconds with 2-20' lengths) stop primer pump, check for air leaks and make sure primer pump is receiving lubricant from its reservoir.

# NOTE: Do not run the primer for more than one minute; it will burn up the motor, if prime is not reached within one minute, repeat the steps above.

Engage "Pump" shift to start pumping water.

When pumping from hydrants, the primer is not needed and must be kept closed.

It may be necessary to use the primer momentarily when pumping from a booster tank when the suction head is insufficient to force all the air out of the pump.

#### LUBRICATING SYSTEM - ELECTRIC PRIMING PUMPS WITH FLUID RESERVOIR

The electric motor rotary van primer pump creates a high vacuum by continuous lubrication of rotor and vanes. Therefore the primer lubricant supply tanks (4 quarts) should be kept full at all times. Recommended primer system lubricant is Darley PRIME GREEN. PRIME GREEN is an environmentally safe, non-toxic, biodegradable lubricant. Its use assures proper primer vane lubricant while minimizing environmental effects.

After the main pump is drained, run the primer motor to drain primer lines and re-lubricate the primer pump.

The vent hole on the lubricant tank cap should be kept open at all times to prevent siphoning lubricant from the tank after the pump is stopped. Do not increase the size of the hole.

Locate the lubricant tank where it may be conveniently inspected and filled.

Should water appear in the lubricant supply tank, the primer valve is leaking. Check and replace valve plug seal oring if necessary.

#### ELECTRIC PRIMING PUMPS WITHOUT FLUID RESERVOIR

- The fluidless electric-motor rotary-vane primer pump creates a high vacuum by using a special material for the vanes and an initial factory applied lubricant film. This film must be present in order for the primer to operate properly and to provide maximum life for the primer components.
- This film will not wash away completely if the pump is used to pump water. If the priming pump is disassembled for any reason, all internal surfaces of the housing and end caps must be coated completely with Dow Corning #111 Silicone valve lubricant prior to operating the primer. If after several years, a degradation of performance is noticed, performance may be restored by re-applying the film in this manner.

After the main pump is drained, run the primer motor to drain primer lines.

![](_page_26_Figure_0.jpeg)

![](_page_27_Figure_0.jpeg)

![](_page_28_Figure_0.jpeg)

![](_page_29_Figure_0.jpeg)

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![](_page_30_Figure_0.jpeg)

![](_page_31_Figure_0.jpeg)

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![](_page_32_Figure_0.jpeg)

![](_page_34_Picture_0.jpeg)

# PARTS LIST FOR MULTIPLE DRAIN VALVE

# **DRAWING NO. DGC0112**

Rep. No.	Name of Part	Qty	Rep. No.	Name of Part	Qty
1	O-ring – Body	2	7	Round Head Cap Screw	4
2	Drain Valve Body	1	8	Drain Valve Stem	1
3	Drain Valve Plug	1	9	Pin	1
4	O-ring – Plug	1	10	Hex Nut	1
5	Drain Valve Mounting Bracket	1	11	Lock Washer	1
6	Lock Washer	4	12	Knob	1

![](_page_35_Figure_0.jpeg)

#### **BALL VALVE QUARTER TURN - SELF LOCKING**

The Darley Ball Valve is a quarter turn, all bronze valve designed for the fire service.

The ball is cast bronze, precision machined *stainless steel ball* for long trouble free service. It is easily serviced in the field.

The lever is self locking and easily adjusted, even under extreme high pressure.

#### TO DISASSEMBLE AND REPAIR THE BALL VALVE ILLUSTRATION DGC0100

### **TOOLS REQUIRED:**

- 3/16" Allen Wrench
- 1-1/8" Wrench
- 3/4" & 1" Wrench
- Vise Grips or Pliers
- 1. Remove cap nut (20) and adjusting nut (16).
- 2. Lever Assembly (11) pulls straight up. Watch for 2 cam balls (12).
- 3. Unbolt and remove clutch ring (9), clutch sleeve (8), valve stem (7), spring (14), and valve stem washer (15). Check clutch ring (9) and sleeve (8) for scoring or excessive wear. Check o-ring (26). Replace if necessary.
- 4. Remove nipple (2). Check Quad Ring (25). Replace if necessary.
- 5. Unscrew ball guide screw (6). Check o-ring (23). Replace if necessary.
- 6. Remove valve ball (3). Check for scratches, corrosion, and wear. Replace if necessary.
- 7. Remove seat assembly (4). Check condition of rubber seat. Replace seat assembly if necessary.

#### REASSEMBLY OF BALL VALVE ILLUSTRATION DGC0100

- 1. Position ball (3) in body so ball guide screw (6) engages bottom of ball as it is screwed into position.
- 2. Put valve stem (7) into position. Make certain stem engages slot on top of ball.
- 3. Slip washer (15), spring (14), and clutch sleeve (8) over the stem. Place clutch ring (9) over the sleeve and secure with the four (4) 1/4" NC x 5/8" socket head cap screws.
- 4. Set the two cam balls (12) into the V grooves in the clutch sleeve (8) and drop lever assembly over them. Tighten the adjusting nut (16) so that approximately 1/8" play is left at the end of a 6" lever. Over tightening this nut will make the clutch lock inoperative. Lock adjusting nut (16) with cap nut (20). Recheck this adjustment after valve is placed in service.
- 5. Place seat assembly (4), seat o-ring (5), and quad ring (25) into position.
- 6. Secure nipple (2) to valve body with eight (8) 1/4'' NC x 5/8'' socket head cap screws.

### If more information is needed, call W.S. DARLEY & CO. at Chippewa Falls, WI at 800-634-7812 or 715-726-2650

![](_page_37_Figure_0.jpeg)

# W.S. DARLEY & CO.

### REMOTE CONTROL PRESSURE RELIEF VALVE WITH MECHANICAL SHUTOFF

### **Refer to Drawing DGC0141**

The relief valve bypasses water from the pump discharge manifold to the suction chamber at a set pump pressure, preventing excessive rise of discharge pressure when hose lines are shut off.

Turning pressure setting hand wheel (14) clockwise raises the relief pressure, and counter clockwise lowers it.

The self-cleaning fine mesh strainer will prevent the entry of solids that could cause the relief valve to malfunction. Open the strainer flush valve to remove small accumulations. This is accomplished by turning the strainer flush valve knob (6) counter clockwise 2 to 3 full turns. Strainer trapped debris will be flushed to the ground. Pump supply pressure should be 50-100 PSI when performing this procedure.

### TO SET RELIEF VALVE

- **1.** Turn four-way valve OFF.
- 2. Open at least one discharge valve and increase engine throttle setting until pressure gage indicates the pressure at which relief valve is to open.
- **3.** Turn four-way valve ON.
- **4.** If gage reading drops below pressure set in step 2, turn hand wheel (14) clockwise until pressure returns to set point.
- **5.** If gage reading does not drop, turn hand wheel (14) counter clockwise until pressure drops 5 to 10 PSI below set point. Then slowly turn hand wheel clockwise until pressure returns to pressure set in step 2.

The relief valve will now prevent the discharge pressure from rising above that for which it is set, and requires no further attention.

Should a higher or lower relief pressure be desired, repeat above procedure.

### CAUTION

With all discharge valves closed, water in the auxiliary pump casing will heat up rapidly. To avoid possible damage, allow a very small stream of water to discharge when the pump is running.

### **REMOTE CONTROL PRESSURE RELIEF VALVE WITH MECHANICAL SHUTOFF**

# MAINTENANCE DRAWING DGC0141

- Open the relief valve strainer flush valve (6) during every operation at 50-100 PSI supply pressure to insure foreign material is not blocking the screen.
- The 3/32" diameter metering orifice and diaphragm chamber at (21) may be back-flushed if necessary while the pump is delivering water by opening the pilot head drain and placing valve handle (9) midway between ON and OFF position.
- The relief valve, pilot unit, and strainer assemblies should be taken apart for inspection and cleaning at least annually, or as often as found necessary to insure trouble free performance.
- To disassemble pilot head, first turn hand wheel (14) counter clockwise to remove spring compression. Remove the four 1/4" screws holding regulator spring housing (18). Lift out diaphragm (23) and pilot valve (51) assembly. Clean and make certain 3/32" diameter orifice hole is free of obstruction.
- When reassembling pilot head, turn hand wheel (14) a few times clockwise to compress spring before tightening four screws holding spring housing. This will properly center valve seat and diaphragm.

The valve piston (40) and spring (44) chamber should be inspected and cleaned.

Replace diaphragm and o-rings if damaged or deteriorated.

- Apply a thin coating of waterproof grease lubricant: to spring housing counterbore that guides the pilot valve (51) and ball (52), to end of tension screw (17), and between piston (40) and center post.
- Self-cleaning strainer (63) can be removed for inspection or replacement by alternately turning valve knob (6) and stop nut (7) counter clockwise until stem is free for removal. To avoid discharging water through opening created by stem (62) removal, pump should be completely shut down before stem (62) is removed. Inspect and clean screen (63) if required. Check quad ring (64) for damage or deterioration. Reverse procedure to reassemble valve. Use case when initially inserting screen into body to avoid damaging quad ring (64) or valve seat.
- To replace flush valve seat (69), remove stem/screen assembly, disconnect tubing lines attached to (61) body half and unscrew (61) body half from (70) body half. Replace (69) valve seat. Reverse procedure to reassemble valve.
- All Darley relief valves can be provided with a micro switch and either one or two pilot lights to indicate when the valve is open or closed.

### IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650

![](_page_40_Figure_0.jpeg)

NAME:SMS OBJECT: DCC0141\_1 DATE:28-Nov-07 08:10:07

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![](_page_42_Figure_0.jpeg)

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# W.S. DARLEY & CO.

# Relief Valve Alarm Installation Instruction

This Alarm is designed to concentrate audible sound in the operator zone only. For optimum Performance, position alarm sound opening so it is facing the operator at a distance of 24 - 36 inches.

Mount unit in 1.12 diameter panel hole. If panel is thicker than .09 inches, invert nut.

Do not mount with sound opening in an upward position. Do not obstruct opening.

Connect to 12 VDC only.

Two (2) wires are required to complete the circuit. The alarm is sensitive to polarity and will not operate if connected with polarity reversed.

# Relief Valve Alarm Installation Instructions

THIS ALARM IS DESIGNED TO CONCENTRATE AUDIBLE SOUND IN THE OPERATOR ZONE ONLY. FOR OPTIMUM PERFORMANCE, POSITION ALARM SOUND OPENING SO IT IS FACING THE OPERATOR AT A DISTANCE OF 24-36 INCHES.

MOUNT UNIT IN 1.12 DIAMETER PANEL HOLE. IF PANEL IS THICKER THAN .09 IN., INVERT NUT. DO NOT MOUNT WITH SOUND OPENING IN AN UPWARD POSITION. DO NOT OBSTRUCT OPENING.

CONNECT TO 12 VDC ONLY.

TWO (2) WIRES ARE REQUIRED TO COMPLETE THE CIRCUIT. THE ALARM IS SENSITIVE TO POLARITY AND WILL NOT OPERATE IF CONNECTED WITH POLARITY REVERSED.

![](_page_43_Figure_12.jpeg)

### IF FURTHER INFORMATION IS NEEDED, CALL W.S. DARLEY & CO. AT CHIPPEWA FALLS, WI. AT 800-634-7812 or 715-726-2650

![](_page_44_Figure_0.jpeg)

![](_page_45_Figure_0.jpeg)

![](_page_46_Figure_0.jpeg)

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![](_page_49_Figure_0.jpeg)

### NAME:RIC OBJECT:5209402\_3 DATE:19-Nov-08 13:40:18