Operator's Manual Table of Contents For Darley Engine Driven HE Portable Fire Pump

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Model:_____ Pump Serial Number: _____

OPERATING INSTRUCTIONS FOR MODEL HE PORTABLE PUMP See Engine Instruction Before Operating This Unit WARNING: DO NOT USE THIS PUMP FOR HOSE TESTING

LUBRICATION

Change the engine oil, & maintain its level according to the recommendations of the engine manufacturer. These recommendations should be found in the engine manual supplied by the engine manufacturer.

PREPARATION FOR PRIMING

Check coupling gaskets and connect hose lines with coupling properly tightened.

Any size of suction and discharge hose may be used, depending on the volume of water required.

- Be certain that the suction hose (or pipe) is absolutely air tight. The pump will not lift water if the suction side of the pump has the slightest air leak.
- A strainer with openings not larger than 1/4" mesh must always be used on the end of the suction line when pumping dirty water.

Avoid air traps in suction hose if possible.

- Keep the suction intake strainer well above the bottom of stream or pond to prevent pickup of soil and other foreign matter. If the strainer must lie on the bottom, a metal plate or pan should be laid under it.
- The suction intake should be submerged several inches to prevent sucking in air. A cover laid over the top of strainer will allow the pump to operate with a minimum of submergence.
- Close drain valve and all other openings into pump casing.
- Do not start the engine until everything is in readiness for pumping, with hose couplings properly tightened, and pump discharge valve partly open.

TO PRIME A PUMP NOT EQUIPPED WITH PRIMER

Install a foot valve and strainer on the submerged end of suction line.

Suction line must slope down all the way from the pump to water.

Pour water into the pump through the discharge opening until pump casing and suction line are completely filled, before starting the engine.

NOTE: If your pump is equipped with a discharge check valve, it will be necessary to prop it open.

RUNNING THE ENGINE

Never start the engine with wide open throttle.

Never run the pump at high speed at any time unless it is discharging water.

Never run the pump at any speed without water longer than the short interval required for priming.

ALL DIESEL ENGINES

- All diesel engines must be throttled back by the operator in high load situations. This must be done to prevent over-fueling the engine as is evident by black exhaust smoke. Careful readjustment of the throttle will not cause a decrease in pump performance. Throttle back until pump performance just begins to decrease.
- **CAUTION:** Over-fueling the engine will cause dilution of the engine oil with diesel fuel and premature wear on the cylinder walls and bearings.







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PRIMING THE PUMP WITH HAND PRIMER

Open primer shut-off valve (handle in line with tubing), briskly cycle hand primer handle in and out until water is discharged from hand primer cylinder cap. Start engine and maintain brisk idling speed. Pump should now be discharging water. If not, cycle hand pump a few more times until discharging begins. Close primer shut-off valve.

Approximate priming time is between 10 and 30 seconds depending upon size and length of suction hose, height above water source, and hand primer cycle rate.

When priming on high lifts or when priming dirty water, it may be necessary to seat the discharge check valve by tightening down gently with the handwheel on top of the discharge head. Unscrew the handwheel as soon as engine is started.

PRIMING THE PUMP WITH AN EXHAUST PRIMER

Open the primer line shut-off valve between the primer jet and pump section. (Valve is open when the lever is in line with the air passage.)

Start the engine and run at a brisk idling speed.

Close the engine exhaust primer valve.

Close primer shut-off valve as soon as primer jet discharges water.

Open exhaust primer valve.

Repeat priming operation if pump fails to hold its prime.

For fast priming, the engine throttle may be held wide open while the engine exhaust port is closed.

When priming on high lifts or when pumping dirty water, it may be necessary to seat the discharge check valve by tightening down gently with the handwheel on top of the discharge head. Unscrew the handwheel as soon as water discharges through the exhaust jet.

If the pump does not deliver water within one minute, stop the engine and check for air leaks or failure of primer jet to produce vacuum.

COLD WEATHER OPERATION

In cold weather it is important to make sure the tubing leading from the exhaust primer to the pump casing is free from water to prevent freezing. Freezing of this tubing will render the exhaust primer inoperative and any damage tubing and fitting.

To remove water from suction tubing: Restart engine after suction line is disconnected. Open primer line shut-off valve and close engine exhaust tightly by lever valve at top of engine for five seconds. Open exhaust valve and shut off engine.

Mechanical Shaft Seal

This pump assembly incorporates high quality mechanical shaft seal(s) separating the pump housing components from atmosphere. Depending on the pump design, there may be one or two seals on each impeller shaft.

The seal size, design type, component materials, and housing configuration have been specifically designed for this pump application and rated operating parameters.

Mechanical Seal Basics

A mechanical seal is a device that houses two highly polished components (known as faces). One face rotates, the other is stationary. A secondary elastomer bellows seals the primary ring to the shaft. An oring or cup seal seals the mating ring in the housing. The polished seal faces of the primary and mating rings are pressed together by a spring mechanism to provide adequate force to affect a seal. The force acting between the seal faces increases in direct proportion to product pressure.

The elastomer bellows seal utilized in this pump has the following design features:

- Mechanical drive of the primary seal ring. The drive band's notch design eliminates overstressing the elastomer sealing bellows.
- Bellows design provides automatic compensation for shaft endplay, run out, and primary ring wear.
- Seal face contact pressure is controlled by a single, non-clogging coil spring. This coil spring has been custom welded per Darley specifications to eliminate high-speed spring distortion.

The seal housing is designed and ported to provide optimal water flow and pressure assuring proper cooling and flushing of the seal components.



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Operation and Maintenance

When operated within rated operating conditions of this pump, these seals will provide trouble free service for extended periods.

Properly selected and applied mechanical shaft seals are leak free and require no adjustment. Should the seal area develop a leak, investigate the cause as soon as possible. Seal failure, leakage, may be the result of; worn seal faces, leaking bellows, or damaged o-rings. These failures may be attributed to bearing failure, impeller blockage, impeller imbalance, seal housing contamination, operating beyond pump design rating, or dry running,

Mechanical shaft seal design relies on the sealed media, in this case, water, to cool and lubricate the sealing surfaces. Therefore, extended dry operation may cause overheating and scoring or damage to the sealing surfaces, resulting in excessive leakage or a much shortened seal life.

To maximize seal life, minimize operation at pump pressures higher than pump rating. While operating at pressures beyond rating will not immediately damage the seal, it will increase sealing surface wear rate.

CAUTION: DO NOT RUN THE PUMP DRY EXCEPT MOMENTARILY AND AT LOW SPEEDS

CAUTION: DO NOT USE THIS PUMP FOR HOSE TESTING

CAUTION: THE MECHANICAL SEAL SHOULD NOT BE RUN DRY, WHILE THE PUMP IS NOT ENTRAINED WITH WATER, FOR A PERIOD LONGER THAN 2 MINUTES. FAILURE TO FOLLOW THIS RECOMMENDATION WILL LEAD TO PREMATURE WEAR AND FAILURE OF YOUR MECHANICAL SHAFT SEAL.

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INSTALLATION OF MECHANICAL FACE SEAL WITH O'RING

SPECIAL HANDLING

Study the engineering layout before installing the seal. This shaft seal is a precision product and should be handled and treated with care. Take special care to prevent scratches on the lapped faces of the primary and mating ring. Provide a very clean work area where the assembly will take place. Clean hands prior to assembly.

INSTRUCTION STEPS:

Instructions for Installing a Mechanical Shaft Seal

1. Inspect mating ring pocket in seal housing ensuring it is clean, free of chips, and nick free, to provide a proper sealing surface. Isopropyl alcohol may be used to clean the surfaces if required.



SEAL HOUSING

2. Inspect the pump shaft surface under the bellows, ensuring it is clean and nick free to provide a proper sealing surface. Isopropyl alcohol may be used to clean surface if required.



3. Lightly lubricate the o-ring on the mating ring with a single drop of P-80 water soluble rubber lubricant (do not over lubricate) and push it into the cavity using the recommended installation tool or other suitable plastic tube free of contaminants, firmly seating the mating ring square.

Note: The polished face of the mating ring must face out – away from the pump's gear case. Try to not touch the polished sealing face with your fingers; the oils from your fingerprint can cause the seal to leak. Remove any P-80 from the sealing face after installation.



4. Clean the mating ring surface with isopropyl alcohol to remove any fingerprints and any other contaminants left on mating ring.

Prepared by: AAN Approved by: TED Revised by: TED (19July2010) Note: Steps 5 - 9 need to all be completed with in 15 minutes or less.

- 5. Apply a small drop of P-80 rubber lubricant or water-soluble lubricant (not soapy water) to the inside diameter of the bellows assembly allowing it to be pushed easily into position.
- 6. Clean the polished sealing face of the primary ring with a clean lint free rag with isopropyl alcohol to remove all fingerprints and other contaminants.
- Slide a seal save, similar to X6550, over the shaft splines to ensure that the seal is not damaged during installation. Place the primary ring and lubricated bellows assembly (without the spring) on the shaft, using a proper pusher - push the assembly into position so that the seal surfaces are in contact. Remove the seal save from the shaft.



- 8. Put the spring in place, seated tight against the spring retainer on the primary ring. Note: Some springs may be slightly tapered, so one end fits the seal better than the other. The end of the spring that best fits the seal should go towards the seal to ensure even spring pressure all the way around.
- 9. Slide impeller onto impeller shaft, engage the spring into the groove of the impeller hub and install impeller washer, impeller nut, and stainless steel cotter key.



** Reference pump configuration for individual mechanical seal instructions. ** Reference pump assembly drawings and pump assembly tips for further assembly.

Note: If the seal leaks slightly after assembly, it may be necessary to run the pump for approximately 30 minutes at 50-60 psi to rinse out excess lubricant and other contaminants.

Once a mechanical seal has been installed, it is recommended that it not be reused.

If further information is needed, call **DARLEY** in Chippewa Falls, WI. at 800-634-7812 or 715-726-2650

The approximate

size of a drop should

be between the sizes

of these two circles.

W. S. DARLEY & CO. DARLEY INJECTION TYPE STUFFING BOX ADJUSTMENT

A Prop 65 Warning: This product contains lead, a chemical known to the State of California to cause cancer, birth defects, and other reproductive harm. Wash hands after handling.

Caution: Do not attempt to use anything but Darley injection packing. Using the wrong packing material in your pump may cause catastrophic failure of the pump shaft sealing components.

Only use W.S. Darley & Co.'s plastallic injection packing material. It is made of a special composition of shredded fibers, and a special bonding and lubricating compound.

It is important that the stuffing box is completely filled solid with packing and compressed firm during adjustment to prevent formation of voids and excessive leakage.

To pack the stuffing box when empty and assembled in the pump, remove the packing screw and nut assembly, and insert pellet form packing into the packing plunger guide. Replace the packing screw assembly and use a hand speed wrench to force the pellets into the gland. <u>DO NOT USE A POWER</u> <u>TOOL!</u> Repeat pellet additions while turning the impeller shaft by hand until resistance to turning is felt when the stuffing box is almost full. Continue turning packing screw by hand using a standard 6" long 9/16" end wrench until 4 lb. of force is felt at the end of the wrench. This is equivalent to 2 ft-lb or 24 in-lb torque. Continue turning until a few flakes of packing are extruded out the opening between the impeller shaft and the stuffing box hole. The gland is now ready for pressure testing or pumping.

After priming the pump with water, start the pump and raise the discharge pressure to 50 psi. Tighten the packing screw using a 6" long 9/16" end wrench until 4 lb. force is felt at the end of the wrench (24 in-lb torque). Continue operating the pump at 50 psi for 5 minutes to dissipate packing pressure against the shaft and permit cooling water to flow between the shaft and stuffing box hole. Make sure that water actually does come through before operating pump at any higher pressure. The normal drip rate may vary between 5 and 60 drops per minute.



Prepared By: EAP Revised By: EAP Approved By: TED Operate the pump for 10 minutes at the highest normal operating pressure flowing sufficient water to prevent overheating. Do not run the pump blocked tight. Lower discharge pressure to 50 psi and repeat the packing screw tightening procedure outlined above.

The pump may now be operated for any time period required within its rated capacity. However, the drip rate should be monitored more frequently during the first few hours, and adjusted if necessary to achieve a stable flow rate. Several more adjustments may be required.



For a list of approximate quantity of packing pellets required by model (completely repacked), see below:

Model	Approximate # Packing Pellets
Α	 6
2BE	 6
EM	 15
н	 8
JM	 8
KD	 10
KS	 8
LD	 15
LS	 9
Р	 10
U2	 5
U4	 10

If further information is needed, call **W.S. DARLEY & CO.** at Chippewa Falls, WI. at 800-634-7812 or 715-726-2650

OPERATING CHARACTERISTICS OF PUMPS

- CENTRIFUGAL PUMPS: A centrifugal pump develops pressure by centrifugal force of the liquid rotating in the impeller wheel. The pressure developed depends upon the peripheral speed of the impeller (increasing as the square of the speed) and it remains fairly constant over a wide range of capacities up to the maximum output of the pump, if speed remains constant.
- If the discharge outlet of a centrifugal pump is entirely shut off, with speed kept constant, there is a small rise in pressure, the water churns in the pump casing and the power drops to a low valve. If the discharge is opened wide, with little resistance to flow the pressure drops while the capacity and power both increase to their maximum.
- A centrifugal pump is an extremely simple mechanism mechanically, but rather complex hydraulically, in that many factors enter into the design of the impeller and water ways which will affect the pump's efficiency.
- DISPLACEMENT PUMPS: Rotary and piston pumps are termed "Positive Displacement" pumps because each revolution displaces or discharge (theoretically) an exact amount of liquid, regardless of the resistance. The capacity is, therefore, proportional to the number of revolutions of the pump per minute and independent of the discharge pressure except as it is reduced by "slip" (leakage past the pistons or rotors). For a given speed the power is directly proportional to the head. If the discharge is completely shut off, the pressure, power, and torque climb indefinitely until the drive power is stalled or breakage occurs.
- Slip is the greatest factor affecting efficiency of a displacement pump, and this factor is greatly influenced by the condition of and wear on the working parts.

DEFINITIONS

- HEAD OF WATER -- vertical depth of water measured in feet or in pressure per unit or area. In hydraulics, head always represents pressure and it is expressed interchangeably in feet of water or pounds per square inch and sometimes in inches of depth of mercury.
- STATIC HEAD -- the pressure that is exerted by a stationary column of water of a given height or depth.
- TOTAL HEAD OR TOTAL DYNAMIC HEAD -- the maximum height above the source of supply to which the pump would elevate the water plus all the resistance to flow in the pipe or hose line.
- DISCHARGE HEAD -- the pressure measured at the discharge outlet of a pump.
- SUCTION HEAD -- the positive pressure measured at the suction entrance of a pump (when pumping from an elevated tank or hydrant).
- VELOCITY HEAD -- the equivalent pressure represented by fluid in motion as measured by means of a Pitot Gage.
- STATIC LIFT -- the vertical height of the center of the pump above the source of supply (when pump from draft).
- TOTAL SUCTION LIFT -- the static lift plus the friction in suction line plus entrance losses.
- NET PUMP PRESSURE -- the total dynamic head of the pump.
- EFFECTIVE NOZZLE PRESSURE -- the pump discharge pressure minus hose friction plus or minus the difference in elevation above or below pump.

- WATER HORSEPOWER the theoretical power required to deliver a given quantity of water per minute against a given head.
- BRAKE HORSEPOWER -- Actual power as delivered by a motor or engine to a driven machine.
- PUMP EFFICIENCY -- The quotient of the water horsepower divided by brake horsepower required to produce it.
- WATER HAMMER -- a series of shock waves produced in a pipeline or pump by a sudden change in water velocity. A sudden change in flow velocity can result from rapid closure of valves. A pressure wave is set up which travels back and forth in the water column at extremely high speed producing rapid vibrations that may be violent and destructive if the water column is long.
- THE MAXIMUM THEORETICAL LIFT of a pump is 34 feet, which is the pressure of the atmosphere at sea level. The maximum practical total lift at sea level is 20 to 25 feet (depending on the type and condition of the pump) and this decreases with drops in barometric pressure.

CONVERSION FACTORS

One pound per square inch	=	2.31 feet of water
	=	2.04 inches of mercury
	=	27.7 inches of water
One foot of water	=	0.43 pounds per square inch
One inch of mercury	=	1.13 feet of water
	=	0.49 pounds per square inch
One cubic foot of water	=	62.4 pounds
	=	7.5 gallons
One gallon of water	=	231 cubic inches
	=	0.13 cubic feet
	=	8.34 pounds
	=	3.8 liters
One Imperial Gallon	=	1.2 U.S. gallons
Atmospheric Pressure (Sea Level)	=	14.8 pounds per square inch
	=	29.9 inches of mercury
	=	34 feet of water

TABLE NO. 2 DISCHARGE FROM SMOOTH BORE NOZZLE Pressures measured by Pitot gage.

Nozzle															
Pressure	1/4	3/8	1/2	5/8	3/4	7/8	1	1 1/8	1 1/4	1 3/8	1 1/2	1 5/8	1 3/4	2	2 1/4
PSI					GAI	LLONS	PER MI	NUTE D	ELIVE	RED					
5	4	9	16	26	37	50	66	84	103	125	149	175	203	266	337
6	4	10	18	28	41	55	72	92	113	137	163	192	223	292	369
7	4	11	19	30	44	59	78	99	122	148	176	207	241	315	399
8	5	11	21	32	47	64	84	106	131	158	188	222	257	336	427
9	5	12	22	34	50	67	89	112	139	168	200	235	273	357	452
10	6	13	23	36	53	71	93	118	146	177	211	248	288	376	477
12	6	15	25	40	58	78	102	130	160	194	231	271	315	412	522
14	7	15	27	43	63	84	110	140	173	210	249	293	340	445	564
16	7	16	29	46	67	90	118	150	185	224	267	313	364	475	603
18	7	17	31	49	71	95	125	159	196	237	283	332	386	504	640
20	8	18	33	51	75	101	132	167	206	250	298	350	407	532	674
22	8	19	34	54	79	105	139	175	216	263	313	367	427	557	707
24	8	20	36	56	82	110	145	183	226	275	327	384	446	582	739
26	9	21	37	59	85	115	151	191	235	286	340	400	464	606	769
28	9	21	39	61	89	119	157	198	244	297	353	415	481	629	799
30	10	22	40	63	92	123	162	205	253	307	365	429	498	651	826
32	10	23	41	65	95	127	167	212	261	317	377	443	514	673	854
34	11	23	43	67	98	131	172	218	269	327	389	457	530	693	880
36	11	24	44	69	100	135	177	224	277	336	400	470	546	713	905
38	11	25	45	71	103	138	182	231	285	345	411	483	561	733	930
40	11	26	46	73	106	142	187	237	292	354	422	496	575	752	954
42	11	26	47	74	109	146	192	243	299	363	432	508	589	770	978
44	12	27	49	76	111	149	196	248	306	372	442	520	603	788	1000
46	12	28	50	78	114	152	200	254	313	380	452	531	617	806	1021
48	12	28	51	80	116	156	205	259	320	388	462	543	630	824	1043
50	13	29	52	81	118	159	209	265	326	396	472	554	643	841	1065
52	13	29	53	83	121	162	213	270	333	404	481	565	656	857	1087
54	13	30	54	84	123	165	217	275	339	412	490	576	668	873	1108
56	13	30	56	86	125	168	221	280	345	419	499	586	680	889	1129
58	13	31	56	87	128	171	225	285	351	426	508	596	692	905	1149
60	14	31	57	89	130	174	229	290	357	434	517	607	704	920	1168
62	14	32	58	90	132	177	233	295	363	441	525	617	716	936	1187
64	14	32	59	92	134	180	237	299	369	448	533	627	727	951	1206
66	14	33	60	93	136	182	240	304	375	455	542	636	738	965	1224
68	14	33	60	95	138	185	244	308	381	462	550	646	750	980	1242
70	15	34	61	96	140	188	247	313	386	469	558	655	761	994	1260
72	15	34	62	97	142	191	251	318	391	475	566	665	771	1008	1278
74	15	35	63	99	144	193	254	322	397	482	574	674	782	1023	1296
76	15	35	64	100	146	196	258	326	402	488	582	683	792	1036	1313
78	15	36	65	101	148	198	261	330	407	494	589	692	803	1050	1330

TABLE NO. 2 DISCHARGE FROM SMOOTH BORE NOZZLE Pressures measured by Pitot gage.

Nozzle															
Pressure	1/4	3/8	1/2	5/8	3/4	7/8	1	1 1/8	1 1/4	1 3/8	1 1/2	1 5/8	1 3/4	2	2 1/4
PSI					GA	LLONS	PER MI	NUTE D	ELIVE	RED					
80	16	36	66	103	150	201	264	335	413	500	596	700	813	1063	1347
82	16	37	66	104	152	204	268	339	418	507	604	709	823	1076	1364
84	16	37	67	105	154	206	271	343	423	513	611	718	833	1089	1380
86	16	37	68	107	155	208	274	347	428	519	618	726	843	1102	1396
88	16	38	69	108	157	211	277	351	433	525	626	735	853	1115	1412
90	17	39	70	109	159	213	280	355	438	531	633	743	862	1128	1429
92	17	39	70	110	161	215	283	359	443	537	640	751	872	1140	1445
94	17	39	71	111	162	218	286	363	447	543	647	759	881	1152	1460
96	17	40	72	113	164	220	289	367	452	549	654	767	890	1164	1476
98	17	40	73	114	166	223	292	370	456	554	660	775	900	1176	1491
100	18	41	73	115	168	225	295	374	461	560	667	783	909	1189	1506
105	18	42	75	118	172	230	303	383	473	574	683	803	932	1218	1542
110	19	43	77	121	176	236	310	392	484	588	699	822	954	1247	1579
115	19	43	79	123	180	241	317	401	495	600	715	840	975	1275	1615
120	19	44	80	126	183	246	324	410	505	613	730	858	996	1303	1649
125	20	45	82	129	187	251	331	418	516	626	745	876	1016	1329	1683
130	20	46	84	131	191	256	337	427	526	638	760	893	1036	1356	1717
135	21	47	85	134	195	262	343	435	536	650	775	910	1056	1382	1750
140	21	48	87	136	198	266	350	443	546	662	789	927	1076	1407	1780
145	21	49	88	139	202	271	356	450	556	674	803	944	1095	1432	1812
150	22	50	90	141	205	275	362	458	565	686	817	960	1114	1456	1843

Approximate Discharge From Different Nozzles at the end of Fifty Feet of Average, 2 1/2" Rubber Lined Fire Hose, for Various Pump Pressures with Discharge Gage Wide Open

				c open			
PUMP	SIZE	OF	NOZZLE	&	GALLONS	PER	MINUTE
PRESSURE	3/4	7/8	1	1 1/8	1 1/4	1 3/8	1 1/2
LBS							
30	90	119	153	187	217	250	282
40	103	137	177	216	253	290	327
50	115	153	198	242	284	325	367
60	126	168	216	265	311	357	402
70	136	182	234	287	337	385	435
80	145	194	250	308	361	414	465
90	154	206	265	325	383	437	492
100	162	217	280	343	405	462	520
110	171	228	295	360	425	485	549
120	179	239	307	377	444	510	572
130	186	249	318	392	462	530	596
140	193	258	330	407	480	549	618
150	200	267	341	421	497	567	
175	215	288	374	455	538		
200	230	309	395	486			
225	243	328	420				
250	257	345					

This table is offered as an aide in testing pump performance where facilities for accurate measurement of capacity are not available. The capacities given above are conservative, and will not vary more than 5% from actual capacities with any of the standard hose that might be used.

TABLE NO. 4

Pump or Hydrant Pressure required to give Effective Nozzle Pressure through various Lengths of best Rubber Lined Hose.

Size of H	lose		2	2 1/2				3		1		1 1/2	2		2
Size of Noz	zzle	3/4	7/8	1	1 1/	4 11	1/2 1	1/4 1 1	1/2	1/4	3/8	1/2	2	5/8	5/8
Nozzle Pressure Pounds		Length of Hose Feet]	PUMP OR	HYDRAN	T PRESSU	J RE - PC	OUNDS	·				
	40	100	44	46	51	64	88	51	62	45	43	48	60	42	50
		200	47	52	60	86	130	59	78	49	46	56	79	43	60
		400	53	62	79	129	212	75	110	58	51	73	118	46	79
		600	59	74	97	172		92	143	67	57	89	158	50	99
		800	65	85	116	215		108	176	76	62	106	196	53	119
		1000	72	96	134	258		124	208	85	68	122	235	56	138
		1500	87	118	181			165		108	72	142		64	187
		2000	103	151	227			205		130	96	204	89	72	226
	60	100	65	69	75	95	132	76	92	67	64	72	117	63	73
		200	70	78	89	126	196	88	115	74	68	84	173	65	86
		400	79	94	116	188		111	161	87	76	107	231	69	112
		600	88	111	143	250		135	208	101	85	131		74	138
		800	98	127	170			158		114	93	153		79	164
		1000	107	143	197			182		127	101	178		83	190
		1500	130	184	264					161	122	237		95	155
		2000	153	225						195	142			106	
	80	100	87	92	99	126	175	101	103	88	85	96	117	83	99
		200	93	103	115	167		116	154	97	91	112	154	86	117
		400	105	125	148	249		147		115	102	143	228	92	154
		600	117	147	181			178		132	112	174		98	191
		800	129	167	214			209		150	123	206		104	228
		1000	141	191	247					167	134	238		110	
		1500	171	245						211	161			125	
		2000	201		125					254	188	120		140	
1	00	100	108	115	150	157		126	152	111	107	139	146	104	123
		200	116	128	200	209		146	190	122	113	177	192	108	145
		400	130	154	250			184		143	127	217	284	115	190
		600	145	180				223		165	140	256		123	235
		800	159	206						186	154			131	
		1000	174	232						208	167			138	
		1500	211							262	200			157	
		2000	253								234			175	

TABLE NO. 5REACH OF FIRE STREAMS

Size of									
Nozzle	1/4"	3/8"	1/2"	5/8"	3/4"	7/8"	1"	1-1/4"	1-1/2"
NOZZLE	E	FFEC	ΓIVE	VERT	ICAL	REA	CH -	Feet	
PRESSURE									
40	30	35	40	50	59	62	64	65	69
60	35	40	45	60	74	77	79	84	87
80	38	42	48	65	81	85	89	94	96
100	40	44	50	68	84	89	94	100	102
NOZZLE	N	IAXIN	IUM '	VERT	[CAL]	REA	CH - H	Feet	
PRESSURE									
40	60	65	70	75	78	79	80	80	80
60	70	75	85	95	105	106	108	110	110
80	78	83	95	105	117	125	132	140	140
100	80	88	100	110	122	135	145	155	155
NOZZLE	E	FFEC	гіуе	HORI	ZONT	'AL F	REAC	H - Fe	et
PRESSURE						-			
40	20	25	30	40	44	50	55	62	66
60	25	32	37	50	54	61	67	75	80
80	28	35	40	57	62	70	76	84	88
100	30	37	42	60	66	76	84	93	95
NOZZLE	N	IAXIN	IUM	HORIZ	ZONT	AL R	EACH	H - Fee	et
PRESSURE									
40	65	80	90	100	108	120	125	138	140
60	80	95	95	120	127	142	156	176	183
80	90	105	105	135	143	160	175	201	210
100	95	110	110	140	153	180	205	215	223

TABLE NO. 6

Friction Loss in Fire Hose

Loss in PSI per 100 Feet of Hose

SIZE HOSE	LINEN	HOSE		BEST RUBER LINED HOSE								
G.P.M.	1 1/2	2	2 1/2	3/4	1	1 1/2	2	2 1/2	3	3 1/2	2-2 1/2	
10	1.0			13.5	3.5	0.5	.1					
15	2.2			29.0	7.2	1.0	0.3					
20	3.6			50.0	12.3	1.7	0.4					
25	5.5			75.0	18.5	2.6	0.6					
30	8.0	1.9		105.0	26.0	3.6	0.9					
40	13.0	3.2		180.0	44.0	6.1	1.5					
50	20.0	4.9	1.6		67.0	9.3	2.3					
60	28.0	7.0	2.2		96.0	13.5	3.3					
70	37.0	9.0	3.1		131.0	17.0	43					
80	47.0	11.5	3.8		171.0	23.0	5.6					
90	59.0	14.5	5.0		217.0	29.0	7.0					
100	72.0	17.5	5.9		268.0	33.0	8.4					
120		25.0	8.3		386.0	47.0	11.7					
140		34.0	11.0			62.0	16.0	5.2	2.0	0.9	1.4	
160		43.0	14.0			78.0	20.0	6.6	2.6	1.2	1.9	
180		53.0	17.7			97.0	25.0	8.3	3.2	1.5	2.3	
200		63.0	21.5			121.0	30.6	10.1	3.9	1.8	2.8	
220						146.0		12.0	4.6	2.1	3.3	
240						173.0		14.1	5.4	2.5	3.9	
260						204.0		16.4	6.3	2.9	4.5	
280						237.0		18.7	7.2	3.3	5.2	
300						272.0		21.2	8.2	3.7	5.9	
320								23.8	9.3	4.2	6.6	
340								26.9	10.5	4./	/.4	
300								30.0	11.5	5.2	8.3	
380								26.2	12.8	5.0	9.2	
400								40.8	14.1	7.0	10.1	
423								40.8	137	7.0	11.5	
430								50.0	17.5	2.9	13.8	
500								55.0	21.2	0.7	15.0	
525								55.0	23.2	10.5	16.6	
550									25.2	11.4	18.1	
575									27.5	12.4	19.6	
600									29.9	13.4	21.2	
650									34.5	15.5	24.8	
700									39.5	17.7	28.3	
750									45.0	20.1	32.2	
800									50.5	22.7	36.2	
850									56.5	25.4	40.7	
900									63.0	28.2	45.2	
1000									76.5	34.3	55.0	

Losses in rough walled, rubber hose may be 50% higher than values given above.

TABLE NO. 7Friction Loss in 15-year-old Steel PipeLoss in PSI per 100 Feet of Pipe

PIPE SIZE	1/8	1/4	3/8	1/2	3/4	1	1 1/4	1 1/2	2	2 1/2	3	4	6	8
G.P.M.														
1	52.0	12.0	2.8	0.9										
2		45.0	10.0	3.2	4.0									
5			55.0	18.0	4.5	1.4	0.4							
10				64.0	16.0	5.0	1.3	0.6						
15				135.0	34.0	11.0	2.7	1.3	0.5					
20					59.0	18.0	4.7	2.2	0.8					
25					89.0	27.0	7.1	3.4	1.2					
30					125.0	39.0	10.0	4.7	1.7	0.6				
35						51.0	13.0	6.3	2.2	0.7				
40						66.0	17.0	8.0	2.9	0.9				
45						82.0	21.0	10.0	3.6	1.2				
50						99.0	26.0	12.0	4.3	1.4	0.6			
60						140.0	38.0	17.0	6.1	2.0	0.8			
70							49.0	23.0	8.0	2.7	1.1			
80							63.0	29.0	10.0	3.4	1.5			
90							78.0	36.0	13.0	4.3	1.8			
100							96.0	44.0	15.0	5.1	2.2	0.5		
125							144.0	66.0	24.0	7.8	3.3	0.8		
150								93.0	33.0	11.0	4.6	1.1		
175								125.0	44.0	15.0	6.1	1.5		
200									56.0	19.0	7.8	1.9		
250									84.0	28.0	12.0	2.9		
300									114.0	40.0	16.0	4.0	0.6	
350										53.0	22.0	5.4	0.8	
400										68.0	28.0	6.9	1.0	
450										84.0	35.0	8.6	1.2	
500										102.0	42.0	10.0	1.4	0.4
600											60.0	15.0	2.1	0.6
800												25.0	3.5	1
1000												37.0	5.2	1.3
1500													11.0	2.7
2000													19.0	4.7
2500													29.0	7.1
3000														10

TABLE NO. 8 Resistance of Fittings Equivalent Lengths of Straight Pine - Feet

Equivalent Lengths of Straight 1 pe - reet													
PIPE SIZE	1/2	3/4	1	1 1/4	1 1/2	2	2 1/2	3	4	5	6	8	
Gate Valve	0.4	0.6	0.8	1.1	1.4	1.8	2.2	2.8	4.1	5.3	6.7	9.4	
Global Valve	3.0	4.5	6.0	8.5	10.5	14.0	17.0	22.0	32.0	42.0	53.0	75.0	
Angle Valve	1.4	2.0	2.7	3.8	4.8	6.3	7.9	10.5	14.5	18.5	23.0	33.0	
Std. Elbow	1.1	1.5	2.0	2.8	3.5	4.7	5.8	7.5	11.0	14.0	18.0	24.0	
45 Elbow	0.6	0.8	1.0	1.4	1.6	2.1	2.5	3.1	4.2	5.2	6.3	8.5	
Long Sweep EI Str Run Tee	0.5	0.8	1.0	1.4	1.7	2.3	2.8	3.7	5.3	7.0	9.0	12.5	
Std. Tee Thru Side Outlet	2.1	2.9	3.9	5.5	6.9	9.1	11.6	14.8	21.0	27.0	34.0	49.0	
SuddenEnlarg or contraction	1.8	2.5	3.2	4.2	5.0	6.5	7.5	9.5	13.0	16.0	19.0	25.0	
Entrance to Pipe	1.0	1.3	1.6	2.2	2.6	3.3	3.9	4.9	6.5	8.2	10.0	13.0	

TABLE NO. 9 To Convert Pounds per Square Inch to Feet Elevation of Water

Feet	5	10	15	20	25	30	35	40	45	50	60	70	80	90
Pounds	2.2	4.3	6.5	8.7	11	13	15	17	20	22	26	30	35	39
Feet	100	120	130	140	150	160	170	180	190	200	220	240	260	280
Pounds	43	52	56	61	65	69	74	78	82	87	95	104	113	121
Feet	300	320	340	360	380	400	425	450	475	500	525	550	600	700
Pounds	130	139	147	156	165	173	184	195	206	217	227	238	260	303

 Table NO. 10

 American National Fire Hose Connection Screw Thread - NH

Size of Hose	3/4	1	1 1/2	2 1/2	3	3 1/2	4	4 1/2	5	6
Thr'ds per inch	8	8	9	7.5	6	6	4	4	4	4
Thread	0.75-8 NH	1-8 NH	1.5-9 NH	2.5-7.5 NH	3-6 NH	3.5-6 NH	4-4 NH	4.5-4 NH	5-4 NH	6-4 NH
Designation										
Max. O.D. Male	1.3750	1.3750	1.9900	3.0686	3.6239	4.2439	5.0109	5.7609	6.2600	7.0250

Ref. NFPA 1963

Underwriters Nozzle Tip Thread: 2.1875 O.D. - 12 threads per inch.